

✓ AD-A148 708

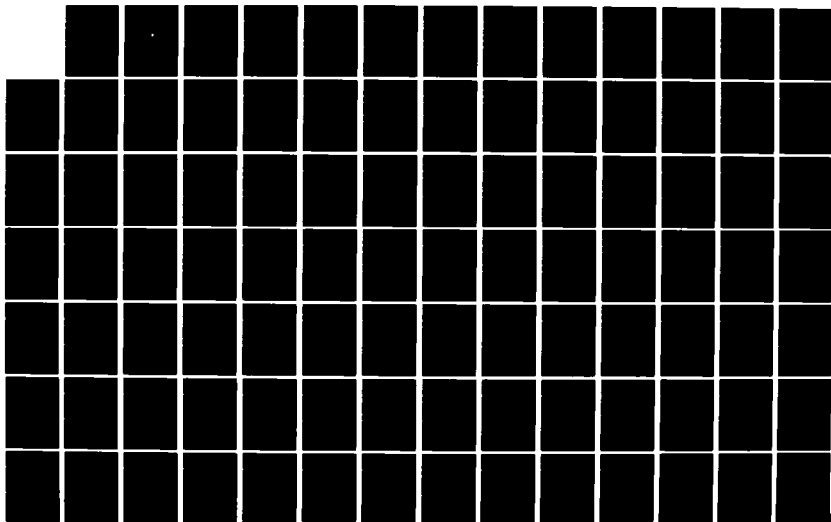
AN ANALYTIC MODEL OF GAS TURBINE ENGINE INSTALLATIONS
(U) NAVAL POSTGRADUATE SCHOOL MONTEREY CA S M EZZELL
SEP 84

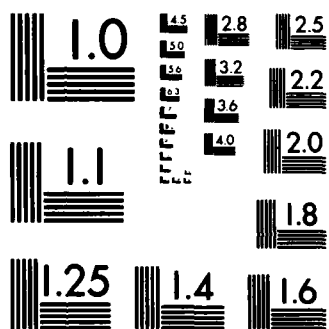
1/3

UNCLASSIFIED

F/G 21/5

NL





MICROCOPY RESOLUTION TEST CHART
NATIONAL BUREAU OF STANDARDS-1963-A

AD-A148 708

2

NAVAL POSTGRADUATE SCHOOL

Monterey, California



DTIC
S
DEC 25 1984
E

THESIS

AN ANALYTIC MODEL OF GAS TURBINE
ENGINE INSTALLATIONS

by

Stephen M. Ezzell

September 1984

Thesis Advisor:

P. F. Pucci

Approved for public release; distribution unlimited.

84 12 12 027

Unclassified

SECURITY CLASSIFICATION OF THIS PAGE (When Data Entered)

REPORT DOCUMENTATION PAGE		READ INSTRUCTIONS BEFORE COMPLETING FORM
1. REPORT NUMBER	2. GOVT ACCESSION NO.	3. REPORT'S CATALOG NUMBER
AD - A148708		
4. TITLE (and Subtitle) An Analytic Model of Gas Turbine Engine Installations		5. TYPE OF REPORT & PERIOD COVERED Master's Thesis; September 1984
		6. PERFORMING ORG. REPORT NUMBER
7. AUTHOR(s) Stephen M. Ezzell		8. CONTRACT OR GRANT NUMBER(s)
9. PERFORMING ORGANIZATION NAME AND ADDRESS Naval Postgraduate School Monterey, California 93943		10. PROGRAM ELEMENT, PROJECT, TASK AREA & WORK UNIT NUMBERS
11. CONTROLLING OFFICE NAME AND ADDRESS Naval Postgraduate School Monterey, California 93943		12. REPORT DATE September 1984
		13. NUMBER OF PAGES 224
14. MONITORING AGENCY NAME & ADDRESS (if different from Controlling Office)		15. SECURITY CLASS. (of this report) Unclassified
		15a. DECLASSIFICATION/DOWNGRADING SCHEDULE
16. DISTRIBUTION STATEMENT (of this Report) Approved for public release; distribution unlimited.		
17. DISTRIBUTION STATEMENT (of the abstract entered in Block 20, if different from Report)		
18. SUPPLEMENTARY NOTES		
19. KEY WORDS (Continue on reverse side if necessary and identify by block number) Gas turbine, ducting		
20. ABSTRACT (Continue on reverse side if necessary and identify by block number) An interactive computer simulation of marine gas turbine installations including intake and exhaust ducting for the engine and module cooling has been developed. A one-dimensional analysis		

DD FORM 1473
1 JAN 73

EDITION OF 1 NOV 68 IS OBSOLETE
S/N 0102-LF-014-6601

1
Unclassified
SECURITY CLASSIFICATION OF THIS PAGE (When Data Entered)

Unclassified

SECURITY CLASSIFICATION OF THIS PAGE (When Data Entered)

was used in determining the pressure losses of the ducting. The pressure losses along with the ambient conditions and desired power setting define a unique operating point for the system. The computer model predicts operating parameters for this point by an iterative matching technique.

Accession For	
NTIS GRA&I	<input checked="checked" type="checkbox"/>
DIC TAB	<input type="checkbox"/>
Unannounced	<input type="checkbox"/>
Justification	
By	
Date	
For Codes	
For	
1	



Approved for public release; distribution unlimited.

An Analytic Model
of
Gas Turbine Engine Installations

by

Stephen M. Ezzell
Lieutenant Commander, United States Navy
B.S., North Carolina State University, 1971

Submitted in partial fulfillment of the
requirements for the degree of

MASTER OF SCIENCE IN MECHANICAL ENGINEERING

from the

NAVAL POSTGRADUATE SCHOOL
September 1984

Author: _____

Approved by: _____

Thesis Advisor

Second Reader

Chairman, Department of Mechanical Engineering

Dean of Science and Engineering

ABSTRACT

An interactive computer simulation of marine gas turbine installations including intake and exhaust ducting for the engine and module cooling has been developed. A one-dimensional analysis was used in determining the pressure losses of the ducting. The pressure losses along with the ambient conditions and desired power setting define a unique operating point for the system. The computer model predicts operating parameters for this point by an iterative matching technique.

TABLE OF CONTENTS

I.	INTRODUCTION	10
II.	THEORY AND ANALYSIS	15
	A. GENERAL	15
	B. THE BERNOULLI EQUATION	15
	C. MODIFIED BERNOULLI EQUATION	16
	D. PRESSURE LOSSES	19
	E. GAS TURBINE/SYSTEM INTERFACE	21
	F. FAN/SYSTEM INTERFACE	23
	G. JUNCTIONS OR WYES	24
	H. EDUCTOR/SYSTEM INTERFACE	26
	I. SYSTEM ANALYSIS	27
	J. TOTAL PRESSURE GRADIENT	33
III.	PROGRAM PROCEDURES	36
	A. GENERAL	36
	B. INTERACTIVE CODE	37
	C. OTHER PROGRAM FEATURES	38
IV.	RESULTS AND RECOMMENDATIONS	40
	A. GENERAL	40
	B. LIMITATIONS	41
	C. RECOMMENDATIONS	42
	APPENDIX A: PROGRAM LISTING	44
	APPENDIX B: FLOW CHARTS	139
	APPENDIX C: USER'S MANUAL	173
	A. GENERAL	173
	B. PRELIMINARY	174

C.	EXECUTING THE PROGRAM	204
1.	IBM 3033 at NPS	204
2.	VAX-11 at NPS	204
D.	BUILDING A DUCT DATA FILE	205
E.	EDITING THE DUCT DATA FILE	214
F.	COMPUTING SYSTEM PERFORMANCE	217
G.	EXAMINING THE OUTPUT	219
	LIST OF REFERENCES	223
	INITIAL DISTRIBUTION LIST	224

LIST OF TABLES

I.	Fittings Available From Program Menu	22
II.	Node Designations	175

LIST OF FIGURES

1.1	Typical Shipboard Inlet and Exhaust Ducting . .	11
2.1	Typical K Values for Fittings	20
2.2	Fan/System Interface	25
2.3	Module Cooling Eductor Schematic	28
2.4	Module Eductor Performance	29
2.5	Eductor/System Interface	30
2.6	System Arrangements and Their Classification . .	32
2.7	Typical Duct Pressure Changes	35

LIST OF SYMBOLS

A	Area, ft ²
AC	Area, cooling flow passage
AM	Area, mixed flow passage
AP	Area, primary flow passage (exhaust)
a }	Duct cross section
b }	dimensions, ft
D	Diameter, ft
e	Absolute roughness factor, ft
f	Friction factor, dimensionless
g	Acceleration due to gravity, ft/sec ²
g _c	Gravitational constant, 32.174 ft-lbm/lbf-sec ²
L	Length, ft
p	Pressure, lbf/ft ²
P _t	Total pressure, lbf/ft ²
Δp _t	Change in total pressure, lbf/ft ²
P _S	Static pressure, lbf/ft ²
P _T	Total pressure, lbf/ft ²
P _V	Velocity pressure, lbf/ft ²
P ₀	Ambient pressure, lbf/ft ² or PSIA
P _B	Engine back pressure, lbf/ft ² or PSIA
Q	Volumetric flow rate, ft ³ /sec
Re	Reynolds Number, dimensionless
V, v	Velocity, ft/sec
W _C	Cooling mass flow rate, lbm/sec
W _B	Exhaust mass flow rate, lbm/sec
z	Potential height, ft
ρ	Density, lbm/ft ³

I. INTRODUCTION

The installation of gas turbine engines in a ship raises several problem areas in the design of the intake and exhaust ducting. The problems relate mainly with the large volume of combustion air required and the properties of the exhaust gases rejected to the atmosphere at high temperatures and velocity. For comparison, a boiler's combustion air requirement is nearly stoichiometric but the gas turbine operates at about 400 percent of stoichiometric. The boiler's exhaust is about 400 degrees F after leaving the last rows of the economizer, but gas turbine exhaust temperatures are frequently as high as 950 degrees F.

In addition to the air that passes through the gas turbine engine there is also a requirement to ventilate the engine enclosure. An adequate and uniformly distributed cooling airflow is required around the engine to maintain engine-mounted components at their proper operating temperatures and to minimize the heat rejected to the engine room thereby reducing the heat exposure of operating personnel. Many current designs branch the engine cooling airflow off the main intakes and/or join heated enclosure cooling air into the engine exhaust ducting. Figure 1.1 shows a typical layout of inlet and exhaust ducting. Since the enclosure cooling airflow is on the order of 20 percent of the engine's full power airflow rate, it is an important part of the ducting design.

The fundamental requirement of an intake design is to provide air to the engine compressor with the minimum total pressure loss and with a minimum of total pressure distortion. The loss of total pressure in the intakes leads to a loss of engine power and an increase in specific fuel

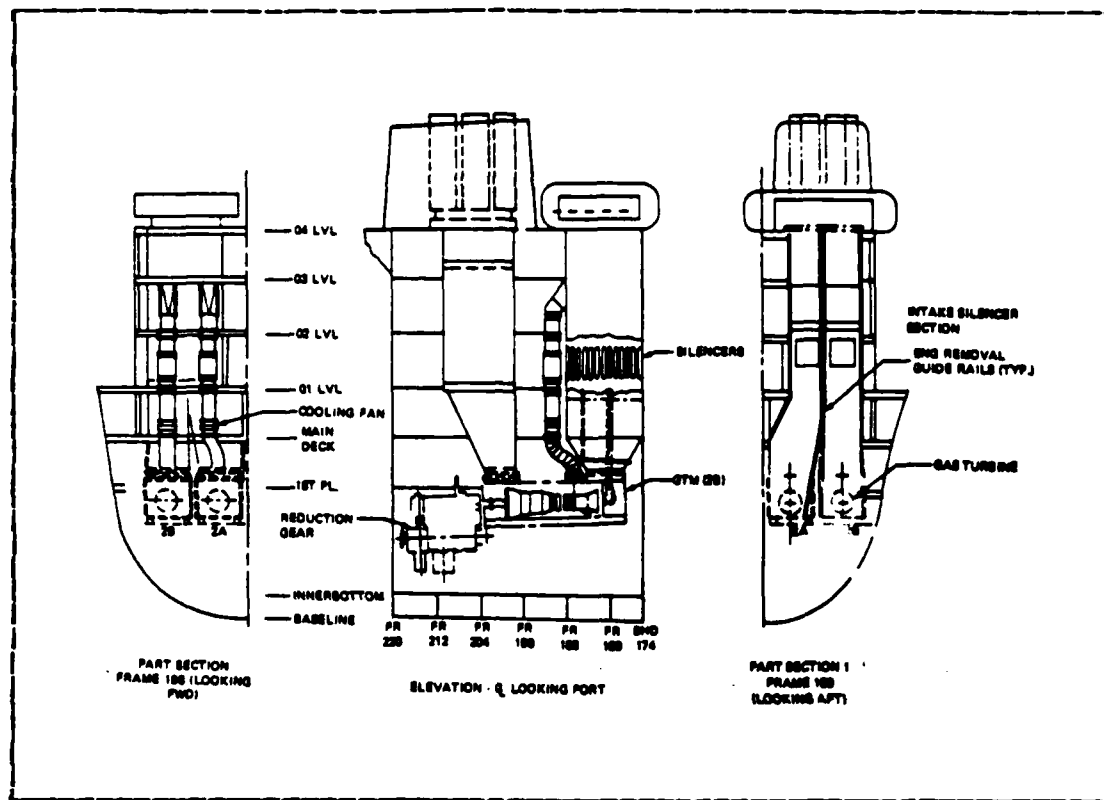


Figure 1.1 Typical Shipboard Inlet and Exhaust Ducting.

consumption. Schwieger reports "Typical exchange rates are that a one percent loss in intake pressure is equivalent to a 2.2 percent loss in power and a 1.2 percent increase in specific fuel consumption" [Ref. 1]. Additionally, total pressure distortion at the compressor face can lead to a risk of compressor blade failure.

Exhaust ducts must also operate with a minimum pressure loss. "The exchange rate is 1.1 percent loss in power and 1.1 percent increase in specific fuel consumption for the one percent increase in total pressure at the power turbine exit" [Ref. 1].

Conflicting with the design objective to reduce losses in the ducting system are several possible requirements to

install components in the ducting system which contribute to the losses but not directly to engine performance. Filters are installed to increase engine life. Silencers are installed to reduce noise. Machinery arrangements dictate the use of certain elbows, contractions, and transitions. The infrared signature of the ship's exhaust plume can be reduced by the installation of an eductor system at the exhaust exit. The eductor also improves the environment of mast mounted equipment and may contribute to flight safety when operating helicopters. Some systems use an eductor arrangement installed at the exhaust plane of the engine to pump cooling air through the engine enclosure. A waste heat recovery boiler may be installed in the exhaust to improve overall efficiency. To reduce pressure losses every attempt should be made to reduce the velocity in the duct. Lower velocities requires larger ducts. Part of the design compromise must balance the large volume of the ship occupied with inlet and exhaust ducts and the volume for other uses such as weapons and habitability. In summary there are many different components that can be utilized within the ducting system and have various effects on the system performance. The effects also vary with the operating point of the system.

It is not a straight forward problem to predict how components in the ducting system will perform. It is an interacting or matching type of problem. Furthermore, it is a dynamic problem as parameters affecting performance can vary over a wide range. For example, one power setting of the gas turbine requires a different mass flow rate of air than another. The variable mass flow rate through the ducting system creates a variable inlet and exhaust duct pressure loss. The variation in exhaust temperature affects the losses in the exhaust duct. Ultimately all losses affect the performance of the gas turbine engine.

One approach to the analysis of ducting system performance is to separate the problem into two areas of concern. The first area should deal with a one-dimensional analysis of the ducting system to determine how pressure losses affect engine performance and how the various components of the system contribute to the sum total of these losses. The second area should deal with the distortion of total pressure across any section of the duct. This area becomes a three-dimensional problem where interest is directed to performance not just at any section of the duct but to within that section to the variation of velocity across the cutting plane. The one-dimensional and three-dimensional areas of the analysis are of course related.

The relationship between the one-dimensional and the three-dimensional aspect of the problem is understood and is dealt with in an empirical manner. The method is to apply a correction factor to the loss developed in the one-dimensional analysis of a particular system component, based on the distortion of the flow assumed to be presented to the component. If the assumptions about flow distortion are made and are accurate much valuable information results from the one-dimensional analysis.

The three-dimensional analysis of a duct system is possible only for a very simple system and requires very large computer assets. It is current practice to deal with three-dimensional analysis of complex systems through model studies. One-dimensional analysis on the other hand is well suited for analysis on a computer.

It is the intent of this study to develop the methodology for a one-dimensional analysis of a gas turbine engine's inlet and exhaust ducting as might be installed on a ship. Then to implement the method in an interactive computer program which allows rapid input of the duct geometry, desired operating point and ambient conditions to

obtain an accurate estimate of performance. The designer can then decide to make changes to components to achieve design objectives and make those changes to the duct geometry through an editing routine and rerun the problem. Once the designer is satisfied with the one dimensional analysis a firm basis exists to provide a design for model studies.

II. THEORY AND ANALYSIS

A. GENERAL

A one dimensional analysis of the flow in duct sections utilizes the Bernoulli Equation modified to account for losses. The term one-dimensional is an adjective often applied to flow situations. The whole flow is considered to be one large streamtube with average velocity V at each cross section. Thus the one dimension is the location down the duct. Losses refers to the pressure loss caused by frictional stresses in the airflow boundary layer and by turbulence. A thorough understanding of these terms and concepts is required to convey the meaning of the results of the duct system analysis.

B. THE BERNOULLI EQUATION

The Bernoulli Equation is discussed in any basic text on fluid mechanics. It was developed to describe the flow work of an ideal incompressible fluid in steady flow through a streamtube. In words it states that the mechanical energy per unit mass along a streamline is conserved. The Bernoulli Equation is:

$$v^2/2g_c + p/\rho + (g/g_c)z = \text{constant.} \quad (\text{eqn 2.1})$$

It relates velocity, pressure, and potential height. The constant may have a different value for each streamline, but for the purposes of duct flow certain simplifying assumptions are valid which make the constant valid for any streamline. The assumptions are that the static pressure is constant at any point in a cross section of the duct. The

next assumption is that because the system uses gases, the effect of variation in potential height at a duct section is so small relative to the other terms that its effect is neglected. This assumption is extended further to include the change in elevation effect at any section relative to any other section.

Alternate forms of the Bernoulli Equation are obtained by multiplying through by either g_c/g or ρ . Of interest to gas flow and duct design is the form obtained by multiplying through by ρ . Applying the above assumptions the resulting equation is:

$$\rho v^2/2g_c + p = \text{constant} \quad (\text{eqn 2.2})$$

In this form the constant has units of foot-pound force/feet³ and expresses the energy per unit volume flow rate. It reduces to pound force/feet² or pressure. Each term in the expression is given a name. The velocity term is the velocity pressure, p is the static pressure, and the constant is the total pressure. In words, the total pressure at a point is the sum of the velocity pressure and the static pressure.

C. MODIFIED BERNOULLI EQUATION

Although equation 2.2 was derived for flow along a streamline of an ideal frictionless flow it can be extended to analyze flow through ducts in real systems by applying the First Law of Thermodynamics. A good development of the application of the First Law of Thermodynamics to pipe flow is found in [Ref. 2]. It results in the modified Bernoulli Equation (2.3). Equation (2.3) incorporates all the assumptions so far and includes the term Δp_t . The flow resistance in a system with a real fluid between stations 1 and 2 is represented by the total pressure loss, Δp_t .

$$\rho v_1^2/2g_c + p_1 = \rho v_2^2/2g_c + p_2 + \Delta p_t \quad (\text{eqn 2.3})$$

The velocity used in the modified Bernoulli Equation will be taken as the mean velocity and then this equation will be assumed valid for any streamline in the duct. Analytically this is not correct because there is a variation of velocity at a duct section from the walls to the center of the duct. The error introduced by this assumption is offset by two circumstances. First, with turbulent flow the velocity profile is nearly uniform which makes the mean velocity a good approximation of the velocity at any point in the cross section. Second, experimentally determined loss coefficients are utilized in computations and this coefficient is applied using the mean velocity. Then if the velocity profile in the system matches the profile of the experiment, the loss will be correctly computed using the mean velocity.

The computer program uses the mean velocity and computes it based on mass flow rates. The mean velocity is computed from the mass flow through a sectional area and the density of the fluid at the section using equation 2.4. Density is computed by the perfect gas law equation (2.5) and is a function of the absolute temperature of the gas and the static pressure of the gas.

$$V_{\text{mean}} = \frac{W}{\rho A} \quad (\text{eqn 2.4})$$

$$\rho = p/RT \quad (\text{eqn 2.5})$$

where p = static pressure
 R = gas constant
 T = absolute temperature

D. PRESSURE LOSSES

There are two types of fluid losses in the ducting system, frictional and dynamic losses. Frictional losses occur along the walls of the entire duct length and are due to fluid viscosity. Dynamic losses result from disturbing the flow such as a change of direction, contraction, or expansion.

The Darcy-Weisbach equation (2.6) calculates the friction loss for straight ducts.

$$\text{Darcy-Weisbach equation} \quad \Delta p_t = f (L/D) \frac{\rho V^2}{2g_c} \quad (\text{eqn 2.6})$$

where Δp_t = frictions loss
in terms of total pressure
 f = friction factor
 L = duct length
 D = duct diameter or
equivalent hydraulic diameter
 $\frac{\rho V^2}{2g_c}$ = velocity pressure

The friction factor, f , used in computing duct losses is taken from a correlation by Swamee and Jain presented in [Ref. 2].

$$f = \frac{0.25}{\left[\log \left(\frac{e}{3.7D} + \frac{5.74}{Re^{1/4}} \right) \right]^2} \quad 10^{-6} \leq \frac{e}{D} \leq 10^{-2} \quad 5000 \leq Re \leq 10^8 \quad (\text{eqn 2.7})$$

The absolute roughness factor, e , is taken to be 0.00015 feet for all air duct components. For rectangular straight duct sections the equivalent hydraulic diameter, D_e , is calculated by equation (2.8) presented in [Ref. 3].

Equations 2.6, 2.7, and 2.8 are utilized in the program for computing friction losses in the straight sections of the duct.

$$C_e = 1.30 \frac{(ab)^{0.625}}{(a+b)^{0.250}} \quad (\text{eqn 2.8})$$

Friction losses occur in all fittings not just in straight duct. There are two techniques to arrive at the friction losses in these other fittings. The decision about which technique to use depends on the whether the fitting is short or long. In short fittings friction is accounted for by measuring the connecting sections of straight duct to the center of the fitting. No attempt is made to include friction in the calculation of fluid resistance for a short fitting. Elbows are short fittings. For long fittings such as diffusers and contractions, friction is included in the computation of the flow resistance coefficient. Therefore, a connecting straight duct length should be measured to the center of an elbow or to the start or end of a diffuser or contraction.

Dynamic losses are sometimes called local or minor losses. In piping systems, losses due to the local disturbances of the flow are often called minor losses. In very long piping systems these losses are usually insignificant in comparison with the friction in the length considered. In the duct used for a gas turbine installation these so-called minor losses actually become major losses because of the short lengths usually encountered. Experimental results are almost always used to account for pressure losses through the duct fittings. Such information is usually given in the form of equation 2.9.

$$\Delta P_e = K \rho v^2 / 2g_c \quad (\text{eqn 2.9})$$

The coefficient K is given for the fitting in numerous handbooks. Figure 2.1 shows some typical representations of the information available.

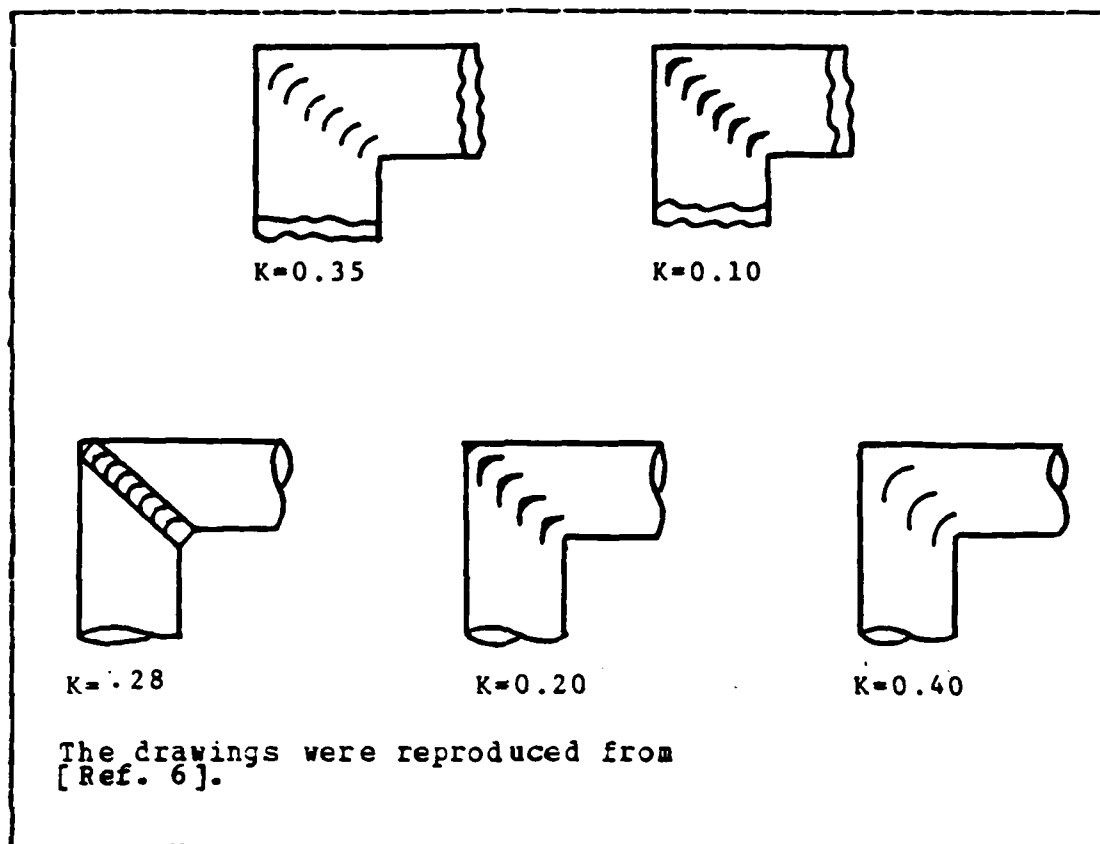


Figure 2.1 Typical K Values for Fittings.

One of the purposes of the program is to provide K coefficients for various fittings selected to represent duct components. K values can vary with the geometry of a fitting. For example, a long smooth radius rectangular elbow has a lower K value than a short smooth radius rectangular elbow. The program takes this into account and is the reason for the various questions about a fitting's geometry in the area of the program where the user is inputting the duct system.

Two fittings in the program's menu do not require geometry inputs to obtain resistance information. The two fittings are filters and the gas turbine module. The reason

for the lack of questions is that the losses are based on manufacturer's data. Filter manufacturers provide pressure loss data based on face velocity and the module is based on the mass flow rate of cooling air. A power curve fits the data and the program uses the curve to model pressure losses for these fittings.

Table I summarizes the fittings available from the program's menu. The fluid resistance coefficients are computed by the program upon input of the required geometry factors for the fitting. Input of the duct fittings is accomplished interactively. The source of the model for each fitting is noted in the program listing in the title block of the fitting subroutine. The program subroutines FIT01 through FIT29 correspond to the fittings listed in table I. A sketch of each fitting is provided in the user's manual for the program. The user's manual is Appendix C.

E. GAS TURBINE/SYSTEM INTERFACE

General Electric Company, the manufacturer of the LM2500 marine gas turbine, publishes performance data for its engine under variable operating conditions. [Ref. 4]. It is important to understand how the shipboard engine is operated under variable operating conditions such as duct losses and ambient temperature, pressure and humidity so that the proper corrections may be applied to the engine performance parameters for these variables.

TABLE I
Fittings Available From Program Menu

<u>Fitting Number</u>	<u>Description</u>
01	Intake shaft, rectangular cross section, side orifices, with or without louvers
02	Straight duct, round or rectangular
03	Smooth radius round elbow
04	Round 90 degree segmented elbow with 3, 4, or 5 pieces
05	Mitered round elbow with or without concentric vanes
06	Mitered rectangular elbow
07	Smooth radius rectangular elbow
08	Smooth radius rectangular elbow with splitters
09	Mitered rectangular elbow with vanes
10	Rectangular elbow with converging or diverging flow
11	90 degree rectangular elbows in a Z-shaped configuration
12	90 degree rectangular elbows in different planes
13	Branch section of a diverging wye
14	Main section of a diverging wye
15	Branch section of a convergent wye
16	Main section of a convergent wye
17	Conical round diffuser
18	Plane in-line diffuser
19	Pyramidal in-line diffuser
20	Transitional diffuser
21	Round contraction
22	Rectangular contraction
23	Screen obstruction in duct
24	Louver entrance
	continued next page

25	Filter element
26	Multi-baffle type silencer
27	Gas turbine module enclosure
28	Waste heat recovery boiler
29	Abrupt exit
30	Fitting not listed

From the shipboard operator's point of view the engine should drive the ship at the desired speed whether it is a hot day or a cold day, or if the inlet duct losses are four inches of water or eight. The engine is operating differently under such conditions to produce the same horsepower and speed. The proper correction factor set to be applied to the tabulated data is the set for constant speed and horsepower. The corrections are applied in the program with each iteration of the duct system performance calculations using the current values of the inlet and exhaust duct losses and ambient conditions. The corrections are very small (less than two percent) and the convergence of the correct engine operating point and duct losses created by the mass flow of air required at the operating point is quite stable.

F. FAN/SYSTEM INTERFACE

The operating point of the fan installed in a duct system is the point where the fan characteristic curve intersects the system characteristic curve. The fan curve shows pressure rise vs. flow rate. With increasing flow the pressure rise across the fan is reduced. The system curve is the opposite, increasing flow in the system increases the resistance to flow. Figure 2.2 represents this situation graphically.

In the iteration process the system curve is estimated as a quadratic fitted to the origin as a minimum point and the other point at the assumed flow and the resulting pressure loss. Similarly the fan curve is also represented as a quadratic with a maximum at maximum pressure attainable and the corresponding flow and another point at zero pressure and maximum flow. The representation of the fan performance for the default condition, the Spruance class destroyer module cooling fan, is excellent. With an equation for both curves the point of intersection can be obtained. The resulting flow is used in the next iteration until the resistance of the system and the pressure rise across the fan is the same for the assumed flow.

G. JUNCTIONS OR WYES

An excellent discussion of the mixing of two streams moving at different velocities was written by Idel'chik and is presented here to develop the background for the eductor/system interface discussion.

The junction of two parallel streams moving at different velocities is characterized by turbulent mixing of the streams, accompanied by pressure losses. In the course of this mixing an exchange of the momentum takes place between the particles moving at different velocities, finally resulting in the equilization of the velocity distributions in the common stream. The jet with higher velocity loses a part of its kinetic energy by transmitting it to the slower jet.

The loss in total pressure before and after mixing is always large and positive for the higher-velocity jet, and increases with an increase in the amount of energy transmitted to the lower velocity jet. Consequently, the resistance coefficient, which is defined as the ratio of the difference of total pressure to the mean dynamic pressure in the given section, will likewise always be positive. As to the lower-velocity jet, the energy stored in it increases as a result of mixing. The loss in total pressure and the resistance coefficient can, therefore, also have negative values for the lower-velocity jet [Ref. 5].

The program incorporates this concept at the junction of the module cooling air and the engine exhaust (if the system is so configured). The program assumes the lower velocity jet

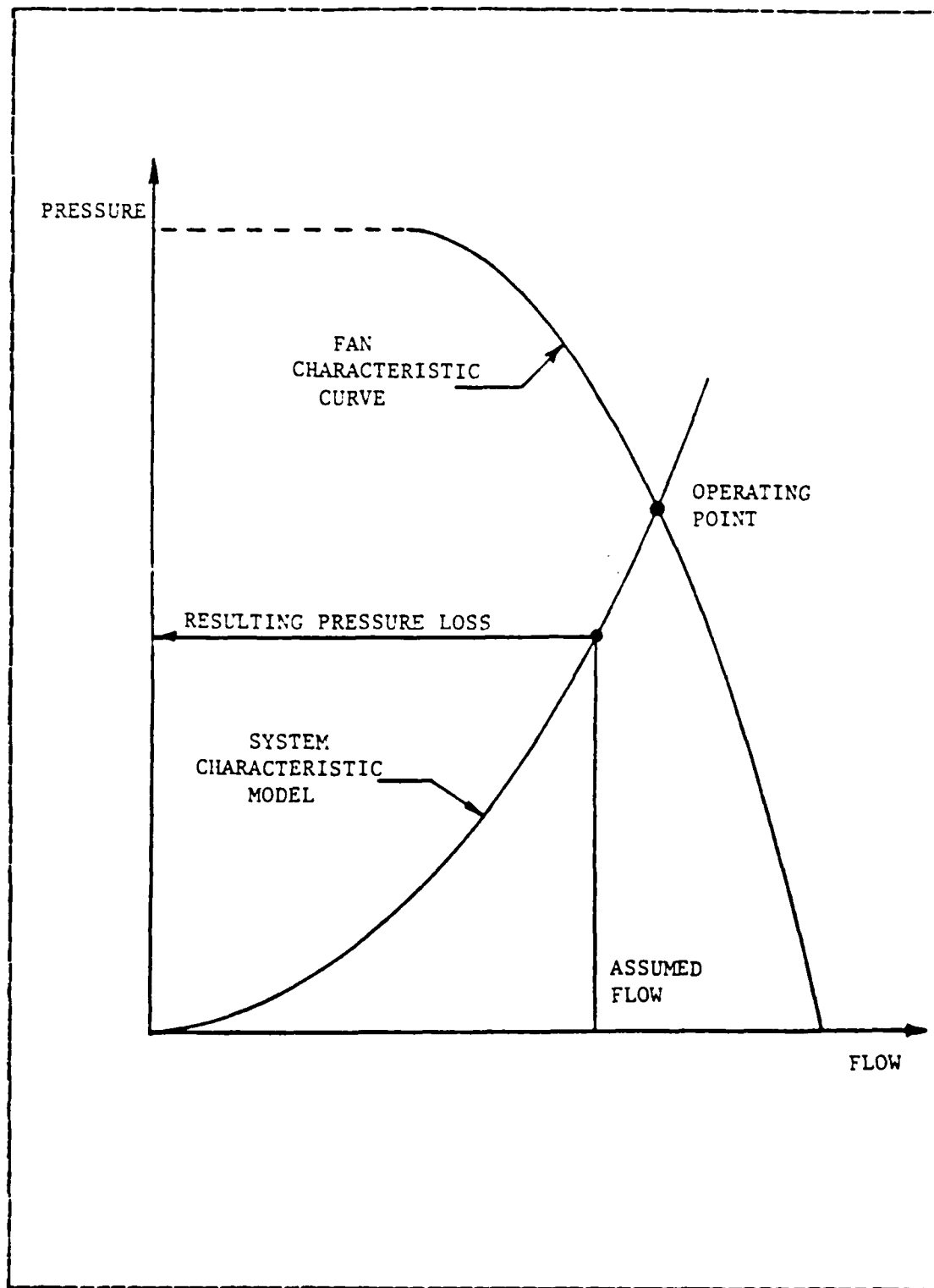


Figure 2.2 Fan/System Interface.

to be the cooling flow and the higher velocity jet to be the exhaust flow.

H. EDUCTOR/SYSTEM INTERFACE

The eductor discussed in this section is used in the engine's exhaust to move cooling air through the cooling ducting and engine enclosure. There is a mixing of the cooling flow and exhaust before it is discharged to the atmosphere. This section does not discuss the eductor installed at the exhaust duct exit. The only component of interest there is the nozzle as a dynamic loss. The effect of the external mixing tube is small and can be neglected.

The module cooling eductor is used on the Oliver Hazard Perry class frigate. It is shown schematically in figure 2.3. The eductor system is illustrated in figure 2.4. This figure shows the geometry and pressure distribution during the mixing of primary flow, engine exhaust, and the secondary flow, module cooling flow. A match point concept can be developed for the eductor much like the fan and system interface concept shown in figure 2.2. One curve is called the gain required and the other the gain available. These curves are shown in figure 2.5. Given the geometry of the mixing area the gain available can be computed by varying the cooling flow while the primary flow, the engine exhaust, remains nearly constant for the desired power setting. The gain available is a maximum at zero cooling flow.

The gain required is computed by dividing the system at the eductor and is analogous to the system characteristic model in figure 2.2. On the downstream side cooling and engine exhaust flows move through the exhaust duct. The cooling flow moves through the upstream duct. Total pressure losses can be computed for both and the sum is the gain required. Since these computations are taking place at

nearly constant primary flow, engine exhaust, the gain required at an operating point is a function of the cooling flow. The gain required at zero cooling flow is the exhaust duct pressure loss under the flow condition represented by the engine exhaust alone. Increasing the cooling flow increases the losses in the exhaust duct and also brings to bear losses in the cooling duct. Therefore the required gain is a minimum at zero cooling flow and increases with increasing cooling flow.

There must be an intersection of the gain required curve and the gain available curve if the system is to operate. This condition occurs if the gain available at zero cooling flow is greater than the gain required at zero cooling flow. The intersection must also be far enough to the right to provide the minimum cooling requirement for the load on the engine. The matching technique is to begin with some minimum cooling flow as specified by the engine manufacturer and march to the right adding a small increment to the cooling flow until gain required equals gain available.

I. SYSTEM ANALYSIS

Sections of the intake and exhaust ductwork will be analyzed from node to node resulting in the pressure loss for the section. The sections will be called branches. A node is the starting or ending point of a branch. The fittings of a branch will be entered into the program in the sequence encountered by the flow along a branch. A node is an entry, diverging wye, fan, the gas turbine engine (not to be confused with the engine enclosure), convergent wye, or an exit. Figure 2.6 shows the six resulting schematic representations of a gas turbine installation and the variations of cooling flow available. The numbered dots are the nodes. Node 1 is always the main inlet entrance. Node 3 is

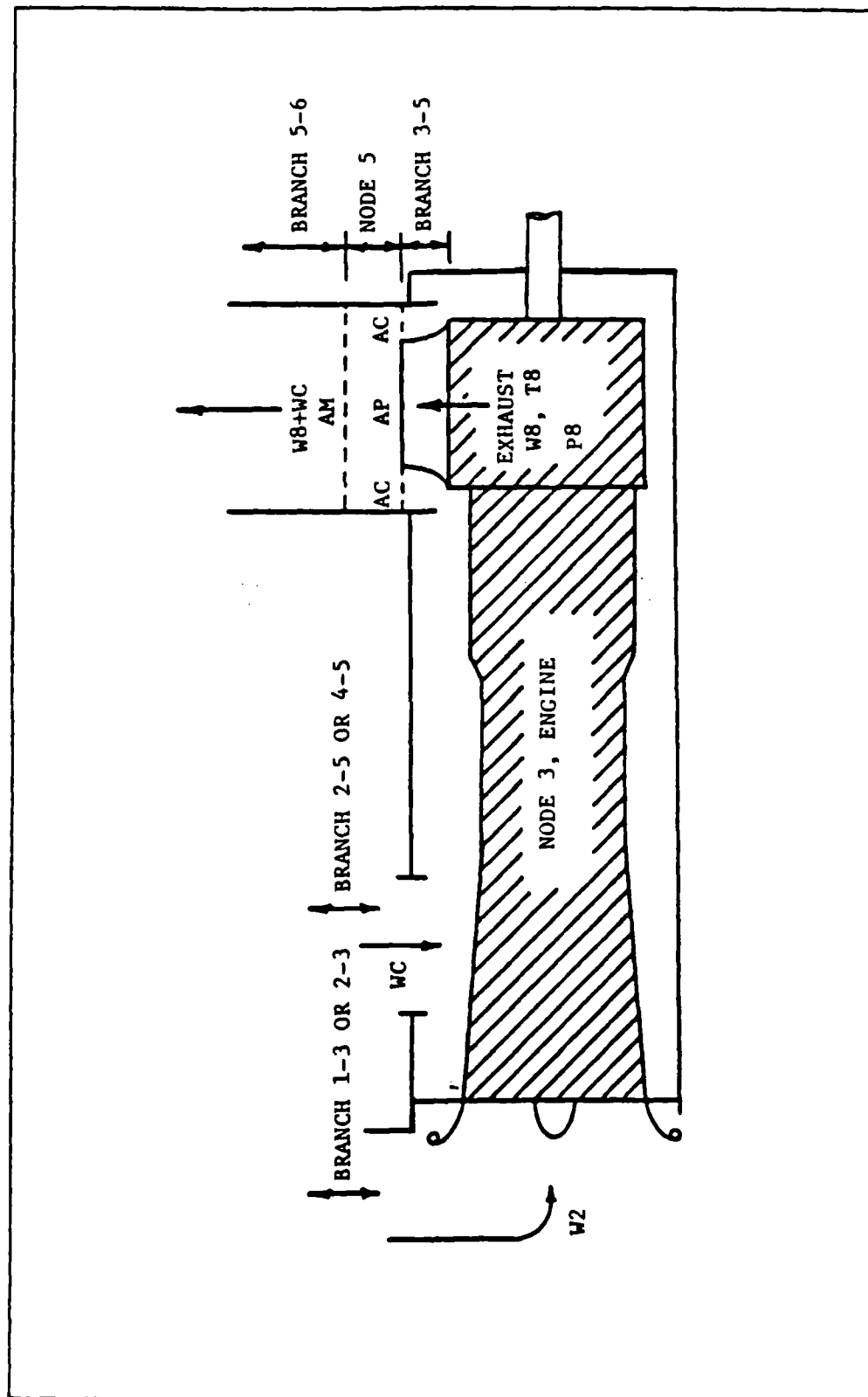


Figure 2.3 Module Cooling Eductor Schematic.

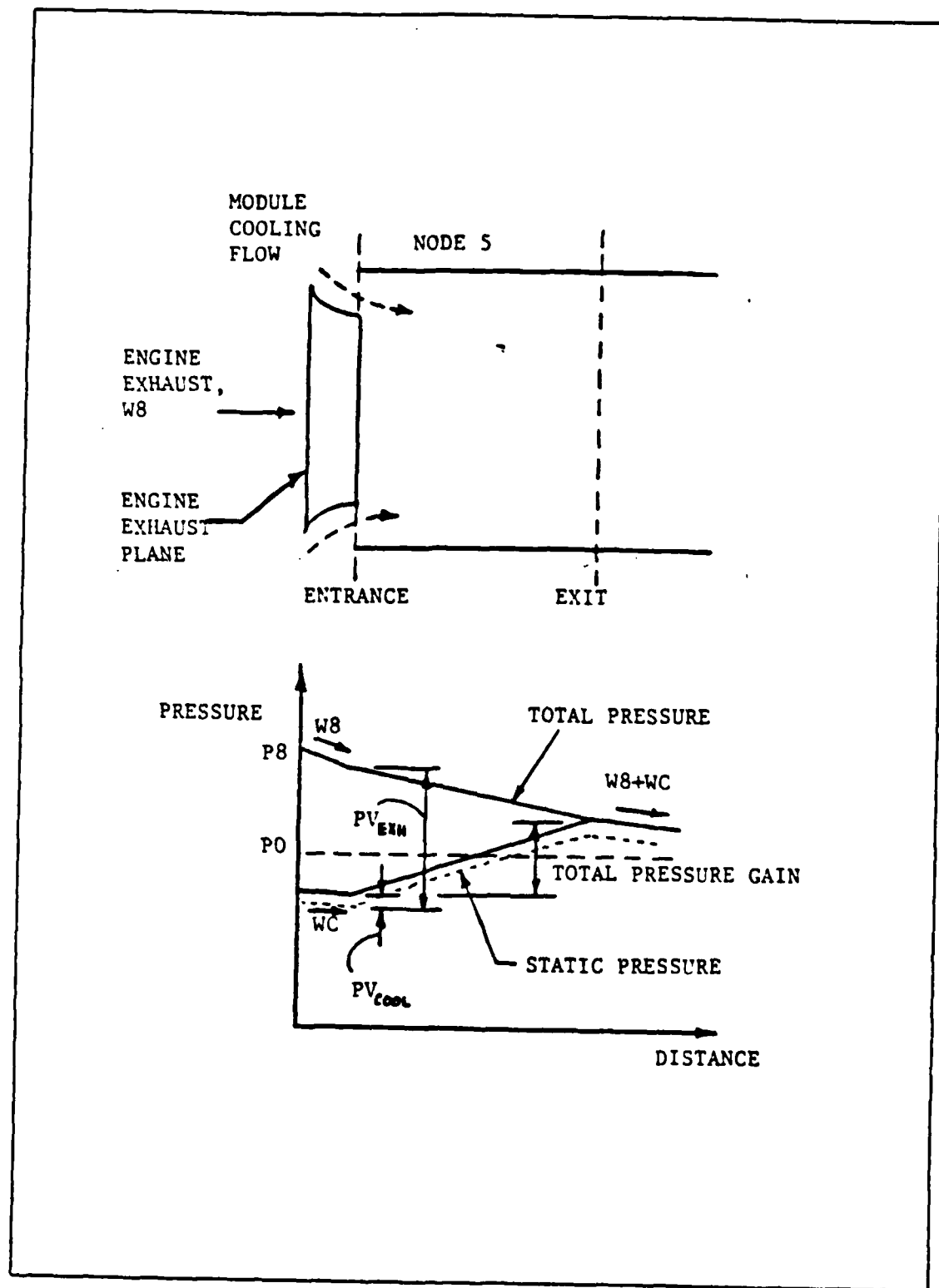


Figure 2.4 Module Eductor Performance.

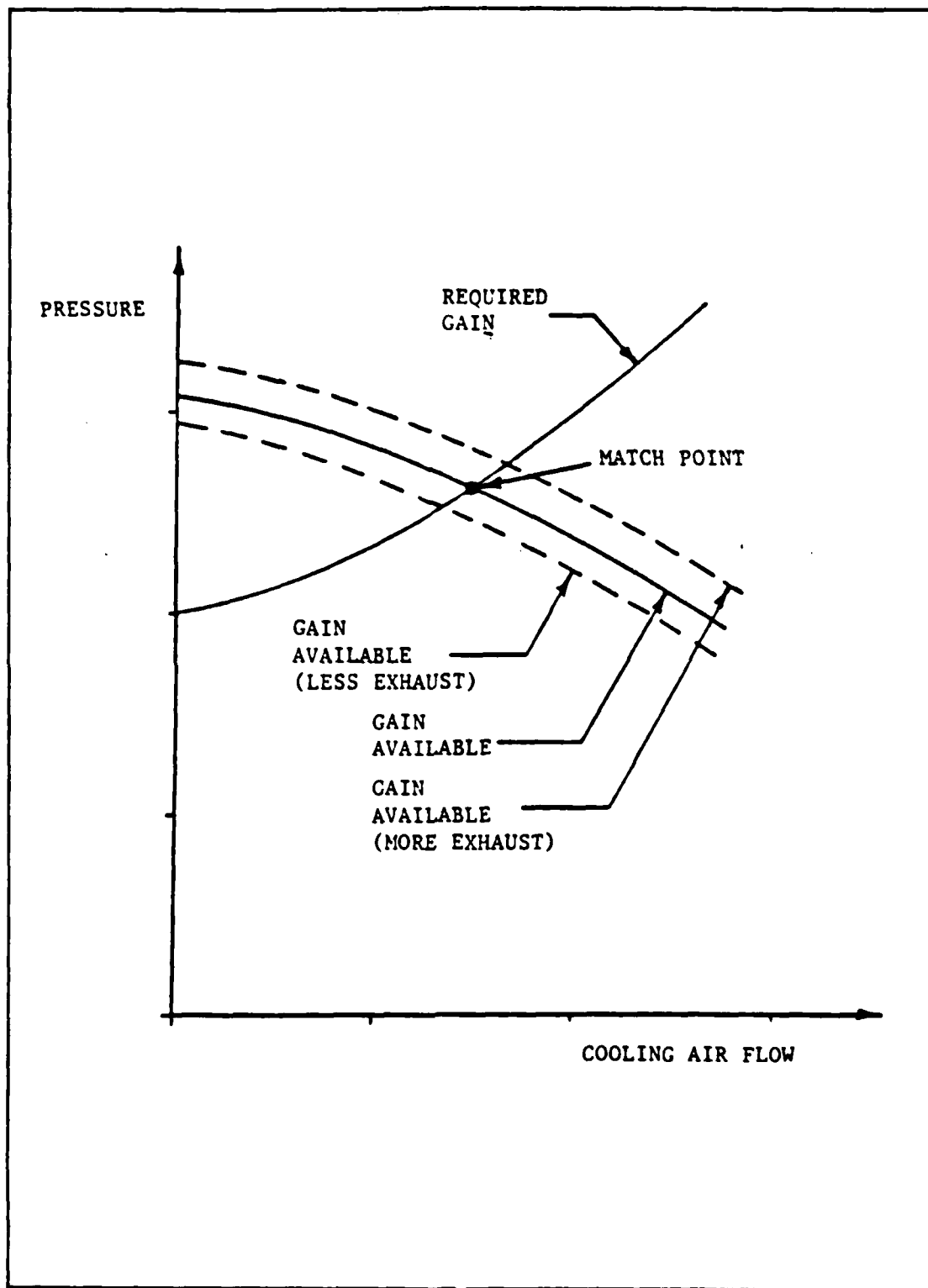


Figure 2.5 Eductor/System Interface.

always the engine. Node 4 is always the cooling fan. Node 6 is always the main exhaust exit. Node 2 may be either an independent entry for the cooling flow or the branch location where the cooling flow diverges from the combined inlet. Node 5 may be either an independent exit for the cooling flow or the junction of cooling flow with the engine exhaust. The hashed area is the engine and the larger rectangle represents the engine module which surrounds the engine and is a fitting in the cooling flow branch. The branches are designated by the node number at the beginning and end of the branch. The reader should refer to the user's manual for a complete description of entry of the fittings into the program.

The system in figure 1.1 would be a class three system. It has the cooling flow branching off the main inlet (divergent wye) and joining the main exhaust near the exhaust exit plane of the engine (convergent wye). It also has a fan installed which differentiates it from the class five system.

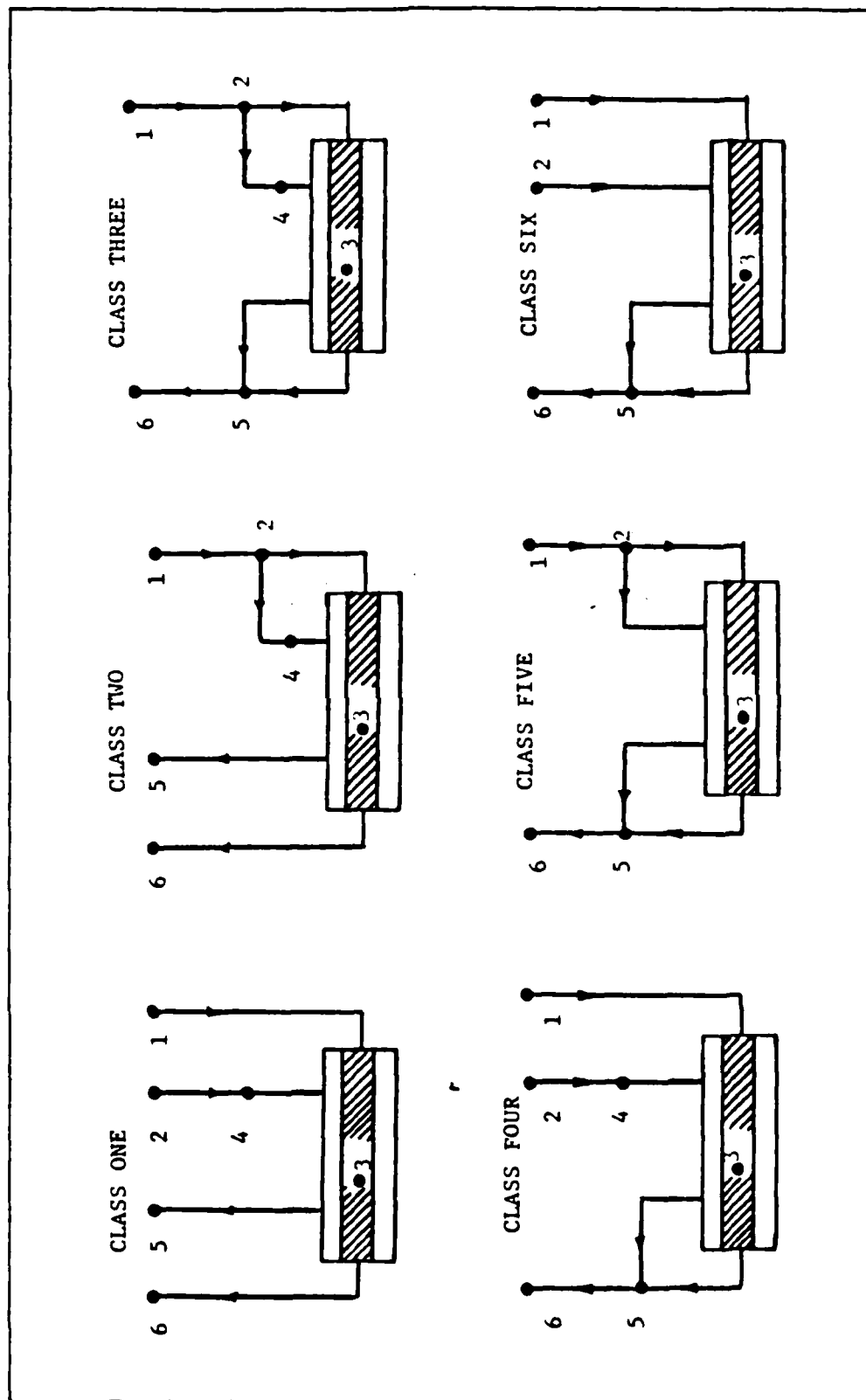


Figure 2.6 System Arrangements and Their Classification.

The basic procedure for system analysis is to assume enough flow and loss information to proceed with the analysis and check the assumptions with continuity of pressure at the nodes with each iteration. If the pressures do not match, new assumptions are made based on the current performance and the iteration is continued until convergence is achieved.

With six different types of systems to match, six different schemes must be implemented in the computer code to handle overall system matching. Each scheme must be tailored to handle the expected components that make it different from any other system. For example, system six has no cooling fan and system one does. System one needs to consider the fan and system interface but system six does not. Appendix A is the complete program listing. Appendix B contains a flow chart of the most complex system in the program, system three, and incorporates all possible component/system interfaces.

J. TOTAL PRESSURE GRADIENT

The total pressure changes represent the energy requirements of the system. Total pressure losses in the intake and exhaust ducts are inputs to the engine performance subroutine in the program and are used to determine the operating parameters of the engine. Fan and system matching is accomplished with the total pressure requirement. Therefore total pressure gradients in the ductwork are most important to analysis. Measurement on the other hand usually produces the static pressure gradient. The static pressure at a point is less than the total pressure at the point. Figure 2.7 shows a typical representation of the pressure changes during flow in a simple duct. Losses in a duct are due to the irreversible transformations of

mechanical energy into heat and the losses are used to plot the total pressure grade line. Note that some fittings such as diffusers and contractions cause a change in the static pressure quite different from the change in total pressure. This is a result of a change in the velocity pressure through a variable area fitting. The sample program output presented in the user's manual, appendix C, can be used to produce similar plots of the pressure grade line.

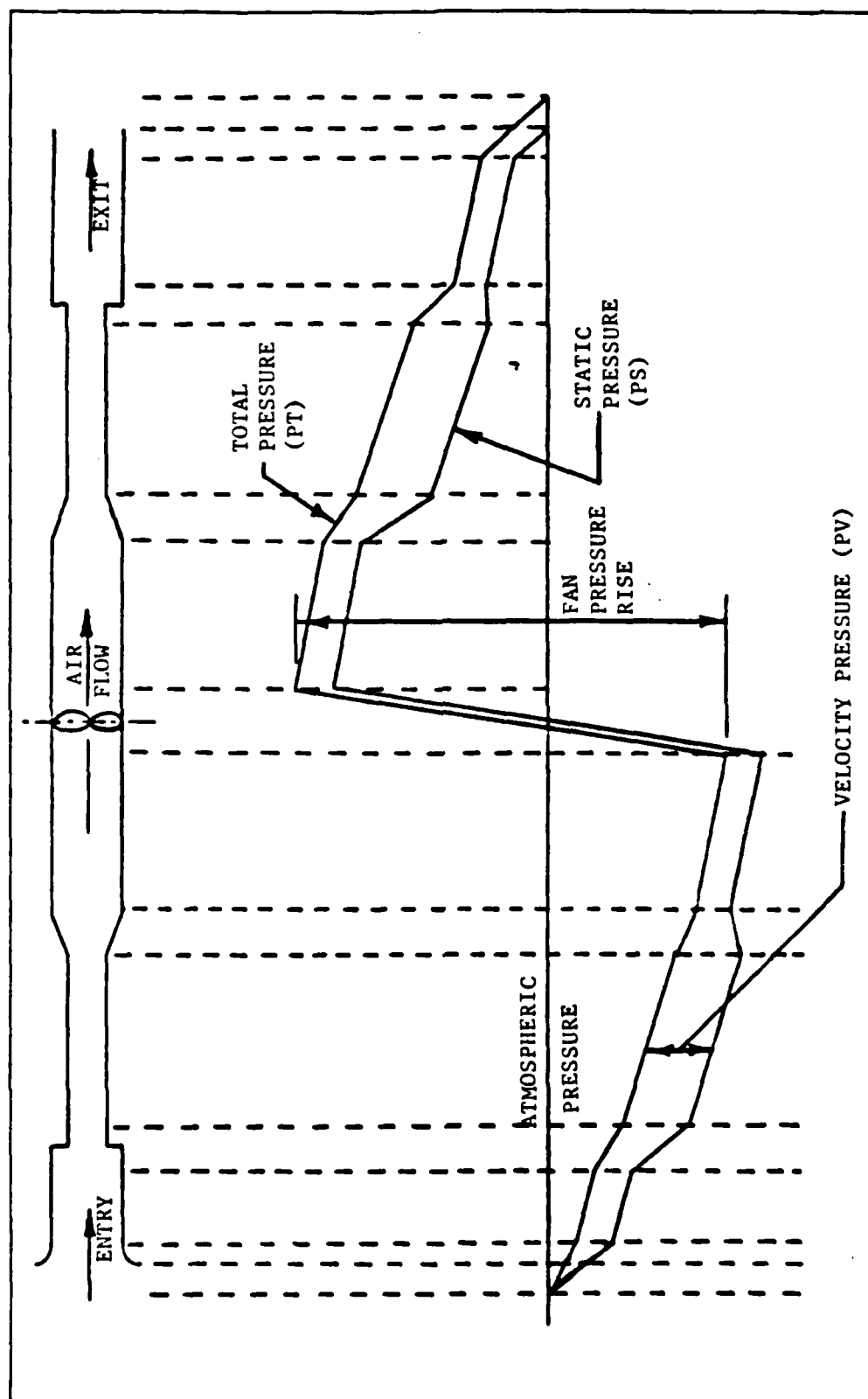


Figure 2.7 Typical Duct Pressure Changes.

III. PROGRAM PROCEDURES

A. GENERAL

The purpose of the program prepared for this study is to translate the geometry of a gas turbine installation including inlet, exhaust, and cooling ducting into a one-dimensional problem to calculate the system's frictional and dynamic resistance to air flow and solve the problem for various operating conditions. The solution will include engine performance parameters such as specific fuel consumption, turbine inlet temperature, and mass flow rates. Additionally a summary of the duct system performance is given by pressure losses for each component and a summary of branch losses. Cooling air flow is predicted by matching the system and the installed fan or module eductor.

Interactive code is utilized for all program inputs. Any number of fittings and combinations of fittings may be selected to represent the user's current design. The system in figure 1.1 can be represented by fittings chosen from the menu. About 30 selections from the menu would be required to model the system. The type and number of selections depends on the system's configuration and complexity. Each fitting may have from one to seven questions posed interactively to establish the required geometry inputs. With the geometry known the program computes areas and coefficients necessary to perform the analysis. This data is stored in a file called duct data and may be saved for future program runs where geometry input is not required. The operating point is defined upon input of ambient temperature and pressure, humidity, horsepower, and power turbine speed. When combined with the duct data file the problem may be solved.

B. INTERACTIVE CODE

Interactive code allows the user to sit at a computer terminal, access a desired program, specify inputs by typing at the terminal keyboard, and execute the program. All inputs are requested by statements appearing on the terminal screen. Resulting output is written to the user's files which may be viewed at the terminal or sent to the printer. The interactive mode of operation is especially valuable because it allows the user, by modifying selected input values, to quickly evaluate the effects of changes to an existing or contemplated design. Modification of a system is accomplished interactively within the editor portion of the program. The editor offers the ability to change a fitting. For example, a mitered round elbow could be modified to add cascaded turning vanes or a different elbow substituted entirely. Also offered is the ability to add or delete a fitting. The addition option does not allow the user to add a new first fitting to a branch, however one may be added anywhere else.

The most important consideration in writing an interactive computer program is what appears on the screen and how it appears. Requests for inputs are in English rather than engineering jargon. Units are all in the English system. All lengths are in feet, etc. All logical choices are accomplished by entry of one letter, the first letter of the choice. For example, "Y" is the reply for yes. All logical choice replies are indicated within parenthesis at the end of the question. Should the user not use one of the choices indicated, the question will be repeated until a proper response is given. Default values are available for many circumstances to minimize the input effort. A default is not available by simply depressing the return key. The user must elect default values by a logical choice. For example

the Hamilton Standard filter system installed on the Spruance class destroyer is available as a default for the filter fitting. The user selects this by answering affirmatively to a question asking if the user would like to use the default filter system.

C. OTHER PROGRAM FEATURES

Another consideration in interactive computer programs is the practice of "user proofing" the inputs. In other words, an interactive computer program should not terminate execution (i.e., "crash") if an improper input value is inadvertently defined by the user. On numerical and logical input two features are incorporated to protect input to the program. First, read statements are protected with error and end of file detection. A problem with input here is handled by asking the user to re-enter the value. On numerical input if it happens again on the same question the program stops execution. Secondly, if an incorrect number is properly defined to the program in the geometry input phase, the user is offered one last chance to re-enter correct fitting data if the user realizes his mistake before he is asked if he wants to load the data for the fitting. The user is assisted here by a check for area continuity from one fitting to the next. A warning is provided if continuity is not maintained. Electing not to load a fitting brings the user back to the menu with the program ready to accept a choice of fittings for use instead of the erroneously entered fitting.

The program is modularized by the extensive use of subroutines. Modularization facilitates program improvements by allowing the upgrade and replacement of individual subroutines. This is a difficult procedure to do if common blocks are used. Therefore common blocks have been

eliminated from the program. The user may decide to change the fittings available in the menu, for example. Internal code documentation shows the areas that must be changed to accomplish this task.

Appendix C is a user's manual and completes the external program documentation. The manual explains how to execute the program as installed on the Naval Postgraduate School's IBM 3033 main frame computer and a smaller VAX computer. A sample case is described and sample output provided. A terminal session is also recorded to show typical screen displays.

IV. RESULTS AND RECOMMENDATIONS

A. GENERAL

It is now possible to analyze system performance of an ordinary marine gas turbine installation. Prior to the development of this program subsections of the system were analyzed and their interaction was neglected. This did not provide serious errors in the estimation of engine performance but it did not provide complete information on system performance. In particular, the prediction of cooling flow was not accurate. This was particularly acute when the system utilized a module eductor.

The process of manually assigning a resistance coefficient to a fitting has been eliminated. Now it is possible for the computer program to analyze the geometry of most fittings rapidly and apply the correct resistance coefficients for the one-dimensional analysis without the user looking up any correlations.

The program flexibility is demonstrated by the ability to quickly change input parameters and analyze a system at any operating point. Previous methods analyzed components at full power and then used a proportionality model where losses were proportional to the square of the engine air mass flow rate. This method consistently under-estimates duct losses at low power because it does not take into account the variation of cooling flow provided with an installed fan or module eductor. At low power the cooling flow can be a significant contributor to duct losses and the previous method can not predict this contribution.

B. LIMITATIONS

It should be emphasized that any one-dimensional analysis does not handle flow distortion well. Suspected problems in this area are still best dealt with by the use of model studies. The limitation of a one-dimensional model is that a fitting's pressure loss may be known for uniform flow distribution, but it is difficult to predict the loss with distorted flow. It is known however that the distorted flow situation will have a larger pressure loss, but how much is not easily determined. A one-dimensional analysis may point to problems with flow distortion. The program recognizes the potential for flow distortion on certain fittings such as diffusers and points out this potential. If a fitting's pressure loss can vary significantly with distortion of flow and the one-dimensional analysis has computed a large pressure loss, the user should flag the fitting for further study by model testing as the pressure loss has probably been underestimated.

Not all possible duct designs can have their fittings modeled by the program. Some fittings will be available from the program menu and others will be similar to fittings listed, but not exactly. Then there are some which may not be listed at all. If the fitting is close, it may be used and expected to give reasonable results. If the fitting is not listed then the user must provide the resistance coefficient by using the "fitting not listed" choice. The data for this entry may come from a published correlation or from tests performed on similar installations. It is in the area of correlations where most benefit can be gained by program modification.

C. RECOMMENDATIONS

The program currently runs as a stand alone program, but some increased utility may be realized by incorporating some of the subroutines in other programs which would then input a ship's horsepower and RPM requirements for an operating profile instead of point by point user input.

The General Electric LM2500 engine is currently the engine within the program. The engine performance in the program is built by table interpolation of the published performance data. General Electric also offers a program which provides performance data and it is recommended that this program be substituted for the engine subroutine currently in the program. This will eliminate any doubts about engine performance predictions and make the parameters more official. Also the General Electric program covers the complete performance map of the engine whereas the engine subroutine used in this analysis was limited to 22,500 horsepower maximum. There is still a little power left beyond this value and the program can not currently operate there. Another modification concerning the engine is improving the module temperature out model used in the FIIDP subroutine. The model used produces reasonable results but is not based on test data but on operator experience.

The biggest improvement in program performance and utility can be made by the incorporation of improved fitting flow resistance correlations of test data. Models and full scale systems should be instrumented to provide duct pressure loss data to check the program's analysis. Where the program prediction is not accurate new fitting correlations should be developed. Potential fittings for improved models are louvers, silencers, diffusers with distorted flow, junctions and wyes (especially where eductor action is desired), and boiler tube bundles. With sufficient data these

fittings could be modeled better and more simply. The overall objective is to increase both the utility and accuracy of the program analysis.

APPENDIX A

PROGRAM LISTING

```

*****
ANALYTIC MODEL OF A GAS TURBINE INSTALLATION ON BOARD A SHIP
PROGRAM WRITTEN BY STEPHEN M. EZZELL, LCDR, USN
VERSION 1.0 DATE MARCH 30, 1984
PURPOSE: TO ANALYZE THE DUCTING AND GAS TURBINE INSTALLATION
          AS MIGHT BE INSTALLED ON A SHIP. INPUT DUCT GEOMETRY,
          AMBIENT CONDITIONS, AND POWER SETTING TO GET PERFORMANCE
          PARAMETERS.
*****
THIS IS THE MAIN CONTROL PROGRAM. ITS SOLE PURPOSE IS TO BRANCH
TO THE AREA OF THE PROGRAM YOU NEED. IF YOU ARE ANALYZING A NEW
SYSTEM YOU WILL NEED TO BUILD A DATA FILE FOR THE SYSTEM. YOU
WILL BE DIRECTED TO THE BUILD SUBROUTINE. IF YOU WANT TO MAKE
SOME CHANGES TO A SYSTEM YOU WILL GO TO THE EDIT SUBROUTINE.
WHEN YOU HAVE A DATA FILE YOU LIKE YOU WILL NEED TO GO TO
THE COMPUTE SUBROUTINE. IN THE COMPUTE SUBROUTINE YOUR DATA FILE
WILL BE READ AND THEN YOU WILL BE ASKED QUESTIONS TO ESTABLISH
THE OPERATING POINT. THEN THE PROGRAM WILL COMPUTE THE OPERATING
PARAMETERS YOU NEED AND OUTPUT THEM TO THE OUTPUT FILE.
NC COMPUTATIONS ARE DONE IN THE MAIN CONTROL PROGRAM.
SUBROUTINES CALLED: BUILD, EDIT, COMPUT, AND FRCHMS
A NOTE ABOUT FRCHMS. YOU WILL NOT FIND IT IN THE LISTING. IT IS
LIBRARY SUBROUTINE AVAILABLE AT NPS AND IS USED TO CALL THE
OPERATING SYSTEM FROM WITHIN THE FORTRAN PROGRAM. I USE IT FOR
TWO PURPOSES. FIRST TO DEFINE MY FILES. SECOND TO CLEAR THE
SCREEN AT YOUR TERMINAL SO THE WRITE FORMATS DON'T GET CHOPPED
UP. IF YOUR SYSTEM DOES NOT HAVE THIS CAPABILITY YOU WILL HAVE
TO SUBSTITUTE AN APPROPRIATE CODE TO ACCOMPLISH THE SAME THINGS.
THIS NOTE APPLIES TO THE IBM 3033 COMPUTER.
*****
      INTEGER ANS,YES,NO,COMPUT,EDIT,QUIT
      DATA YES/'Y',NC/'N',COMPUT/'C',EDIT/'E',QUIT/'Q'/

NPS IBM 3033 MAIN FRAME COMPUTER PROGRAM REQUIREMENTS

HERE IS WHERE I SET UP THE FILE DEFINITIONS USING THE LIBRARY
SUBROUTINE "FRCHMS". THERE ARE NO OTHER FILEDEF'S REQUIRED.

      READING TERMINAL INPUT
      CALL FRCHMS ('FILEDEF','05','TERMINAL')
      WRITING TO THE TERMINAL
      CALL FRCHMS ('FILEDEF','06','TERMINAL')
      STORAGE FILE FOR THE DUCT GEOMETRY DEPENDENT VARIABLES
      CALL FRCHMS ('FILEDEF','08','DISK','DUCT')
      *
      *
      *
      STORAGE FILE FOR THE PERFORMANCE DATA OUTPUT
      CALL FRCHMS ('FILEDEF','04','DISK','OUTPUT')
      *
      *
      *
      CALL FRCHMS ('CLRSCRN')
      INTRODUCTION. IS THERE A DUCT DATA FILE ???
      WRITE (6,600)

      EVERY READ IS PROTECTED AGAINST A NULL ENTRY AND AN ERROR IN
      INPUT. THIS IS ACCOMPLISHED WITH "END=XX,ERR=XX". YOUR SYSTEM
      MAY NOT HAVE THIS CAPABILITY, IN WHICH CASE DELETE IT OR
      SUBSTITUTE AND EQUIVALENT CODE.

      READ (5,601,END=12,ERR=12) ANS
      CALL FRCHMS ('CLRSCRN')

```



```

603 *   FOR A PROPER ANSWER !!!!!!!)
      FORMAT (/, 'DO YOU WANT TO EDIT THE FILE OR USE IT FOR COMPUTATION
604 * (E/C)?')
      FORMAT (' DO YOU WANT TO COMPUTE WITH THE FILE OR QUIT (C/Q)?')
605 FORMAT (' DO YOU WANT TO EDIT THE DUCT DATA FILE OR QUIT (E/Q)?')
      STOP
      END

```



```

4  GEOM(1)=413101
   GEOM(2)=424001
   GEOM(3)=435101
   GEOM(4)=445001
   GEOM(5)=456201
   BRANCH=5
   CALL FRTCMS ('CLRSCRN ')
   WRITE (6,603)
   GO TO 10
5  GEOM(1)=512201
   GEOM(2)=523101
   GEOM(3)=5345001
   GEOM(4)=545101
   GEOM(5)=556201
   BRANCH=5
   CALL FRTCMS ('CLRSCRN ')
   WRITE (6,604)
   GO TO 10
6  GEOM(1)=613101
   GEOM(2)=625001
   GEOM(3)=635101
   GEOM(4)=656201
   BRANCH=4
   CALL FRTCMS ('CLRSCRN ')
   WRITE (6,605)
10  CONTINUE
   M=0
   WRITE (6,606)

C  READI IS AN INTEGER READ SUBROUTINE TO PROTECT THE PROGRAM FROM
C  CRASHING ON NULL INPUT OR ERROR INPUT. IT ALSO ALLWS FREE
C  FORMAT INPUT.
C
C  CALL READI(DUMMY,5)
C  CALL FRTCMS('CLRSCRN ')
C
C  NOW EACH BRANCH WILL BE FILLED UP WITH THE FITTINGS. BRANCHES
C  ARE TAKEN IN NUMERICALLY ASCENDING ORDER.
C
C  DO 40 I=1, BRANCH
20  CALL MENU (I,TERM,TYPE,GEOM(I))
C  THE MENU CHOICES ARE 0 THRU 30, CHANGE THE NUMBER OF FITTINGS
C  AND YOU MUST CHANGE THE FOLLOWING IF CONDITION ACCORDINGLY
C  IF ((TYPE.EQ.0).AND.(TYPE.LT.31)) GO TO 30
C  CALL FRTCMS('CLRSCRN ')
C  WRITE(6,607)
C  GO TO 20
C 30  ZERO MEANS NO MORE FITTINGS THIS BRANCH
C  IF (TYPE.EQ.0) GO TO 40
C  M=M+1
C
C  A FITTING HAS BEEN SELECTED, NOW GO TO THE BRANCHING SUBROUTINE
C  TO ENTER THE FITTING.
C
C  CALL SELECT (M,SORI,GEOM(I),TYPE,WORKI,WORKR)
C  CALL FRTCMS ('CLRSCRN ')
C  GEOM(I)=GEOM(I)+1
C  GO TO 20
40  CONTINUE
C
C  ALL THE FITTINGS HAVE BEEN ENTERED AND THE DATA FILE IS ABOUT
C  TO BE WRITTEN.
C  CALL SUMOUT(WORKI,WORKR,M)
600  FORMAT(' SYSTEM IS CLASS ONE, SEPARATE ENGINE/COOLING FLOWS. '//
*  YOU WILL BE ENTERING FITTINGS FOR FOUR BRANCHES. '//
*  1. ENGINE INLET TO THE ENGINE. '//
*  2. COOLING INLET TO THE COOLING FAN. '//
*  3. ENGINE EXHAUST TO THE ATMOSPHERE. '//
*  4. COOLING FAN EXHAUST TO THE ATMOSPHERE, VIA GT MODULE.')

```

```

601  FORMAT(' SYSTEM IS CLASS TWO, COMBINED INLET FOR ENGINE AND //
      COOLING FLOW AND SEPARATE FLOWS FOR ENGINE EXHAUST AND MODULE //
      COOLING HOT EXHAUST. YOU WILL BE ENTERING FITTINGS FOR FIVE //
      BRANCHES. //
      1. COMBINED INLET TO THE COMBINED SECTION OF A DIVERGENT WYE
      2. MAIN SECTION OF THE DIVERGENT WYE TO THE ENGINE. //
      3. BRANCH SECTION OF THE DIVERGENT WYE TO THE COOLING FAN. //
      4. ENGINE EXHAUST TO ATMOSPHERE. //
      5. COOLING FAN EXHAUST TO THE ATMOSPHERE VIA GT MODULE. //
602  FORMAT(' SYSTEM IS CLASS THREE, COMBINED INLETS AND EXHAUST //
      FLOWS FOR THE ENGINE AND MODULE COOLING. A COOLING FAN IS //
      INSTALLED. YOU WILL BE ENTERING FITTINGS FOR SIX BRANCHES. //
      1. COMBINED INLET TO THE COMBINED SECTION OF A DIVERGENT WYE
      2. MAIN SECTION OF THE DIVERGENT WYE TO THE ENGINE. //
      3. BRANCH SECTION OF THE DIVERGENT WYE TO THE COOLING FAN. //
      4. ENGINE EXHAUST TO MAIN SECTION OF A CONVERGENT WYE. //
      AN EDUCTOR INSTALLED AT THE EXHAUST PLANE OF THE ENGINE //
      IS CONSIDERED TO BE A CONTRACTION FOLLOWED BY THE MAIN //
      SECTION OF A CONVERGENT WYE FOR THE PURPOSES OF THIS //
      PROGRAM. //
      5. COOLING FAN EXHAUST TO THE BRANCH SECTION OF A CONVERGENT
      WYE. //
      6. COMBINED SECTION OF A CONVERGENT WYE TO THE ATMOSPHERE. //
603  FORMAT(' SYSTEM IS CLASS FOUR, SEPARATE INLETS FOR THE ENGINE //
      AND COOLING FLOWS, COMBINED FLOWS FOR THE ENGINE EXHAUST AND //
      HOT MODULE COOLING. A COOLING FAN IS INSTALLED. //
      ENTER FITTINGS FOR FIVE BRANCHES. //
      1. ENGINE INLET TO THE ENGINE. //
      2. COOLING INLET TO THE COOLING FAN. //
      3. ENGINE EXHAUST TO MAIN SECTION OF A CONVERGENT WYE. //
      AN EDUCTOR INSTALLED AT THE EXHAUST PLANE OF THE ENGINE //
      IS CONSIDERED TO BE A CONTRACTION FOLLOWED BY THE MAIN //
      SECTION OF A CONVERGENT WYE FOR THE PURPOSES OF THIS //
      PROGRAM. //
      4. COOLING FAN EXHAUST TO THE BRANCH SECTION OF A CONVERGENT
      WYE. //
      5. COMBINED SECTION OF A CONVERGENT WYE TO THE ATMOSPHERE. //
604  FORMAT(' SYSTEM IS CLASS FIVE, COMBINED INLET AND EXHAUST FLOW. //
      AN EDUCTOR SYSTEM IS USED TO PUMP COOLING AIR. //
      ENTER FITTINGS FOR FIVE BRANCHES. //
      1. COMBINED INLET TO THE COMBINED SECTION OF A DIVERGENT WYE
      2. MAIN SECTION OF THE DIVERGENT WYE TO THE ENGINE. //
      3. THE EDUCTOR ONLY. THIS PROGRAM CONSIDERS THIS BRANCH TO //
      CONSIST OF ONLY TWO COMPONENTS, A CONTRACTION AND THE //
      MAIN SECTION OF A CONVERGENT WYE INSTALLED AT THE EXHAUST
      PLANE OF THE ENGINE. //
      4. BRANCH SECTION OF A DIVERGENT WYE VIA THE GT MODULE TO //
      THE EDUCTOR. THE PROGRAM CONSIDERS THIS PART OF THE //
      EDUCTOR TO BE THE BRANCH SECTION OF A CONVERGENT WYE. //
      5. COMBINED SECTION OF A CONVERGENT WYE TO THE ATMOSPHERE. //
      INSTALLATION OF A WASTE HEAT BOILER IS NOT RECOMMENDED. //
605  FORMAT(' SYSTEM IS CLASS SIX, SEPARATE INLETS FOR THE ENGINE //
      AND COOLING FLOWS, COMBINED FLOWS FOR THE ENGINE EXHAUST AND //
      HOT MODULE COOLING. AN EDUCTOR IS INSTALLED. //
      ENTER FITTINGS FOR FOUR BRANCHES. //
      1. ENGINE INLET TO THE ENGINE. //
      2. COOLING INLET TO THE EDUCTOR VIA THE GT MODULE. //
      THE PROGRAM CONSIDERS THIS PART OF THE EDUCTOR TO BE //
      THE BRANCH SECTION OF A CONVERGENT WYE. //
      3. THE EDUCTOR ONLY. THIS PROGRAM CONSIDERS THIS BRANCH TO //
      CONSIST OF ONLY TWO COMPONENTS, A CONTRACTION AND THE //
      MAIN SECTION OF A CONVERGENT WYE INSTALLED AT THE EXHAUST
      PLANE OF THE ENGINE. //
      4. THE COMBINED SECTION OF A CONVERGENT WYE TO THE ATMOSPHERE
      WYE. //
606  FORMAT(' /// ENTER ZERO TO CONTINUE. //
607  PCORNA: YOU DID NOT ENTER A CORRECT FITTING ID NUMBER. //
      RETURN
      END

```

```

C*****
C***** EDITING SUBROUTINE: USED TO ALTER THE DUCT DATA FILE *****C
C*****
C***** WITH THIS PART OF THE PROGRAM YOU CAN CHANGE, DELETE, OR ADD A
C***** FITTING TO THE DATA FILE. IT WOULD BE HANDY TO HAVE A COPY OF
C***** IT WITH YOU WHEN YOU MAKE THE CHANGES. ALSO THE DATA FILE IS
C***** IS PERMANENTLY CHANGED, TO SAVE A COPY, MAKE A COPY OF IT UNDER
C***** A DIFFERENT FILE NAME. YOU STILL MUST HAVE A FILE NAMED "DUCT
C***** DATA" TO EDIT. EACH DUCT DATA FILE IS SERIALIZED BY THE USER
C***** AND A NEW SERIAL NUMBER CAN BE ASSIGNED TO THE CHANGED FILE.
C***** THE SERIAL NUMBER APPEARS OF THE COMPUTED OUTPUT FILE OF SYSTEM
C***** PERFORMANCE. CHANGES ARE MADE BY THE INDEX NUMBER OF THE FITTING
C***** IN THE DUCT DATA FILE. THE INDEX NUMBER IS THE NUMBER IN THE
C***** FIRST COLUMN.
C*****
C***** THIS SUBROUTINE DOES NOT CHANGE THE SYSTEM CLASSIFICATION.
C***** TO GET A DIFFERENT SYSTEM YOU MUST BUILD IT WITH THE BUILD
C***** PART OF THE PROGRAM.
C*****
C***** SUBROUTINE ED
C***** REAL A,WORKR
C***** INTEGER N,INDEX,ANS,CHANGE,DELETE,ADD,L,M,S,YES,NO,WORKI,P,Z,
C***** + FITID
C***** DIMENSION INDEX(200),WORKR(200,4),WORKI(200,2)
C***** DATA CHANGE/'C',DELETE/'D',ADD/'A',YES/'Y',NO/'N'/
C***** READ (8,600) SERIAL,N
C***** DO 10 I=1,N
C***** + READ (8,601) INDEX(I),WORKI(I,1),WORKI(I,2),WORKR(I,1),
C***** + WORKR(I,2),WORKR(I,3),WORKR(I,4)
10 CONTINUE
20 REWIND 3
WRITE (6,602)
READ (5,603,END=22,ERR=22) ANS
IF ((ANS.EQ.CHANGE).OR.(ANS.EQ.DELETE).OR.(ANS.EQ.ADD)) GO TO 30
22 REWIND 3
WRITE (6,604)
GO TO 20
30 IF (ANS.EQ.CHANGE) GO TO 40
IF (ANS.EQ.DELETE) GO TO 80
IF (ANS.EQ.ADD) GO TO 150
C*****
C***** FITTING IS TO BE CHANGED, A NEW FITTING SUBSTITUTED FOR THE OLD
C***** WHAT INDEX NUMBER, N ???
40 WRITE (6,605)
CALL READI(M,5)
DO YOU NEED A MENU ???
50 WRITE (6,606)
READ (5,603,END=52,ERR=52) ANS
CALL FRICHS ('CIRSCRN ')
IF ((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 60
52 REWIND 5
WRITE (6,604)
GO TO 50
60 CONTINUE
IF (ANS.EQ.YES) GO TO 62
WRITE (6,607)
CALL READI(TYPE,5)
GO TO 64
C*****
C***** CALL THE MENU AND MAKE THE CHANGE
C*****
62 CALL MENU (0,0,TYPE,WORKI(M,1))
64 CALL SELECT (1,1,WORKI(M,1),TYPE,WORKI,WORKR)
C***** ANY MORE CHANGES ???
66 WRITE (6,608)
READ (5,603,END=68,ERR=68) ANS

```

```

68 IF ((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 70
    REWIND 5
    WRITE (6,604)
    GO TO 66
70 CCNTINUE
    IF (ANS.EQ.YES) GO TO 40
    GO TO 250
C
C
C A FITTING IS TO BE DELETED
30 WRITE (6,605)
    CALL READI (M,5)
    IF (M.EQ.N) GO TO 120
    N=N-1
    Z=0
C REWORK THE ID NUMBERS AND RELOAD THE FILE
DO 110 I=M,N
    TEST=WORKI(I,1)-WORKI(I,1)
    IF (TEST.GT.1) Z=Z+1
    IF ((TEST.EQ.1).AND.(Z.EQ.0)) GO TO 90
    WORKI(I,1)=WORKI(I+1,1)
    GO TO 100
90 CONTINUE
100 WORKI(I,2)=WORKI(I+1,2)
    WORKI(I,1)=WORKI(I+1,1)
    WORKI(I,2)=WORKI(I+1,2)
    WORKI(I,3)=WORKI(I+1,3)
    WORKI(I,4)=WORKI(I+1,4)
110 CCNTINUE
120 CONTINUE
    WRITE (6,609)
    ANY MORE DELETIONS ???
130 WRITE (6,610)
    READ (5,603,END=132,ERR=132) ANS
    IF ((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 140
132 REWIND 5
    WRITE (6,604)
    GO TO 130
140 CONTINUE
    IF (ANS.EQ.YES) GO TO 80
    GO TO 250
C
C
C A FITTING IS TO BE ADDED
150 WRITE (6,611)
    CALL READI (M,5)
    FITID=WORKI(M,1)+1
    S=N-M
    N=N+1
    M=M+1
C OPEN UP THE DATA FILE TO ADD THE NEW FITTING
DO 160 I=1,S
    WORKI(N+1-I,1)=WORKI(N-I,1)
    WORKI(N+1-I,2)=WORKI(N-I,2)
    WORKI(N+1-I,3)=WORKI(N-I,3)
    WORKI(N+1-I,4)=WORKI(N-I,4)
    WORKI(N+1-I,1)=WORKI(N-I,1)
    WORKI(N+1-I,2)=WORKI(N-I,2)
    WORKI(N+1-I,3)=WORKI(N-I,3)
    WORKI(N+1-I,4)=WORKI(N-I,4)
160 CONTINUE
    Z=0
C REWORK THE ID NUMBERS
DO 180 I=P,N
    TEST=WORKI(I,1)-WORKI(I-1,1)
    IF ((TEST.LT.100).AND.(Z.EQ.0)) GO TO 170
    Z=Z+1
    GO TO 180
    WORKI(I,1)=WORKI(I,1)+1
170 CCNTINUE
180 DO YOU NEED A MENU ???

```



```

C*****
C***** COMPUTE SUBROUTINE: PRODUCES PERFORMANCE DATA OF SYSTEM *****C
C***** THE DUCT DATA FILE IS READ AND THEN THE USER MUST INPUT THE *****C
C***** DESIRED OPERATING POINT. INPUT THE AMBIENT TEMPERATURE *****C
C***** (DEGREES F), THE AMBIENT PRESSURE (PSIA), AND HUMIDITY (GRAINS), *****C
C***** HORSEPOWER, AND POWER TURBINE SPEED. OUTPUT IS THE ENGINE *****C
C***** PERFORMANCE AND DUCT RESISTANCES. THE OUTPUT GOES TO YOUR DISK *****C
C***** UNDER FILE OUTPUT DATA. *****C
C*****
C***** SUBROUTINE COME *****
C***** REAL WORKR(10,20),HUMID,HP,NET,ACFB,ACFM,ACWC,ADWB,ADWC, *****
C***** ADWM,ALFAD,ALFAC,ABOSTD,CNFD,CFMAX,OPMAX,K *****
C***** + INTEGER N,INDEX,WORKI,CLASS,BRANCH,FIT1ST,I,TEST,NBR,OFF,SERIAL, *****
C***** + ANS,YES,NC *****
C***** DIMENSION INDEX(200),WORKI(200,2),WORKR(200,4),FIT1ST(7),NBR(6) *****
C***** DATA YES/'Y',NC/'N', *****
C***** CALL FRTCMS('CIRSCRN') *****
C***** READ FILE SERIAL NUMBER AND HOW MANY FITTINGS ARE IN THE FILE *****
C***** READ (8,600) SERIAL,N *****
C***** READ INDEX,ID NUMBER, FITTING TYPE, AND FOUR ELEMENTS OF DATA *****
C***** FOR EACH FITTING *****
C***** DO 10 I=1,N *****
C***** + READ (8,601) INDEX(I),WORKI(I,1),WORKI(I,2),WORKR(I,1), *****
C***** + WORKR(I,2),WORKR(I,3),WORKR(I,4) *****
10 CONTINUE
REWIND 8
C***** FIND OUT WHAT CLASS SYSTEM IS IN THE DUCT DATA FILE *****
C***** CLASS=WORKI(1,1)/100000 *****
C***** SET UP FOR THE CORRECT NUMBER OF BRANCHES FOR THE SYSTEM *****
C***** IF (CLASS.EQ.1) BRANCH=4 *****
C***** IF (CLASS.EQ.2) BRANCH=5 *****
C***** IF (CLASS.EQ.3) BRANCH=6 *****
C***** IF (CLASS.EQ.4) BRANCH=5 *****
C***** IF (CLASS.EQ.5) BRANCH=5 *****
C***** IF (CLASS.EQ.6) BRANCH=4 *****
C***** IF (CLASS.EQ.1) GO TO 80 *****
C***** SEARCH FOR WYE AREAS. MUST BE KNOWN FOR MATCHING. SEARCH IS *****
C***** DONE BY LOOKING FOR THE "WYE" FITTING TYPES, 13,14,15,16. *****
20 DO 70 I=1,N *****
C***** IF (WORKI(I,2).EQ.13) GO TO 30 *****
C***** IF (WORKI(I,2).EQ.14) GO TO 40 *****
C***** IF (WORKI(I,2).EQ.15) GO TO 50 *****
C***** IF (WORKI(I,2).EQ.16) GO TO 60 *****
C***** GO TO 70 *****
30 ADWC=WORKR(I,1) *****
C***** ADWB=WORKR(I,2) *****
C***** ALFAD=WORKR(I,3) *****
C***** GO TO 70 *****
40 ADWM=WORKR(I,1) *****
C***** GO TO 70 *****
50 ACWC=WORKR(I,1) *****
C***** ACWB=WORKR(I,2) *****
C***** ALFAC=WORKR(I,3) *****
C***** GO TO 70 *****
60 ACWM=WORKR(I,1) *****
70 CONTINUE *****
80 CONTINUE *****
C***** M=2 *****
C***** THE INDEX NUMBER OF THE FIRST FITTING OF THE BRANCH MUST BE KNOWN. *****
C***** IT IS USED IN THE DO LOOPS OF THE SYSTEM ANALYSIS TO FIND BRANCH *****
C***** PRESSURE DROPS. NEXT LOOP SEARCHES FOR THESE INDEXES. LOOP WILL *****
C***** FIND THE INDEX WHEN ID NUMBERS DIFFER BY MORE THAN ONE. *****
C***** FIT1ST(1)=1 *****
C***** DO 90 I=1,N *****
C***** TEST=WORKI(I+1,1)-WORKI(I,1) *****
C***** IF (TEST.EQ.1) GO TO 90 *****
C***** FIT1ST(I)=INDEX(I+1) *****
C***** M=M+1 *****
90 CONTINUE *****

```



```

90  CCNTINUE
C  GET THE OPERATING CONDITIONS AND POWER REQUIREMENTS.
C  CALL OPTCOND(TO,PO,HUMID)
C  IF A FAN IS INSTALLED SET FAN CHARACTERISTICS
  IF (CLASS.GT.4) GO TO 98
  CALL FAN (RHOSID,CFM0,CFMMAX,DPMAX,K)
  CALL FANPT (HP,NPT,TO,PO)
95  GO TO THE SYSTEM SUBROUTINE TAILORED FOR THE SYSTEM
C  GO TO (100,150,200,250,300,350),CLASS
98  CALL SYS1 (SERIAL,N,WORKI,WORKR,HP,NPT,FIT1ST,TO,PO,HUMID,
100  RHOSID,CFM0,CFMMAX,DPMAX,K)
  GO TO 400
150  CALL SYS2 (SERIAL,N,WORKI,WORKR,HP,NPT,FIT1ST,TO,PO,HUMID,
  ALFAC,ADWB,ADWC,ADWM,
  RHOSID,CFM0,CFMMAX,DPMAX,K)
  GO TO 400
200  CALL SYS3 (SERIAL,N,WORKI,WORKR,HP,NPT,FIT1ST,TO,PO,HUMID,
  ALFAC,ADWB,ADWC,ADWM,ALFAC,ACWB,ACWC,ACWM,
  RHOSID,CFM0,CFMMAX,DPMAX,K)
  GO TO 400
250  CALL SYS4 (SERIAL,N,WORKI,WORKR,HP,NPT,FIT1ST,TO,PO,HUMID,
  ALFAC,ACWB,ACWC,ACWM,
  RHOSID,CFM0,CFMMAX,DPMAX,K)
  GO TO 400
300  CALL SYS5 (SERIAL,N,WORKI,WORKR,HP,NPT,FIT1ST,TO,PO,HUMID,
  ALFAC,ADWB,ADWC,ADWM,ALFAC,ACWB,ACWC,ACWM)
  GO TO 400
350  CALL SYS6 (SERIAL,N,WORKI,WORKR,HP,NPT,FIT1ST,TO,PO,HUMID,
  ALFAC,ACWB,ACWC,ACWM)
400  CCNTINUE
C  DO YOU WANT TO COMPUTE WITH DIFFERENT OPERATING CONDITIONS ???
410  WRITE (6,602)
  READ (5,603,END=420,ERR=420) ANS
  IF ((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 430
420  REWIND 5
  WRITE (6,604)
  GO TO 410
430  CCNTINUE
  IF (ANS.EQ.YES) GO TO 95
600  FORMAT (I6,/,I3)
601  FORMAT (I3,3X,I6,3X,I2,3X,F10.4,3X,F10.4,3X,F10.4,3X,F10.4)
602  FORMAT (' DO YOU WANT TO COMPUTE WITH DIFFERENT OPERATING CONDITIO
  NS (Y/N) ?')
603  FORMAT (A1)
604  FORMAT (' YOU MUST ENTER A LETTER INDICATED IN THE BRACKETS.')
```

```

C*****INSTRUCTIONS SUBROUTINE: LONG OR SHORT, CRT OR TYPEWRITER*****C
C*****THIS SUBROUTINE IS CALLED IN THE BUILD SUBROUTINE. IT DOES NOT*****C
C*****COMPUTE ANYTHING. IT IS AN ADMINISTRATIVE PART OF THE PROGRAM*****C
C*****TO SEE IF THE USER WANTS LONG OR SHORT INSTRUCTIONS, AND IF*****C
C*****THE USER IS USING A CRT TERMINAL OR TYPEWRITER TERMINAL.*****C
C*****TYPEWRITER TERMINALS DO NOT GET THE MENUS OVER AND OVER.*****C
C*****
SUBROUTINE INST(SOCL, TERM)
  INTEGER SOCL, TERM, ANS, LCNG, SHORT, CRT, TYPE
  DATA LONG, SHORT, CRT, TYPE / 'L', 'S', 'C', 'T' /
5  WRITE (6, 600)
  READ (5, 601, ENL=7, ERR=7) ANS
  IF ((ANS.EQ.LONG).OR.(ANS.EQ.SHORT)) GO TO 10
7  REWIND 5
  WRITE (6, 602)
  GO TO 5
10 IF (ANS.EQ.SHORT) GO TO 20
  SOCL=1
  WRITE (6, 603)
  GO TO 30
20 SOCL=0
30 CCNTINUE
40 WRITE (6, 604)
  READ (5, 601, ENL=42, ERR=42) ANS
  IF ((ANS.EQ.CRT).OR.(ANS.EQ.TYPE)) GO TO 50
42 REWIND 5
  WRITE (6, 602)
  GO TO 40
50 IF (ANS.EQ.CRT) GO TO 60
  TERM=0
  WRITE (6, 605)
  GO TO 70
60 TERM=1
  WRITE (6, 606)
70 CCNTINUE
600 FORMAT(' DO YOU WANT LONG OR SHORT INSTRUCTIONS (L/S)?')
601 FORMAT(A1)
602 FORMAT(' YOU MUST ENTER THE LETTER INDICATED IN THE BRACKETS.')
603 FORMAT(' YOU HAVE SELECTED THE LONG INSTRUCTIONS.')
604 FORMAT(' ARE YOU WORKING ON A CRT OR TYPEWRITER TERMINAL (C/T)?')
605 FORMAT(' YOU ARE WORKING ON A TYPEWRITER TERMINAL.')
606 FORMAT(' YOU ARE WORKING ON A CRT TERMINAL.')
RETURN
END

```

```

C*****
C SYSTEM SUBROUTINE: DETERMINES WHICH SYSTEM, 1, 2, 3, 4, 5, OR 6
C*****
C CALLED BY THE BUILD SUBROUTINE. USED TO SET UP THE PROGRAM FOR
C THE VARIATIONS IN DUCT SYSTEMS AVAILABLE. THE EDITING SUBROUTINE
C CAN NOT CHANGE THE SYSTEM TYPE. ONCE SET UP HERE ANOTHER RUN OF
C THE BUILD SUBROUTINE IS REQUIRED TO GET A DIFFERENT SYSTEM.
C*****
C SUBROUTINE SYSTEM(SORL, CLASS)
C INTEGER SORL, CLASS, ANS1, ANS2, ANS3, YES, NO, SHORT
C DATA YES/'Y'/, NO/'N'/, SHORT/'S'/
C IF (SORL.EQ.SHORT) GO TO 30
C DOES THE COOLING AIR BRANCH OFF THE MAIN INLET ???
C 5 WRITE(6,600)
C READ(5,601,END=7,ERR=7)ANS1
C IF ((ANS1.EQ.YES).OR.(ANS1.EQ.NO)) GO TO 10
C 7 REWIND 5
C WRITE(6,602)
C GO TO 5
C DOES THE COOLING AIR JOIN THE MAIN EXHAUST ???
C 10 WRITE(6,603)
C READ(5,601,END=12,ERR=12)ANS2
C IF ((ANS2.EQ.YES).OR.(ANS2.EQ.NO)) GO TO 15
C 12 WRITE(6,602)
C GO TO 10
C 15 IF (ANS2.EQ.YES) GO TO 20
C ANS3=YES
C GO TO 25
C 20 IS THERE A COOLING FAN INSTALLED ???
C WRITE(6,604)
C READ(5,601,END=22,ERR=22)ANS3
C IF ((ANS3.EQ.YES).OR.(ANS3.EQ.NO)) GO TO 25
C 22 REWIND 5
C WRITE(6,602)
C GO TO 20
C SYSTEM CLASSIFICATION DEPENDS ON THE CONFIGURATION OF THE SYSTEM
C AND IF A COOLING FAN IS INSTALLED. CONFIGURATION MEANS HOW THE
C DUCT ARE JOINED TOGETHER IN THE SYSTEM.
C 25 IF ((ANS1.EQ.NO).AND.(ANS2.EQ.NO)) CLASS=1
C IF ((ANS1.EQ.YES).AND.(ANS2.EQ.NO)) CLASS=2
C IF ((ANS1.EQ.YES).AND.(ANS2.EQ.YES).AND.(ANS3.EQ.YES)) CLASS=3
C IF ((ANS1.EQ.NO).AND.(ANS2.EQ.YES).AND.(ANS3.EQ.YES)) CLASS=4
C IF ((ANS1.EQ.YES).AND.(ANS2.EQ.YES).AND.(ANS3.EQ.NO)) CLASS=5
C IF ((ANS1.EQ.NO).AND.(ANS2.EQ.YES).AND.(ANS3.EQ.NO)) CLASS=6
C GO TO 40
C 30 SHORT INSTRUCTIONS...JUST ENTER THE SYSTEM CLASSIFICATION NUMBER
C WRITE(6,605)
C CALL READ1(CLASS,5)
C IF ((CLASS.GT.0).AND.(CLASS.LT.7)) GO TO 40
C WRITE(6,606)
C GO TO 30
C 40 CONTINUE
C 600 FORMAT(' DOES THE MODULE COOLING AIR BRANCH OFF THE MAIN INLET?
C (Y,N)')
C 601 FORMAT('A1)
C 602 FORMAT(' YOU MUST ENTER A LETTER IN THE BRACKETS.')
C 603 FORMAT(' DOES THE MODULE COOLING AIR JOIN THE MAIN ENGINE EXHAUST
C (Y,N)')
C 604 FORMAT(' IS THERE A COOLING FAN INSTALLED?')
C 605 FORMAT(' ENTER THE SYSTEM CLASSIFICATION: 1, 2, 3, 4, 5, OR 6')
C 606 FORMAT(' YOU MUST ENTER A 1, 2, 3, 4, 5, OR 6')
C RETURN
C END

```

```

C*****C
C      MENU SUBROUTINE: PRINTS MENU AND FINDS OUT WHICH FITTING TO USE C
C*****C
C      CALLED BY BUILD AND EDIT SUBROUTINES. C
C      CHANGING THE NUMBER OF FITTINGS REQUIRES CHANGING THE MENU. C
C      JUST REVISE THE FORMAT STATEMENTS, WATCH THAT IT DOES NOT C
C      OVERFLOW THE SCREEN. C
C*****C
SUBROUTINE MENU(M,TERM,TYPE,FITID)
INTEGER FITID,M,TERM,TYPE,CRT,IYPT
DATA IYPT/0/
C
IF USER IS ON A TYPEWRITER TERMINAL, THE MENU IS PRINTED ONLY ONCE
IF((M.GT.0).AND.(TERM.EQ.IYPT)) GO TO 10
WRITE(6,600)
WRITE(6,601)
WRITE(6,602)
WRITE(6,603)
WRITE(6,604)
WRITE(6,605) FITID
CALL READI(TYPE,5)
GO TO 20
10  WRITE(6,606)
    CALL READI(TYPE,5)
20  CONTINUE
    CALL FRTCHS('CLRSCHN ')
600  FORMAT('00 NO MORE FITTINGS THIS BRANCH' 6X, '* 14 DIVERGI
+NG WYE, MAIN SECTION' // '01 INTAKE SHAFT, RECT SECTION, SIDE *
+15 CONVERGENT WYE, BRANCH SECTION' // '02 STRAIGHT DUCT' LO
+UVERS * 16 CONVERGENT WYE, MAIN SECTION' // '03 ELBOW, SMOOTH RAD
+21X' * 17 DIFFUSER, CONICAL ROUND SECTION' // '04 ELBOW, 90 DEG:
+IUS ROUND' 3X, '* 18 DIFFUSER, PLANE, IN-LINE' // '05 ELBO
+3.4, 3 PCS: ROUND * 19 DIFFUSER, PYRAMIDAL, IN-LINE' //
+H, MITERED, ROUND, W&W/O VANES* 20 DIFFUSER, TRANSITIONAL (ROUND T
+O' // '06 ELBOW, MITERED, RECTANGULAR * RECT OR RECT TO RO
+UND')
601  FORMAT('07 ELBOW, SMOOTH RADIUS, RECTANGULAR * 21 CONTRACTION RO
+UND' // '08 ELBOW, SMOOTH RADIUS, WITH * 22 CONTRACTION RECT
+ANGULAR' // '09 ELBOW, MITERED WITH VANES, RECT * 23 OBSTRUCTION
+SCREEN IN DUCT' // '10 ELBOW, MITERED WITH VANES, RECT * 24 LOUVE
+R ENTRANCE')
602  FORMAT('11 ELBOW, CONVERGING OR DIVERGING * 25 FILTER' //
+FLOW, RECTANGULAR' 14X, '* 26 MULTI-BAFFLE SILENCER' //
+12 ELBOWS, 90 DEG, Z-SHAPED, RECT * 27 GT MODULE ')
603  FORMAT('13 ELBOWS, 90 DEG, IN DIFFERENT * 28 WASTE HEAT BOI
+LER' // '14 ELBOWS, 90 DEG, IN DIFFERENT * 29 EXIT ABRUPT' //
+15 DIVERGING WYE, BRANCH SECTION' 11X, '* 30 FITTING NOT LISTED')
604  FORMAT(' / *****USE TWO DIGIT NUMBER, PRESS ENTER*****
+*****')
605  FORMAT(' >> YOU ARE WORKING ON FITTING NUMBER >> ',I6)
606  FORMAT(' ENTER THE FITTING TYPE NUMBER FROM THE MENU.')
RETURN
END

```

```

C*****C
C SELECT SUBROUTINE: BRANCHES TO FITTING SELECTED IN MENU C
C*****C
C CALLED BY BUILD AND EDIT SUBROUTINES C
C THIS SUBROUTINE CALLS LOAD A SUBROUTINE THAT TRANSFERS THE C
C DATA OF A FITTING TO THE SYSTEM STORAGE ARRAYS WORKI AND WORKR C
C*****C
C SUBROUTINE SELECT (N, SORL, GEOM, TYPE, WORKI, WORKR)
C REAL WORKR, WKR
C INTEGER I, WKI, WORKI, SORL, GEOM, TYPE
C DIMENSION WORKI(200,2), WORKR(200,4), WKI(2), WKR(3)
C CHANGING THE NUMBER OF FITTINGS REQUIRES A CHANGE TO THE FOLLOWING
C GO TO STATEMENT AND THE ADDITION OF A CALL TO THE SUBROUTINE
C THAT HANDLES THE NEW FITTING.
C GO TO (1,2,3,4,5,6,7,8,9,10,11,12,13,14,15,16,17,18,19,20,
C 21,22,23,24,25,26,27,28,29),TYPE
1 CALL FIT01 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
2 CALL FIT02 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
3 CALL FIT03 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
4 CALL FIT04 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
5 CALL FIT05 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
6 CALL FIT06 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
7 CALL FIT07 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
8 CALL FIT08 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
9 CALL FIT09 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
10 CALL FIT10 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
11 CALL FIT11 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
12 CALL FIT12 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
13 CALL FIT13 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
14 CALL FIT14 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
15 CALL FIT15 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
16 CALL FIT16 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
17 CALL FIT17 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)
GO TO 40
18 CALL FIT18 (SORL,GEOM,WKI,WKR)
CALL LOAD (N,GEOM,WKI,WKR,WORKI,WORKR)

```

```

GO TO 40
19 CALL FIT19 (SORI,GEOM,WKI,WKR)
   CALL LOAD (N,GECH,WKI,WKR,WORKI,WORKR)
   GO TO 40
20 CALL FIT20 (SORI,GEOM,WKI,WKR)
   CALL LOAD (N,GECH,WKI,WKR,WORKI,WORKR)
   GO TO 40
21 CALL FIT21 (SORI,GEOM,WKI,WKR)
   CALL LOAD (N,GECH,WKI,WKR,WORKI,WORKR)
   GO TO 40
22 CALL FIT22 (SORI,GEOM,WKI,WKR)
   CALL LOAD (N,GECH,WKI,WKR,WORKI,WORKR)
   GO TO 40
23 CALL FIT23 (SORI,GEOM,WKI,WKR)
   CALL LOAD (N,GECH,WKI,WKR,WORKI,WORKR)
   GO TO 40
24 CALL FIT24 (SORI,GEOM,WKI,WKR)
   CALL LOAD (N,GECH,WKI,WKR,WORKI,WORKR)
   GO TO 40
25 CALL FIT25 (SORI,GEOM,WKI,WKR)
   CALL LOAD (N,GECH,WKI,WKR,WORKI,WORKR)
   GO TO 40
26 CALL FIT26 (SORI,GEOM,WKI,WKR)
   CALL LOAD (N,GECH,WKI,WKR,WORKI,WORKR)
   GO TO 40
27 CALL FIT27 (SORI,GEOM,WKI,WKR)
   CALL LOAD (N,GECH,WKI,WKR,WORKI,WORKR)
   GO TO 40
28 CALL FIT28 (SORI,GEOM,WKI,WKR)
   CALL LOAD (N,GECH,WKI,WKR,WORKI,WORKR)
   GO TO 40
29 CALL FIT29 (SORI,GEOM,WKI,WKR)
   CALL LOAD (N,GECH,WKI,WKR,WORKI,WORKR)
   GO TO 40
30 CALL FIT30 (SORI,GEOM,WKI,WKR)
   CALL LOAD (N,GECH,WKI,WKR,WORKI,WORKR)
C 40 A NEW FITTING WOULD REQUIRE ANOTHER CALL STATEMENT HERE
   CONTINUE
   RETURN
   END

```

```

C*****
C FITTING 01: VERT. INTAKE SHAFT, SIDE ORIFACES, WITH (OUT) LOUVERS
C*****
C REF. HANDBOOK OF HYDRAULIC RESISTANCE, I.E. IDEL'CHIK, PAGE 103
C THE TABULATED VALUES ARE LISTED IN AN ARRAY "A". THE PROPER VALUE
C IS EXTRACTED BY ANSWERING CERTAIN QUESTIONS ABOUT CONFIGURATION.
C THE REFERENCE AREA IS THE SHAFT AREA. THIS FITTING IS FOR
C DYNAMIC RESISTANCE. THE DUCT CONNECTED TO IT SHOULD START JUST
C BELOW THE ORIFACES.
C*****
C SUBROUTINE FIT01(SORL,GEOM,WKI,WKR)
C REAL WKI,A,AREA
C INTEGER N,M,ANS1,ANS2,YES,NO,GEOM,SORL,WKI,OPP,ADJ
C DIMENSION WKI(2),WKR(4),A(2,5)
C DATA YES/'Y'/,NO/'N'/,OPP/'O'/,ADJ/'A'/
C HOW MANY ORIFACES ???
C 2 WRITE(6,600)
C CALL READI(N,5)
C IF((N.LT.1).OR.(N.GT.4)) GO TO 2
C IF(N.EQ.2) GO TO 10
C IF(N.EQ.3) GO TO 10
C IF(N.EQ.4) GO TO 10
C IF(N.EQ.5) GO TO 10
C 5 WRITE(6,602)
C READ(5,603,END=7,ERR=7) ANS1
C IF((ANS1.EQ.OPP).OR.(ANS1.EQ.ADJ)) GO TO 10
C 7 REWIND 5
C WRITE(6,604)
C GO TO 5
C ARE THERE LOUVERS INSTALLED ???
C 10 WRITE(6,601)
C CALL READR(AREA,5)
C 15 WRITE(6,605)
C READ(5,603,END=17,ERR=17) ANS2
C IF((ANS2.EQ.YES).OR.(ANS2.EQ.NO)) GO TO 18
C 17 REWIND 5
C WRITE(6,604)
C GO TO 15
C 18 IF((N.EQ.2).AND.(ANS1.EQ.OPP)) N=2
C IF((N.EQ.2).AND.(ANS1.EQ.ADJ)) N=3
C IF((N.EQ.3).AND.(ANS1.EQ.OPP)) N=4
C IF((N.EQ.3).AND.(ANS1.EQ.ADJ)) N=5
C IF((N.EQ.4).AND.(ANS1.EQ.OPP)) N=6
C IF((N.EQ.4).AND.(ANS1.EQ.ADJ)) N=7
C IF((N.EQ.5).AND.(ANS1.EQ.OPP)) N=8
C IF((N.EQ.5).AND.(ANS1.EQ.ADJ)) N=9
C M=N
C GO TO 30
C 20 M=2
C 30 CONTINUE
C DATA FROM IDEL'CHIK'S HANDBOOK
C A(1,1)=12.0
C A(1,2)=3.6
C A(1,3)=4.2
C A(1,4)=1.8
C A(1,5)=1.2
C A(2,1)=17.5
C A(2,2)=5.4
C A(2,3)=6.3
C A(2,4)=3.2
C A(2,5)=2.5
C 35 ENTER DATA INTO TRANSFER ARRAYS WKI,WKR
C WKI(1)=GEOM
C WKI(2)=1
C WKR(1)=AREA
C WKR(2)=0.0
C WKR(3)=A(M,N)
C WKR(4)=AREA
C 600 FORMAT(' YOU HAVE SELECTED A VERTICAL INTAKE SHAFT OF ' //
C + ' RECTANGULAR SECTION WITH SIDE ORIFACES AT THE TOP.' //
C + ' IT MAY OR MAY NOT HAVE LOUVERS OVER THE ORIFACES.' //
C + ' FILTERS ARE A SEPARATE FITTING.' // '***FIRST, ENTER THE NO
C + 'MBER OF ORIFACES. (1,2,3,OR 4)***')
C 601 FORMAT(' ENTER THE CROSS SECTIONAL AREA OF THE VERTICAL SHAFT.')

```

```

602  FORMAT(' SINCE THERE ARE TO BE TWO ORIFACES, ARE THE ORIFACES OPP
+OSITE OR ADJACENT (O/A)?')
603  FORMAT(A1)
604  FORMAT(' YOU MUST ENTER A LETTER IN THE BRACKETS.')
605  FORMAT(' LAST QUESTION, ARE LOUVERS MOUNTED ON THE ORIFACES? (Y/
+N)')
      RETURN
      END

```



```

C***** FITTING J2: STRAIGHT DUCT, ROUND OR RECTANGULAR C*****
C***** NO REFERENCE, ONLY THE DUCT GEOMETRY IS INPUT HERE. LATER ON IN C*****
C***** THE COMPUTED PART OF THE PROGRAM A COEFFICIENT BASED ON F*L/D WILL C*****
C***** BE DEVELOPED TO DETERMINE THE RESISTANCE OF THE DUCT. F IS THE C*****
C***** FRICTION FACTOR. SEE FITOP FOR THE CORRELATION USED. C*****
C***** SUBROUTINE FITO2 (SORL,GEOM,WKI,WKR) C*****
      REAL A,B,L,D,WKR
      INTEGER SORL,GEOM,WKI,ANS1,CIR,REC,SHORT
      DIMENSION WKI(2),WKR(4)
      DATA CIR/'C'/,REC/'R'/,SHORT/0/
      IS DUCT CIRCULAR OR RECTANGULAR ???
5      WRITE(6,600)
      READ(5,601)END=6,ERR=6) ANS1
      IF((ANS1.EQ.CIR).OR.(ANS1.EQ.REC)) GO TO 7
      REWIND 5
      WRITE(6,608)
      GO TO 5
7      IF(ANS1.EQ.CIR) GO TO 30
      IF(SORL.EQ.SHORT) GO TO 10
      WRITE(6,602)
      CALL READER(A,5)
      WRITE(6,603)
      CALL READER(B,5)
      WRITE(6,604)
      CALL READER(L,5)
      GO TO 20
10     WRITE(6,605)
      CALL READER(A,5)
      CALL READER(B,5)
      CALL READER(L,5)
20     CONTINUE
      AREA=A*B
      C      SINCE THE DUCT IS RECTANGULAR, THE EQUIVALENT CIRCULAR DIAMETER
      C      IS REQUIRED. THIS IS FROM THE ASHRAE HANDBOOK, CHAPTER 33, DUCTS
      D=1.3*((A*B)**0.625)/(A+B)**0.250
      R=L/D
      GO TO 100
30     IF(SORL.EQ.0) GO TO 40
      WRITE(6,606)
      CALL READER(D,5)
      WRITE(6,604)
      CALL READER(L,5)
      GO TO 50
40     WRITE(6,607)
      CALL READER(D,5)
      CALL READER(L,5)
50     AREA=3.14*(D**2/4.0)
100    WKI(1)=GEOM
      WKI(2)=2
      WKR(1)=AREA
      WKR(2)=D
      WKR(3)=L
      WKR(4)=AREA
600    FORMAT(' YOU HAVE SELECTED STRAIGHT DUCT. IT MAY BE ROUND OR REC
+TANGULAR.'/'*****FIRST QUESTION, IS THE DUCT CIRCULAR OR RECTANGULA
+R (C/R) ?')
601    FORMAT(A1)
602    FORMAT(' THE DUCT IS RECTANGULAR, ENTER FIRST CROSS-SECTIONAL DIM
+ENSION. (FEET)')
603    FORMAT(' SECOND DIMENSION (FEET)')
604    FORMAT(' ENTER THE LENGTH OF THIS DUCT SECTION. (FEET)')
605    FORMAT(' ENTER THE RECTANGULAR DUCT DIMENSIONS. (FEET)')
      +* FORMAT: FIRST DIMENSION SAMPLE: 10.1
      +* SECOND DIMENSION 8.35
      +* LENGTH 18.3
606    FORMAT(' THE DUCT IS CIRCULAR, ENTER THE DIAMETER (FEET)')

```

```

607 FORMAT(' ENTER THE DIMENSIONS (FEET) OF THE CIRCULAR DUCT. '/
+ '      FORMAT:  DIAMETER      SAMPLE:  5.65 '/
+ '      LENGTH      20')
608 FORMAT(' YOU MUST ENTER A LETTER IN THE BRACKETS. ')
RETURN
END

```

```

C*****FITTING 03: ELBOW, SMOOTH RADIUS, ROUND CROSS-SECTION*****C
C*****REF. ASHRAE HANDBOOK, PAGE 33.33, TABLE B-1, FITTING 3-1*****C
C*****CUBIC FIT TO THE TABULATED DATA*****C
C*****FRICTION LOSSES NOT INCLUDED, CONNECTING DUCTS*****C
C*****C.F. CENTER OF THIS FITTING*****C
C*****
SUBROUTINE FIT03(SCRL,SECM,WKI,WKR)
REAL R,D,THETA,KTHETA,C,AREA,CPRIME,WKR
INTEGER GEOM,SCRL,WKI
DIMENSION WKI(2),WKR(4)
WRITE(6,600)
CALL READR(D,5)
WRITE(6,601)
CALL READR(R,5)
WRITE(6,602)
CALL READR(THETA,5)
AREA=0.7854*D**2
KTHETA=0.0305*THETA**0.7825
CPRIME=0.02946*EXP(2.5627*(1.57138-(R/D)))+0.11746
C=KTHETA*CPRIME
WKI(1)=GEOM
WKI(2)=3
WKR(1)=AREA
WKR(2)=0.0
WKR(3)=C
WKR(4)=AREA
600  FORMAT(' YOU HAVE SELECTED A SMOOTH RADIUS ROUND CROSS-SECTION' /
+ ' ELBOW.' /) **FIRST QUESTION, WHAT IS THE CROSS-SECTION DIAMETER
+ ' (FEET) **'
601  FORMAT(' ENTER THE RADIUS OF THE TURN OF THE ELBOW MEASURED TO' /
+ ' CENTERLINE OF THE DUCT.' /)
602  FORMAT(' LAST QUESTION, ENTER THE ANGLE OF THE ELBOW TURN. (DEGR
+ ' ) **'
RETURN
END

```

```

C*****C
C FITTING 04: ELBOW, SEGMENTED ROUND CROSS-SECTION, 90 DEGREE C
C*****C
C REF. ASHRAE HANDBOOK, PAGE 33.33, TABLE B-3, FITTING 3-2 C
C CURVE FIT TO THE TABULATED DATA FOR EACH NUMBER OF SEGMENTS. C
C THIS IS A SHORT FITTING, FRICTION LOSSES NOT INCLUDED, MEASURE C
C CONNECTING DUCTS TO THE CENTER OF THIS FITTING. C
C*****C
C SUPERROUTINE FIT04(SORL,GEOM,WKI,WKR)
C REAL D,R,WKR
C INTEGER SORL,GEOM,WKI,N,M
C DIMENSION WKI(2),WKR(4)
5 WRITE(6,600)
C CALL READR(N,5)
C IF((N.LT.3).OR.(N.GT.5)) GO TO 5
C WRITE(6,601)
C CALL READR(D,5)
C WRITE(6,602)
C CALL READR(R,5)
C AREA=0.7854*D**2
C M=N-2
C GO TO (10,20,30),M
10 C=4.4022*EXP(3.9394*(0.00282-R/D))+0.32829
C GO TO 40
20 C=1.8428*EXP(2.4861*(-0.02393-R/D))+0.22798
C GO TO 40
30 C=1.0456*EXP(1.74313*(0.01219-R/D))+0.15776
40 CONTINUE
C WKI(1)=GEOM
C WKI(2)=4
C WKR(1)=AREA
C WKR(2)=0.0
C WKR(3)=C
C WKR(4)=AREA
600 FORMAT(' YOU HAVE SELECTED A SEGMENTED ROUND CROSS-SECTION 90 DEG
+REE ELBOW. '/' **FIRST QUESTION, HOW MANY SEGMENTS, INCLUDE ENTRY
+AND EXIT? (3,4,OR 5) **')
601 FORMAT(' ENTER THE CROSS-SECTIONAL DIAMETER.')
602 FORMAT(' LAST QUESTION, WHAT IS THE RADIUS OF THE TURN OF THE ELB
+OW '/' MEASURED TO THE CENTERLINE OF THE DUCT?')
C RETURN
C END

```

```

C***** FITTING 05: ELBOW MITERED CIRCULAR CROSS-SECTION *****C
C***** REF. ASHRAE HANDBOOK, PAGE 33.33, TABLE 8-3. FITTING 3-3 *****C
C CURVE FIT TO DATA.
C THIS IS A SHORT FITTING. CONNECTING DUCTS SHOULD BE MEASURED
C TO THE CENTER OF THIS FITTING.
C*****
SUBROUTINE FIT05 (SORL,GEOM,WKI,WKR)
REAL J,THETA,CPRIME,AREA,WKE,K
DIMENSION SORL(2),WKI(2),WKE(4),ANS,YES,NO
DATA YES/1,NO/2,NC/3,N/4
WRITE (6,600)
CALL READR (J,5)
WRITE (6,601)
CALL READR (THETA,5)
K=1.0
10 WRITE (6,602)
READ (5,603,END=12,ERR=12) ANS
IF ((ANS.EQ.YES).OR.(ANS.EQ.NC)) GO TO 20
12 REWIND 5
WRITE (6,604)
GO TO 10
20 CONTINUE
IF (ANS.EQ.YES) K=0.27
CPRIME=(3.74E-4)*(THETA**1.7852)*K
AREA=0.7854*D**2
WKI(1)=GEOM
WKI(2)=5
WKE(1)=AREA
WKE(2)=D
WKE(3)=CPRIME
WKE(4)=AREA
600 FORMAT(' YOU HAVE SELECTED A MITERED ROUND ELBOW.'/
* ' FIRST QUESTION: WHAT IS THE CROSS-SECTIONAL DIAMETER?')
601 FORMAT(' WHAT IS THE ANGLE OF THE ELBOW BURN?')
602 FORMAT(' LAST QUESTION: ARE OR ROUND NUMBER OF CONCENTRIC VANES'/
* ' INSTALLED TO REDUCE RESISTANCE AND TURBULANCE (Y/N)?')
603 FORMAT(A1)
604 FORMAT(' YOU MUST ENTER A LETTER IN THE BRACKETS.')
RETURN
END

```

```

C***** FITTING J6: ELBOW MITERED RECTANGULAR CROSS-SECTION *****C
C***** REF. ASHRAE HANDBOOK, PAGE 33.33, TABLE B-3, FITTING 3-6 AND *****C
C***** THIS HANDBOOK OF HYDRAULIC RESISTANCE, FEDL'CHIK. *****C
C***** CURVE FITS TO THE DATA. THIS IS A SHORT FITTING, MEASURE DUCT *****C
C***** CONNECTED TO IT TO THE CENTER OF THIS FITTING. *****C
C*****
SUBROUTINE FIT06(SORL,GEOM,WKI,WKR)
REAL H,W,THETA,C1,A,PHI,CPRIME,RAD,AREA,DH,WKR
INTEGER SORL,GEOM,WKI
DIMENSION WKI(2),WKR(4)
WRITE(6,600)
CALL READR(H,5)
WRITE(6,601)
CALL READR(W,5)
10 WRITE(6,602)
CALL READR(THETA,5)
IF (THETA-2.90.0) GO TO 20
WRITE(6,603)
GO TO 10
20 RAD=THETA*3.1416/180.0
DH=2.0*(H*W)/(H+W)
AREA=H*W
C1=0.23097*EXP(0.38896*(1.87338-(H/W)))+0.67819
A=1.2+1.0381*((1.5708-RAD)/1.0472)**1.8233
PHI=0.95*((SIN(RAD/2.0))**2)+2.05*((SIN(RAD/2.0))**4.0)
CPRIME=C1*A*PHI
WKI(1)=GEOM
WKI(2)=0
WKR(1)=AREA
WKR(2)=DH
WKR(3)=CPRIME
WKR(4)=AREA
600 FORMAT(' YOU HAVE SELECTED A MITERED, RECTANGULAR CROSS-SECTION,
+ ELBOW. ',' **FIRST QUESTION, WHAT IS THE HEIGHT OF THE ELBOW?'/
+ ' (THE DIMENSION PARALLEL TO THE TURN AXIS)')
601 FORMAT(' WHAT IS THE WIDTH OF THE ELBOW CROSS-SECTION?'/
+ ' (THE DIMENSION IN THE PLANE OF THE TURN)')
602 FORMAT(' LAST QUESTION, WHAT IS THE ANGLE OF THE ELBOW TURN (0 -
+ 90 DEGREES)?')
603 FORMAT(' ELBOW TURN ANGLE MUST NOT BE GREATER THAN 90 DEGREES.')
RETURN
END

```

```

C*****C
C FITTING 07: ELBOW SMOOTH RADIUS RECTANGULAR WITHOUT VANES C
C*****C
C REF. ASHRAE HANDBOOK, PAGE 33.31, TABLE 3-3, FITTING 3-5 C
C USES TWO DIMENSIONAL TABLE TO PROVIDE COEFFICIENT. CALL TABLE C
C SUBROUTINE A TABLE LOOKUP AND INTERPOLATION SUBROUTINE. C
C SHORT FITTING, MEASURE CONNECTING DUCTS TO THE CENTER OF FITTING C
C*****C
C SUBROUTINE FIT07 (SORL,GEOM,WKI,WKR)
C REAL WKR,H,W,R,THETA,T,X,KTHETA,C,CPRIME,DH,AREA
C INTEGER WKI,SORL,GEOM,XOUT
C DIMENSION WKI(2),WKR(4),T(61),X(2)
C TABLE 25 LISTED AS FOLLOWS, NUMBER OF X'S, NUMBER OF Y'S, THE X'S
C DATA T/ 9.00,5.00, 0.25,0.50,0.75,1.00,1.50,2.00,3.00,4.00,5.00,
C THE Y'S
C 0.50,0.75,1.00,1.50,2.00,
C THE TABLE INCREASING X TO THE RIGHT, INCREASING Y DOWN
C 1.30,1.30,1.20,1.20,1.10,1.10,0.98,0.92,0.89,
C 0.57,0.52,0.48,0.44,0.40,0.39,0.39,0.40,0.42,
C 0.27,0.25,0.23,0.21,0.19,0.18,0.18,0.19,0.20,
C 0.22,0.20,0.19,0.17,0.15,0.14,0.14,0.15,0.16,
C 0.20,0.18,0.16,0.15,0.14,0.13,0.13,0.14,0.14/
10 WRITE (6,600)
WRITE (6,601)
CALL READR(H,5)
WRITE (6,602)
CALL READR(W,5)
WRITE (6,603)
CALL READR(R,5)
WRITE (6,604)
CALL READR(THETA,5)
X(1)=H/W
X(2)=R/W
CALL TABLE (T,X,XOUT,C)
IF (XOUT.GT.0) GO TO 20
WRITE (6,605)
20 KTHETA=0.0306*THETA**0.7825
DH=2.0*(H+W)/(R+W)
CPRIME=C*KTHETA
AREA=H*W
WKI(1)=GEOM
WKI(2)=7
WKR(1)=AREA
WKR(2)=DH
WKR(3)=CPRIME
WKR(4)=R/W
600 FORMAT(' YOU HAVE SELECTED A SMOOTH RADIUS RECTANGULAR ELBOW WITH
+OUT VANES.')
601 FORMAT(' **FIRST QUESTION, WHAT IS THE HEIGHT OF THE ELBOW?')
602 FORMAT(' THE CROSS-SECTIONAL DIMENSION PARALLEL TO THE TURN AXIS(')
603 FORMAT(' WHAT IS THE WIDTH OF THE ELBOW (THE CROSS-SECTIONAL
+ DIMENSION IN THE PLANE OF THE TURN)?')
604 FORMAT(' WHAT IS THE RADIUS OF THE ELBOW, MEASURED TO THE CENTER
+ OF THE ELBOW CROSS-SECTION?')
605 FORMAT(' LAST QUESTION, WHAT IS THE ANGLE OF THE TURN (0-90 DEGR
+ ES)?')
RETURN
END

```

```

C***** FITTING J8: ELBOW SMOOTH RADIUS RECTANGULAR WITH SPLITTERS *****C
C***** REF. ASHRAE HANDBOOK, PAGE 33.32 & 33.33, TABLE B-3, FITTING 3-7 *****C
C***** USES TABLE INTERPOLATION SCHEME *****C
C***** THIS IS A SHORT FITTING, MEASURE CONNECTING DUCT TO THE CENTER *****C
C***** OF THIS FITTING TO INCLUDE FRICTION. *****C
C***** SUBROUTINE FITOB (SORL,GEOM,WKI,AKR) *****C
C***** REAL WKR,H,W,R,THETA,KTHETA,AREA,X,C,CPRIME,T1,T2,T3 *****C
C***** INTEGER WKI,N,XOUT,SORL,GEOM *****C
C***** DIMENSION WKI(2),WKR(4),X(2),T1(100),T2(100),T3(100),XOUT(2) *****C
C***** ONE SPLITTER *****C
C***** DATA T1/8.00,10.00,0.25,0.50,1.00,1.50,2.00,3.00,4.00,5.00, *****C
C***** 0.55,0.60,0.65,0.70,0.75,0.80,0.85,0.90,0.95,1.00, *****C
C***** 0.36,0.27,0.25,0.23,0.20,0.18,0.16,0.14,0.13,0.12, *****C
C***** 0.28,0.21,0.18,0.16,0.14,0.12,0.11,0.10,0.09,0.08, *****C
C***** 0.22,0.16,0.14,0.12,0.11,0.10,0.09,0.08,0.07,0.06, *****C
C***** 0.18,0.13,0.11,0.09,0.07,0.06,0.05,0.04,0.03,0.02, *****C
C***** 0.15,0.09,0.08,0.07,0.06,0.05,0.04,0.03,0.02,0.01, *****C
C***** 0.11,0.08,0.07,0.06,0.05,0.04,0.03,0.02,0.01,0.00, *****C
C***** 0.10,0.07,0.06,0.05,0.04,0.03,0.02,0.01,0.00,0.00, *****C
C***** 0.09,0.06,0.05,0.04,0.03,0.02,0.01,0.00,0.00,0.00, *****C
C***** TWO SPLITTERS *****C
C***** DATA T2/8.00,10.00,0.25,0.50,1.00,1.50,2.00,3.00,4.00,5.00, *****C
C***** 0.55,0.60,0.65,0.70,0.75,0.80,0.85,0.90,0.95,1.00, *****C
C***** 0.36,0.27,0.25,0.23,0.20,0.18,0.16,0.14,0.13,0.12, *****C
C***** 0.17,0.13,0.10,0.08,0.06,0.05,0.04,0.03,0.02,0.01, *****C
C***** 0.09,0.07,0.06,0.05,0.04,0.03,0.02,0.01,0.00,0.00, *****C
C***** 0.08,0.05,0.04,0.03,0.02,0.01,0.00,0.00,0.00,0.00, *****C
C***** 0.06,0.04,0.03,0.02,0.01,0.00,0.00,0.00,0.00,0.00, *****C
C***** 0.05,0.03,0.02,0.01,0.00,0.00,0.00,0.00,0.00,0.00, *****C
C***** 0.04,0.03,0.02,0.01,0.00,0.00,0.00,0.00,0.00,0.00, *****C
C***** 0.03,0.02,0.01,0.00,0.00,0.00,0.00,0.00,0.00,0.00, *****C
C***** THREE SPLITTERS *****C
C***** DATA T3/8.00,10.00,0.25,0.50,1.00,1.50,2.00,3.00,4.00,5.00, *****C
C***** 0.55,0.60,0.65,0.70,0.75,0.80,0.85,0.90,0.95,1.00, *****C
C***** 0.11,0.10,0.12,0.13,0.14,0.15,0.16,0.17,0.18,0.19, *****C
C***** 0.07,0.05,0.06,0.06,0.06,0.06,0.06,0.06,0.06,0.06, *****C
C***** 0.05,0.04,0.04,0.04,0.04,0.04,0.04,0.04,0.04,0.04, *****C
C***** 0.03,0.02,0.02,0.02,0.02,0.02,0.02,0.02,0.02,0.02, *****C
C***** 0.03,0.02,0.02,0.02,0.02,0.02,0.02,0.02,0.02,0.02, *****C
C***** 0.02,0.02,0.02,0.02,0.02,0.02,0.02,0.02,0.02,0.02, *****C
C***** 0.02,0.01,0.01,0.01,0.01,0.01,0.01,0.01,0.01,0.01, *****C
C***** 0.01,0.01,0.01,0.01,0.01,0.01,0.01,0.01,0.01,0.01, *****C
C***** 0.01,0.01,0.01,0.01,0.01,0.01,0.01,0.01,0.01,0.01, *****C
C***** WRITE(6,600) *****C
C***** HOW MANY SPLITTERS ??? *****C
C***** WRITE(6,601) *****C
C***** CALL READI(N,5) *****C
C***** IF(N.EQ.1).OR.(N.GT.3) GO TO 10 *****C
C***** WRITE(6,602) *****C
C***** CALL READR(H,5) *****C
C***** WRITE(6,603) *****C
C***** CALL READR(W,5) *****C
C***** WRITE(6,604) *****C
C***** CALL READR(R,5) *****C
C***** WRITE(6,605) *****C
C***** CALL READR(THETA,5) *****C
C***** KTHETA=0.0306*THETA**0.7825 *****C
C***** X(1)=H/W *****C
C***** X(2)=R/W *****C
C***** AREA=H*W *****C
C***** GO TO (20,30,40),N *****C

```



```

20  CALL TABLE (T1,X,XOUT,CPRIME)
30  CALL TABLE (T2,X,XOUT,CPRIME)
40  CALL TABLE (T3,X,XOUT,CPRIME)
50  CCNT=NUS
   IF ((XOUT(1).GT.0).OR.(XOUT(2).GT.0)) GO TO 60
   WRITE (9,606)
   GO TO 10
60  C=CPRIME*KTHETA
   NKT(1)=GEOM
   NKT(2)=3
   WKRR(1)=AREA
   WKRR(2)=J.0
   WKRR(3)=C
   WKRR(4)=AREA
600  FORMAT(' YOU HAVE SELECTED A SMOOTH RADIUS RECTANGULAR ELBOW WITH
* SPLITTERS.'/' IT MAY HAVE 1, 2, OR 3 SPLITTERS.')
601  FORMAT(' **FIRST QUESTION, HOW MANY SPLITTERS ARE IN THE ELBOW (1
* OR 3)?')
602  FORMAT(' WHAT IS THE HEIGHT OF THE ELBOW?'/
* (THE CROSS-SECTIONAL DIMENSION PARALLEL TO THE TURN AXIS))
603  FORMAT(' WHAT IS THE WIDTH OF THE ELBOW (THE CROSS-SECTIONAL
* DIMENSION IN THE PLANE OF THE TURN)?')
604  FORMAT(' WHAT IS THE RADIUS OF THE ELBOW, MEASURED TO THE CENTER
* OF THE ELBOW CROSS-SECTION?')
605  FORMAT(' LAST QUESTION, WHAT IS THE ANGLE OF THE TURN (0-90 DEGR
* ES)?')
606  FORMAT(' CROSS-SECTION EXTREMELY NARROW, RE-ENTER BETTER DATA.')
      RETURN
      END

```



```

C***** FITTING 10: ELBOW RECTANGULAR WITH CONVERGING OR DIVERGING FLOW *****C
C***** REF. ASHRAE HANDBOOK, PAGE 33.32, TABLE B-3, FITTING 3-10 *****C
C***** TABLE INTERPOLATION *****C
C***** SHORT FITTING, DYNAMIC LOSSES ONLY. MEASURE CONNECTING DUCT TO *****C
C***** THE CENTER OF THIS FITTING TO INCLUDE FRICTION. *****C
C*****
SUBROUTINE FIT10 (SOR1,GEOM,WKI,WKR)
REAL WKR,C,AREA,DH,X,CPRIME,W0,W1,H0
INTEGER WK1,SOR1,GEOM,N
DIMENSION WK1(2),WKR(4),T(36),X(2),XOUT(2)
DATA 1/6.00,4.00,0.60,0.80,1.20,1.40,1.60,2.00,
+ 0.25,1.00,4.00,1000.00,
+ 1.80,1.40,1.10,1.10,1.10,1.10,
+ 1.70,1.40,1.00,0.95,0.90,0.88,
+ 1.50,1.10,0.81,0.76,0.72,0.68,
+ 1.50,1.00,0.69,0.63,0.60,0.55/
10 WRITE (6,600)
   WRITE (6,601)
   CALL HEADR(H0,5)
   WRITE (6,602)
   CALL HEADR(W0,5)
   WRITE (6,603)
   CALL HEADR(W1,5)
   X(1)=W1/W0
   X(2)=H0/W0
   CALL TABLE(T,X,XOUT,CPRIME)
   IF ((XOUT(1).GT.0).OR.(XOUT(2).GT.0)) GO TO 20
   GO TO 10
20 DH=2.0*(H0*W0)/(H0+W0)
   AREA=H0*W0
   WK1(1)=GEOM
   WK1(2)=10
   WKR(1)=AREA
   WKR(2)=DH
   WKR(3)=CPRIME
   WKR(4)=W1*H0
600 FORMAT(' YOU HAVE SELECTED A 90 DEGREE RECTANGULAR ELBOW WITH'/
+ ' EITHER CONVERGING OR DIVERGING FLOW. THE HEIGHT (DIMENSION'/
+ ' PARALLEL TO THE TURN AXIS) SHOULD REMAIN CONSTANT.')
601 FORMAT(' **FIRST QUESTION, WHAT IS THE CROSS-SECTIONAL INLET HEIG
+ H?'')
602 FORMAT(' WHAT IS THE CROSS-SECTIONAL OUTLET HEIGHT (DIMENSION IN
+ THE PLANE OF THE TURN)?')
603 FORMAT(' LAST QUESTION, WHAT IS THE CROSS-SECTIONAL EXIT WIDTH?')
604 FORMAT(' CROSS-SECTION EXTREMELY NARROW, RE-ENTER BETTER DATA.')
RETURN
END

```

```

C***** FITTING 11: ELBOWS 90 DEGREE RECTANGULAR IN Z-SHAPED CONFIG. C
C***** REF. ASHRAE HANDBOOK, PAGE 33.32, TABLE B-3, FITTING 3-11 C
C***** CURVE FIT TO THE TABLE DATA C
C*****
SUBROUTINE FIT11 (SORL, GEOM, WKI, WKR)
REAL WKR, C, AREA, JH, L, W, H, CPRIME, X, Y, K
INTEGER SORL, GEOM, WKI
DIMENSION WKI(2), WKR(4)
WRITE (6, 600)
CALL READR (H, 5)
WRITE (6, 601)
CALL READR (W, 5)
WRITE (6, 602)
CALL READR (L, 5)
X=L/H
Y=W/H
IF ((X.GT.0.0).AND.(X.LT.2.8)) GO TO 10
C=3.4547-0.0992*X
GO TO 20
10 C=((0.85045*X)-5.21052)*X+9.1399)*X-2.168)*X+0.0545
20 CONTINUE
K=0.4704*EXP(-0.3558*Y)+0.67
CPRIME=C*K
JH=2.0*(H*W)/(H+W)
AREA=H*W
WKI(1)=GEOM
WKI(2)=11
WKR(1)=AREA
WKR(2)=JH
WKR(3)=CPRIME
WKR(4)=AREA
600 FORMAT(' YOU HAVE SELECTED A SERIES 90 DEGREE RECTANGULAR ELBOW'//
+ ' SET IN A Z-SHAPED CONFIGURATION.'/' **FIRST QUESTION, WHAT IS
+ ' THE HEIGHT OF THE ELBOW CROSS-SECTION?'/' (DIMENSION IN THE PLA
+ ' NE OF THE TURN)')
601 FORMAT(' WHAT IS THE WIDTH OF THE ELBOW CROSS-SECTION?'/
+ ' (THE DIMENSION PARALLEL TO THE TURN AXIS)')
602 FORMAT(' LAST QUESTION, WHAT IS THE LENGTH BETWEEN CENTERLINES'//
+ ' OF THE "Z" ENTRANCE AND "Z" EXIT?')
RETURN
END

```

```

C*****
C FITTING 12: ELBOWS 90 DEGREE IN DIFFERENT PLANES
C*****
C REF. ASHRAE HANDBOOK, PAGE 33.33, TABLE B-3, FITTING B-12
C CURVE FIT TO THE TABULATED DATA
C*****
SUBROUTINE FIT12 (SORL,GEOM,WKI,WKR)
REAL WKR,C,AREA,DH,L,W,H,CPRIME,X,Y,K
INTEGER SORL,SECN,WKI
DIMENSION WKI(2),WKR(4)
WRITE(6,600)
CALL READR(H,5)
WRITE(6,601)
CALL READR(W,5)
WRITE(6,602)
CALL READR(L,5)
X=L/H
W=W/H
IF ((X.GT.0.0).AND.(X.LT.1.4)) GO TO 10
IF ((X.GE.1.4).AND.(X.LT.2.0)) GO TO 20
IF ((X.GE.2.0).AND.(X.LT.4.0)) GO TO 30
C=3.4-0.10*X
GO TO 40
10 C=((1.79343*X)-5.47366)*X+3.5957)*X+2.29846)*X+1.20
GO TO 40
20 C=((1.34166*X)-5.35713)*X+8.60118)*X-1.0057
GO TO 40
30 C=((0.30983*X)-0.246799)*X+1.154425)*X+1.702965
40 CCNT=1
K=0.4704*EXP(-0.3558*Y)+0.67
CPRIME=C*K
DH=2.0*(H*W)/(H+W)
AREA=H*W
WKI(1)=GEOM
WKI(2)=12
WKR(1)=AREA
WKR(2)=DH
WKR(3)=CPRIME
WKR(4)=AREA
600 FORMAT(' YOU HAVE SELECTED A SET OF 90 DEGREE RECTANGULAR ELBOWS
* IN DIFFERENT PLANES.'/' **FIRST QUESTION, WHAT IS THE HEIGHT OF T
* HE ELBOW CROSS-SECTION?'/ (DIMENSION IN THE PLANE OF THE TURN
*)
601 FORMAT(' WHAT IS THE WIDTH OF THE ELBOW CROSS-SECTION?'/
* (THE DIMENSION PARALLEL TO THE TURN AXIS)')
602 FORMAT(' LAST QUESTION, WHAT IS THE LENGTH BETWEEN CENTERLINES'/
* OF THE "2" ENTRANCE AND "2" EXIT?')
RETURN
END

```

```

C*****
C FITTING 13: BRANCH SECTION DIVERGING WYE
C*****
C REF. IDEL'CHIK, HANDBOOK OF HYDRAULIC RESISTANCE, SECTION SEVEN
C PAGES 247-253
C*****
C SUPERROUTINE FIT 13(SORL,GEOM,WKI,WKR)
C REAL WKR,ALFAD,AC,AB
C INTEGER SORL,SECS,WKI
C DIMENSION WKI(2),WKR(4)
C WRITE(6,600)
C CALL READR(ALFAD,5)
C WRITE(6,601)
C CALL READR(AC,5)
C WRITE(6,602)
C CALL READR(AB,5)
C WKI(1)=GEOM
C WKI(2)=13
C WKR(1)=AC
C WKR(2)=AB
C WKR(3)=ALFAD
C WKR(4)=B
600 *C ***** YOU HAVE SELECTED THE BRANCH SECTION OF A DIVERGENT WYE.
* / THE MODULE COOLING AIR SHOULD BE BRANCHING OFF THE MAIN /
* / INLET AND FLOWING THROUGH THIS SECTION. THIS SHOULD BE THE /
* / FIRST FITTING OF THIS BRANCH. /
* / **FIRST QUESTION, WHAT IS THE ANGLE BETWEEN THE MAIN FLOW /
* / AXIS AND THE BRANCH FLOW AXIS (DEGREES) ? /
601 *C ***** WHAT IS THE CROSS-SECTIONAL AREA OF THE COMBINED FLOW /
* / SECTION? THIS IS WHERE BOTH ENGINE AIR AND COOLING AIR FLOW /
* / JUST UPSTREAM OF THE BRANCH. /
602 *C ***** LAST QUESTION, WHAT IS THE CROSS-SECTIONAL AREA OF THE B
* / RANCH? /
*C *****
C RETURN
C END

```

```

C*****C
C FITTING 14: MAIN SECTION DIVERGING WYE C
C*****C
C REF. IDEL'CHIK, HANDBOOK OF HYDRAULIC RESISTANCE, SECTION SEVEN C
C PAGES 247-253 C
C*****C
SUBROUTINE FIT14(SORL,GEOM,WKI,WKR)
REAL WKR,AM
INTEGER SORL,GEOM,WKI
DIMENSION WKI(2),WKR(4)
WRITE(6,000)
CALL READR(AM,5)
WKI(1)=GEOM
WKI(2)=14
WKR(1)=AM
WKR(2)=0.0
WKR(3)=0.0
WKR(4)=AM
000 FORMAT(' YOU HAVE SELECTED THE MAIN SECTION OF A DIVERGING WYE. '//
* ' THE AIR TO THE ENGINE SHOULD BE FLOWING THRCJGH THIS SECTION. '//
* ' JUST ONE QUESTION, WHAT IS THE CROSS-SECTIONAL AREA OF THE '//
* ' MAIN SECTION? THIS SHOULD BE THE AREA JUST DOWNSTREAM OF THE '//
* ' JUNCTION AND DIRECTS FLOW TO THE ENGINE. IT ALSO SHOULD BE '//
* ' THE FIRST FITTING OF THE BRANCH.')
RETURN
END

```

```

C*****
C***** FITTING 15: BRANCH SECTION CONVERGING WYE *****C
C***** REF. IDEL'CHIK, HANDBOOK OF HYDRAULIC RESISTANCE, SECTION SEVEN *****C
C***** PAGES 247-253 *****C
C*****
SUBROUTINE FIT15(SORL,GEOM,WKI,WKR)
REAL WKR,ALFAC,AC,AB
INTEGER SORL,GEOM,WKI
DIMENSION WKI(2),WKR(4)
WRITE(6,600)
CALL READER(ALFAC,5)
WRITE(6,601)
CALL READER(AC,5)
WRITE(6,602)
CALL READER(AB,5)
WKI(1)=GEOM
WKI(2)=15
WKR(1)=AC
WKR(2)=AB
WKR(3)=ALFAC
WKR(4)=AB
600  FORMAT(' YOU HAVE SELECTED THE BRANCH SECTION OF A CONVERGENT '
+ ' WYE. THE HOT MODULE COOLING AIR SHOULD BE JOINING THE MAIN '
+ ' ENGINE EXHAUST IN THIS WYE. THIS FITTING SHOULD BE THE LAST '
+ ' FITTING IN THE BRANCH. '
+ ' **FIRST QUESTION, WHAT IS THE ANGLE BETWEEN THE MAIN FLOW '
+ ' AXIS AND THE BRANCH AXIS (DEGREES) ?')
601  FORMAT(' WHAT IS THE CROSS-SECTIONAL AREA OF THE COMBINED FLOW '
+ ' SECTION? THIS IS WHERE ENGINE EXHAUST AND MODULE COOLING AIR '
+ ' FLOW JUST DOWNSTREAM OF THE BRANCH. ')
602  FORMAT(' LAST QUESTION, WHAT IS THE CROSS-SECTIONAL AREA OF THE '
+ ' BRANCH? ')
RETURN
END

```



```

C*****
C      FITTING 16: MAIN SECTION CONVERGING WYE
C*****
C      REF. IDEL'CHIK, HANDBOOK OF HYDRAULIC RESISTANCE, SECTION SEVEN
C      PAGES 247-253
C*****
C      SUBROUTINE FIT16(SORL,GEOM,WKI,WKR)
      REAL WKR,AM
      INTEGER SORL,GEOM,WKI
      DIMENSION WKI(2),WKR(4)
      WRITE(16,600)
      CALL HEADR(AM,5)
      WKI(1)=GEOM
      WKI(2)=16
      WKR(1)=AM
      WKR(2)=0.0
      WKR(3)=0.0
      WKR(4)=AM
600  FORMAT(' YOU HAVE SELECTED THE MAIN SECTION OF A CONVERGING'//
+ ' WYE. THE ENGINE EXHAUST ALONG SHOULD BE FLOWING THROUGH'//
+ ' THIS SECTION. IT SHOULD BE THE LAST FITTING OF THE BRANCH.'//
+ ' **JUST ONE QUESTION, WHAT IS THE CROSS-SECTIONAL AREA OF THE'//
+ ' MAIN BRANCH?')
      RETURN
      END

```

```

C*****
C FITTING 17: CONICAL DIFFUSER
C*****
C REF. IDEL'CHIK, HANDBOOK OF HYDRAULIC RESISTANCE, SECTION FIVE,
C PAGE 167
C*****
C SUBROUTINE FIT 17 (SORL, GEOM, WKI, WKR)
C REAL WKR, L, D0, D1, K1, K2, A0, A1, THETA, CEXP, CFPRI
C INTEGER GEOM, SCRL, WKI, ANS, YES, NO
C DIMENSION WKI(2), WKR(4)
C DATA YES/'Y', NO/'N'/
C WRITE(6,600)
C CALL READR(L,5)
C WRITE(6,601)
10 C CALL READR(D0,5)
C WRITE(6,602)
12 C CALL READR(D1,5)
C WRITE(6,603)
C READ(5,604,END=14,ERR=14) ANS
14 IF ((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 16
C REWIND 5
C WRITE(6,608)
C GO TO 12
16 CONTINUE
K1=1.0
IF (ANS.EQ.YES) K1=0.8
A0=0.7854*D0**2
A1=0.7854*D1**2
IF (A1.GT.A0) GO TO 20
C WRITE(6,605)
C GO TO 13
20 THETA=2.0*ATAN((D1-D0)/(2.0*L))
IF (THETA.LT.0.524) GO TO 30
22 C WRITE(6,606)
C READ(5,604,ERR=24,END=24) ANS
24 IF ((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 26
C REWIND 5
C WRITE(6,608)
C GO TO 22
26 CONTINUE
K2=1.0
IF (ANS.EQ.YES) K2=0.65
30 IF (THETA.GT.3.7) GO TO 40
CEXP=1.3454*(THETA**1.2)*(1.0-A0/A1)**2
C GO TO 60
40 IF (THETA.GT.1.05) GO TO 50
CEXP=(((0.3637*THETA)-0.8715)*THETA+3.0218)*THETA-0.8410)*
*(1.0-A0/A1)**2
C GO TO 60
50 CEXP=(((0.0061*THETA)-0.0139)*THETA-0.09293)*THETA+1.2623)*
*(1.0-A0/A1)**2
60 CONTINUE
C WRITE(6,607)
CFPRI=(1.0-(A0/A1)**2)/(8.0*SIN(THETA/2.0))
WKI(1)=GEOM
WKI(2)=17
WKR(1)=A0
WKR(2)=CFPRI*K2
WKR(3)=CEXP*K1*K2
WKR(4)=A1
600 FORMAT(' YOU HAVE SELECTED A CONICAL DIFFUSER WITH CIRCULAR '/
' INLET AND OUTLET SECTIONS.'/
' **FIRST QUESTION: WHAT IS THE LENGTH OF THE DIFFUSER?')
601 FORMAT(' WHAT IS THE INLET DIAMETER?')
602 FORMAT(' WHAT IS THE OUTLET DIAMETER?')
603 FORMAT(' IS THERE A NON-UNIFORM VELOCITY DISTRIBUTION AT THE INLE
' T (Y/N)?')

```

```

004  FORMAT(A1)
005  FORMAT('
+  DOWNSTREAM AREA IS DIVIDED INTO SEVEN UPSTREAM AREA.
+  SINCE THE AREA IS DIVIDED INTO SEVEN UPSTREAM AREA.
+  THE RESULTS OF THE DIVISION ARE AS FOLLOWS:
+  DIVIDING THE AREA INTO SEVEN UPSTREAM AREA.
007  FORMAT('
008  FORMAT('
END

```

```

*****
FITTING 18: PLANE IN-LINE DIFFUSER
*****
REF. IDEL'CHIK, HANDBOOK OF HYDRAULIC RESISTANCE, SECTION FIVE,
PAGE 171
*****
SUBROUTINE FIT18(SCRL,GECM,WKI,WKE)
REAL WKR,L,H,A0,A1,K1,K2,A0,A1,THETA,CEXP,CPRPRI
INTEGER GEOM,K1,K2,ANS,YES,NO
DIMENSION SCRL(4),WKR(4)
DATA YES/1,NO/0,N'/N/
WRITE(6,600)
CALL READR(L,5)
WRITE(6,601)
CALL READR(H,5)
10 WRITE(6,602)
CALL READR(A0,5)
WRITE(6,603)
CALL READR(A1,5)
12 WRITE(6,604)
READ(5,605) END=14,ERR=14) ANS
IF((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 16
14 PRINT
WRITE(6,609)
GO TO 12
16 CONTINUE
K1=1.0
IF((ANS.EQ.YES) K1=6.3
A0=10*H
A1=11*H
IF(A1.GT.A0) GO TO 20
WRITE(6,606)
GO TO 10
20 THETA=2.0*ATAN((W1-W0)/(2.0*L))
IF(THETA.LT.0.524) GO TO 30
WRITE(6,607)
READ(5,608) END=24,ERR=24) ANS
IF((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 26
24 PRINT
WRITE(6,609)
GO TO 22
26 CONTINUE
K2=1.0
IF((ANS.EQ.YES) K2=0.65
IF(THETA.GT.0.7) GO TO 40
CEXP=1.3454*(THETA**1.2)*(1.0-A0/A1)**2
GO TO 60
40 IF(THETA.GT.1.35) GO TO 50
CEXP=((((-0.3637*THETA)-0.3715)*THETA+3.0218)*THETA-0.5410)*
*((1.0-A0/A1)**2)
GO TO 60
50 CEXP=((((-0.0061*THETA)-0.0139)*THETA-0.0929)*THETA+1.2623)*
*((1.0-A0/A1)**2)
60 CONTINUE
CPRPRI=((W0/H)*(1.0-A0/A1)+0.5*(1.0-(A0/A1)**2))/
(4.0*SIN(THETA/2.0))
WRITE(6,608)
WKI(1)=GEOM
WKI(2)=18
WKR(1)=A0
WKR(2)=CPRPRI*K2
WKR(3)=CEXP*K1*K2
WKR(4)=A1
600 FORMAT(' YOU HAVE SELECTED A PLANE INLINE DIFFUSER WITH ONE',
+ ' DIMENSION CONSTANT THROUGHOUT AND RECTANGULAR INLET',
+ ' ANSWER:',
+ ' **FIRST QUESTION, WHAT IS THE LENGTH OF THE DIFFUSER?')

```

```

601  FORMAT(' WHAT IS THE CONSTANT HEIGHT OF THE INLET AND OUTLET ' /
      + ' CROSS SECTIONAL AREA? ' /
602  ' WHAT IS THE INLET CROSS SECTIONAL AREA? ' /
603  ' WHAT IS THE OUTLET CROSS SECTIONAL AREA? ' /
604  ' IS THERE A NON-UNIFORM VELOCITY DISTRIBUTION AT THE INLET ' /
      + ' (Y/N)? ' /
605  ' ' /
606  ' DOWNSTREAM AREA IS SMALLER THAN UPSTREAM AREA. ' /
      + ' PRINTING THIS NOT A DIFFUSION FLOW. ENTER DATA. ' /
607  FORMAT(' SINCE THIS IS A DIFFUSION FLOW, THE PROPER ' /
      + ' INSTALLATIONS OF DIVIDING WALLS OR Baffles CAN REDUCE ' /
      + ' THE RESISTANCE OF THE DIVIDING WALLS OR Baffles. DO YOU WANT TO INSTALL ' /
      + ' DIVIDING WALLS OR Baffles? ' /
608  ' (Y/N)? ' /
609  ' NO MORE INSTALLATIONS FOR THIS FLOW. ' /
      + ' YOU MUST ENTER IN THE BRACKETS. ' )
      + ' ' /
      + ' END ' /

```

```

C*****
C***** FITTING 19: PYRAMIDAL DIFFUSER, IN LINE *****
C***** REF. IDEL'CHIK, HANDBOOK OF HYDRAULIC RESISTANCE, SECTION FIVE, *****
C***** PAGE 169 *****
C*****
SUBROUTINE FIT19(SORL, GECM, WKI, WKR)
REAL WKR, L, H0, W0, H1, A1, K1, K2, A0, A1, ALFA, BETA, THETA, CEXP, CFRPRI
INTEGER GECM, SORL, WKI, ANS, YES, NO
DIMENSION WKI(2), WKR(4)
DATA YES/'Y' /, NO/'N' /
WRITE(6,600)
CALL READR(L,5)
10  WRITE(6,601)
CALL READR(H0,5)
WRITE(6,602)
CALL READR(W0,5)
WRITE(6,603)
CALL READR(H1,5)
WRITE(6,604)
CALL READR(W1,5)
12  WRITE(6,605)
READ(5,606,END=14,ERR=14) ANS
IF ((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 16
14  WRITE(6,610)
GO TO 12
16  CONTINUE
K1=1.0
IF (ANS.EQ.YES) K1=6.8
A0=W0*H0
A1=W1*H1
IF (A1.GT.A0) GO TO 20
WRITE(6,607)
GO TO 10
20  ALFA=2.0*ATAN((W1-W0)/(2.0*L))
BETA=2.0*ATAN((H1-H0)/(2.0*L))
THETA=AMAX1(ALFA,BETA)
THETA=1.0-0.324*THETA GO TO 30
22  WRITE(6,608)
READ(5,606,END=24,ERR=24) ANS
IF ((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 26
24  WRITE(6,610)
GO TO 22
26  CONTINUE
K2=1.0
IF (ANS.EQ.YES) K2=0.65
IF (THETA.GT.0.44) GO TO 40
CEXP=1.0818*(THETA**1.2)*(1.0-A0/A1)**2
GO TO 60
40  THETA=1.05 GO TO 50
CEXP=(((3.3598*THETA)-9.7924)*THETA+9.4559)*THETA-1.9220)*
*(1.0-A0/A1)**2
GO TO 60
50  CEXP=1.1*(1.0-A0/A1)**2
60  CONTINUE
CFRPRI=(1.0-(A0/A1)**2)/(8.0*SIN(THETA/2.0))
WRITE(6,609)
WKI(1)=GECM
WKI(2)=19
WKR(1)=A0
WKR(2)=CFRPRI*K2
WKR(3)=CEXP*K1*K2
WKR(4)=A1
600  PRINT ' YOU HAVE SELECTED A PYRAMIDAL INLINE RECTANGULAR DIFFUSE
      ' FIRST QUESTION, WHAT IS THE LENGTH OF THE DIFFUSER/'

```

```

601 *FORMAT('WHAT IS THE SMALLER DIMENSION OF THE INLET AREA?')
602 *FORMAT('WHAT IS THE SMALLER DIMENSION OF THE INLET AREA?')
603 *FORMAT('WHAT IS THE SMALLER DIMENSION OF THE OUTLET AREA PARALLEL TO THE SMALLER DIMENSION OF THE INLET AREA?')
604 *FORMAT('WHAT IS THE SMALLER DIMENSION OF THE OUTLET AREA PARALLEL TO THE SMALLER DIMENSION OF THE INLET AREA?')
605 *FORMAT('IS THERE A NON-UNIFORM VELOCITY DISTRIBUTION AT THE INLE')
606 *FORMAT('Y/N?')
607 *FORMAT('A')
608 *FORMAT('DOWNSTREAM AREA IS NOT GREATER THAN UPSTREAM AREA.')
609 *FORMAT('SINCE THIS IS A DIFFICULT CASE, PLEASE ENTER DATA.')
610 *FORMAT('INSTALLATIONS OF DIVIDING WALLS OR BAFFLES CAN REDUCE')
611 *FORMAT('THE RESISTANCE OF THIS SYSTEM. DO YOU WANT TO INSTALL')
612 *FORMAT('DIVIDING WALLS OR BAFFLES?')
613 *FORMAT('Y/N?')
614 *FORMAT('NO MORE QUESTIONS THIS TIME.')
615 *FORMAT('YOU MUST ENTER A LETTER IN THE BRACKETS.')
616 *FORMAT('END')

```



```

*****
FITTING 21: CIRCULAR CONTRACTION
*****
REF: ASHRAE HANDBOOK, PAGE 33.34, TABLE B-5, FITTING 5-1
TABLE INTERPOLATION
*****
SUBROUTINE FIT21(SORL,GEOM,WKI,WKR)
REAL WKR,D0,D1,L,THETA,T,C,A1,A0,X
INTEGER SORL,GEOM,WKI,XOUT
DIMENSION WKI(2),WKR(4),T(55),X(2),XOUT(2)
DATA T/8.0,5.0,
+ 0.0,10.0,30.0,55.0,90.0,120.0,150.0,180.0,
+ 1.0,2.0,4.0,6.0,10.0,
+ 0.0,3.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0,
+ 0.0,0.0,0.0,0.05,0.06,0.12,0.18,0.24,0.26,
+ 0.0,0.0,0.0,0.04,0.07,0.17,0.27,0.33,0.41,
+ 0.0,0.0,0.0,0.04,0.07,0.18,0.28,0.36,0.42,
+ 0.0,0.0,0.05,0.05,0.08,0.19,0.29,0.37,0.43/
WRITE(6,600)
CALL READER(1,5)
IF(L.LT.0.05) L=0.05
WRITE(6,601)
CALL READER(D1,5)
WRITE(6,602)
CALL READER(D0,5)
THETA=114.59156*ATAN((D1-D0)/(2.0*L))
A1=0.7854*D1**2
A0=0.7854*D0**2
X(1)=THETA
X(2)=A1/A0
CALL TABLE(T,X,XOUT,C)
WKI(1)=GEOM
WKI(2)=21
WKR(1)=A1
WKR(2)=0.0
WKR(3)=C
WKR(4)=A0
600 FORMAT(' YOU HAVE SELECTED A CIRCULAR CONTRACTION.'/
+ '** FIRST QUESTION, WHAT IS THE LENGTH OF THE CONTRACTION?')
601 FORMAT(' WHAT IS THE UPSTREAM DIAMETER?')
602 FORMAT(' WHAT IS THE DOWNSTREAM DIAMETER?')
RETURN
END

```

```

*****
***** FITTING 22: RECTANGULAR CONTRACTION *****
*****
***** REF: ASHRAE HANDBOOK, PAGE 33.34, TABLE 3-5, FITTING 5-1 *****
*****
*****
SUBROUTINE FIT22(SCRL,SECM,WKI,WKR)
REAL WKR,X,T,A1,B1,A0,B0,AREA1,AREA0,C,L,THETA1,THETA2,THETA
INTEGER SCRL,SECM,WKI,XOUT
DIMENSION WKI(2),WKR(4),T(55),X(2),XOUT(2)
DATA T/8.0,5.0,
* 0.0,10.0,30.0,55.0,90.0,120.0,150.0,180.0,
* 1.0,2.0,3.0,4.0,5.0,6.0,7.0,8.0,9.0,10.0,
* 0.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0,
* 0.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0,
* 0.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0,
* 0.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0,0.0/
WRITE(6,600)
CALL READER(L,5)
L=L-0.05
WRITE(6,601)
CALL READER(A1,5)
WRITE(6,602)
CALL READER(B1,5)
WRITE(6,603)
CALL READER(A0,5)
WRITE(6,604)
CALL READER(B0,5)
THETA1=ATAN((A1-A0)/(2.0*L))
THETA2=ATAN((B1-B0)/(2.0*L))
THETA=AMAX1(THETA1,THETA2)
AREA1=A1*B1
AREA0=A0*B0
X(1)=THETA
X(2)=AREA1/AREA0
CALL TABLE(T,X,XOUT,C)
WKI(1)=SCRM
WKI(2)=SECM
WKR(1)=AREA1
WKR(2)=0.0
WKR(3)=C
WKR(4)=AREA0
600 FORMAT('** YOU HAVE SELECTED A RECTANGULAR CONTRACTION. /
* FIRST CROSS-SECTION SHAPE IS: ',X(1),X(2),XOUT(1),XOUT(2),
601 FORMAT('** WHAT IS THE GRATE SPACING? ',X(1),X(2),XOUT(1),XOUT(2),
602 FORMAT('** WHAT IS THE GRATE SPACING? ',X(1),X(2),XOUT(1),XOUT(2),
603 FORMAT('** WHAT IS THE GRATE SPACING? ',X(1),X(2),XOUT(1),XOUT(2),
604 FORMAT('** LAST CROSS-SECTION SHAPE IS: ',X(1),X(2),XOUT(1),XOUT(2),
* CROSS-SECTION DIMENSION?')
RETURN
END

```

```

C*****
C FITTING 23: SCREEN
C*****
C REF. ASHRAE HANDBOOK, PAGE 33-42, TABLE B-7, FITTING 7-3
C CURVE FIT TO TABULATED DATA, BASED ON DUCT AREA AND SCREEN
C FREE FLOW AREA.
C*****
C SUBROUTINE FIT23 (SORL,GEOM,WKI,WKR)
C REAL WKR,DUCTA,SCRNA,N,C
C INTEGER SORL,GEOM,WKI
C DIMENSION WKI(2),WKR(4)
C PRINT* (6,600)
C CALL READR (DUCTA,5)
C PRINT* (6,601)
C CALL READR (SCRNA,5)
C N=SCRNA/DUCTA
C=(((97.9021*N)-92.445)*N+32.066)*N-1.9557)*N+0.025
C WKI(1)=GEOM
C WKI(2)=23
C WKR(1)=DUCTA
C WKR(2)=0.0
C WKR(3)=C
C WKR(4)=DUCTA
600 * FORMAT(' YOU HAVE SELECTED A SCREEN OBSTRUCTION IN THE DUCT.'/
* ' FIRST QUESTION, WHAT IS THE DUCT CROSS-SECTIONAL AREA?')
601 * FORMAT(' LAST QUESTION, WHAT IS THE FREE FLOW AREA OF THE SCREEN?')
* RETURN
END

```

```

*****
***** FITTING 24: LOUVER ENTRANCE *****
***** REM. HANDBOOK OF HYDRAULIC RESISTANCE, IDEL'CHIK *****
***** CURVE FIT TO DYNAMIC LOSS INFORMATION, NO FRICTION INCLUDED *****
*****
SUBROUTINE FIT24(SOBL,SECM,NKI,NKR)
  DIMENSION DB(4),DBL(4),DUCTA(4),C
  DIMENSION N(4),N1(4),N2(4),N3(4),N4(4)
  DIMENSION N1(2),N2(2),N3(2),N4(2)
  DIMENSION N1(2),N2(2),N3(2),N4(2)
  CALL READR(DX,S)
  CALL READR(DB,S)
  CALL READR(DBL,S)
  CALL READR(N,S)
  CALL READR(N1,S)
  CALL READR(N2,S)
  CALL READR(N3,S)
  CALL READR(N4,S)
  CALL READR(DUCTA,S)
  C=62.144*EXP(-4.47543*F)
  N1(1)=SECM
  N1(2)=24
  N2(1)=DUCTA
  N2(2)=0.0
  N3(1)=C
  N3(2)=DUCTA
  N4(1)=C
  N4(2)=DUCTA
600  FORMAT(' YOU HAVE SELECTED A LOUVERED ENTRANCE.'/
  *      ' **FIRST QUESTION, WHAT IS THE DISTANCE ACROSS THE '/
  *      ' LOUVER OPENINGS?')
601  FORMAT(' WHAT IS THE DISTANCE BETWEEN THE LOUVERS, USE THE '/
  *      ' CLOSEST DISTANCE.')
602  FORMAT(' HOW MANY OPENINGS ARE THERE BETWEEN THE LOUVERS?')
603  FORMAT(' **SECOND QUESTION, WHAT IS THE AREA OF THE DUCT JUST'/
  *      ' INSIDE THE LOUVER ENTRANCE?')
  RETURN
END

```

```

*****
C***** FITTING 25: INLET FILTER *****C
C***** REF. NAVSEA INLET DESIGN HANDBOOK *****C
C***** DD963 TYPE FILTER CURVE FIT 2C DATA *****C
C***** CONICAL FILTER POWER CURVE FITTER IS MADE TO PRESSURE LOSS DATA *****C
C***** BASED ON FACE VELOCITY ON FILTER. DATA SUPPLIED BY USER. *****C
C*****
SUBROUTINE FIT25(SCRL,GEOM,WKI,WKR)
REAL*4 AREA,VEL,DELP,XX,YY,SUMX,SUMY,SUMX2,SUMY2,SUMXY,A,B
INTEGER SCRL,GEOM,WKI,WKR,N,ANS,IPTS,NPTS,NC
DIMENSION WKI(2),WKR(4),VEL(10),DELP(10),XX(10),YY(10)
DATA YES/'Y',NO/'N'
WRITE(6,600)
CALL READR(AREA,5)
WRITE(6,601)
READ(5,602,END=4,ERR=4) ANS
IF((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 6
REWIND 5
WRITE(6,607)
GO TO 2
CONTINUE
IF(ANS.EQ.YES) GO TO 30
WRITE(6,603)
CALL READI(NPTS,5)
DO 10 I=1,NPTS
WRITE(6,604) I
CALL READR(VEL(I),5)
WRITE(6,605) I
CALL READR(DELP(I),5)
XX(I)=A*LOG(VEL(I))
YY(I)=A*LOG(DELP(I))
10 CONTINUE
SUMX=0.0
SUMY=0.0
SUMX2=0.0
SUMY2=0.0
SUMXY=0.0
DO 20 I=1,NPTS
SUMX=SUMX+XX(I)
SUMY=SUMY+YY(I)
SUMX2=SUMX2+XX(I)**2
SUMY2=SUMY2+YY(I)**2
SUMXY=SUMXY+XX(I)*YY(I)
20 CONTINUE
N=IPTS*(IPTS+1)/2
B=(N*SUMXY-SUMX*SUMY)/(N*SUMX2-(SUMX**2))
A=EXP(SUMY/N-B*SUMX/N)
WRITE(6,606)
GO TO 40
30 A=0.0167
B=1.6287
WRITE(6,606)
40 CONTINUE
WKI(1)=SCRL
WKI(2)=GEOM
WKR(1)=AREA
WKR(2)=A
WKR(3)=B
WKR(4)=AREA
600 FORMAT(' YOU HAVE SELECTED THE INLET FILTER. '//
+ ' FIRST QUESTION: WHAT IS THE TOTAL FACE AREA OF THE FILTER?')
601 FORMAT(' DO YOU WANT TO USE THE DD963 TYPE FILTER IN THE DRY COND'
+ ' ITION (Y/N)?')
602 FORMAT(A1)
603 FORMAT(' THE OPERATING CHARACTERISTICS OF YOUR FILTER WILL BE '//
+ ' DEFINED BY A POWER CURVE FIT OF THE FORM: '//
+ ' DELTA PRESSURE = COEFA * FACE VELOCITY** COEFS '//
+ ' APPLIED TO PERFORMANCE DATA TO BE INPUT BY THE USER. '//

```



```

*****
FITTING 26: SILENCER MULTI-BAFFLE TYPE
*****
REF. NAVSEA INLET DESIGN HANDBOOK
COMPOSITE LOSS COEFFICIENT BASED ON A SUDDEN CONTRACTION,
FRICTION AND A SUDDEN EXPANSION
*****
SUBROUTINE FIT26(SCRL,GEOM,WKI,FKR)
REAL WKR(3),L,E,CX,A0,A1,DH,R,C1,C2,C3,C,N
INTEGER SOAL,JECH,WKI
DIMENSION WKI(2),WKR(4)
WRITE(6,600)
CALL READR(G,5)
WRITE(6,601)
CALL READR(T,5)
WRITE(6,602)
CALL READR(L,5)
WRITE(6,603)
CALL READR(H,5)
WRITE(6,604)
CALL READR(CX,5)
WRITE(6,605)
CALL READR(N,5)
A0=CX*H
A1=N*G*H
DH=2.0*(H/(G+H))
R=(CX-N*G)/(2.0*N)
SUDDEN CONTRACTION
C1=0.114*((R/DH+0.1)**14.4405)*(1.0-A1/A0)
FRICTION
C2=0.05*L/DH
SUDDEN EXPANSION
C3=0.47*(1.0-N*G/CX)**2+0.02
COMPOSITE COEFFICIENT
C=C1+C2+C3
WKI(1)=SCRL
WKI(2)=JECH
WKR(1)=A0
WKR(2)=0.0
WKR(3)=C
WKR(4)=A0
600 FORMAT(' YOU HAVE SELECTED A MULTI-BAFFLE TYPE SILENCER.'/
+ ' EACH BAFFLE HAS A STREAMLINED SHAPE. IT IS THE TYPE'/
+ ' USED IN THE INLETS OF THE DDG63.'/
+ '**FRICTION QUESTION, WHAT IS THE GAP BETWEEN THE BAFFLES?')
601 FORMAT(' WHAT IS THE THICKNESS OF THE BAFFLES?')
602 FORMAT(' WHAT IS THE LENGTH OF THE BAFFLES?')
603 FORMAT(' WHAT IS THE DIMENSION OF THE BAFFLES PARALLEL TO THE GAP')
604 FORMAT(' WHAT IS THE DIMENSION OF THE MAIN DUCT ACROSS THE GAPS?')
605 FORMAT(' LAST QUESTION, HOW MANY GAPS ARE THERE?')
RETURN
END

```



```

*****
***** FITTING 27: GAS TURBINE MODULE *****
*****
***** GENERAL ELECTRIC DATA, LOSSES IN THE MODULE BASED ON THE *****
***** MASS FLOW THROUGH THE MODULE. NO LOSS HAS BEEN PASSED HERE. *****
***** SUBROUTINE JUST LOCATES THE MODULE. NO LOSS HAS BEEN PASSED HERE. *****
***** FLOW PATH NO. THROUGH THE ENGINE. LOSSES WILL BE IN THE COOLING FLOW. *****
*****
SUBROUTINE FIT27(SORL,GEOM,WKI,WKR)
  REAL WKR,AO
  INTEGER SORL,JEOM,WKI
  DIMENSION WKI(2),WKR(4)
  WKI(1)=6.000
  WKI(2)=GEOM
  WKR(1)=2.7
  WKR(2)=1.0
  WKR(3)=1.0
  WKR(4)=1.0
600  FORMAT(' YOU HAVE SELECTED THE GAS TURBINE MODULE AS A PART OF'//
  + ' THE COOLING FLOW PASSAGE. NO QUESTIONS, JUST NEEDED'//
  + ' TO KNOW WHERE YOU WANTED THE MODULE.')
  RETURN
END

```

```

*****
***** RITTING 28: WASTE HEAT RECOVERY BOILER *****
***** REF. EXTENDED SURFACE HEAT TRANSFER, D.Q. KERNS AND A.D. KRAUSS *****
***** PAGES 582-589 *****
***** PRELIMINARY DRAWINGS ON THE RACER SYSTEM *****
*****

SUBROUTINE FIT28(SOCL,GEOM,WKL,WKR)
REAL WKR,DV,DE,ST,N,TL,VFACT,TEST,CPRIME,AREA
INTEGER SOCL,GEOM,WKL,ANS,STAG,INLINE,YES,NO
COMMON /GEOM/ WKL(2),WKE(4)
DATA STAG/'S'/,INLINE/'I'/,YES/'Y'/,NO/'N'/'
WRITE(6,600)
READ(5,601) END=20,ERR=20) ANS
IF((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 30
WRITE(6,602)
GO TO 40
30 CONTINUE
AREA=68.75
VFACT=1.35095
CPRIME=335.22027
DV=0.03987
IF(ANS.EQ.YES) GO TO 80
WRITE(6,603)
CALL READR(DV,5)
WRITE(6,604)
CALL READR(DE,5)
WRITE(6,605)
CALL READR(ST,5)
40 WRITE(6,606)
READ(5,607) END=50,ERR=50) ANS
IF((ANS.EQ.STAG).OR.(ANS.EQ.INLINE)) GO TO 60
WRITE(6,602)
GO TO 40
60 CONTINUE
WRITE(6,607)
CALL READR(N,5)
WRITE(6,608)
CALL READR(SL,5)
WRITE(6,609)
CALL READR(L,5)
WRITE(6,610)
CALL READR(XL,5)
C PRINT*,HIGHEST VELOCITY OF THE TUBE BANK
VFACT=ST/(ST-DE)
IF(ANS.EQ.INLINE) GO TO 70
VFACT=(ST+DE)/2.0
SQR=SQRT(SL**2+(ST/2.0)**2)
VFACT=ST/(2.0*(SD-DE))
70 CONTINUE
X=((DV/ST)**0.4)*((SL/ST)**0.6)*SL*N/DV
80 COMMON /FAC/ WKL(1)=GEOM
WRITE(6,611)
WRITE(6,612)
WRITE(6,613)
WRITE(6,614)
WRITE(6,615)
WRITE(6,616)
600 FORMAT(' YOU HAVE SELECTED A WASTE HEAT BOILER. DO YOU WANT TO /
* USE THE PROPOSED RACER DESIGN DEVELOPED BY SOLAR ?')
601 FORMAT(A1)
602 FORMAT(' YOU MUST USE A LETTER IN THE BRACKETS.')
603 FORMAT(' A NUMBER OF QUESTIONS ARE REQUIRED ABOUT THE
* BUNDLE GEOMETRY TO OBTAIN LOSS COEFFICIENTS.')

```

AD-A148 708

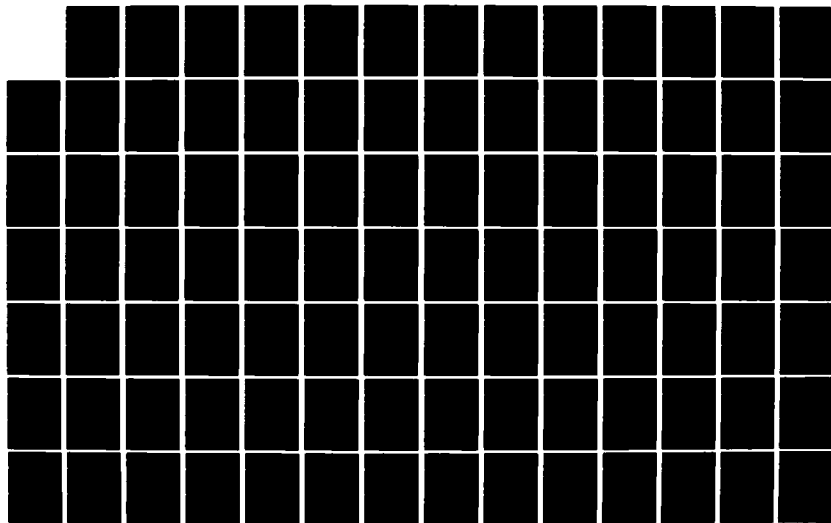
AN ANALYTIC MODEL OF GAS TURBINE ENGINE INSTALLATIONS
(U) NAVAL POSTGRADUATE SCHOOL MONTEREY CA S M EZZELL
SEP 84

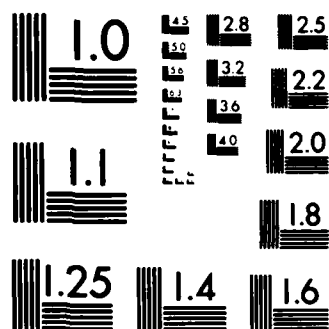
2/3

UNCLASSIFIED

F/G 21/5

NL





MICROCOPY RESOLUTION TEST CHART
NATIONAL BUREAU OF STANDARDS-1963-A

```

*      : USE CONSISTENT UNITS (FEET). // VOLUMETRIC HYDRAULIC //
*      : **FIRST QUESTION, WHAT IS THE VOLUMETRIC HYDRAULIC //
*      : DIAMETER? IF YOU DO NOT KNOW 0.340 FEET IS A GOOD //
*      : GUESS FOR THIS APPLICATION. //
604 * FORMAT('WHAT IS THE DIAMETER OR EQUIVALENT DIAMETER OF A //
*      : FINNET TUBE IN THE BUNDLE (FEET)? IF YOU DO NOT //
*      : KNOW 1.4 TIMES THE BARE TUBE DIAMETER IS A GOOD //
*      : GUESS.')
605 * FORMAT('WHAT IS THE TUBE SPACING IN A BANK OF TUBES (FEET)? //
*      : TUBE CENTERLINE TO TUBE CENTERLINE.')
606 * FORMAT('ARE THE TUBE BANKS STAGGERED OR INLINE (S/I) ?')
607 * FORMAT('HOW MANY TUBE BANKS ARE THERE ?')
608 * FORMAT('WHAT IS THE DISTANCE BETWEEN THE TUBE BANKS //
*      : FROM THE PLANE OF A TUBE CENTERLINE TO TUBE CENTERLINE //
*      : PLANE OF THE NEXT BANK.')
609 * FORMAT('WHAT IS THE DUCT DIMENSION PARALLEL TO THE TUBES ?')
610 * FORMAT('WHAT IS THE DUCT DIMENSION ACROSS THE TUBES ?')
      RETURN
      END

```

```

C*****
C FITTING 29: ABRUPT EXIT
C*****
C REF. ASHRAE HANDBOOK PAGE 33.29, TABLE B-2, FITTING 2-1
C THIS SHOULD ALWAYS BE USED FOR THE FASTER FITTING OF THE ENGINE
C EXHAUST BRANCH, NODE SIX. IT MAY BE REQUIRED FOR THE COOLING
C FLOW IT GOES DIRECTLY TO THE ATMOSPHERE (CLASS 10).
C*****
SUBROUTINE FIT29(SORL,GEOM,*K1,*K2)
REAL *K1,*K2
INTEGER SORL,GEOM,*K1
DIMENSION *K1(2),*K2(4)
WRITE(6,600)
CALL READR(AR2A,5)
*K1(1)=GEOM
*K1(2)=29
*K2(1)=AREA
*K2(2)=0.0
*K2(3)=0.0
*K2(4)=1.0
600 FORMAT(' YOU HAVE SELECTED AN ABRUPT EXIT TO THE ATMOSPHERE.'/
+ ' **JUST ONE QUESTION, WHAT IS THE AREA OF THE EXIT PLANE?
+ ')
RETURN
END

C*****
C FITTING 30: FITTING OF YOUR CHOICE, NOT ON MENU
C*****
C NO REFERENCE. THIS IS INTENDED TO BE A CATCH ALL FITTING FOR
C THOSE FITTINGS NOT LISTED ON THE MENU. IT INPUTS A CONSTANT
C COEFFICIENT FOR MULTIPLICATION TO THE PRESSURE VELOCITY. THE
C VELOCITY IS COMPUTED THROUGH THE AREA INPUT REQUESTED.
C*****
SUBROUTINE FIT30(SORL,GEOM,*K1,*K2)
REAL *K1,*K2
INTEGER SORL,GEOM,*K1
DIMENSION *K1(2),*K2(3)
WRITE(6,600)
CALL READR(A1,5)
WRITE(6,601)
CALL READR(C,5)
WRITE(6,602)
CALL READR(AO,5)
*K1(1)=GEOM
*K1(2)=30
*K2(1)=A1
*K2(2)=C
*K2(3)=AO
600 FORMAT('
+ ' SINCE THE PROGRAM IS LIMITED IN THE NUMBER OF FITTINGS'
+ ' FOR WHICH IT CAN PRODUCE PERFORMANCE CHARACTERISTICS'
+ ' THIS OPTION ALLOWS THE USER TO INPUT CHARACTERISTICS'
+ ' OF A FITTING NOT LISTED.'
+ ' **FIRST QUESTION, WHAT IS THE CHARACTERISTIC AREA OF'
+ ' THE FITTING? THROUGH THIS AREA THE FLOW PRODUCES'
+ ' A VELOCITY USED TO CALCULATE THE VELOCITY PRESSURE.')
601 FORMAT('
+ ' WHAT IS THE MULTIPLIER COEFFICIENT USED IN THE'
+ ' VELOCITY PRESSURE EXPRESSION: '
+ ' P=CO*PHO*(VELOCITY**2)/(2*GCG) ? ')
602 FORMAT('
+ ' LAST QUESTION, WHAT IS THE OUTLET AREA?')
RETURN
END

```

```

C*****
C      TABLE INTERPOLATION SUBROUTINE:  PRODUCES VALUE FROM 2-D TABLE
C*****
C      INPUT A ONE DIMENSIONAL ARRAY "M", CONTAINING THE FOLLOWING
C      INFORMATION:  NUMBER OF X'S, NUMBER OF Y'S, THE X'S, THE Y'S, THE
C      TABLE STARTING WITH THE SMALLEST X-Y VALUE INPUT BY ROW
C      INCREASING X VALUES WITH ROWS INPUT WITH INCREASING Y VALUES.
C*****
C      SUBROUTINE TABLE(T,X,XOUT,FF)
C      INPUT:  X(200),X(2),NN(2),XOUT(2),F(100)
C      REAL NEW
C      INTEGER V(2),XINIT(2),YINC(2)
C      NXI=1
C      NN(1)=3
C      NN(2)=3
C      IO=3
C      NXI=3
C      N=1
C      LOOP DETERMINES STARTING POINTS IN T ARRAY FOR INTERPOLATION
C      DO 20 I=1,2
C      K=NXI+T(I)-1
C      IF (X(I).GE.T(NXI).AND.X(I).LE.T(K)) GO TO 32
C      IF X OUT OF RANGE, INFORM USER THAT TABLE INTERPOLATION IS NOT
C      POSSIBLE.
C      XOUT(I)=0
C      GO TO 999
C 32 XOUT(I)=1
C      J=NXI
C      L=(J+K)/2
C 21 IF ((X(I)-T(J))*(X(I)-T(L)).GT.0.) GO TO 23
C      K=L
C      GO TO 24
C 23 J=L
C 24 IF ((K-J).GT.1) GO TO 21
C      L=K-NN(I)/2
C      IF (L.LE.NXI) GO TO 25
C      K=NXI+T(I)-NN(I)
C      IF (L.GT.K) L=K
C      GO TO 26
C 25 L=NXI
C 26 I=IO+T(I)+(L-NXI)*N
C      X=0
C      IF (I.NE.1) IA=NN(I)
C      YINC(I)=M*(T(I)-IA)
C      XINIT(I)=L
C      NYI=NXI+T(I)
C      N=NXI+T(I)
C 20 CONTINUE
C      V(2)=1
C      L=NN(1)
C      NXX=NN(1)
C      INTERPOLATE IN FIRST DIMENSION
C      V(1)=NXX
C      DO 11 J=1,NXX
C      F(J)=NEW(X(1),T,XINIT(1),T,IO,L)
C      INTERPOLATE IN SECOND DIMENSION
C 11 L=XC+YINC(1)
C      N=NN(1)
C      F(N+V(2))=NEW(X(2),T,XINIT(2),F,1,N)
C      FF=F(N+V(2))
C      RETURN
C 999 PF=1000.0
C      RETURN
C      END
C      FUNCTION TO RETURN INTERPOLATED VALUE FROM TABLE
C      FUNCTION NEW(X,AX,NX,AY,NY,N)
C      FUNCTION NEW PERFORMS AN "N" POINT INTERPOLATION FOR X
C      STARTING AT ABSCISSA ARRAY AX(NX) AND ORDINATE ARRAY

```

```

C      AY(NY) WITH THE INTERPOLATED VALUE RETURNED IN NEV
      DIMENSION AX(1),AY(1),F(100)
      REAL NEV
      DO 10 J=1,N
10      F(J)=AY(NY+J-1)
      N1=N-1
      DO 20 J=1,N1
      NJ=N-J
      DO 20 I=1,NJ
      KI=NY+I-1
20      F(I)=(F(I+1)-F(I))*(X-AX(KI))/(AX(KI+J)-AX(KI))+F(I)
      CONTINUE
      NEV=F(1)
      RETURN
      END

```



```

C*****
C DUCT DATA FILE OUTPUT SUBROUTINE
C*****
C WRITES THE SYSTEM ARRAYS WORK1 AND WORKR TO THE DUCT DATA FILE.
C ALLOWS THE USER TO SERIALIZE EACH FILE CREATED.
C *WARNING* WRITES OVER OLD FILES, SAVE THEM UNDER A DIFFERENT NAME.
C*****
SUBROUTINE SUNCUT(WORK1,WORKR,N)
REAL WORKR
INTEGER WORK1,N,SERIAL
DIMENSION WORK1(200,2),WORKR(200,4)
WRITE(6,600)
CALL READI(SERIAL,5)
WRITE(6,601) SERIAL
WRITE(6,602) N
DO 10 I=1,N
  WRITE(8,603) I,WORK1(I,1),WORK1(I,2),WORKR(I,1),WORKR(I,2),
    & WORKR(I,3),WORKR(I,4)
10 CONTINUE
REWIND 8
600 FORMAT(' WHAT SERIAL NUMBER WOULD YOU LIKE TO GIVE THIS DUCT DATA
  & FILE?',/,' YOU MAY USE UP TO A SIX DIGIT INTEGER NUMBER.')
```

```

601 FORMAT(I6)
602 FORMAT(I3)
603 FORMAT(I3,3X,I6,3X,I2,3X,F10.4,3X,F10.4,3X,F10.4,3X,F10.4)
RETURN
END
```

```

C*****
C REAL NUMBER REAL SUBROUTINE: FREE FORMAT
C PREVENTS THE INADVERTENT ENTRY OF NULL DATA (HITTING THE RETURN
C KEY WITH NO ENTRY) AND INCORRECT DATA, THIS ROUTINE IS USED.
C IT ALLOWS FREE FORMAT INPUT. TWO NULLS KILLS THE PROGRAM.
C*****
SUBROUTINE REALR (ANSR,FD)
REAL ANSR
INTEGER COUNT,FE
COUNT=0
10 CONTINUE
COUNT=COUNT+1
IF (COUNT.EQ.3) GO TO 20
CALL FETCHS ('CLASCRN ')
WRITE(6,600)
GO TO 40
20 CONTINUE
READ (FD,*,END=30,ZRR=30) ANSR
RETURN
30 REWIND FD
WRITE(6,601)
GO TO 10
40 CONTINUE
STOP
600 FORMAT (////' PROGRAM KILLED - TWO NULL STRINGS ENTERED!'/)
601 FORMAT (' WARNING: NULL STRINGS ARE NOT ALLOWED, ENTER A NUMERIC
  & VALUE.')
```

END


```

C*****C
C POWER POINT INPUT SUBROUTINE (HORSEPOWER, POWER TURBINE SPEED) C
C DCISE A PRELIMINARY TEST TO INSURE TJSOUR LIMITS ARE NOT EXCEED C
C A TIME-DEPENDENT OPERATING POINT. THIS IS THE CASE. C
C*****C
SUBROUTINE PWRPT (HP, NPT, T0, T1, T2, T3, T4, T5, T6, T7, T8, T9, T10, T11, T12, T13, T14, T15, T16, T17, T18, T19, T20, T21, T22, T23, T24, T25, T26, T27, T28, T29, T30, T31, T32, T33, T34, T35, T36, T37, T38, T39, T40, T41, T42, T43, T44, T45, T46, T47, T48, T49, T50, T51, T52, T53, T54, T55, T56, T57, T58, T59, T60, T61, T62, T63, T64, T65, T66, T67, T68, T69, T70, T71, T72, T73, T74, T75, T76, T77, T78, T79, T80, T81, T82, T83, T84, T85, T86, T87, T88, T89, T90, T91, T92, T93, T94, T95, T96, T97, T98, T99, T100)
10  RETURN HP, NPT, T0, T1, T2, T3, T4, T5, T6, T7, T8, T9, T10, T11, T12, T13, T14, T15, T16, T17, T18, T19, T20, T21, T22, T23, T24, T25, T26, T27, T28, T29, T30, T31, T32, T33, T34, T35, T36, T37, T38, T39, T40, T41, T42, T43, T44, T45, T46, T47, T48, T49, T50, T51, T52, T53, T54, T55, T56, T57, T58, T59, T60, T61, T62, T63, T64, T65, T66, T67, T68, T69, T70, T71, T72, T73, T74, T75, T76, T77, T78, T79, T80, T81, T82, T83, T84, T85, T86, T87, T88, T89, T90, T91, T92, T93, T94, T95, T96, T97, T98, T99, T100
20  CALL READR (HP, 5)
    WRITE (6, 601)
    CALL READR (NPT, 5)
    IF ((NPT.GE.1200.0).AND.(NPT.LE.3600.0)) GO TO 30
    WRITE (6, 603)
    GO TO 20
30  CONTINUE
    HP = (5177.5 - 7.0527 * T0) + (8.3275 - 0.0012 * T0) * NPT
    IF (HP.LT.HPTP) GO TO 40
    WRITE (6, 602)
    GO TO 10
40  CONTINUE
    CALL FRICHS ('CLPSCHN ')
600  FORMAT (' INPUT THE POWER SETTING YOU DESIRE. /'
    * ' ***WHAT IS THE HORSEPOWER?')
601  FORMAT (' ***WHAT IS THE POWER TURBINE SPEED (RPM)?')
602  FORMAT (' HORSEPOWER IS NOT ON THE PERFORMANCE MAP, PICK A LOWER H'
    * 'ORSEPOWER.')
603  FORMAT (' POWER TURBINE RPM IS NOT REASONABLE. IT SHOULD BE' /
    * ' 1200 TO 3600 RPM. RE-ENTER.')
    RETURN
    END

```



```

10 GO TO 30
   ADELPI = 0.00075
20 GO TO 30
   ADELPI = 0.001667
30 GO TO 30
   ADELPI = 0.000475
   ADELPI = 0.005423/100.0
   CF = (1.0 + ADELPI * INLOSS) * (1.0 + ADELH * HUMID) *
   * DELTA / SQRT (THETA)
   C = CF
   RETURN
END

C FUNCTION TO CORRECT T8
FUNCTION CFT8 (T8, THETA, INLOSS, EXLOSS, HUMID, NGC)
REAL T8, THETA, INLOSS, EXLOSS, HUMID, NGC, ADELPI, ADELPE, ADELH,
   * CF, T8
   IF (NGC - 9100.0) GO TO 10
   IF (NGC - 9200.0) GO TO 20
   ADELPI = 0.00105 + (NGC - 9100.0) * 0.001242/100.0
10 ADELPI = 0.00105
20 ADELPI = 0.002292
30 ADELPI = 0.00095
   ADELPE = 0.000643/100.0
   CF = (1.0 + ADELPI * INLOSS) * (1.0 + ADELH * HUMID) *
   * DELTA
   C = CF
   RETURN
END

C FUNCTION TO CORRECT T8
FUNCTION CFT8 (T8, THETA, INLOSS, EXLOSS)
REAL T8, THETA, INLOSS, EXLOSS, CF, T8
   CF = (1.0 + 0.00241 * EXLOSS) * DELTA
   C = CF
   RETURN
END

C FUNCTION TO CORRECT T54
FUNCTION CFT54 (T54, THETA, INLOSS, EXLOSS, HUMID, NGC)
REAL T54, THETA, INLOSS, EXLOSS, HUMID, NGC, ADELPI, ADELPE, ADELH,
   * CF, T54
   IF (NGC - 9100.0) GO TO 10
   IF (NGC - 9200.0) GO TO 20
   ADELPI = 0.000958 + (NGC - 9100.0) * 0.001/100.0
10 ADELPI = 0.000958
20 ADELPI = 0.001958
30 ADELPI = 0.00056
   ADELPE = 0.002057/100.0
   CF = (1.0 + ADELPI * INLOSS) * (1.0 + ADELH * HUMID) *
   * DELTA
   C = CF
   RETURN
END

```


[illegible]

107


```

C*****FAN CHARACTERISTICS INPUT SUBROUTINE*****C
C
C   THE DEFAULT FAN CHARACTERISTIC WAS PROVIDED BY JOY MANUFACTURING
C   COMPANY AND IS FOR THE FAN INSTALLED ON THE SPRUANCE CLASS
C   DESTROYER. OTHER FANS ARE MODELED AS A QUADRATIC EQUATION
C   WITH A MAXIMUM AT MAXIMUM FAN PRESSURE AND DISCHARGE
C   AND ANOTHER POINT AT MAXIMUM DISCHARGE AND ZERO FAN PRESSURE.
C*****C
C   SUBROUTINE FAN(RHOSTD,CFM0,CFMMAX,DPMAX,K)
C   REAL RHOSTD,CFM0,CFMMAX,DPMAX,K
C   INTEGER YES,ANS,NO
C   DATA YES/'Y'/,NO/'N'/
C   DO YOU WANT THE DEFAULT FAN, THE DD 963 CLASS DESTROYER FAN ???
C   2  WRITE(6,600)
C   READ(5,601,END=4,ERR=4) ANS
C   4  IF((ANS.EQ.YES).OR.(ANS.EQ.NO)) GO TO 3
C   WRITE(6,602)
C   GO TO 2
C   8  CONTINUE
C   IF(ANS.EQ.YES) GO TO 10
C   A DIFFERENT FAN HAS BEEN SELECTED, INPUT REQUIRED PARAMETERS
C   WRITE(6,602)
C   CALL READR(RHOSTD,5)
C   WRITE(6,603)
C   CALL READR(CFM0,5)
C   WRITE(6,604)
C   CALL READR(CFMMAX,5)
C   WRITE(6,605)
C   CALL READR(DPMAX,5)
C   K=-1.0*DPMAX/(CFM0-CFMMAX)**2
C   GO TO 20
C   10  CALCULATING ARE VALUES FOR THE DEFAULT FAN
C   RHOSTD=0.071
C   CFM0=24000.0
C   CFMMAX=11000.0
C   DPMAX=27.7
C   K=-1.0*DPMAX/(CFM0-CFMMAX)**2
C   20  CONTINUE
C   600  FORMAT(' YOU HAVE SELECTED A SYSTEM WITH A COOLING FAN. THE'//
C   + ' DEFAULT SPECIFICATIONS ARE FOR THE FAN INSTALLED ON'//
C   + ' THE DD963 CLASS SHIP.'//
C   + ' DO YOU WANT TO USE THE DEFAULT SPECIFICATIONS (Y/N)?')
C   601  FORMAT('A')
C   602  + ' THE PROGRAM WILL APPROXIMATE YOUR FAN WITH A QUADRATIC'//
C   + ' EQUATION. TWO POINTS ARE REQUIRED FROM THE FAN PERFORMANCE'//
C   + ' CURVES AND THE REFERENCE AIR DENSITY OF THE CURVES. WHAT'//
C   + ' IS THE REFERENCE AIR DENSITY (LBM/FT3)?'
C   603  FORMAT(' WHAT IS THE FLOW AT ZERO GAGE PRESSURE (CFM)?')
C   604  FORMAT(' WHAT IS THE FLOW AT MAXIMUM GAGE PRESSURE (CFM)?')
C   605  + ' D(CFM)/DP=0.'//
C   + ' WHAT IS THE MAXIMUM FAN DISCHARGE GAGE PRESSURE (INCHES'//
C   + ' W.G.)?'//
C   + ' D(CFM)/DP=0.'//
C   RETURN
C   END

```

202


```

C      EXIT (CONDITION CONDITION IS NOT AMBIENT PRESSURE REPEAT
      GO TO 70) GO TO 70
C      PRESSURE RISE
C 70)  PRESSURE RISE / (144.0 * 0.033609)
C      OR FOR EXIT INFORMATION WITH WCM
C      OR FOR EXIT INFORMATION WITH WCM
C      MATCHES MATCHED WHEN YOU ARE FINISHED
      GO TO 1) GO TO 45
C      INFORMATION TO AN OUTPUT FILE
      C, HP, DP, FITPV, INLOSS, EXLOSS, WC,
      C, NG, SERIAL, MOD)
      196996
      196996
      196996
      196996
C      SUMMARY OF BRANCH LOSSES
      1213, DP36, DP24, DP45
500  CONTINUE
600  FOR EACH POINT IS NOT ON THE PERFORMANCE MAP.
001  / 5X, LOSS BRANCH 1-3: ', F12.3, / 5X, LOSS BRANCH 3-6: ', F12.2,
      / 5X, LOSS BRANCH 2-4: ', F12.3, / 5X, LOSS BRANCH 4-5: ', F12.2,
      END

```



```

      DP(I)=DELP
      DP35=DP35+DELP
      FITPV(I)=PV
      PTIN=PTOUT
      TIN=TOUT
C      LOSS IN THE MAIN ENGINE EXHAUST FLOW FOR NODE 5, ENERGY IS
C      TRANSFERRED TO THE MODULE COOLING FLOW AS A PRESSURE GAIN.
      IF (TYPE.EQ.16) LOSS=DELP
35  CONTINUE
C      COMPARE EXIT PRESSURE TO PT5, REVISE INLET CONDITION TO BRANCH
C      J-5 AND REPEAT IF NECESSARY
      TEST=ABS(PTOUT-PT5)
      IF (TEST.LT.1.0) GO TO 40
      PTIN=PQ*144.0+DP35+DP56
      GO TO 30
40  CONTINUE
C      COMPARE ASSUMED LOSSES TO COMPUTED LOSSES REPEAT ITERATION
C      IF REQUIRED
      INLOSS=INLOSS*5.19696
      EXLOSS=EXLOSS*5.19696
      TEST1=ABS(DP12+DP23-INLOSS)
      TEST2=ABS(DP35+DP56-EXLOSS)
      INLOSS=(DP12+DP23)/5.19696
      EXLOSS=(DP35+DP56)/5.19696
C      IF ((TEST1.GT.1.0).OR.(TEST2.GT.1.0)) GO TO 5
45  INITIALIZE LOSSES FOR BRANCH 2-4
      PTIN=PT2
      DP24=0.0
      TIN=TJ+459.7
C      COMPUTE FITTING LOSSES
      DO 50 I=BR
      TYPE=WORKI(I,2)
      DATA1=WORKR(I,1)
      DATA2=WORKR(I,2)
      DATA3=WORKR(I,3)
      DATA4=WORKR(I,4)
      CALL FITDP(WC,HP,PTIN,TIN,PTOUT,TOUT,PV,DATA1,DATA2,DATA3,
      DATA4,TYPE,DELP,ALFAC,0.,0.,0.,0.,ADWB,ADWM,ADWC,0.,0.,0.,
      VDNB,VLMH,VDMC,0.,T0)
      DP(I)=DELP
      DP24=DP24+DELP
      FITPV(I)=PV
      PTIN=PTOUT
      TIN=TOUT
      IF (TYPE.EQ.26) THOD=TOUT
50  CONTINUE
C      INITIALIZE INLET CONDITIONS FOR BRANCH 4-5
      T4=TOUT
      PTIN=PTOUT+(K*((WC/RHOSTD*60.0)-CFMMAX)**2+DPMAX)*5.19696
      TIN=T4
      DP45=0.0
C      COMPUTE FITTING LOSSES
      DO 60 I=BR
      TYPE=WORKI(I,2)
      DATA1=WORKR(I,1)
      DATA2=WORKR(I,2)
      DATA3=WORKR(I,3)
      DATA4=WORKR(I,4)
      CALL FITDP(WC,HP,PTIN,TIN,PTOUT,TOUT,PV,DATA1,DATA2,DATA3,
      DATA4,TYPE,DELP,ALFAC,0.,ACWB,ACWM,ACWC,0.,0.,0.,VCWB,VCWM,
      VCWC,0.,0.,0.,RHOCC,T0)
      DP(I)=DELP
      DP45=DP45+DELP
      FITPV(I)=PV
      PTIN=PTOUT
      TIN=TOUT
C      THE MODULE FLOW GET A BOOST IN PRESSURE BY A TRANSFER
C      OF MOMENTUM FROM THE HIGHER VELOCITY MAIN EXHAUST FLOW
      IF (TYPE.EQ.15) GAIN=DELP

```

```

60 IF (TYPE.E2.26) TMOD=TOUT
C CCMM=NO
C TEST=ABS(P5OUT-PT5)
C IF (TEST.LT.1.0) GO TO 70
C COOLING FLOW IN INCREASED BY SMALL STEPS UNTIL SYSTEM IS
C MATCHED
C WC=WC+0.1
C GO TO 5
C SYSTEM IS MATCHED PRINT RESULTS
70 CALL OUTPUT(TO,PO,HUMID,HP,NPT,N,WORKI,DP,FITPV,INLOSS,EXLOSS,WC,
+ DP12,DP23,DP35,DP56,DP24,DP45,SFC,I54,NG,SERIAL,TMOD)
C DP12=DP12/5.19696
C DP23=DP23/5.19696
C DP35=DP35/5.19696
C DP56=DP56/5.19696
C DP24=DP24/5.19696
C DP45=DP45/5.19696
C PRINT BRANCH LOSS SUMMARY
C WRITE (4,601) DP12,DP23,DP35,DP56,DP24,DP45
500 CONTINUE
800 FORMAT(' POWER POINT IS NOT ON THE PERFORMANCE MAP.')
601 FORMAT(/5X,'LOSS BRANCH 1-2:',F12.2,/5X,'LOSS BRANCH 2-3:',F12.2,
+ /5X,'LOSS BRANCH 3-5:',F12.2,/5X,'LOSS BRANCH 5-6:',F12.2,
+ /5X,'LOSS BRANCH 2-4:',F12.2,/5X,'LOSS BRANCH 4-5:',F12.2)
C RETURN
C END

```



```

C*****
C SYSTEM FOUR MATCHING SUBROUTINE
C*****
C THIS SYSTEM HAS SEPARATE INLETS FOR THE ENGINE AIR FLOW AND
C MODULE COOLING. NODE 5 IS THE JUNCTION OF MODULE AIR AND
C ENGINE EXHAUST. FOR THE ASSUMED FLOW THE PRESSURE AT NODE
C 5 IS COMPUTED FROM THE COMBINED EXHAUST. THEN THE EXIT
C PRESSURE FROM BRANCHES 3-5 AND 4-5 SHOULD MATCH P75. IF NOT
C THE ITERATION PROCESS CONTINUES.
C*****
C SUBROUTINE SYS4(SERIAL,N,WORK1,WORKR,HP,NPT,FIT1ST,TO,PO,HUMID,
C   ALFAC,ACHB,ACWC,ACWM,
C   RHOSTD,CENO,CFMAY,DPMAX,K)
C   REAL WORKR,HP,NPT,TO,PO,HUMID,CFMAY,CFMNO,DPMAX,K,T2,W8,P8,T9,
C   INLOSS,EXLOSS,PANDP,DP13,DP24,DP35,DP45,PV,PITV,PTOUT,DATA1,
C   DATA2,DATA3,TEST,DP,FITPV,WC,RHOSTD,TEST1,TEST2,WCN,TMOD,
C   TS4,SFC,NG,DP56
C   DATA4,R,ALFAC,ACHB,ACWC,ACWM,VCWB,VCWC,VCWM,TMAIN,TMOD,PSEC,
C   HMAIN,HSTACK,T4,TS4,GAIN,LOSS,PSEC,PMAIN,PIT1,PT5,TEST3,
C   PV8,PVC,PVM,SSB,PSC,PSM,RHOCBT,RHOCCT,RHOCMT,TEST1,TEST2
C   INTEGER WORK1,FIT1ST,CFM,N,PP,QQ,RR,SS,TT,A,B,C,D,E,SERIAL,TYPE,
C   IND
C   DIMENSION WORK1(200,2),WORKR(200,4),FIT1ST(6),DP(200),FITPV(200)
C   GAS CONSTANT
C   DATA R/53.3424/
C   THE STARTING AND STOPPING INDEX FOR A BRANCH IS COMPUTED
C   PP=FIT1ST(2)-1
C   QQ=FIT1ST(3)-1
C   RR=FIT1ST(4)-1
C   SS=FIT1ST(5)-1
C   A=FIT1ST(1)
C   B=FIT1ST(3)
C   C=FIT1ST(4)
C   D=FIT1ST(5)
C   INITIALIZE THE INLET AND EXHAUST LOSSES
C   INLOSS=.0
C   EXLOSS=.0
C   INITIALIZE THE GAIN AND LOSS AT NODE 5
C   GAIN=-30.0
C   LOSS=30.0
C   INITIALIZE THE COOLING FLOW
C   WC=CFMAY*RHOSTD/60.0
C   INITIALIZE THE BRANCH LOSSES
C   DP45=100.0
C   DP56=100.0
C   INITIALIZE THE MODULE TEMPERATURE
C   TMD=710.0
C   INITIALIZE THE PRESSURES AT NODE 5
C   P75=PO*144.0+DP56
C   PMAIN=P75+LOSS
C   PSEC=P75+GAIN
C   SEARCH FOR A WASTE HEAT BOILER IN BRANCH 3-5
C   IND=0
C   DO 4 I=SS
C     TYPE=WORK1(I,2)
C     IF(TYPE.EQ.24) IND=1
C   CONTINUE
C   GEN INITIAL PERFORMANCE OF ENGINE WITH ASSUMED CONDITIONS
C   CALL ENGINE(INLOSS,EXLOSS,TO,PO,HUMID,HP,NPT,W2,W8,P8,T8,SFC,
C   TS4,NG,OFF)
C   IF(OFF.EQ.0) GC TO 6
C   WRITE(6,600)
C   GC TO 500
C   CONTINUE
C   INITIALIZE INLET CONDITIONS FOR BRANCH 1-3
C   DP13=0.0
C   P71N=PO*144.0
C   T71N=100+59.7

```



```

      FITPV(I)=PV
      PTIN=PTOUT
      TIN=TOUT
      IF (TYPE.EQ.26) TMOD=TOUT
20  CONTINUE
      FI4=PTOUT
      T4=TOUT
C   INITIALIZE INLET CONDITIONS FOR BRANCH 4-5
      PTIN=PT4+(K*((WC/RHOSTD*60.0)-CFMMAX)**2*DPMAX)*5.19696
      TIN=T4
      DP45=0.0
C   COMPUTE FITTING LOSSES
      DO 60 I=B,RR
        TYPE=WORKR(I,2)
        DATA1=WORKR(I,1)
        DATA2=WORKR(I,2)
        DATA3=WORKR(I,3)
        DATA4=WORKR(I,4)
        CALL FITDP(WC,HP,PTIN,TIN,PTOUT,TOUT,PV,DATA1,DATA2,DATA3,
          * DATA4,TYPE,DELP,ALFAC,0.,ACFB,ACWM,ACWC,0.,0.,0.,VCFB,VCFM,
          * VCFWC,0.,0.,RHOCC,I))
        DP(I)=DELP
        DP45=DP45+DELP
        FITPV(I)=PV
        PTIN=PTOUT
        TIN=TOUT
C   GAIN IS RESULT OF MOMENTUM TRANSFER FROM EXHAUST FLOW
        IF (TYPE.EQ.15) GAIN=DELP
        IF (TYPE.EQ.26) TMOD=TOUT
60  CONTINUE
C   EXIT PRESSURE SHOULD BE PT5 OR REPEAT ITERATION
      TEST=ABS(PTOUT-PT5)
      IF (TEST.LT.1.0) GO TO 70
C   ADD A SMALL INCREMENT TO THE COOLING FLOW AND REPEAT ITERATION
      WC=WC+0.1
      GO TO 5
C   SYSTEM IS MATCHED, OUTPUT RESULTS
70  CALL OUTPUT(20,PO,HUMID,HE,NPT,N,WORKR,DP,FITPV,INLOSS,EXLOSS,WC,
    * H2,H8,S2,S8,SFC,154,NG,SERIAL,TMOD)
      DP13=DP13/5.19696
      DP24=DP24/5.19696
      DP35=DP35/5.19696
      DP45=DP45/5.19696
      DP56=DP56/5.19696
C   OUTPUT BRANCH LOSS SUMMARY
      WRITE (4,601) DP13,DP24,DP35,DP45,DP56
600  CONTINUE
500  FORMAT(' POWER POINT IS NOT ON THE PERFORMANCE MAP.')
601  FORMAT('/5X, LOSS BRANCH 1-3:',F12.2,/5X, LOSS BRANCH 2-4:',F12.2,
    * /5X, LOSS BRANCH 3-5:',F12.2,/5X, LOSS BRANCH 4-5:',F12.2,
    * /5X, LOSS BRANCH 5-6:',F12.2)
      RETURN
      END

```

```

*****
SYSTEM FIVE MATCHING SUBROUTINE
*****
THIS SYSTEM HAS COMBINED INLETS AND EXHAUST FLOWS FOR THE ENGINE
AND THE MODULE COOLING. THERE IS NO COOLING FAN. THE MOVEMENT
OF COOLING AIR IS ACCOMPLISHED BY AN EDUCTOR ARRANGEMENT AT THE
ENGINE EXHAUST PLANE. THERE IS A TRANSFER OF MOMENTUM FROM A
HIGH SPEED JET (ENGINE EXHAUST) THROUGH A NOZZLE TO A LOW
JET (MODULE COOLING FLOW). THE SCHEME IS TO START WITH A SMALL
COOLING FLOW AND SEE IF THERE IS ENOUGH GAIN AVAILABLE FROM THE
EDUCTOR ARRANGEMENT TO MOVE THE AIR. A PROPERLY DESIGNED SYSTEM
WILL HAVE EXCESS GAIN AT THIS LOW FLOW AND THE ITERATION PROCESS
CAN CONTINUE, INCREASING THE COOLING FLOW UNTIL THE SYSTEM IS
MATCHED.
*****
SUBROUTINE SYS5(SERIAL,N,WORKI,WORKR,HP,NPT,FIT1ST,TO,PO,HUMID,
+ALFAD,ADWB,ADWC,ADWM,ALFAC,ACWB,ACWC,ACWM)
REAL WORKR,HP,NPT,TO,PO,HUMID,CENMAX,CENMO,DPMAX,K22,W8,P8,T8
+INLOSS,EXLOSS,T1NDP,DP13,DP24,DP35,DP45,PV,PATN,PTOUT,DATA1,
+DATA2,DATA3,TEST,DP,FITPV,WC,HOSTO,TEST1,TEST2,ACN,INOD,
+T54,SFC,NG,DP56
+DATA4,ALFAC,ACWB,ACWC,ACW4,VCWB,VCWC,VCWM,IMAIN,IMOD,HSEC,
+HMAIN,HSTACK,T4,T5,GAIN,LOSS,PSEC,PMAIN,P14,PT5,TEST3,
+PVB,PVC,PVM,PSB,PSC,PSM,RHOCBT,RHOCCT,RHOCNT,TEST1,TEST2
+INTEGER WORKI,FIT1ST,OFF,N,PP,QQ,RR,SS,A,B,C,D,SERIAL,TYPE,
+IND
DIMENSION WORKI(200,2),WORKR(200,4),FIT1ST(6),DP(200),FITPV(200)
GAS CCNSTANT
DATA R/53.3424/
C COMPUTE THE STARTING AND STOPPING POINTS FOR THE BRANCH FITTINGS.
C FP=FIT1ST(2)-1
C CP=FIT1ST(3)-1
C HP=FIT1ST(4)-1
C SP=FIT1ST(5)-1
C A=FIT1ST(6)
C B=FIT1ST(3)
C C=FIT1ST(4)
C D=FIT1ST(5)
C INITIALIZE THE INLET AND EXHAUST LOSSES (INCH WG)
C INLOSS=4.0
C EXLOSS=8.0
C INITIALIZE THE GAIN OR PRESSURE RISE TO MODULE AIR FLOW IN THE
C EDUCTOR
C GAIN=-30.0
C LOSS=30.0
C INITIALIZE THE COOLING FLOW TO THE MINIMUM REQUIRED FOR THE ENGINE
C WC=7.5
C INITIALIZE PARAMETERS TO START ITERATION
C DP56=100.0
C IMOD=710.0
C FT5=PO*144.0+DP56
C PMAIN=PT5+LOSS
C PSEC=PT5+GAIN
C SEARCH FOR A WASTE HEAT BOILER, THERE PROBABLY IS NOT A BOILER
C INSTALLED IN THIS SYSTEM.
C IND=0
C DO 4 I=1,SS
C TYPE=WORKI(I,2)
C IF (TYPE.EQ.27) IND=1
C CONTINUE
C GET ENGINE PERFORMANCE BASED ON ASSUMED CONDITIONS
C CALL ENGINE(INLOSS,EXLOSS,TO,PO,HUMID,HP,NPT,22,W8,P8,T8,SFC,
+T54,NG,OFF)
C IF (OFF.EQ.0) GC TO 6
C WRITE(6,600)
C GO TO 500
C CONTINUE
C INITIALIZE INLET CONDITIONS FOR BRANCH 1-2

```



```

TEST=ABS(PTOUT-PT5)
IF (TEST.LT.1.0) GO TO 40
PTIN=20*144.0+DP35+DP56
GO TO 30
40 CONTINUE
C COMPARE ASSUMED LOSSES WITH COMPUTED LOSSES
INLOSS=INLOSS*5.19696
EXLOSS=EXLOSS*5.19696
TEST1=ABS(DP12+DP23-INLOSS)
TEST2=ABS(DP35+DP56-EXLOSS)
INLOSS=(DP12+DP23)/5.19696
EXLOSS=(DP35+DP56)/5.19696
C IF (TEST1.GT.1.0).OR.(TEST2.GT.1.0)) GO TO 5
INITIALIZE INLET CONDITIONS FOR BRANCH 2-5, NO FAN (MODE 4)
PTIN=PT2
TIN=T2+459.7
DP25=0.0
DO 60 I=3,RR
TYPE=WORKI(I,2)
DATA1=WORKR(I,1)
DATA2=WORKR(I,2)
DATA3=WORKR(I,3)
DATA4=WORKR(I,4)
CALL FITDP(WC,HP,PTIN,TIN,PTOUT,TCUT,PV,DATA1,DATA2,DATA3,
+ DATA4,TYPE,DELP,ALFAC,0.,ACWB,ACWM,ACWC,0.,0.,0.,VCFB,VCFM,
+ VCHC,0.,0.,0.,RHOCC,10)
DP(I)=DELP
DP25=DP25+DELP
FITPV(I)=PV
2.TIN=PTCUT
TIN=TCUT
C GAIN IS FROM MOMENTUM TRANSFER FROM ENGINE EXHAUST
IF (TYPE.EQ.15) GAIN=DELP
IF (TYPE.EQ.26) TMOD=TCUT
60 CONTINUE
C EXHAUST PRESSURE SHOULD BE PT5, IF NOT REPEAT ITERATION
TEST=ABS(PTOUT-PT5)
IF (TEST.LT.1.0) GO TO 70
C INCREASE COOLING FLOW UNTIL SYSTEM IS MATCHED
WC=WC+0.1
GO TO 5
C SYSTEM IS MATCHED, OUTPUT RESULTS
70 CALL OUTPUT(TO,20,HUMID,HP,NPT,N,WORKI,IP,FITPV,INLOSS,EXLOSS,WC,
+ DP12,DP23,DP35,DP56,SFC,154,NG,SERIAL,TMOD)
DP12=DP12/5.19696
DP23=DP23/5.19696
DP35=DP35/5.19696
DP56=DP56/5.19696
DP25=DP25/5.19696
C OUTPUT BRANCH LOSS SUMMARY
WRITE(4,601) DP12,DP23,DP35,DP56,DP25
500 CCN=CCN+1
600 FORMAT(' POWER POINT IS NOT ON THE PERFORMANCE MAP.')
601 FORMAT('/5X, LOSS BRANCH 1-2:',F12.2,'/5X, LOSS BRANCH 2-3:',F12.2,
+ '/5X, LOSS BRANCH 3-4:',F12.2,'/5X, LOSS BRANCH 5-6:',F12.2,
+ '/5X, LOSS BRANCH 2-5:',F12.2)
RETURN
END

```



```

*****
SYSTEM SIX MATCHING SUBROUTINE
*****
THIS SYSTEM HAS SEPARATE INLETS FOR COOLING FLOW AND ENGINE AIR.
THE TWO FLOW JUNCTIONS CAN BE CONNECTED BY AN EDUCTOR OR EXHAUST
PIPER. THERE IS NO COOLING FLOW INSTALLED. THE EDUCTOR PROVIDES
ALL THE PUMPING ACTION BY MOMENTUM TRANSFER FROM A HIGH VELOCITY
JET (ENGINE EXHAUST THROUGH A NOZZLE) TO A LOW VELOCITY JET
(MODULE COOLING FLOW).
*****
SUBROUTINE SYS6(SERIAL, J, WORKI, WORKR, H2, NPT, FIT1ST, T0, P0, HUMID,
+ ALFAC, ACMB, ACWC, ACWM)
+ REAL WORKR, HP, NPT, P0, HUMID, ACMB, ACWC, ACWM, XFM0, DPMAK, K, W2, W8, P8, T8,
+ INLOSS, EXLOSS, DP13, DP25, DP35, DP55, DP65, PTCUT, DATA1,
+ DATA2, DATA3, TEST1, DP, FITV, AC, RHOSID, TEST2, NCN, TMOD,
+ SFC, NG,
+ DATA1, B, ALFAC, ACMB, ACWC, ACWM, VCM0, VCM1, VCM2, VCM3, TMAIN, TMOD, HSEC,
+ HMA1, HSTACK, S, W, GAIN, LOSS, PSEC, PMAIN, P15, TEST3,
+ PVB, PVB, PVB, PVB, PVB, PVB, PVB, PVB, PVB, PVB, PVB, PVB, PVB, PVB, PVB, PVB,
+ INTEGER WORKI, FIT1ST, CFF, W, P2, C, RR, A, B, C, SERIAL, TYPE,
+ IND
+ DIMENSION WORKI(200,2), WORKR(200,4), FIT1ST(6), DP(200), FITPV(200)
+ GAS CONSTANT
+ DATA P/53.3424/
+ STARTING AND STOPPING POINTS FOR THE BRANCH FITTING INDEX
+ PP=FIT1ST(1)-1
+ QC=FIT1ST(2)-1
+ MC=FIT1ST(3)-1
+ ME=FIT1ST(4)-1
+ A=FIT1ST(5)-1
+ B=FIT1ST(6)-1
+ C=FIT1ST(7)-1
+ INITIALIZE THE INLET AND EXHAUST DUCT LOSSES (INCH WG)
+ INLOSS=4.0
+ EXLOSS=3.0
+ INITIALIZE THE GAIN AND LOSS IN THE EDUCTOR (PSF)
+ GAIN=-30.0
+ LOSS=30.0
+ INITIALIZE THE COOLING FLOW TO THE MINIMUM REQUIRED
+ NC=7
+ INITIALIZE OTHER VALUES
+ DP0=100.0
+ TMD=710.0
+ P15=P0*144.0+DP56
+ PMAIN=P15+LOSS
+ PSEC=P15+GAIN
+ SEARCH FOR A BOILER, THERE PROBABLY IS NOT ONE INSTALLED FOR THIS
+ SYSTEM
+ IND=0
+ DO 4 I=C, RR
+ 4 P2=WORKI(I,2)
+ 4 CFF=(TYPE-20.27)/IND=1
+ 4 CONTINUE
+ 4 GET ENGINE PERFORMANCE FOR THE ASSUMED CONDITIONS
+ 5 CALL ENGINE(INLOSS, EXLOSS, T0, P0, HUMID, HP, NPT, W2, W8, P8, T8, SFC,
+ 5 T84, NG, CFF)
+ IF (OFF.30.0) GO TO 6
+ 6 H2=2.0(0.600)
+ GO TO 500
+ 6 CONTINUE
+ INITIALIZE THE INLET CONDITIONS FOR BRANCH 1-3
+ DP13=0.0
+ P1IN=20*144.0
+ T1N=T0+459.7
+ COMPUTE FITTING LOSSES
+ DO 8 I=1, PP
+ 8 TYPE=WORKI(I,2)
+ DATA1=WORKR(I,1)
+ DATA2=WORKR(I,2)

```



```

12      TOUT=TOUT
C      CONTINUE
      TEMPERATURE SHOULD BE ATMOSPHERIC IF NOT REPEAT ITERATION
      TEST=ABS(P5OUT-P0*144.0)
      IF (TEST.LT.1.0) GO TO 14
      GC TO 11
14      CONTINUE
      P1=P0*144.0+DP56
C      INITIALIZE INLET CONDITIONS FOR BRANCH 3-5
      PTIN=P8*144.0
      TIN=T8
      DP35=0.0
C      COMPUTE FITTING LOSSES
      DO 20 I=B,R
      TYPE=WORKR(I,2)
      DATA1=WORKR(I,1)
      DATA2=WORKR(I,2)
      DATA3=WORKR(I,3)
      DATA4=WORKR(I,4)
      CALL FITDP(WC,HP,PTIN,TIN,PTOUT,TOUT,PV,DATA1,DATA2,DATA3,
      * DATA4,TYPE,DELP,ALFAC,0.,ACWB,ACWM,ACWC,0.,0.,0.,VCWB,VCWM,
      * VCWC,0.,0.,RHOCC,T0)
      DP(I)=DELP
      DP35=DP35+DELP
      P1PV(I)=PV
      PTIN=PTOUT
      TIN=TOUT
      IF (TYPE.EQ.16) LOSS=DELP
20      CONTINUE
C      COMPARE ASSUMED LOSSES AND COMPUTED LOSSES, IF NOT THE SAME REPEAT
      INMLOSS=INLOSS*5.19636
      EXMLOSS=EXLOSS*5.19636
      TEST1=ABS(DP12+DP23-INLOSS)
      TEST2=ABS(DP35+DP56-EXLOSS)
      INMLOSS=(DP12+DP23)/5.19636
      EXMLOSS=(DP35+DP56)/5.19636
      IF (TEST1.LT.1.0) .OR. (TEST2.LT.1.0) GO TO 5
C      INITIALIZE INLET CONDITIONS FOR BRANCH 2-5
      PTIN=P0*144.0
30      TIN=T0+459.7
      DP25=0.0
C      COMPUTE FITTING LOSSES
      DO 35 I=A,C
      TYPE=WORKR(I,2)
      DATA1=WORKR(I,1)
      DATA2=WORKR(I,2)
      DATA3=WORKR(I,3)
      DATA4=WORKR(I,4)
      CALL FITDP(WC,HP,PTIN,TIN,PTOUT,TOUT,PV,DATA1,DATA2,DATA3,
      * DATA4,TYPE,DELP,ALFAC,0.,ACWB,ACWM,ACWC,0.,0.,0.,VCWB,VCWM,
      * VCWC,0.,0.,RHOCC,T0)
      DP(I)=DELP
      DP25=DP25+DELP
      P1PV(I)=PV
      PTIN=PTOUT
      TIN=TOUT
C      GAIN IS RESULT OF MOMENTUM TRANSFER FROM EXHAUST TO COOLING
      FLCH
      IF (TYPE.EQ.15) GAIN=DELP
      IF (TYPE.EQ.26) IMOD=TOUT
35      CONTINUE
C      EXIT PRESSURE SHOULD BE PT5, IF NOT REPEAT ITERATION
      TEST=ABS(PTOUT-PT5)
      IF (TEST.LT.1.0) GO TO 40
C      NEXT ITERATION IS DONE WITH INCREASED COOLING FLOW, INCREASE
C      UNTIL SYSTEM IS MATCHED
      WC=WC+0.1
      GC TO 5
40      CONTINUE

```



```

C*****FAN MATCHING SUBROUTINE*****C
C      FAN MATCHING SUBROUTINE
C      THIS SUBROUTINE PRODUCES THE NEXT GUESS AT COOLING FLOW BY
C      LOCATING THE INTERSECTION OF THE SYSTEM MODEL CURVE AND THE
C      FAN CHARACTERISTIC CURVE.
C*****C
C      SUBROUTINE PANMAT(WC, TO, PO, FANDP, RHOSTD, CFM0, CFMMAX, DPMAX, K, WCN)
C      REAL CFMSTD, DPSTD, WC, RHOSTD, FANDP, PO, TO, C, CFM, WCN, CFM0, CFMMAX,
C      *      DPMAX, K, R
C      GAS CONSTANT
C      DATA Z/53.3424/
C      CONVERT MASS FLOW TO STANDARD VOLUME FLOW (CFM)
C      CFMSTD=WC/RHOSTD*60.0
C      CONVERT FAN DELTA PRESSURE TO STANDARD CONDITIONS FOR THE FAN
C      DPSTD=FANDP*(RHCSTD*R*(TO+459.7)/(PO*144.0))**2
C      C IS THE PROPORTIONALITY CONSTANT FOR THE QUADRATIC MODEL ASSUMED
C      TO REPRESENT THE SYSTEM
C      C=DPSTD/CFMSTD**2
C      CFM IS THE INTERSECTION OF THE FAN CHARACTERISTIC AND SYSTEM
C      MODEL
C      CFM=(2.0*K*CFMMAX-SQRT((2.0*K*CFMMAX)**2-4.0*(K-C)*(K*CFMMAX**2
C      *      +DPMAX)))/(2.0*(K-C))
C      CONVERT CFM TO MASS FLOW
C      WCN=RHOSTD*CFM/60.0
C      RETURN
C      END

```

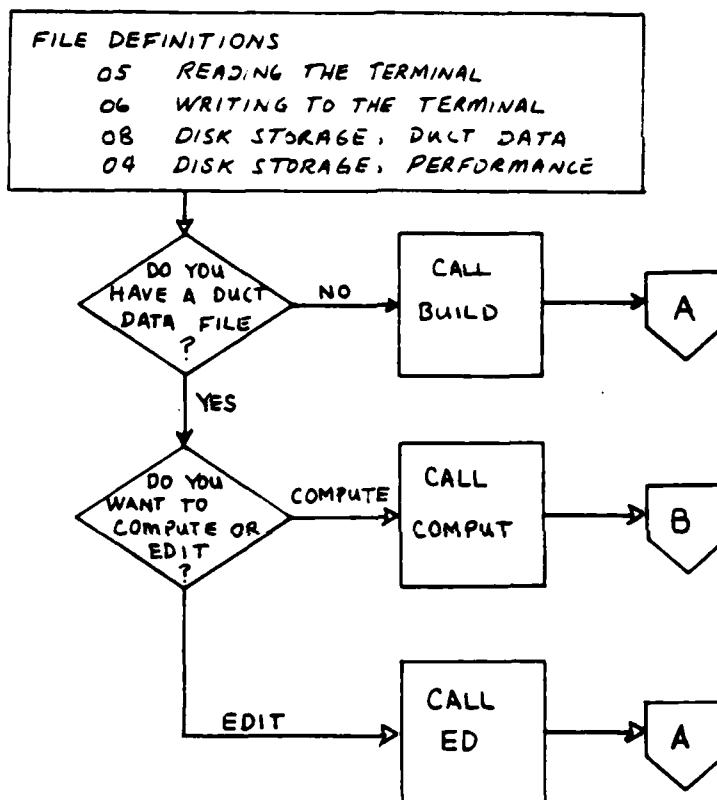
```

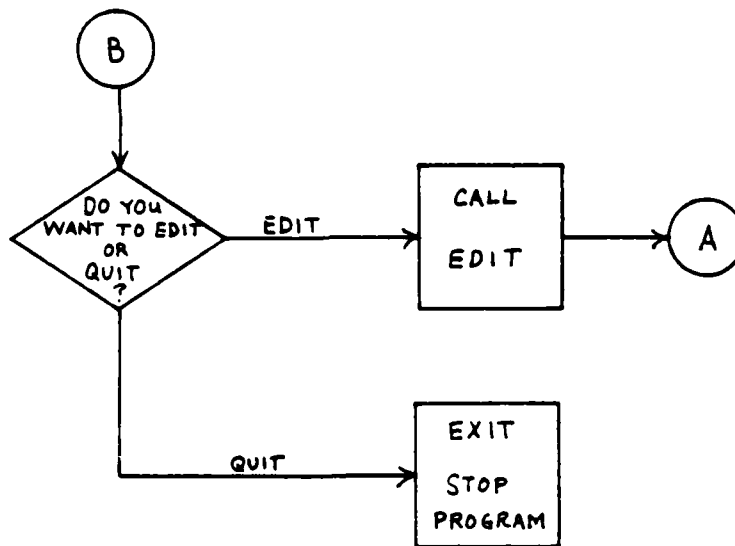
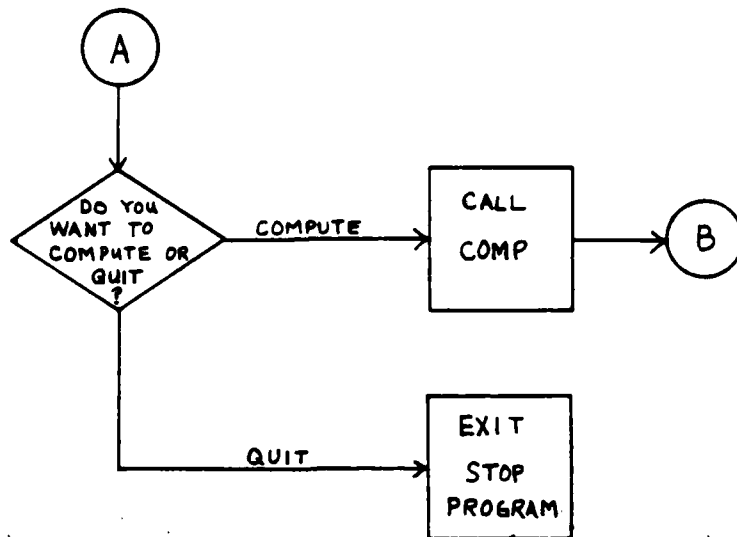
C*****
C      COMPUTE OUTPUT SUBROUTINE: PRINTS SYSTEM DATA
C*****
C      THIS SUBROUTINE WRITES TO THE OUTPUT FILE.  IF YOU HAVE AN OUTPUT
C      FILE ALREADY IT WILL BE WRITTEN OVER BY THIS PROGRAM.  IF YOU
C      WANT TO SAVE THE PREVIOUS RESULTS, RENAME THE FILE.  IF YOU ADD
C      OR CHANGE FITTINGS YOU MUST MAKE SOME CHANGES HERE.
C*****
C      SUBROUTINE OUTPUT (20, P0, HUMID, HP, NPT, N, WORKI, DP, FITPV, INLOSS,
C      * EXLOSS, WC, W2, W8, P8, T8, SFC, T54, NG, SERIAL, TMOD)
C      * REAL TO, P0, HUMID, HP, NPT, DP, FITPV, INLOSS, EXLOSS, WC, W2, TMOD,
C      * P8, T8, SFC, T54, NG
C      * INTEGER N, WORKI, SERIAL, TYPE
C      * DIMENSION DP (200), FITPV (200), WORKI (200, 2)
C      * WRITE (4, 600) SERIAL, P0, HUMID, HP, NPT
C      * WRITE (4, 601) INLOSS, EXLOSS
C      * TMOD=INMOD-459.7
C      * WRITE (4, 602) WC, W2, W8, P8, T8, SFC, T54, NG, TMOD
C      * DC 100 I=1, N
C      *   DP(I)=DP(I)/5.19696
C      *   TYPE=WORKI(I, 2)
C      *   FITPV(I)=FITPV(I)/5.19696
C      *   GO TO (1, 2, 3, 4, 5, 6, 7, 8, 9, 10, 11, 12, 13, 14, 15, 16, 17, 18, 19, 20,
C      *   * 21, 22, 23, 24, 25, 26, 27, 28, 29, 30), TYPE
C      *   *
1  WRITE (4, 603) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
2  GO TO 100
3  WRITE (4, 604) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
4  GO TO 100
5  WRITE (4, 605) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
6  GO TO 100
7  WRITE (4, 606) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
8  GO TO 100
9  WRITE (4, 607) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
10 GO TO 100
11 WRITE (4, 608) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
12 GO TO 100
13 WRITE (4, 609) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
14 GO TO 100
15 WRITE (4, 610) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
16 GO TO 100
17 WRITE (4, 611) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
18 GO TO 100
19 WRITE (4, 612) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
20 GO TO 100
21 WRITE (4, 613) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
22 GO TO 100
23 WRITE (4, 614) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
24 GO TO 100
25 WRITE (4, 615) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
26 GO TO 100
27 WRITE (4, 616) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
28 GO TO 100
29 WRITE (4, 617) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
30 GO TO 100
31 WRITE (4, 618) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
32 GO TO 100
33 WRITE (4, 619) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
34 GO TO 100
35 WRITE (4, 620) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
36 GO TO 100
37 WRITE (4, 621) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
38 GO TO 100
39 WRITE (4, 622) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
40 GO TO 100
41 WRITE (4, 623) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
42 GO TO 100
43 WRITE (4, 624) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
44 GO TO 100
45 WRITE (4, 625) WORKI(I, 1), WORKI(I, 2), DP(I), FITPV(I)
46 GO TO 100

```


APPENDIX B
FLOW CHARTS

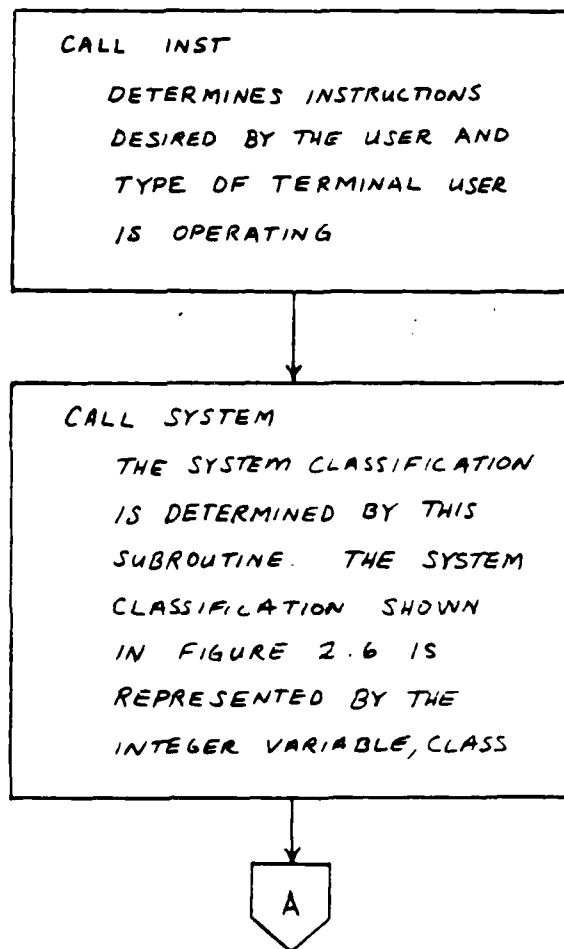
I. MAIN PROGRAM NO INPUT OR OUTPUT VARIABLES

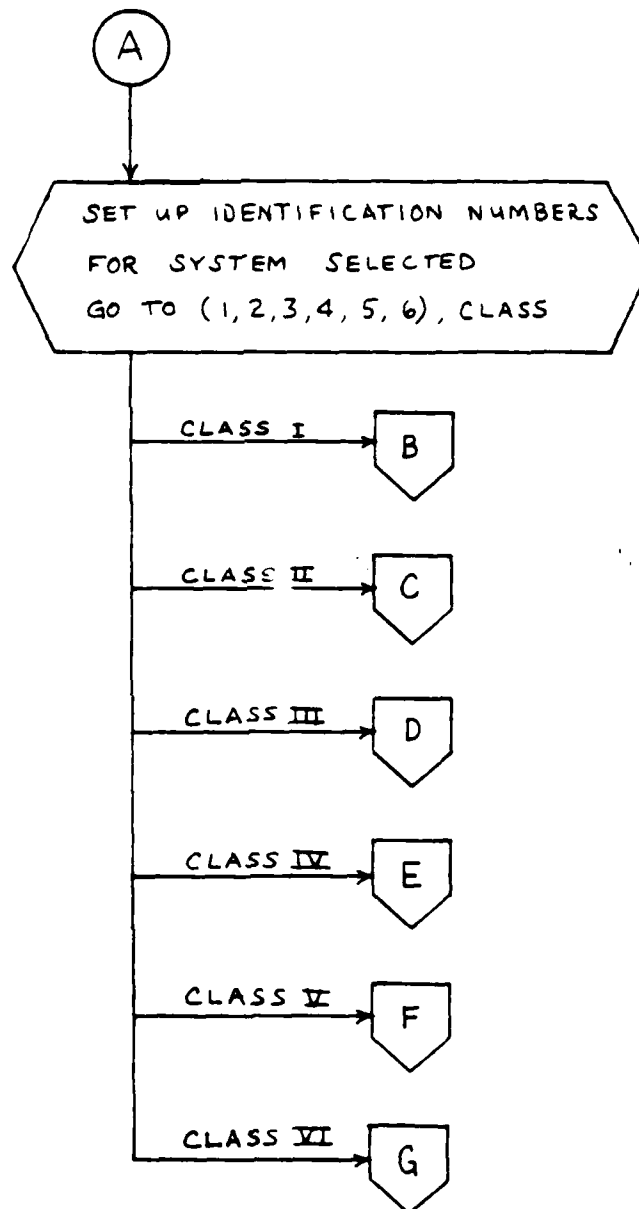




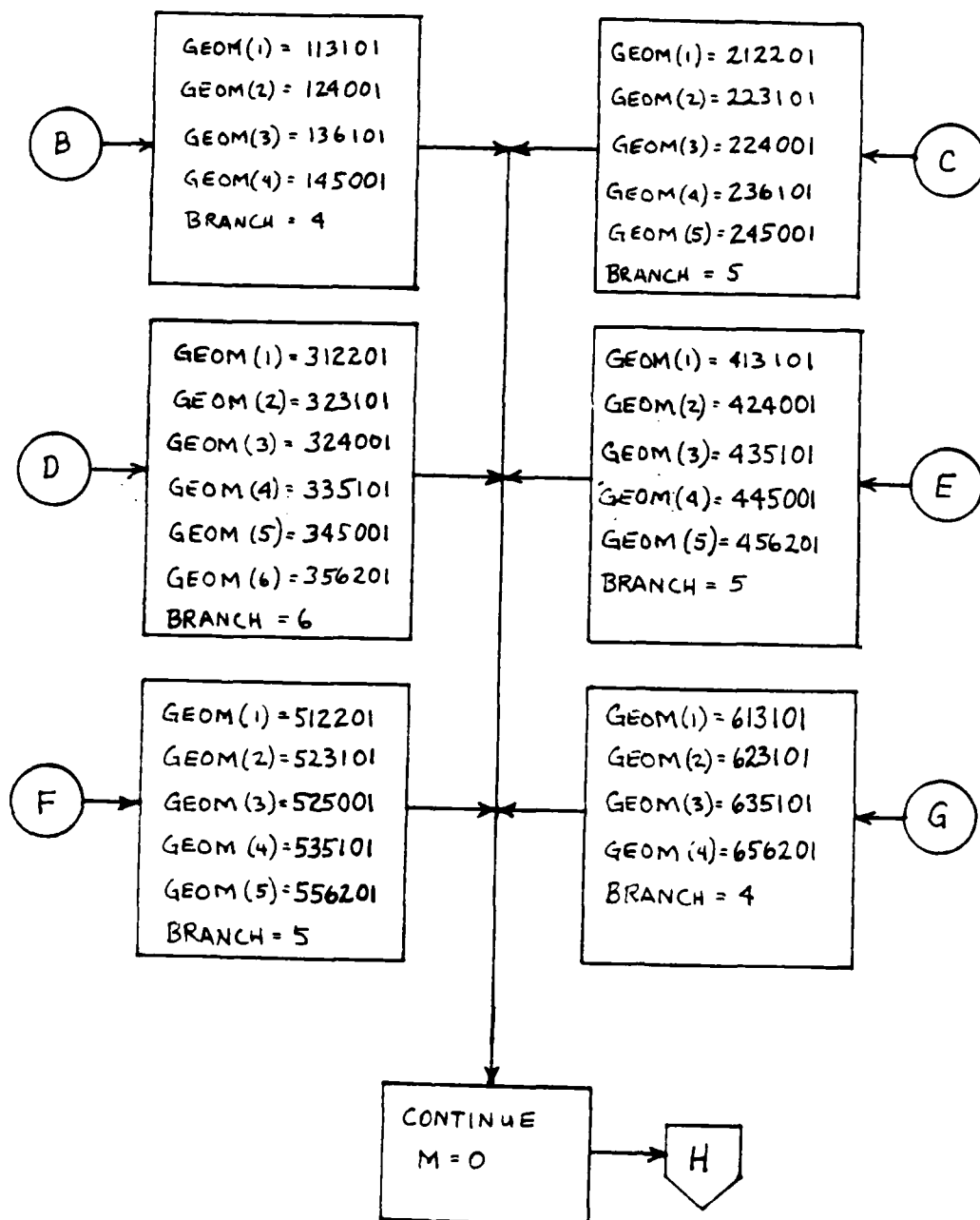
II. BUILD SUBROUTINE

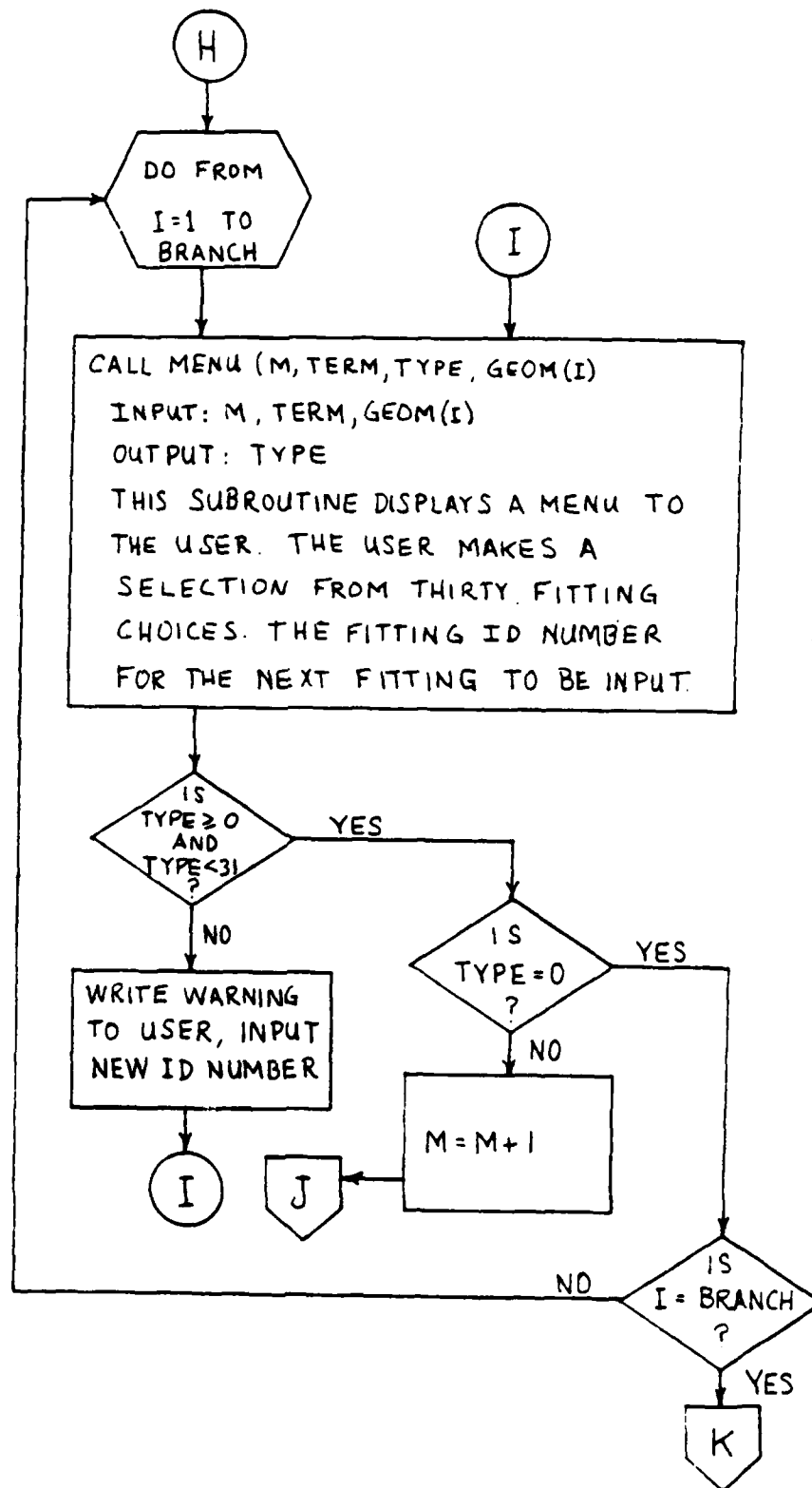
THERE ARE NO INPUT OR OUTPUT VARIABLES FOR THIS SUBROUTINE, HOWEVER SUBROUTINES CALLED BY THE BUILD SUBROUTINE DO HANDLE INPUT AND OUTPUT DATA.

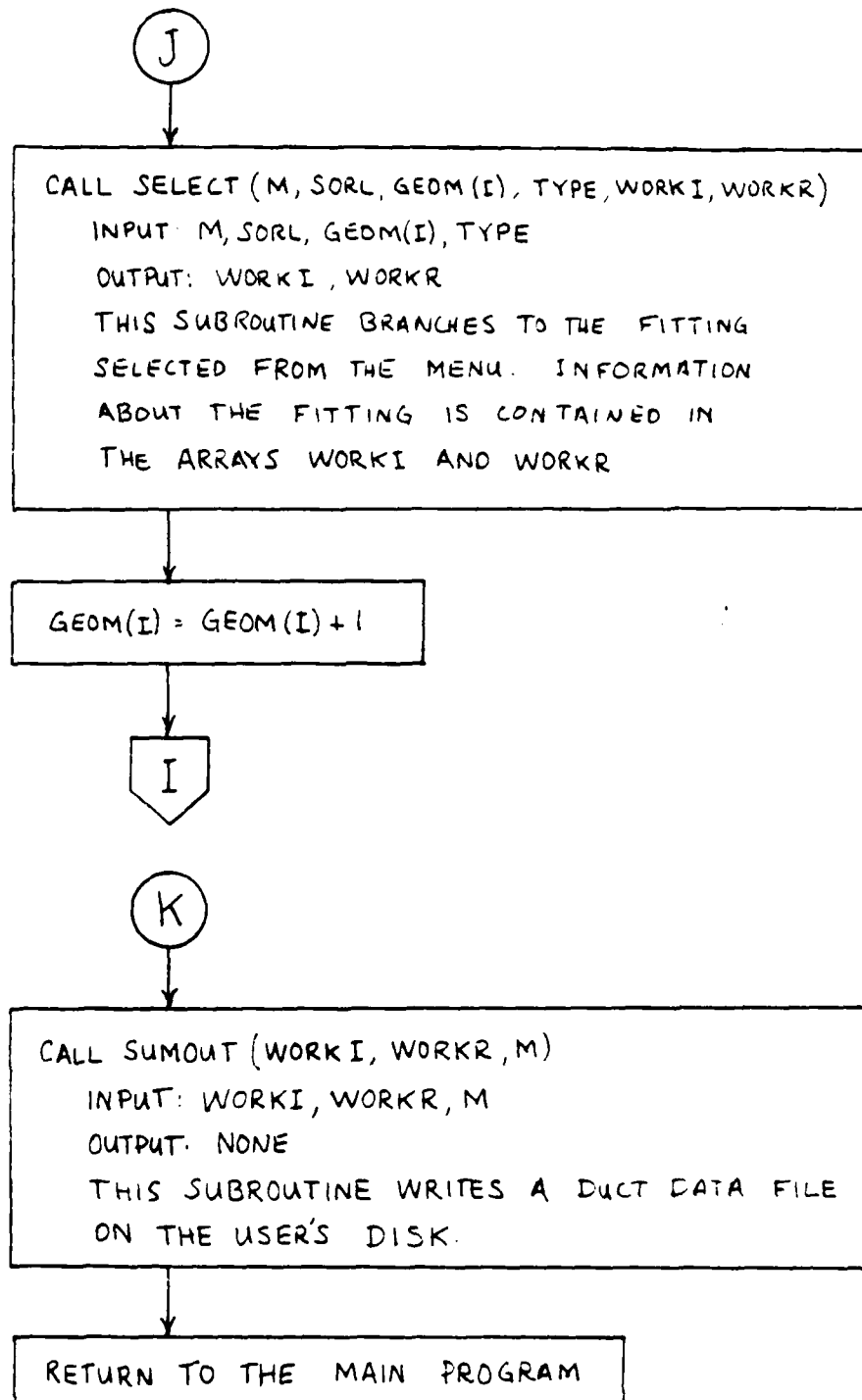




SEE THE PRELIMINARY SECTION OF THE
USERS MANUAL FOR EXPLANATION OF
IDENTIFICATION NUMBERS.

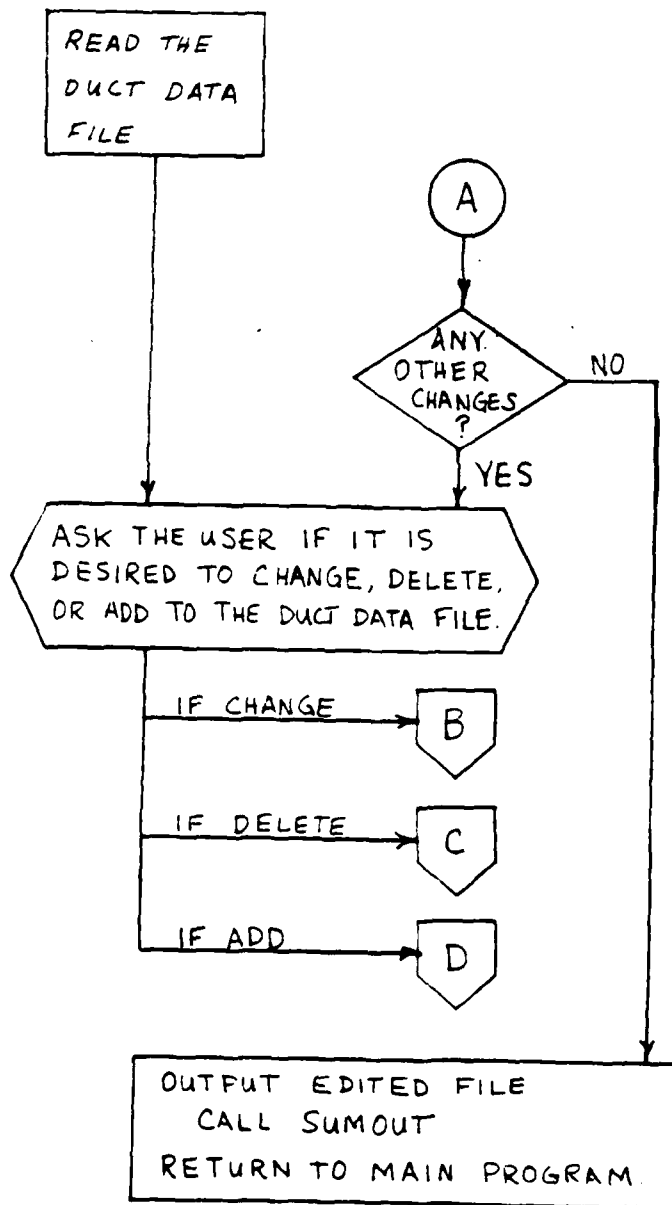


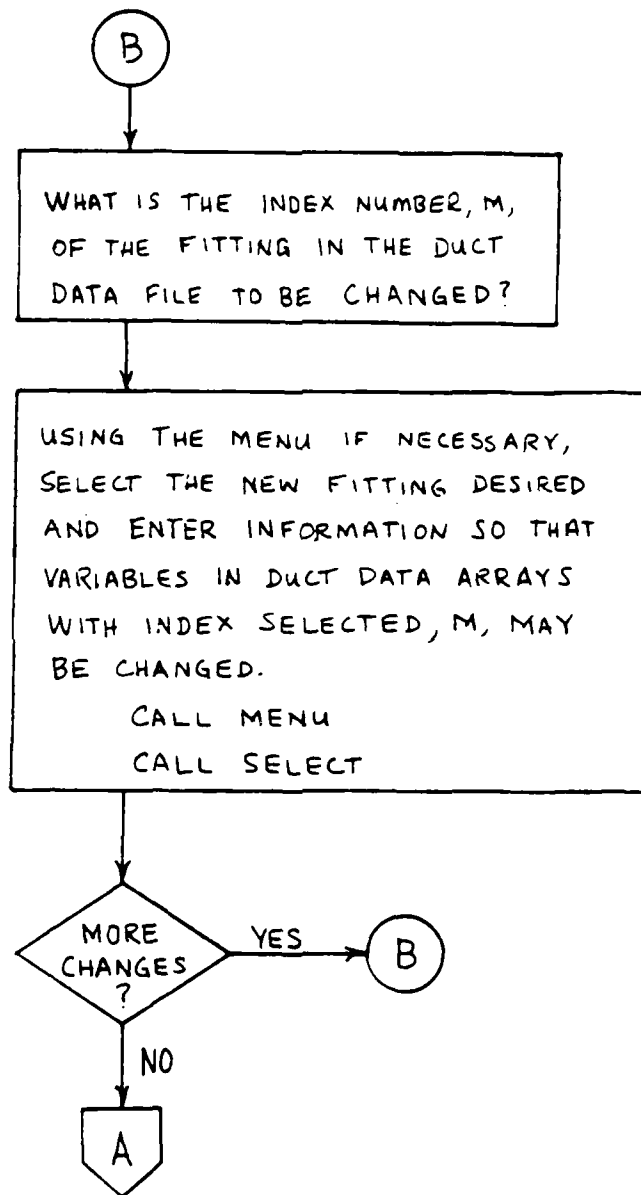


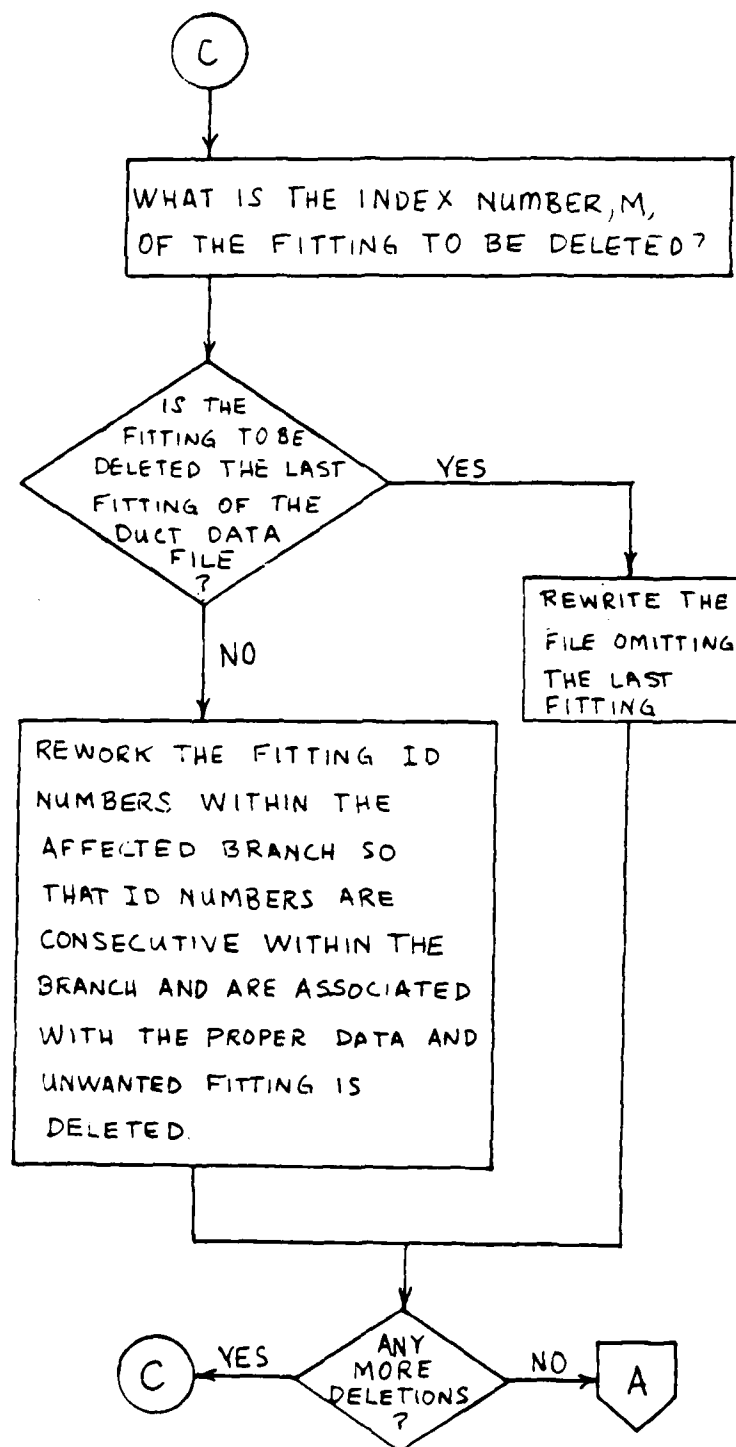


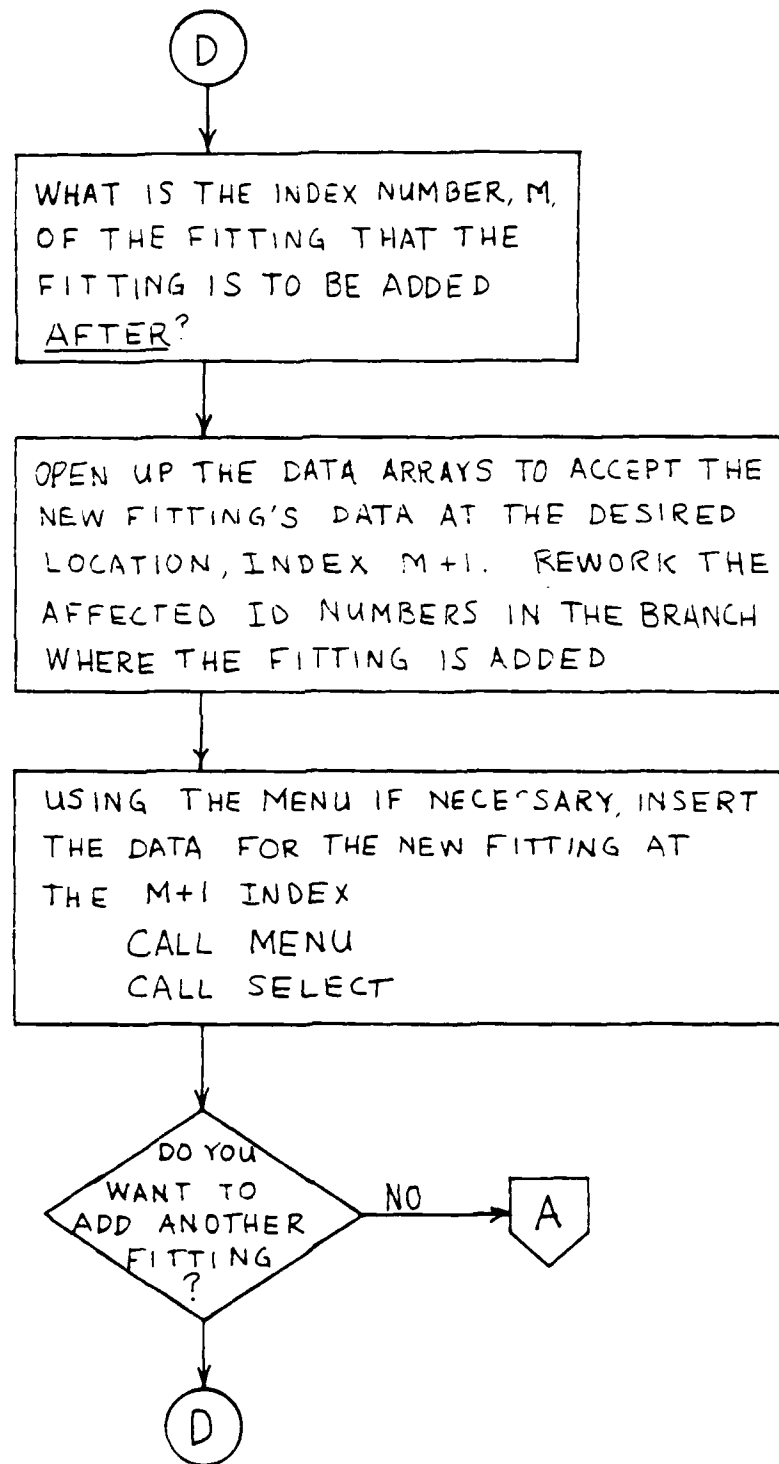
III. EDITING SUBROUTINE (ED)

THERE ARE NO INPUT OR OUTPUT VARIABLES FOR THIS SUBROUTINE, HOWEVER SUBROUTINES CALLED BY THE ED SUBROUTINE DO HANDLE INPUT AND OUTPUT DATA.

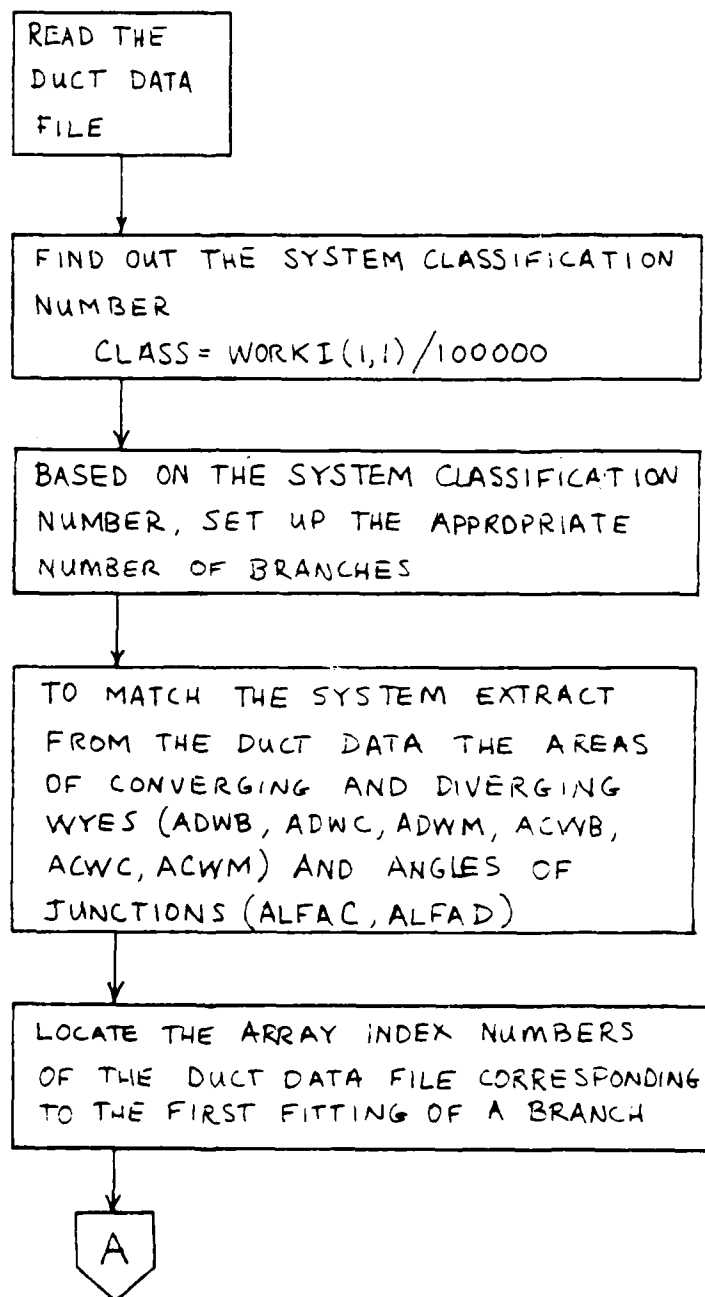


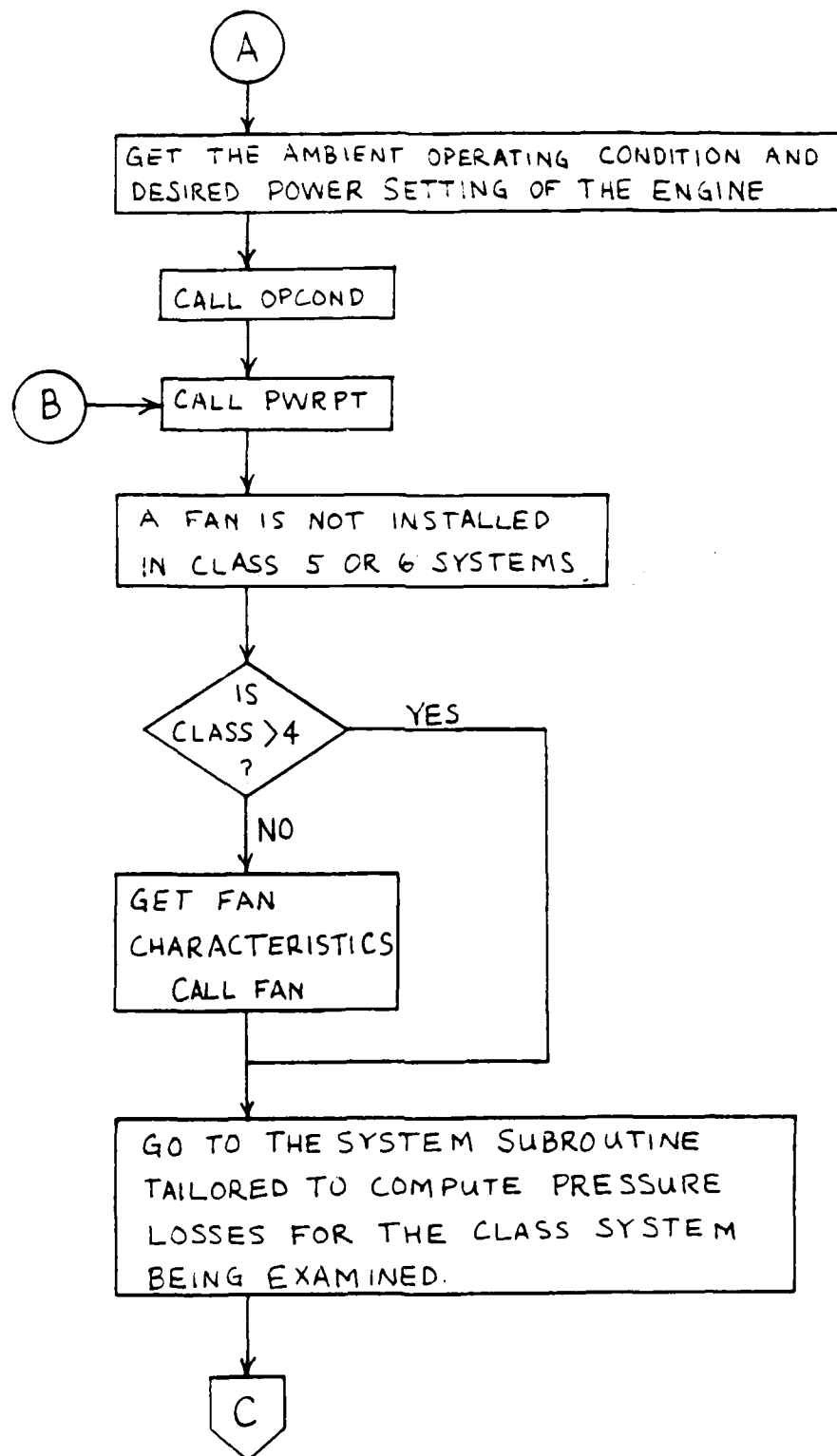


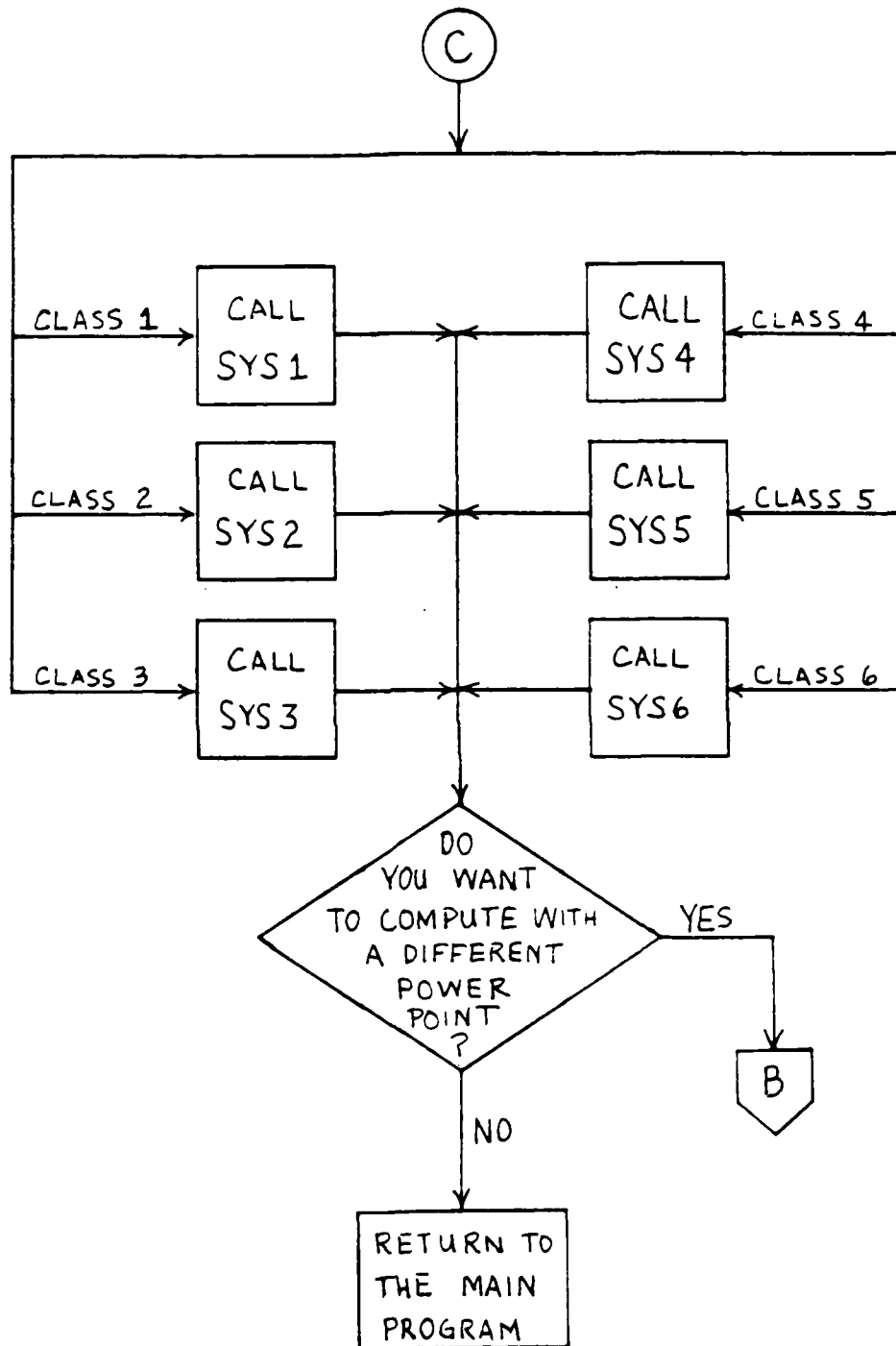




IV. COMPUTE SUBROUTINE







II. SYSTEM THREE MATCHING SUBROUTINE (SYS3)

THIS SUBROUTINE IS CALLED BY THE COMPUTE SECTION OF THE PROGRAM. ALL VARIABLES ARE INPUT FROM COMP SUBROUTINE. THERE ARE NO OUTPUT VARIABLES RETURNED TO COMP, ALL OUTPUT IS WRITTEN TO THE PERFORMANCE FILE.

SET UP THE STARTING AND STOPPING INDEXES FOR THE DATA ARRAYS FOR THE BRANCHES

INITIALIZE SYSTEM VARIABLES FOR START OF ITERATION

DUCT LOSSES

INLOSS = 4.0 (INCH WG)

EXLOSS = 8.0 (INCH WG)

EDUCTOR GAIN

GAIN = -30.0 (PSF)

COOLING FLOW PASSAGE LOSS

LOSS = 30.0 (PSF)

COOLING FLOW

$WC = CFMAX * RHSTD / 60.0$

BRANCH INFORMATION

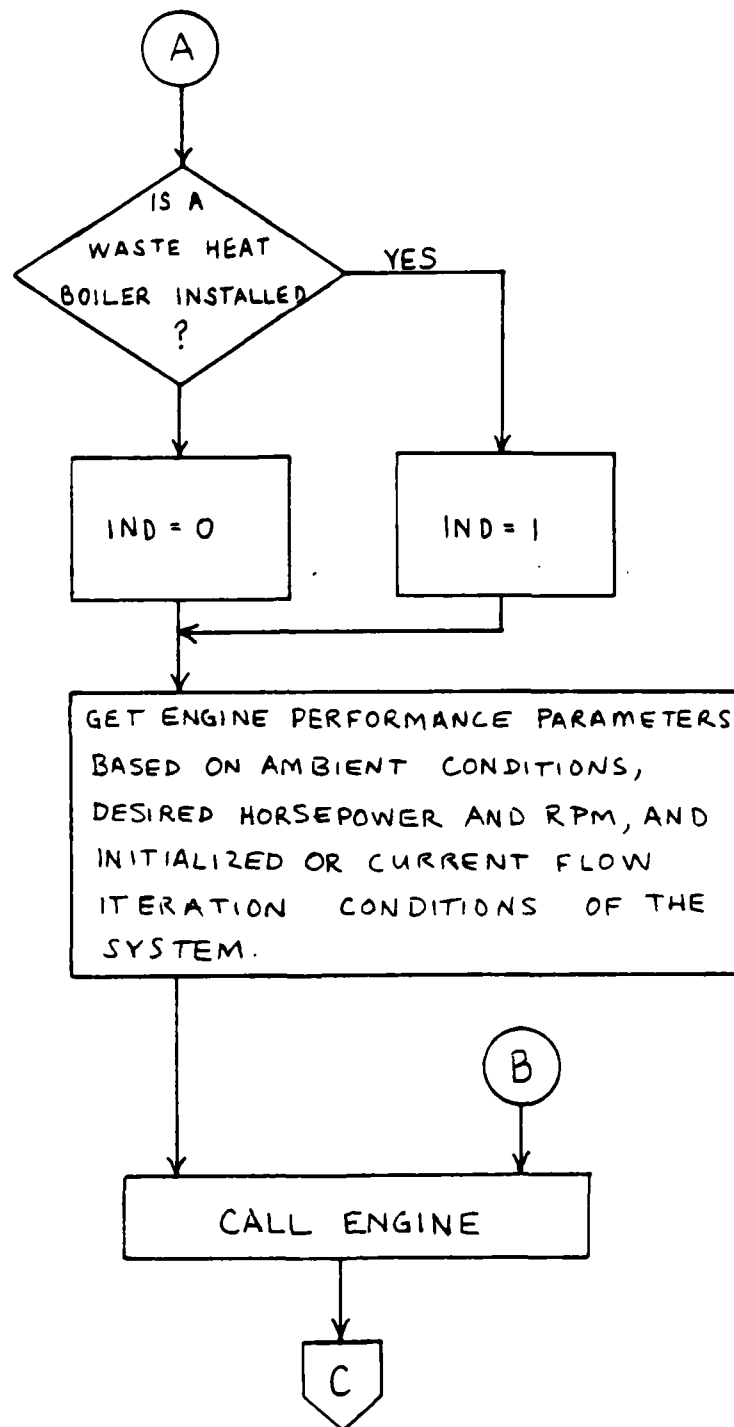
DP45 = 100.0 (PSF)

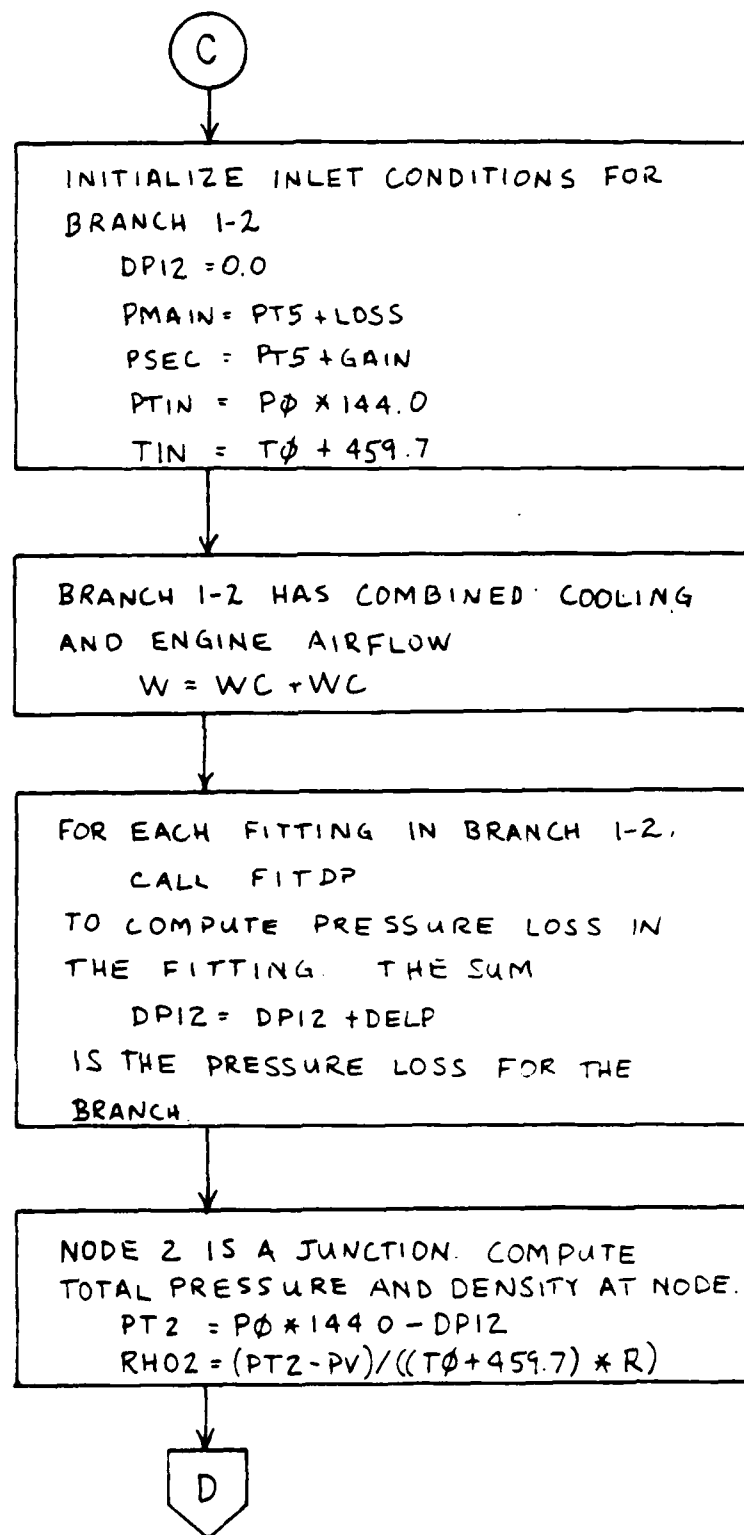
DP56 = 100.0 (PSF)

TMOD = 710.0 (°R)

$PT5 = P0 * 144.0 + DP56$

A





D

COMPUTE THE AVERAGE VELOCITIES IN THE THREE BRANCHES ENTERING AND LEAVING NODE 2, A DIVERGENT WYE.

BRANCH COOLING AIR: $VDWB = WC / (RH02 * ADWB)$

COMBINED INLET: $VDWC = (WC + W2) / (RH02 * ADWC)$

MAIN ENGINE: $VDWM = W2 / (RH02 * ADWM)$

COMPUTE NODE 5 PARAMETERS. NODE 5 IS A CONVERGENT WYE, MIXING STREAMS OF DIFFERENT TEMPERATURES. IF NO WASTE HEAT BOILER IS INSTALLED TEMPERATURE OF THE MAIN BRANCH, EXHAUST FROM THE ENGINE IS:

$TMAIN = T8$ ELSE,

$TMAIN = 770.0 + (370 * 10^{-3} * HP)$

COMPUTE TEMPERATURE IN COMBINED EXHAUST STACK BASED ON MIXING ENTHALOPY OF COOLING AND EXHAUST STREAMS.

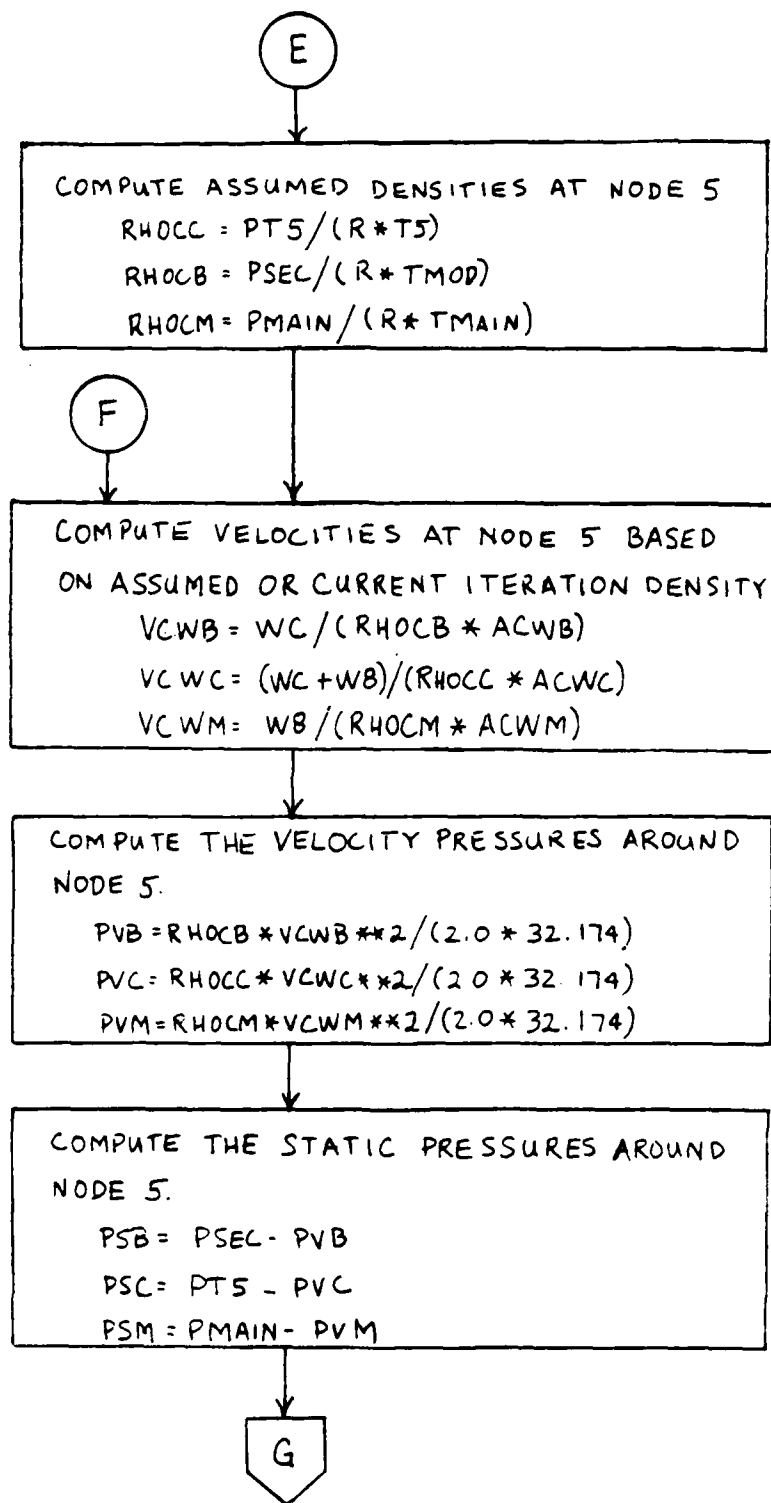
COOLING ENTHALOPY: $HSEC = (1.421385E-5 * TMOD + .221091) * TMOD + 5.6373$

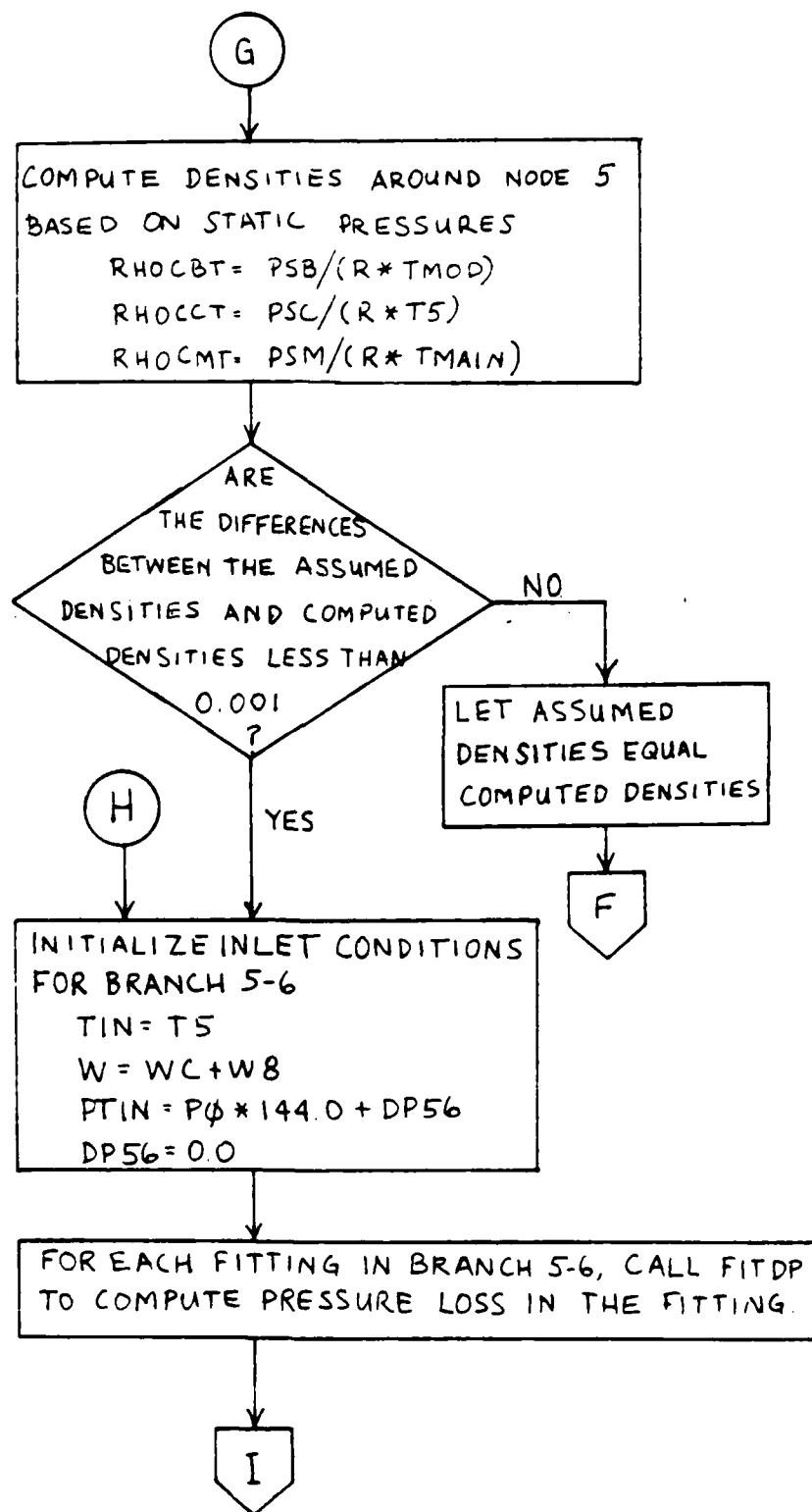
EXHAUST ENTHALOPY: $HMAIN = (1.56051E-5 * TMAIN + .22388) * TMAIN + 4.75396$

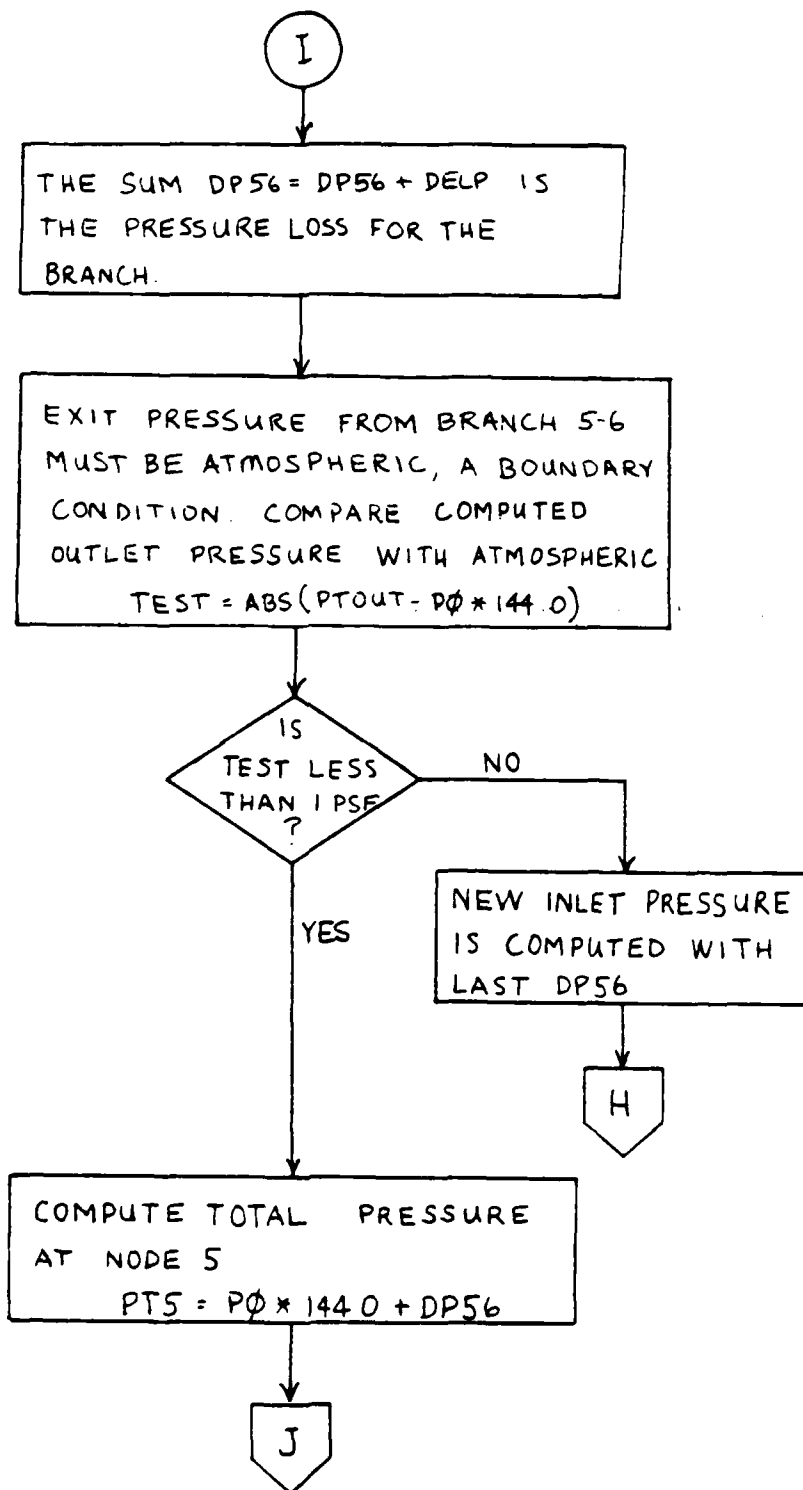
COMBINED ENTHALOPY: $HSTACK = (WB / (WB + WC)) * HMAIN + (WC / (WB + WC)) * HSEC$

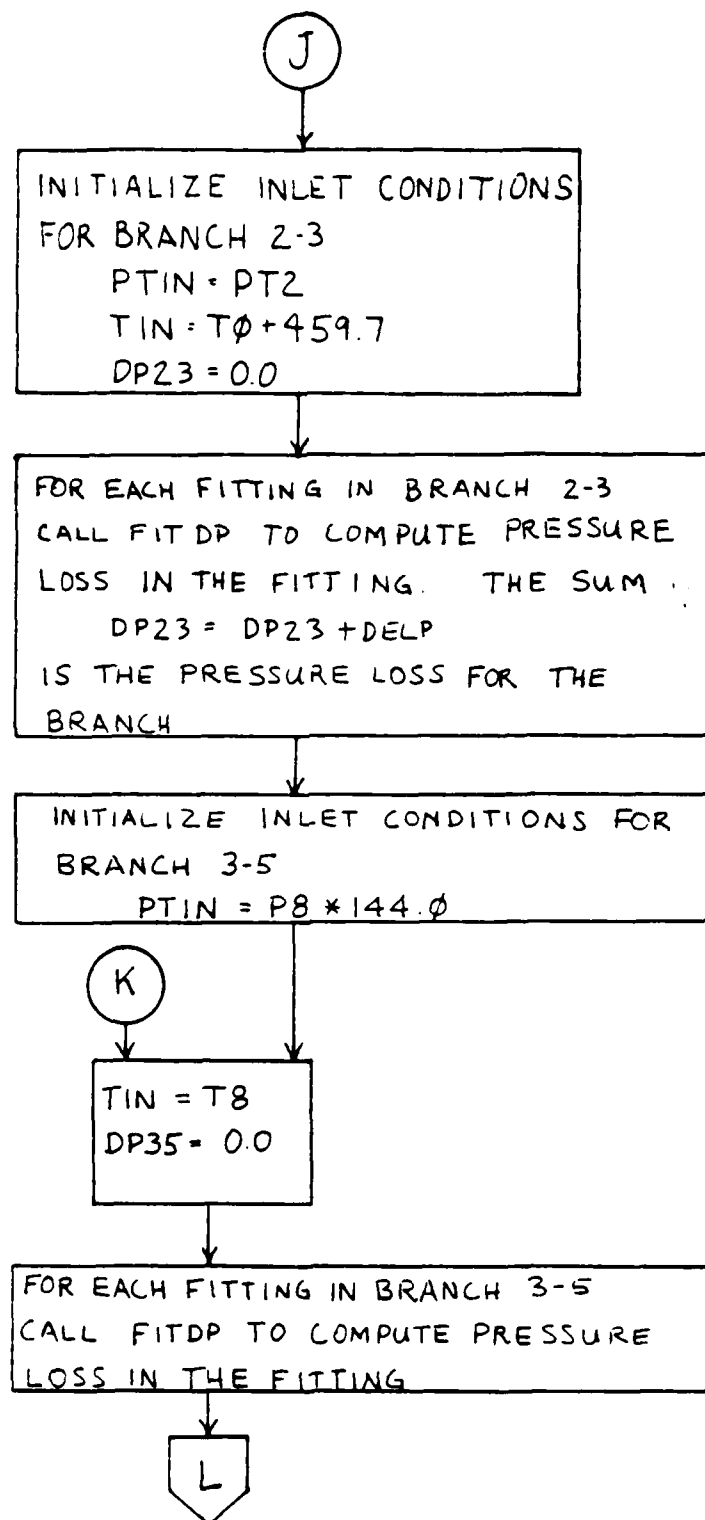
EXHAUST TEMPERATURE: $T5 = (0.000841) * HSTACK + 4.33577 * HSTACK - 9.5778$

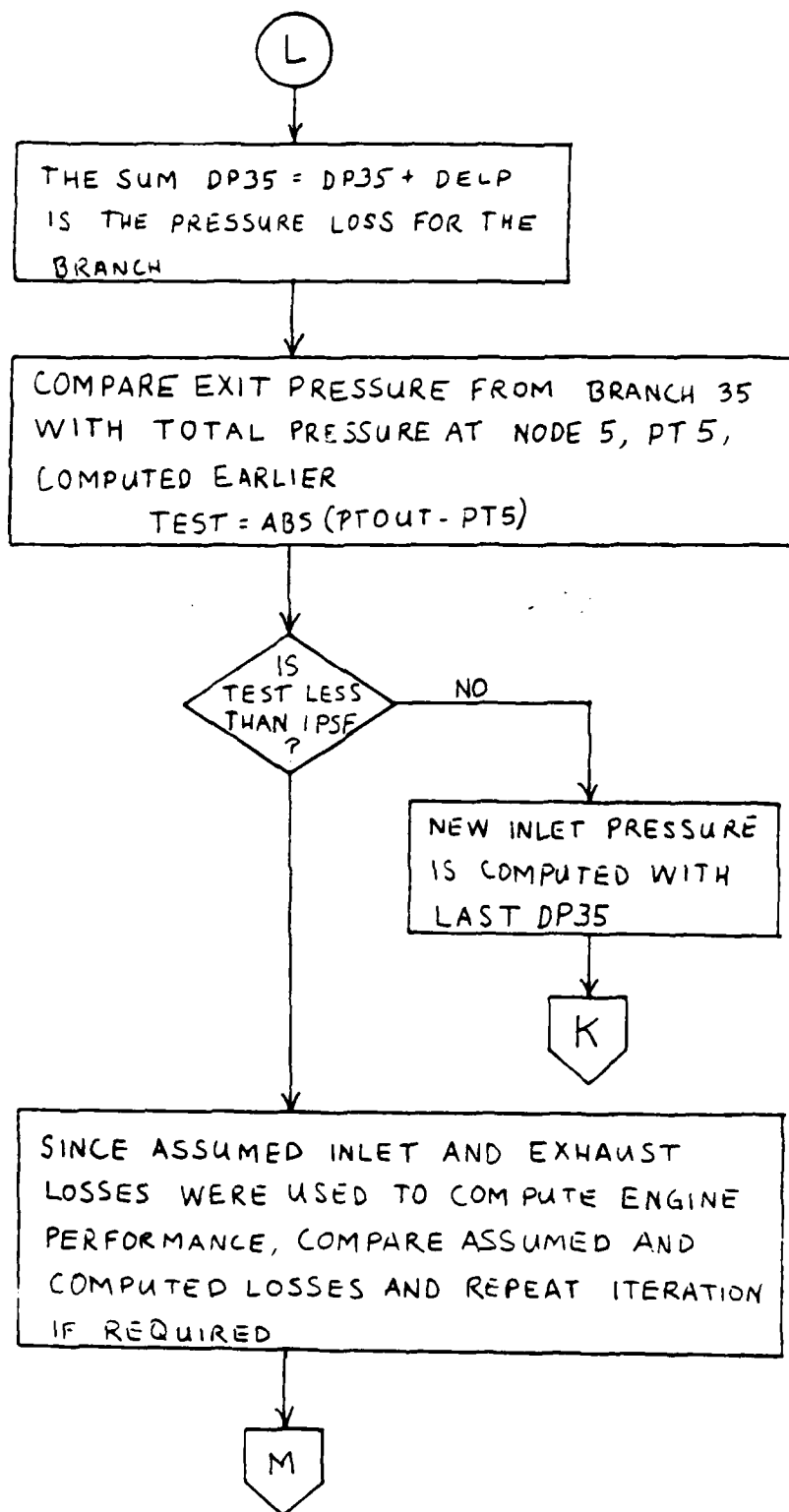
E

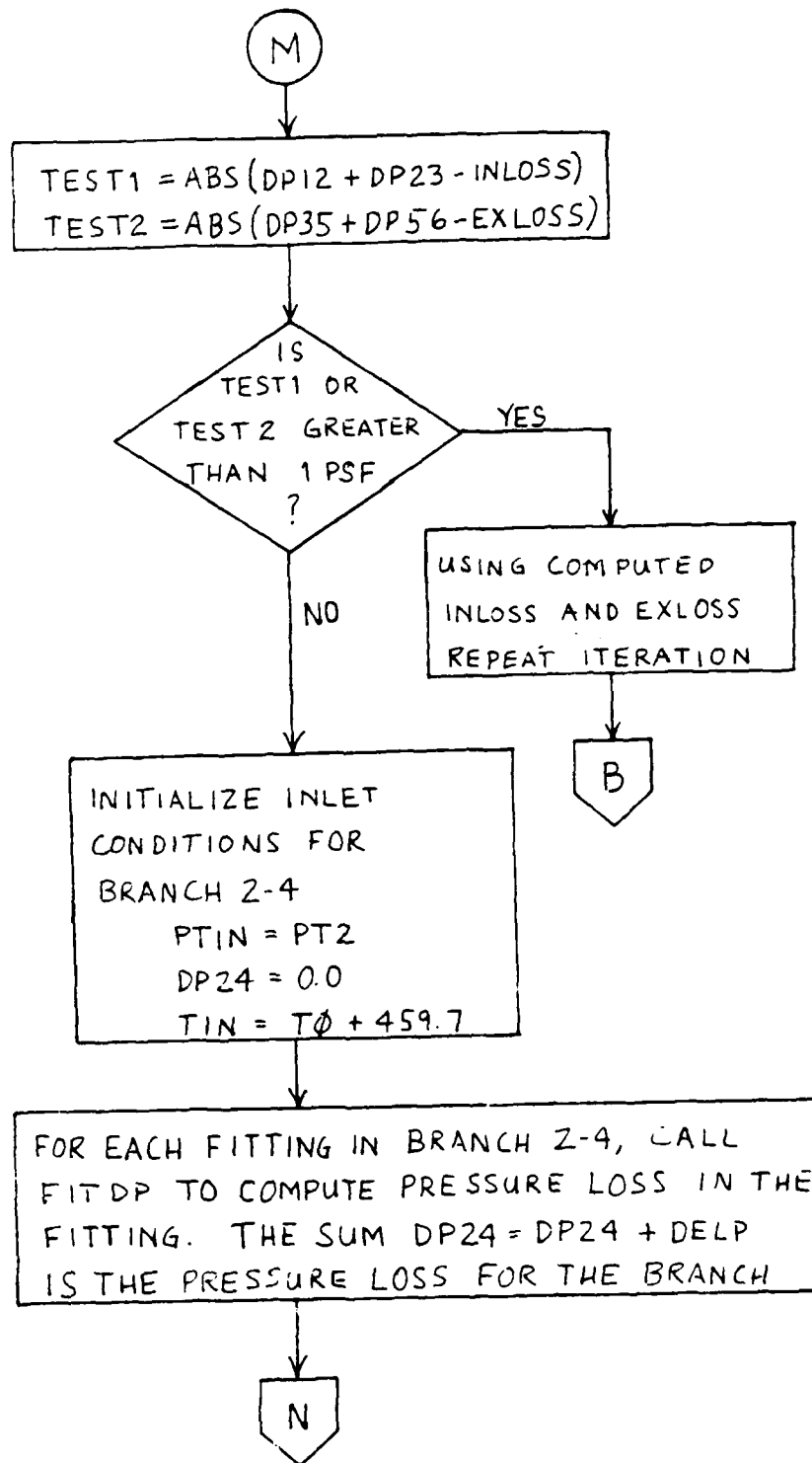


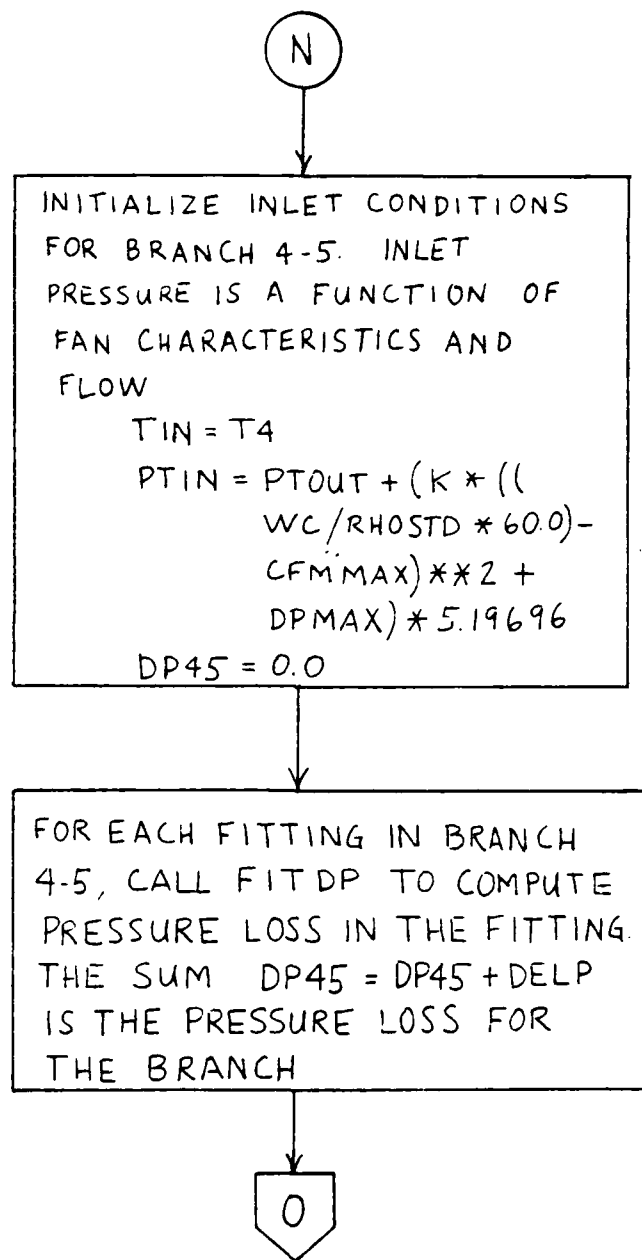


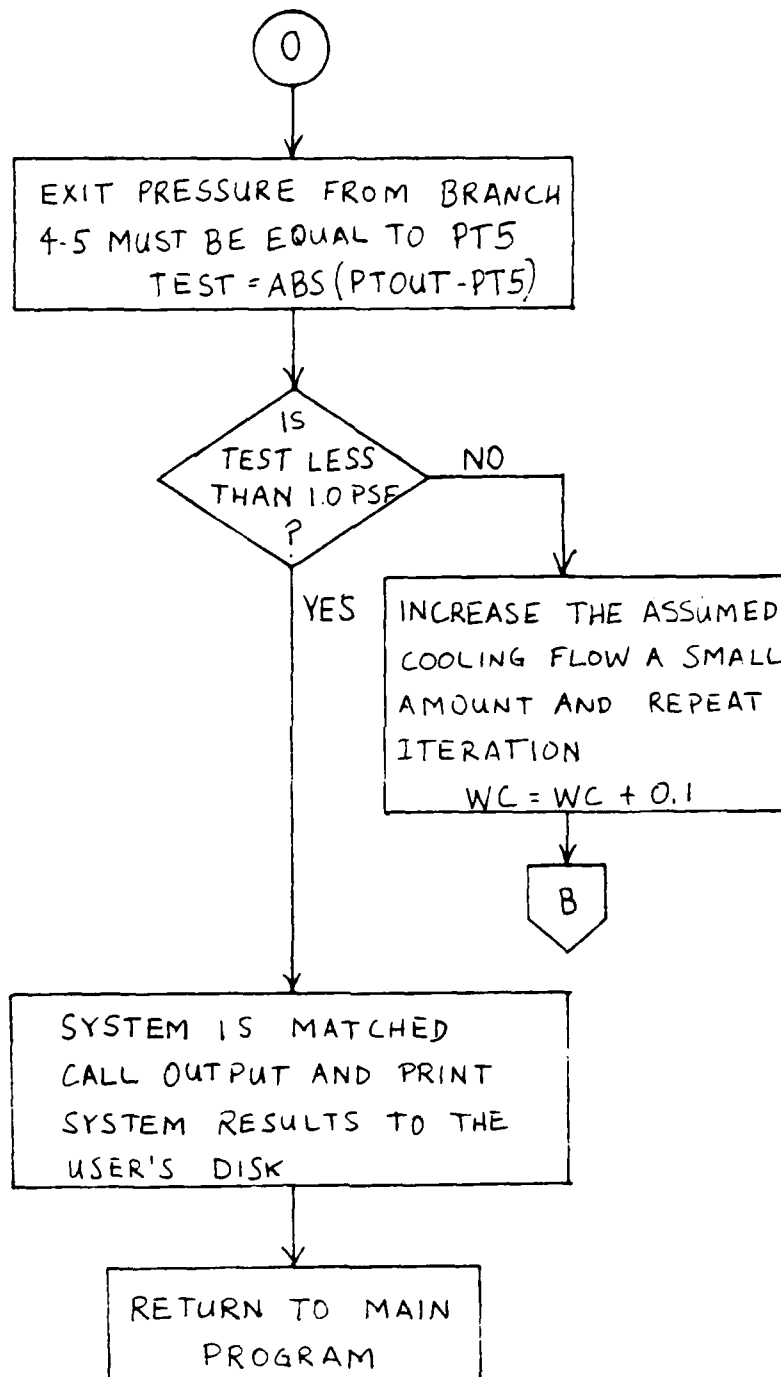




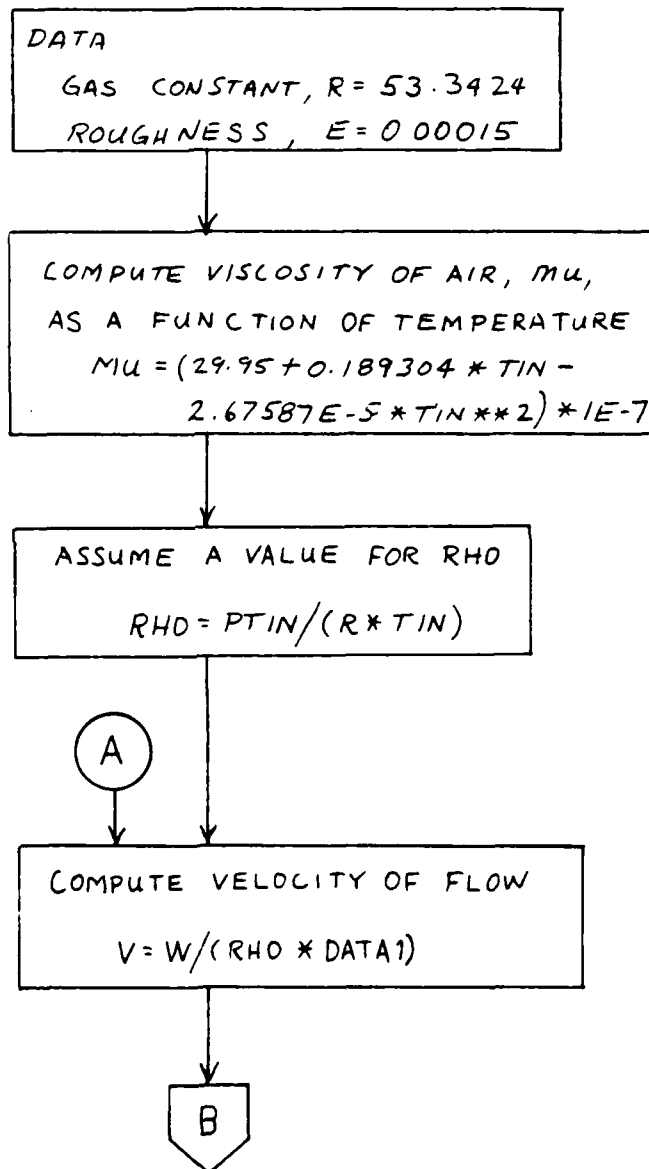


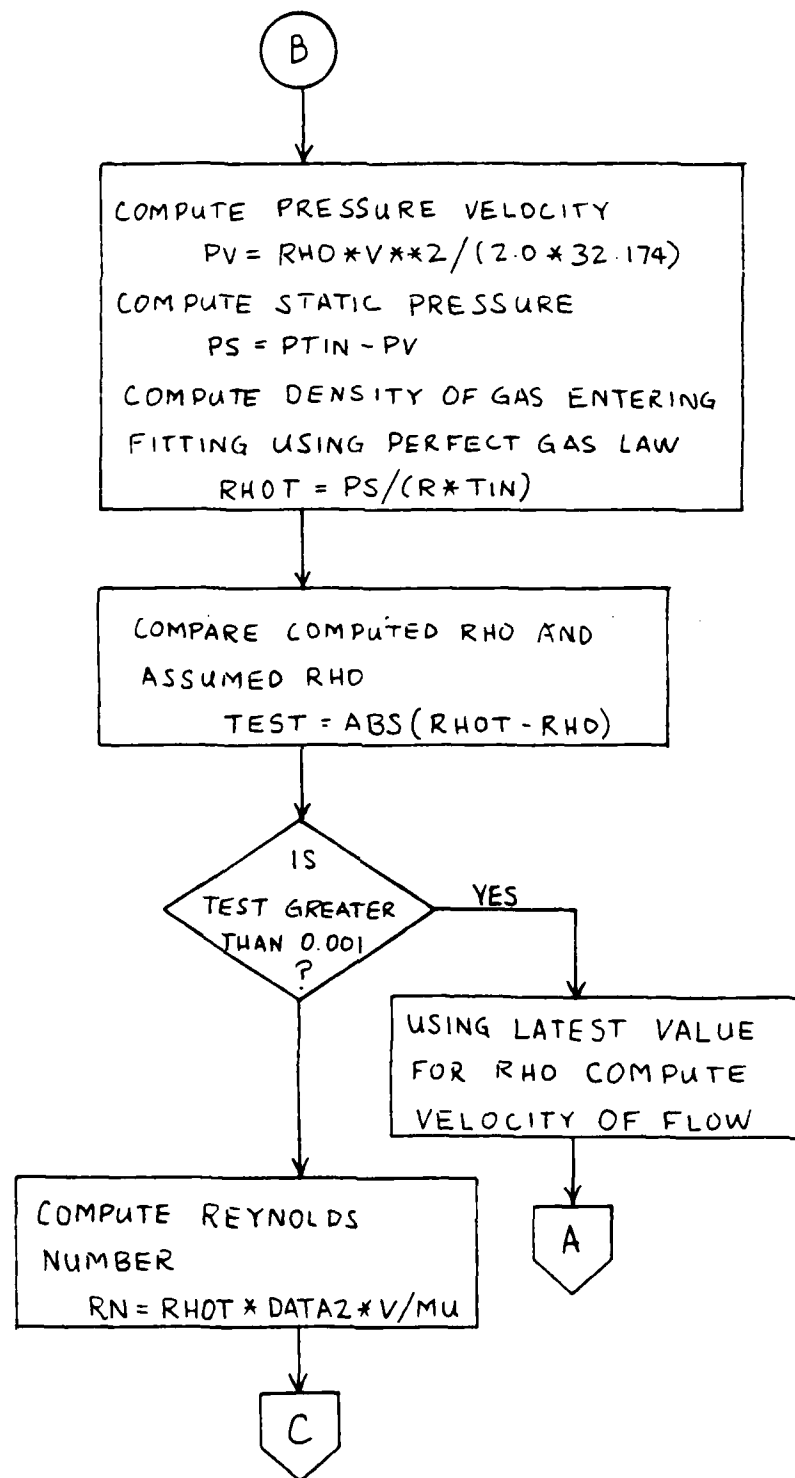


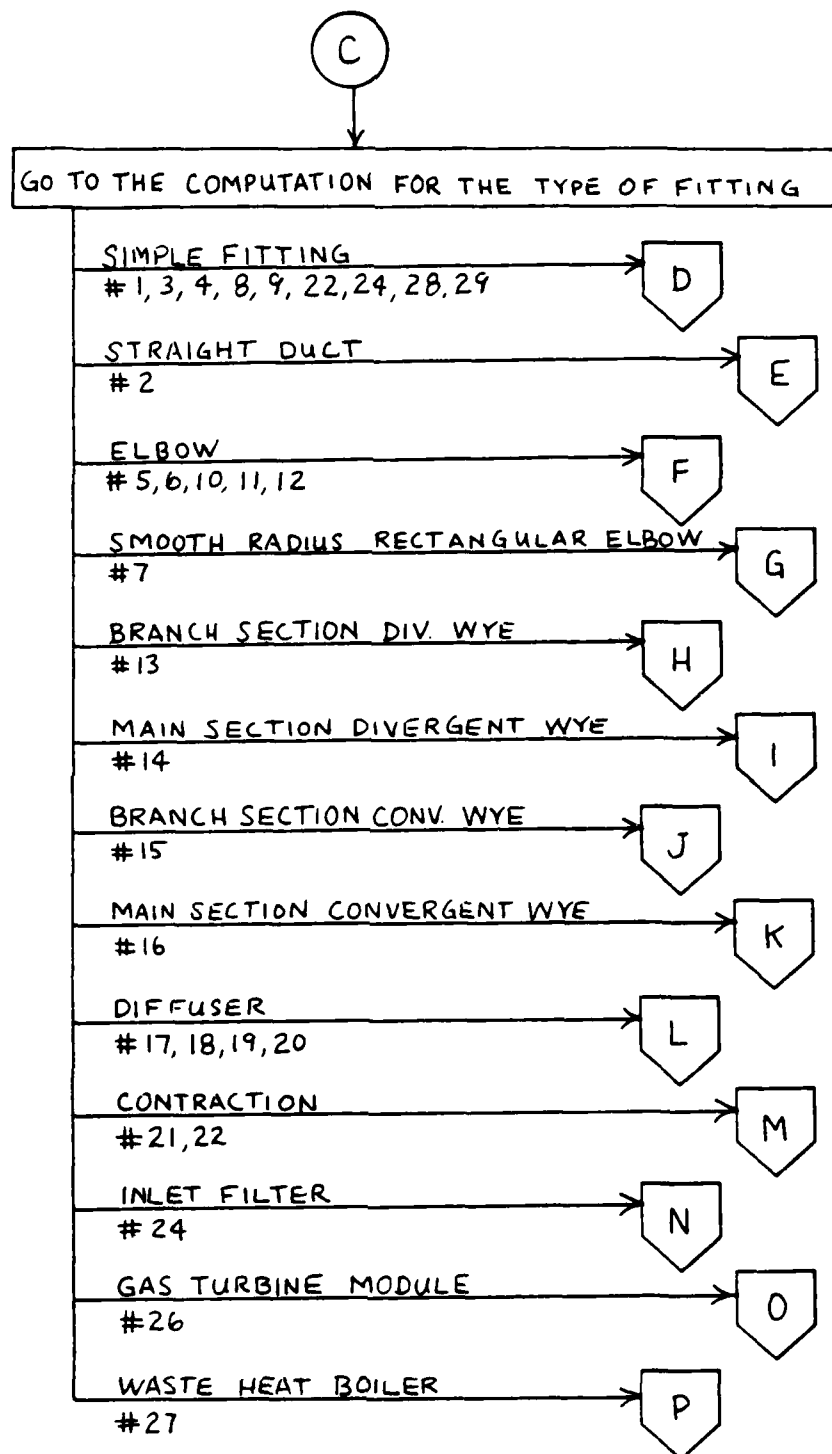


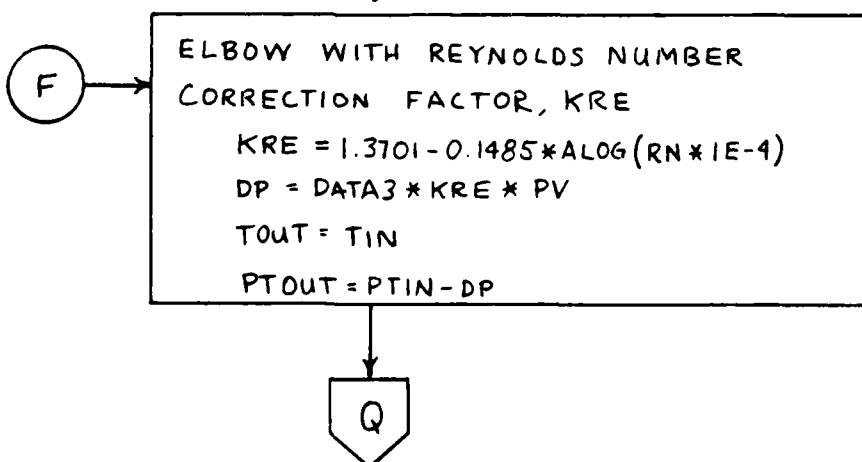
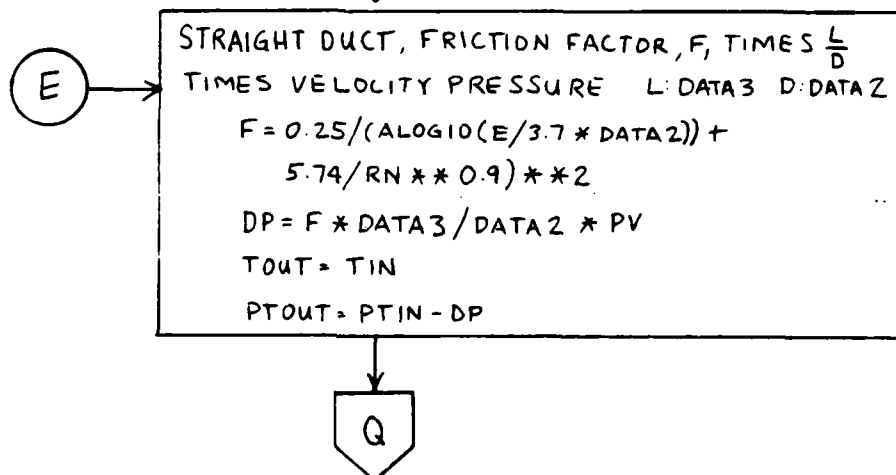
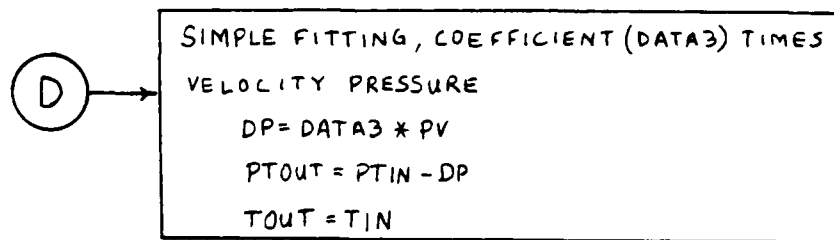


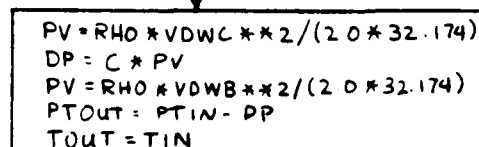
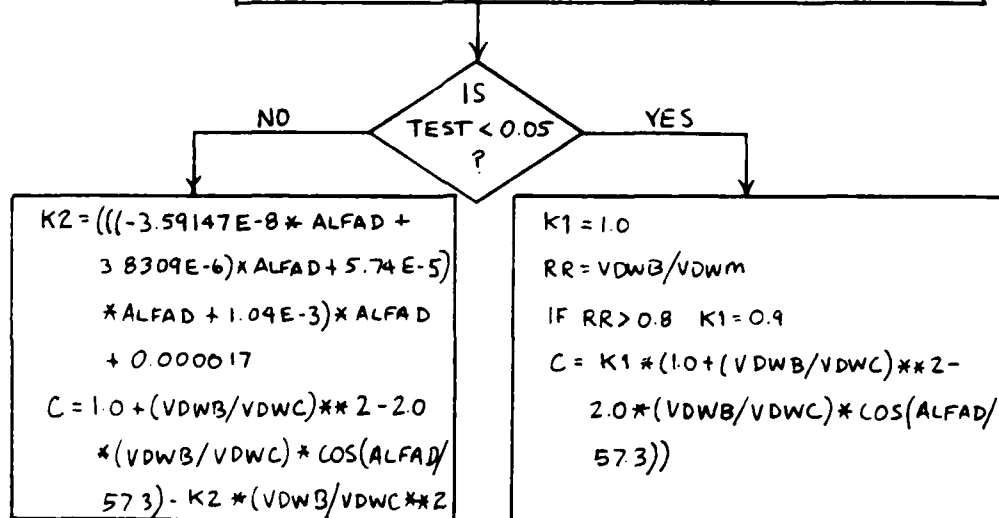
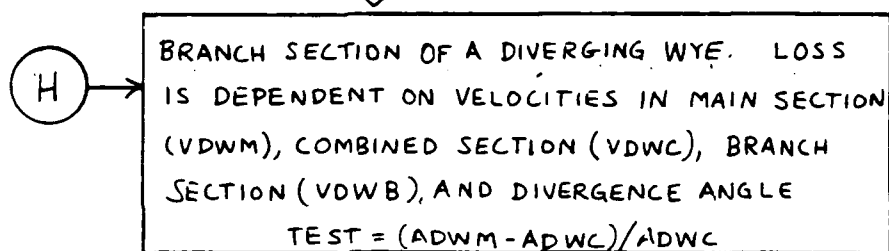
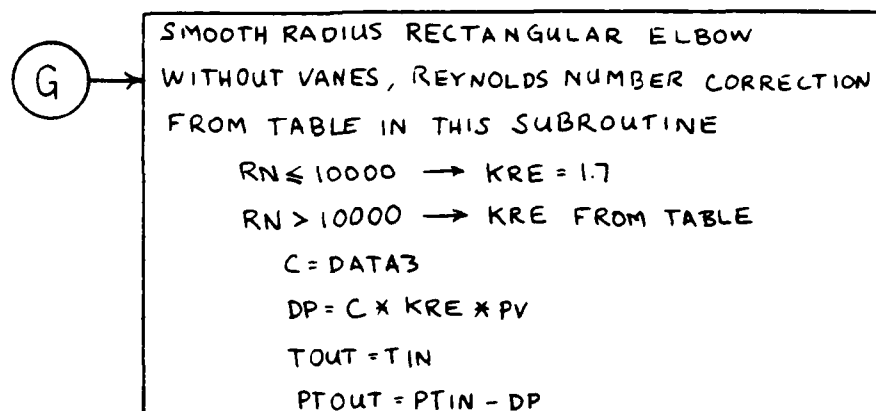
VI. FITTING PRESSURE LOSS CALCULATION
SUBROUTINE. SET UP TO COMPUTE
PRESSURE LOSS AND VELOCITY DATA FOR
30 FITTINGS LISTED IN THE MENU

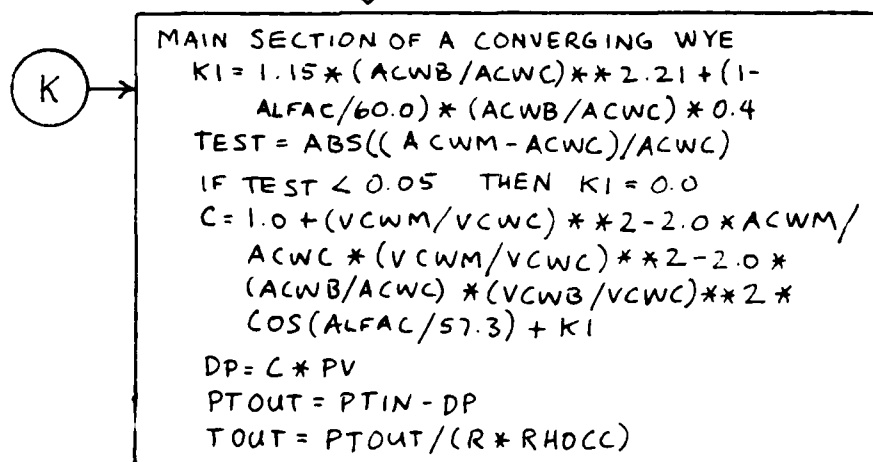
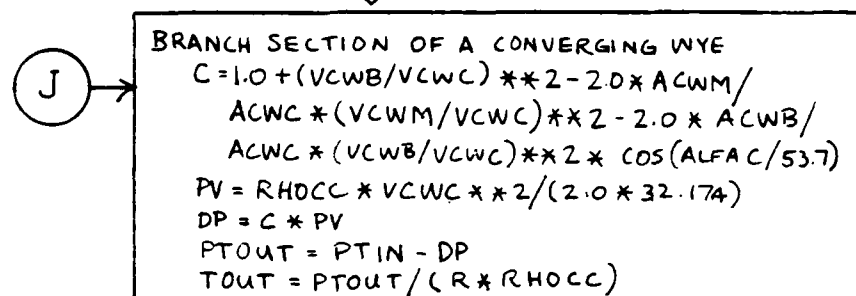
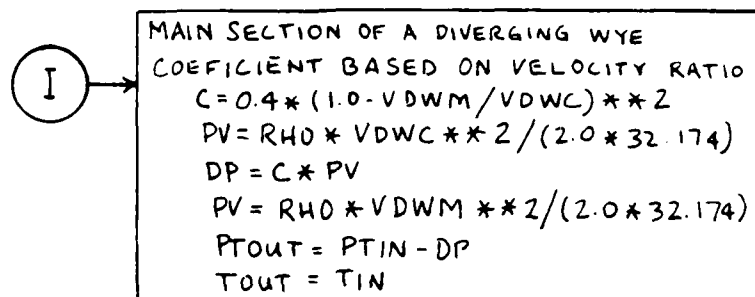


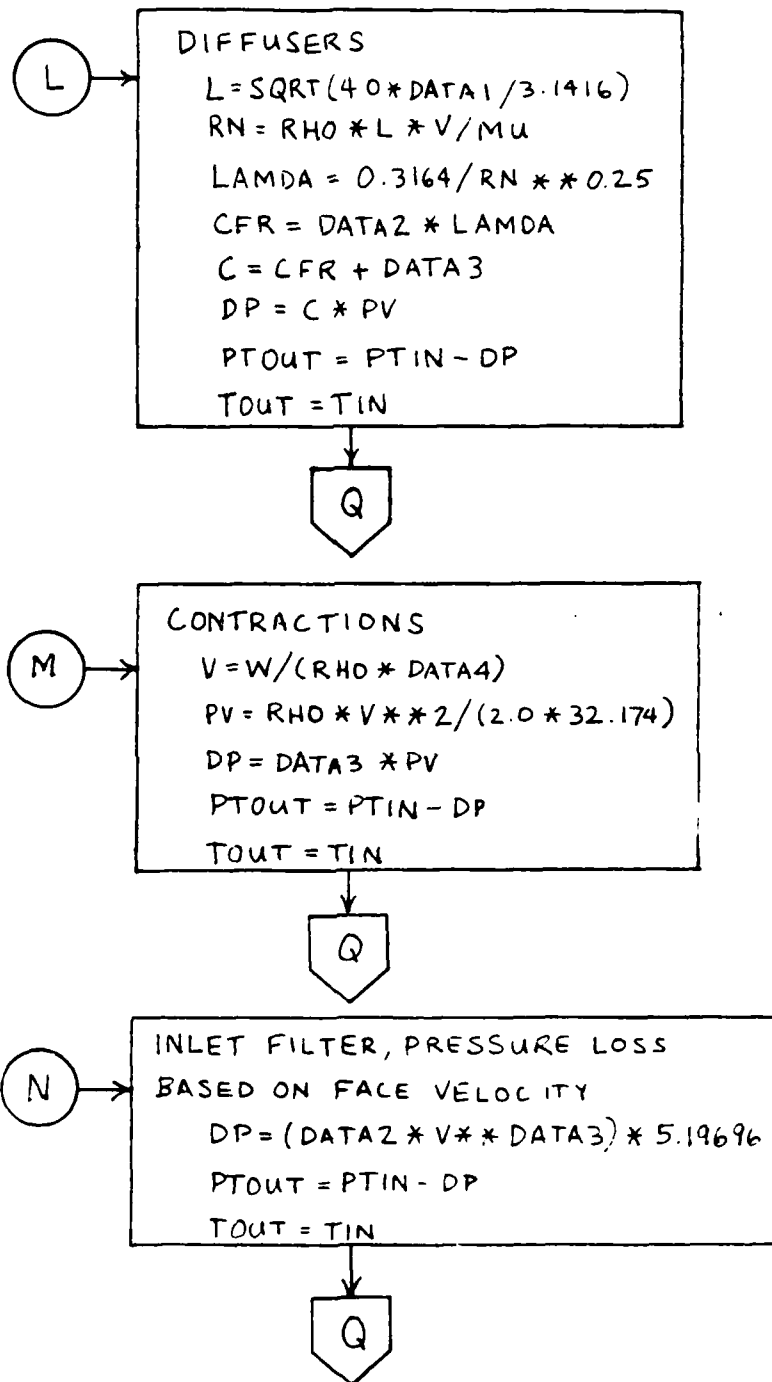


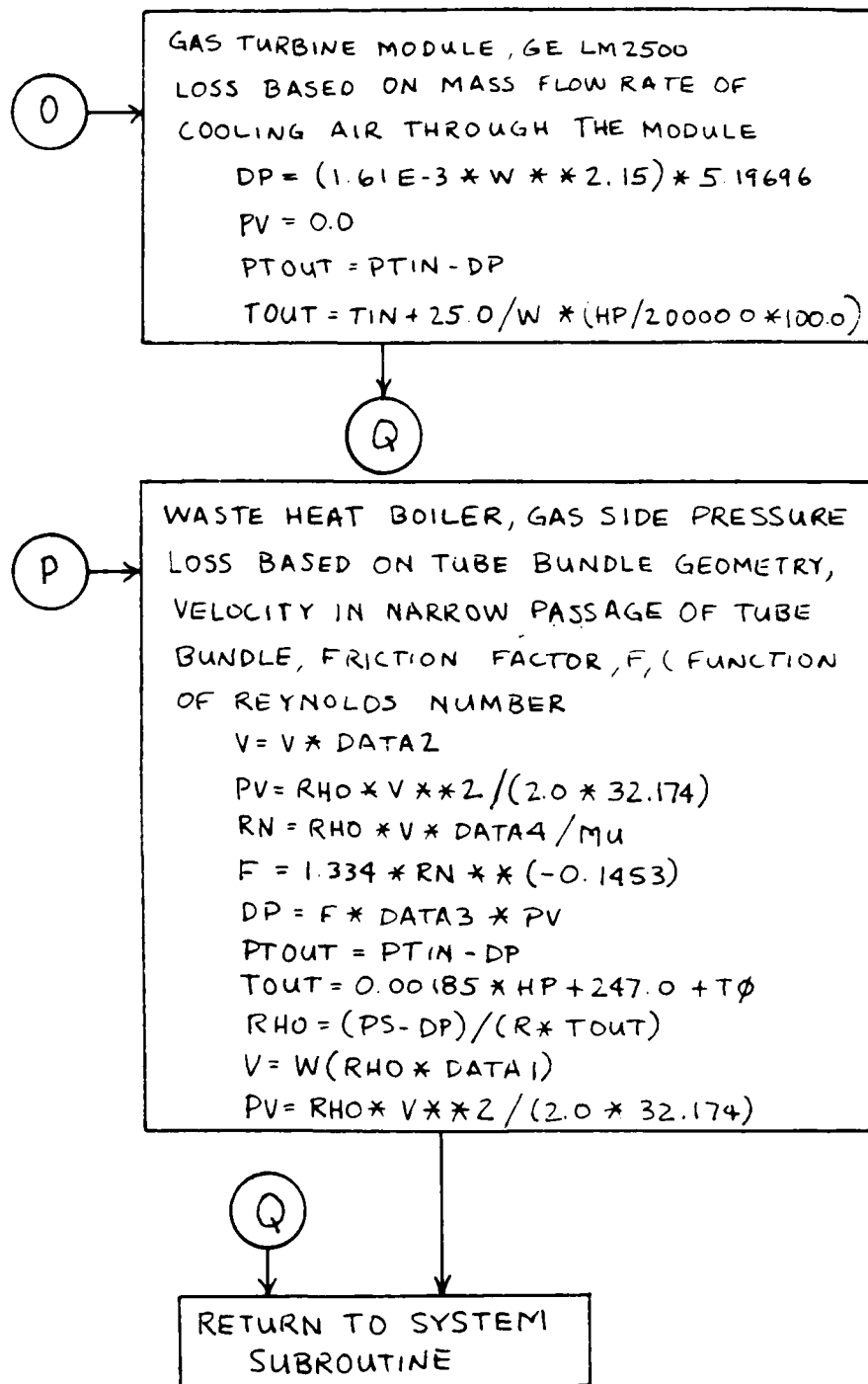












APPENDIX C

USER'S MANUAL

A. GENERAL

The purpose of this program is to analyze a marine gas turbine installation on board a ship complete with inlet, exhaust, and cooling ductwork. The duct geometry must be input to the program to accomplish this. The program makes a file called "duct data" which contains resistance information on each fitting entered. This file may be edited with the built in editor or if the user is satisfied with the current design the file is read by the program and used in the COMPUTE section of the program. COMPUTE uses the duct data file and inputs dealing with the operating point of the engine to produce the performance parameters of the system. Performance includes both engine parameters and duct losses. All procedures in the program are accomplished using an interactive terminal session.

There are two versions of the program discussed in this user's manual. Version 1.0 is implemented on the NPS IBM 3033 computer. Version 1.1 is implemented on the NPS VAX-11 computer.

This user's manual will discuss the questions posed by the program. Familiarity with the program sections and the questions asked in each section will facilitate program execution and help produce reasonable results. The most critical area for familiarity is in the BUILD and EDIT sections of the program. It is not so critical in the COMPUTE section of the program because only two questions are asked for each operating point run after the ambient conditions are input.

B. PRELIMINARY

The program does not design ducts or read mechanical drawings. The user plays a vital role by interpreting the system for the program. Some fittings are easy to recognize such as elbows, straight duct, transitions, diffusers and contractions. Some are harder to understand, like diverging and converging wyes. Each fitting listed in the menu is sketched for the user. The sketches show a typical view but remember that the dimensions shown on the drawings are variable inputs so the configuration can change drastically by looking at a fitting over the range of variable dimensions.

Before running the program the user should become familiar with the fitting sketches. Comparing the sketch to the fitting to be modeled will assist the user in preparing a list of fittings for the system. The user should note the dimensions and be prepared to input them to the program.

The program looks for fittings in a definite sequence. Branches are groups of fittings or sections of the ductwork. Branches run from node to node. A node is an entry, exit, junction, fan, or engine. Refer to figure 2.6 for the various system configurations. Nodes are indicated in this figure by the numbered black dots. Nodes have numbers from one to six. The branches get their number designation from the end point nodes. The user should become familiar with the system schematics then it will be easy to understand the order that the program will be asking for fittings. Branches are entered in a sequence from the lowest number node to the next lowest number node until all fittings are entered. For example, a class three system enters branches in the following order; 1-2, 2-3, 2-4, 3-5, 4-5, 5-6. To assist the user when entering fittings the program displays the current fitting identification number on the screen with the menu. The ID number is a six digit number where the

first digit is the system class, the next two digits are the branch number and the last two numbers are the sequence number of the fitting in the branch. A terminal session has been recorded and the printout annotated to show this number.

It would be helpful to pencil in the node numbers in the system drawings. The following table may help.

TABLE II
Node Designations

- | | |
|---|---|
| 1 | Main air inlet (engine only or combined) |
| 2 | Cooling air inlet or divergent wye
off main inlet |
| 3 | The engine |
| 4 | A fan |
| 5 | Cooling air exit or convergent wye
with main exhaust |
| 6 | Main exhaust (engine only or combined) |

The user should prepare a list of fittings organized by branches and continuous with regard to the sequence of fittings. It's the old "toe bone connected to the foot bone" idea. As an example, the following list may help.

node 1

vert intake, 3 orifices, with louvers
straight duct
rectangular contraction
smooth radius rect elbow

node 3

etc.

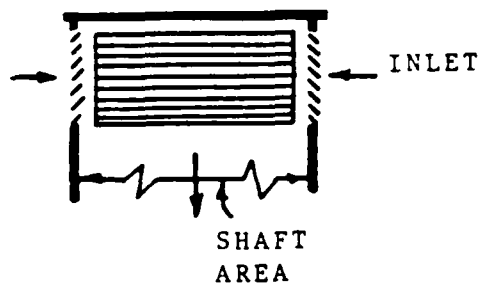
Do not forget to include abrupt exits where they appear. Sometimes it is easy to overlook an obvious fitting such as the engine module as part of the cooling air ductwork.

Only the class one system does not have either a divergent wye or a convergent wye. Class three and five have both. The divergent wye is fairly straight forward. The user only needs to enter the areas indicated in the sketch and the angle of divergence (0-90). The branch section of the divergent wye is the first fitting in branch 2-4 (2-5 if no fan) and the main section (combustion air) is the first fitting in branch 2-3. The combined area and the divergence angle are data entered when entering the branch of the diverging wye. The convergent wye is a more complex. It is located at node five. The branch of a convergent wye should be the last fitting of branch 4-5 (2-5 if no fan). It will usually be the fitting after the module. The main section (engine exhaust) of the convergent wye is the last fitting of branch 3-5. Usually there are just two fittings in branch 3-5. The first is the nozzle or extension bolted to the exhaust plane flange of the engine, and the last is the main section of the convergent wye. The combined area and convergence angle are data entered with the branch section. The convergence angle is usually zero and the combined area is about equal to the sum of the main and branch areas.

FITTING NAME:
Vertical intake shaft with side orifices,
with or without louvers

NUMBER:
01

SKETCH:



	OPENINGS				
WITHOUT LOUVERS	1	2	2	3	4
WITH LOUVERS	1	2	2	3	4

INPUT REQUIREMENTS:

1. The number of orifices (1,2,3, or 4)
2. The cross section area of the vertical shaft
3. With two orifices, whether they are adjacent or opposite
4. If there are louvers installed

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
shaft area	0.0	resistance coefficient	shaft area

REMARKS:

The louvers are flat plates of standard configuration. The opening areas are not required but should be approximately proportional to those shown in the sketch.

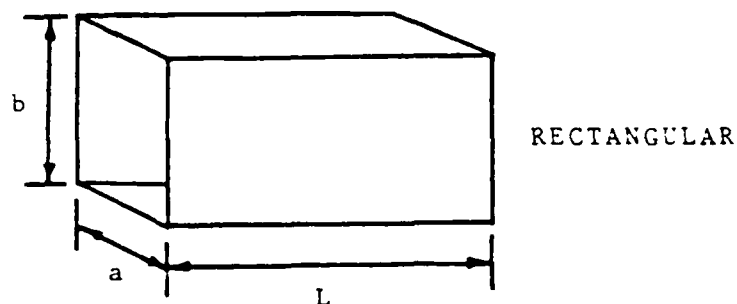
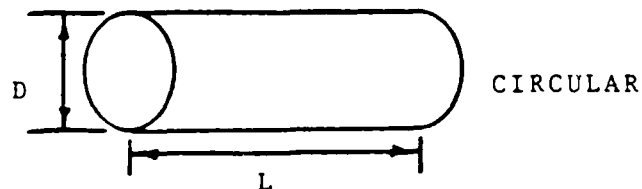
REFERENCE:

Handbook of Hydraulic Resistance, Idel'chik

FITTING NAME:
Straight duct, round or rectangular

NUMBER:
02

SKETCH:



INPUT REQUIREMENTS:

1. Round: diameter and length
2. rectangular: cross section dimensions (a, b)
length

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
section	diameter or	length	section
area	equivalent		area

REMARKS:

Darcy-Wiesbach Equation used for resistance.
Friction factor by correlation by Swamee & Jain.
Equivalent circular diameter computed for rectangular
sections. Length should be measured to the center of
short fittings and to the start or end of a long
fitting.

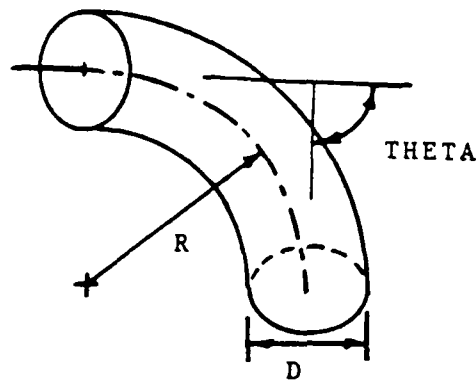
REFERENCE:

Mechanics of Fluids, Shames

FITTING NAME:
Smooth radius round cross section elbow

NUMBER:
03

SKETCH:



INPUT REQUIREMENTS:

1. Cross section diameter
2. Radius of the turn measured to the centerline of the section
3. The turn angle

DUCT DATA FILE ENTRIES:

WORKR(I,1)
section
area

WORKR(I,2)
0.0

WORKR(I,3)
resistance
coefficient

WORKR(I,4)
section
area

REMARKS:

Turn angle should be from 0 to 90 degrees.

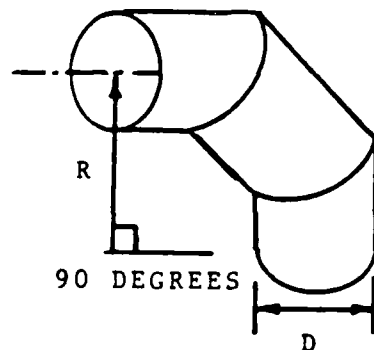
REFERENCE:

ASHRAE FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Segmented round cross section elbow
3, 4, or 5 segments, 90 degree turn

NUMBER:
34

SKETCH:



THREE SEGMENTS SHOWN
(THERE MAY ALSO BE
FOUR OR FIVE SEGMENTS)

INPUT REQUIREMENTS:

1. Number of segments
2. Cross section diameter
3. Radius of the turn measured to the centerline of the turn

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
section	0.0	resistance	section
area		coefficient	area

REMARKS:

Note that the number of segments includes the entry and exit segments.

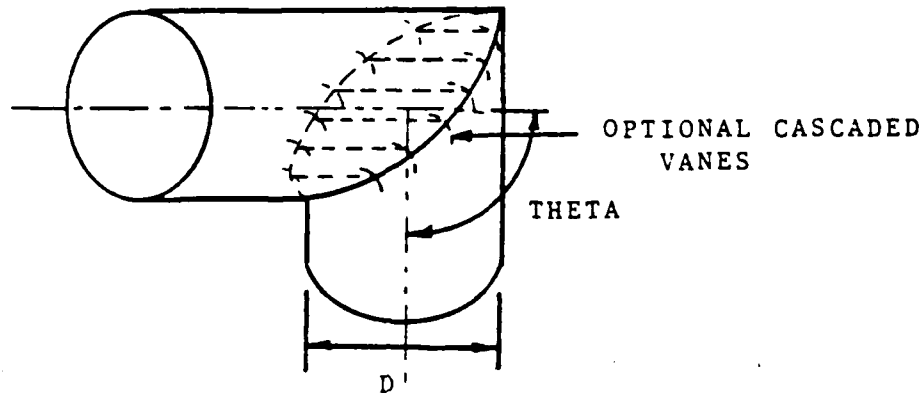
REFERENCE:

ASHRAE FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Mitered round cross section elbow

NUMBER:
05

SKETCH:



INPUT REQUIREMENTS:

1. Cross section diameter
2. Turn angle
3. Whether or not concentric guide vanes are installed

DUCT DATA FILE ENTRIES:

WORKR(I,1)
section
area

WORKR(I,2)
diameter

WORKR(I,3)
resistance
coefficient

WORKR(I,4)
section
area

REMARKS:

A Reynolds number correction is applied to this fitting.

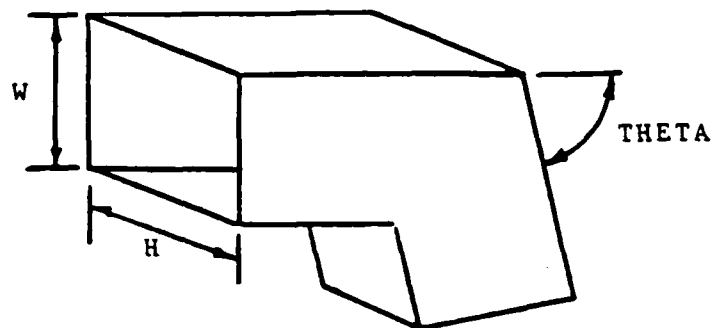
REFERENCE:

ASHRAE FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Mitered rectangular cross section elbow
without turning vanes

NUMBER:
06

SKETCH:



INPUT REQUIREMENTS:

1. Height of the elbow, dimension parallel to turn axis
2. Width of the elbow, dimension in the turn plane
3. Turn angle

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
section	hydraulic	resistance	section
area	diameter	coefficient	area

REMARKS:

This fitting has a Reynolds number correction applied to the resistance coefficient.

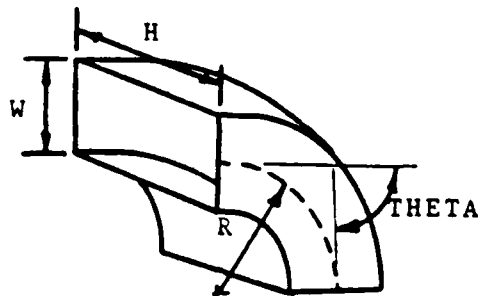
REFERENCE:

ASHRAE FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Smooth radius rectangular elbow without
guide vanes

NUMBER:
07

SKETCH:



INPUT REQUIREMENTS:

1. Height of the elbow, the dimension parallel to the turn axis
2. Width of the elbow, the dimension in the turn plane.
3. Radius of the elbow measured to the centerline of the elbow.
4. Turn angle

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
section	hydraulic	resistance	radius/
area	diameter	coefficient	width

REMARKS:

This fitting has a Reynolds number correction.
The correction also varies with the R/W ratio.

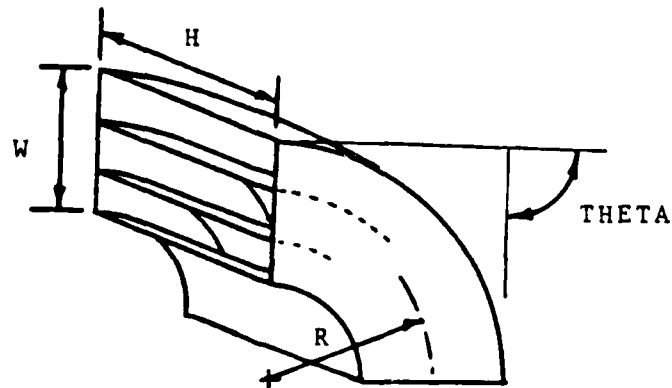
REFERENCE:

ASHRAE FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Smooth radius rectangular elbow with
splitters

NUMBER:
08

SKETCH:



TWO SPLITTERS SHOWN
(THERE MAY ALSO BE
ONE OR THREE)

INPUT REQUIREMENTS:

1. Number of splitters, 1, 2, or 3
2. Height, distance parallel to turn axis
3. Width, distance in turn plane
4. Radius of elbow to section centerline
5. Turn angle

DUCT DATA FILE ENTRIES:

WORKR(I,1)
section
area

WORKR(I,2)
0.0

WORKR(I,3)
resistance
coefficient

WORKR(I,4)
section
area

REMARKS:
None

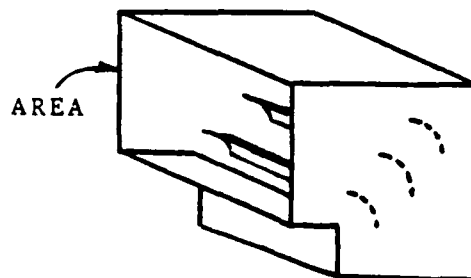
REFERENCE:

ASHRAE FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Mitered rectangular elbow with vanes

NUMBER:
09

SKETCH:



THREE VANES SHOWN

(THERE MAY ALSO BE
ONE OR TWO)

INPUT REQUIREMENTS:

1. Number of vanes (1, 2, or 3)
2. Cross section area

DUCT DATA FILE ENTRIES:

WORKR(I,1)
section
area

WORKR(I,2)
0.0

WORKR(I,3)
resistance
coefficient

WORKR(I,4)
section
area

REMARKS:

Flat plate turning vanes are used.

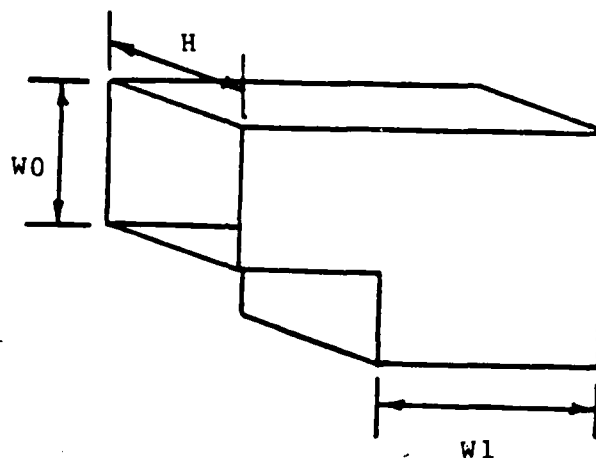
REFERENCE:

ASHRAE FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Rectangular elbow with converging or
diverging flow

NUMBER:
10

SKETCH:



INPUT REQUIREMENTS:

1. Inlet height, dimension parallel to turn axis
2. Exit height, dimension parallel to turn axis
3. Constant width, dimension in turn plane

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
inlet	inlet hyd.	resistance	outlet
area	diameter	coefficient	area

REMARKS:

Elbow should have a 90 deg turn.
The width should remain constant in the elbow.

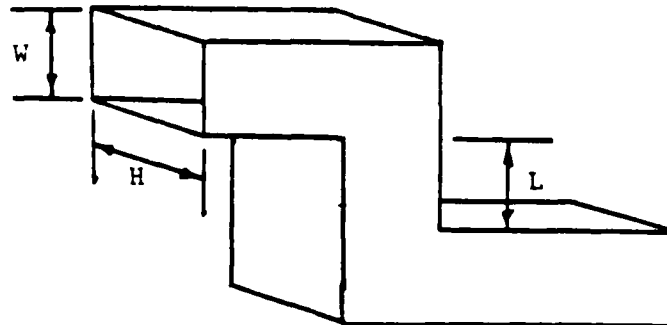
REFERENCE:

ASRHAЕ FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Two 90 degree rectangular elbows in a
Z-shaped configuration

NUMBER:
11

SKETCH:



INPUT REQUIREMENTS:

1. Height of elbows, dimension parallel to turn axis
2. Width of elbows, dimension in turn axis
3. The distance between the centerlines of the offset duct

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
section	hydraulic	resistance	section
area	diameter	coefficient	area

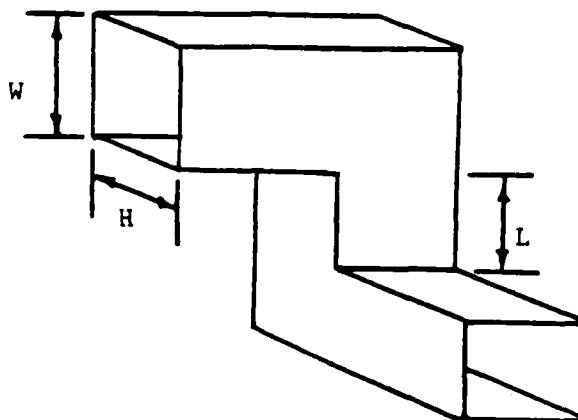
REMARKS:
None.

REFERENCE:
ASHRAE FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Two 90 degree elbows in different planes

NUMBER:
12

SKETCH:



INPUT REQUIREMENTS:

1. Height of elbow, dimension parallel to turn axis
2. Width of elbow, dimension in the plane of the turn
3. Distance between the centerlines of the duct connected to this arrangement

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
section	Hydraulic	resistance	section
area	Diameter	coefficient	area

REMARKS:

Resistance coefficient is a curve fit to the tabulated data.

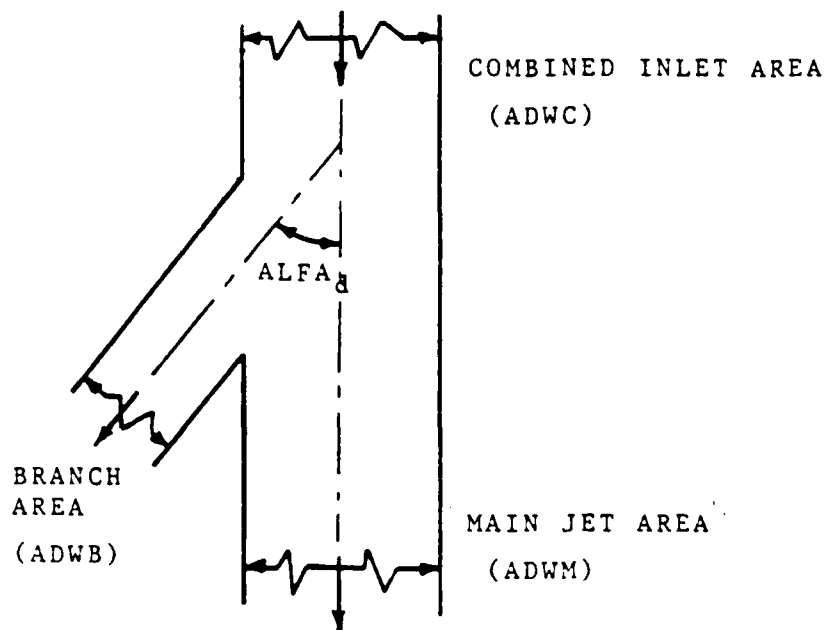
REFERENCE:

ASHRAE FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Diverging wye, branch and main sections

NUMBER:
13 & 14

SKETCH:



INPUT REQUIREMENTS:

- A. Branch section
1. combined area
 2. branch area
 3. divergence angle

- B. Main section
1. main area

DUCT DATA FILE ENTRIES: (fitting 13)

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
combined	branch	divergence	branch
area	area	angle	area

REMARKS:

The divergence angle should follow some centerline streamline. The areas are cross section areas perpendicular to the streamline in the sections away from the dividing location. Cooling air flows through the branch section. Main inlet air to the engine flows through the main section. Both flow through the combined section.

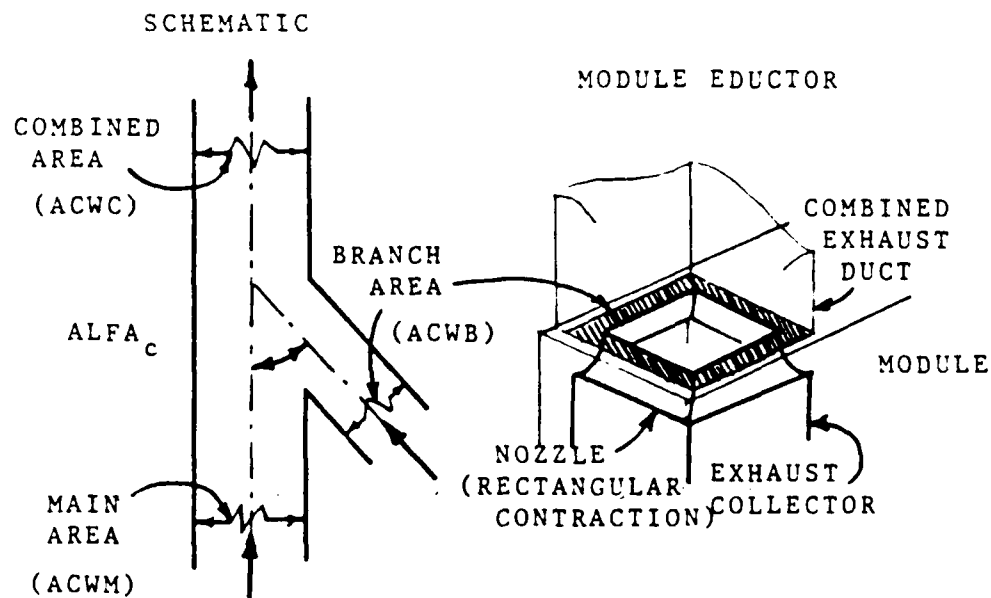
REFERENCE:

Handbook of Hydraulic Resistance, Idel'chik

FITTING NAME:
Convergent wye, branch and main sections

NUMBER:
15 8 16

SKETCH:



INPUT REQUIREMENTS:

A. Branch section

1. branch area
2. combined area
3. convergence angle

B. Main section

1. main area

DUCT DATA FILE ENTRIES: (fitting 15)

WORKR(I,1)
combined
area

WORKR(I,2)
branch
area

WORKR(I,3)
convergence
angle

WORKR(I,4)
branch
area

REMARKS:

The branch area has module cooling air flowing through it. The main area has engine exhaust flowing through it. The combined area has both. The angle should be measured to representative streamlines at the plane where the two flows meet.

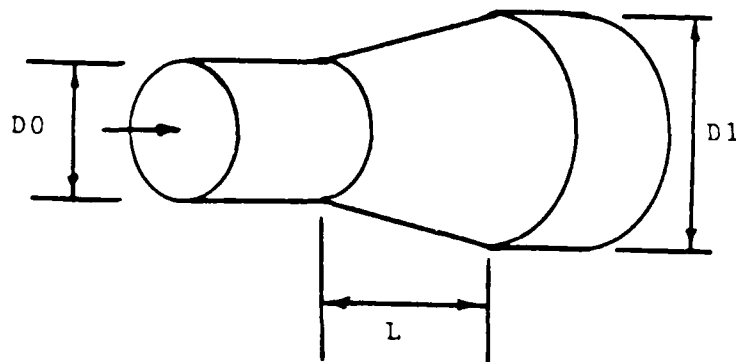
REFERENCE:

Handbook of Hydraulic Resistance, Idel'chik

FITTING NAME:
Conical diffuser

NUMBER:
17

SKETCH:



INPUT REQUIREMENTS:

1. Length of the diffuser
2. Inlet diameter
3. Outlet diameter
4. Is there distorted flow at the inlet
5. Are there dividing wall or baffles installed to reduce resistance

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
inlet	friction	flow	outlet
area	coefficient	coefficient	area

REMARKS:

The program recognizes a wide diverging angle and warns the user. Resistance in this case may be reduced by 35 % with installation of baffles.

REFERENCE:

Handbook of Hydraulic Resistance, Idel'chik

AD-A148 708

AN ANALYTIC MODEL OF GAS TURBINE ENGINE INSTALLATIONS
(U) NAVAL POSTGRADUATE SCHOOL MONTEREY CA S M EZZELL
SEP 84

3/3

UNCLASSIFIED

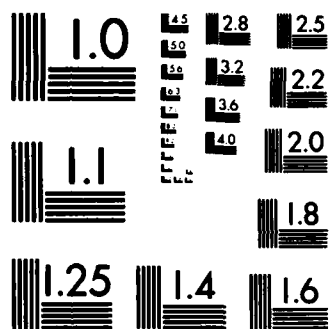
F/G 21/5

NL

END

FILMED

DTIC

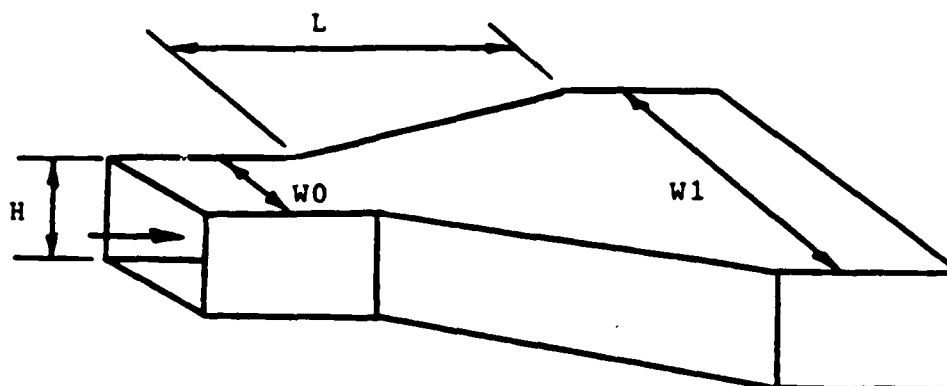


MICROCOPY RESOLUTION TEST CHART
NATIONAL BUREAU OF STANDARDS-1963-A

FITTING NAME:
Flare in-line diffuser

NUMBER:
18

SKETCH:



INPUT REQUIREMENTS:

1. Length of the diffuser
2. The constant height of the diffuser
3. The inlet width
4. The outlet width
5. Distorted flow
6. Installation of baffles

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
inlet	friction	flow	outlet
area	coefficient	coefficient	area

REMARKS:

The divergence is assumed to be uniform with respect to the main centerline. A wide divergence angle is recognized and the user is asked if dividing walls or baffles are installed to reduce resistance.

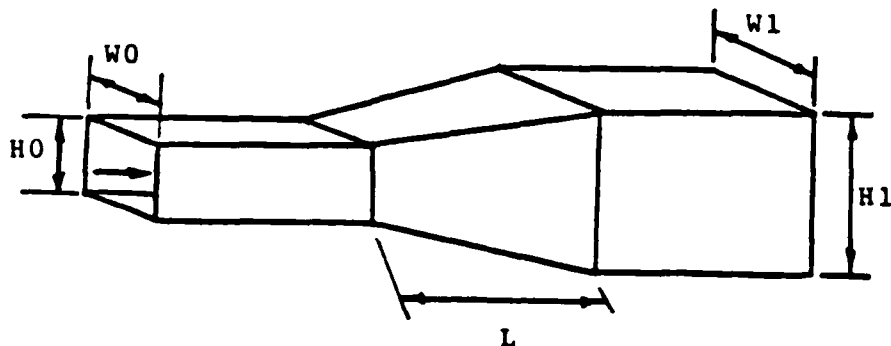
REFERENCE:

Handbook of Hydraulic Resistance, Idel'chik

FITTING NAME:
Pyramidal in-line diffuser

NUMBER:
19

SKETCH:



INPUT REQUIREMENTS:

1. length of the diffuser
2. Smaller inlet dimension, larger inlet dimension
3. Dimensions parallel to inlet dimensions
4. Non-uniform velocity profile
5. Are baffles installed

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
inlet	friction	flow	outlet
area	coefficient	coefficient	area

REMARKS:

A uniform divergence with respect to the centerline is assumed. Wide divergence angle is recognized by the program. With a wide angle the flow resistance can be reduced by 35% with baffles or dividing walls.

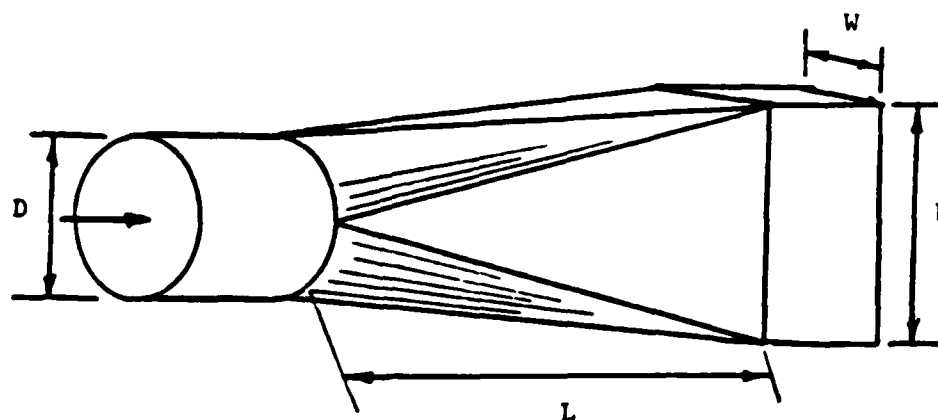
REFERENCE:

Handbook of Hydraulic Resistance, Idel'chik

FITTING NAME:
Transition diffuser, round to rectangular
or rectangular to round

NUMBER:
20

SKETCH:



INPUT REQUIREMENTS:

1. Manner of transition
2. Diameter
3. rectangular dimensions
4. length of the diffuser
5. Non-uniform velocity distribution
6. Installation of baffles or dividing walls

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
inlet	friction	flow	outlet
area	coefficient	coefficient	area

REMARKS:

Uniform divergence with respect to the centerline is assumed. Wide divergence angle is recognized and if baffles or dividing walls are installed the resistance is reduced by 35%.

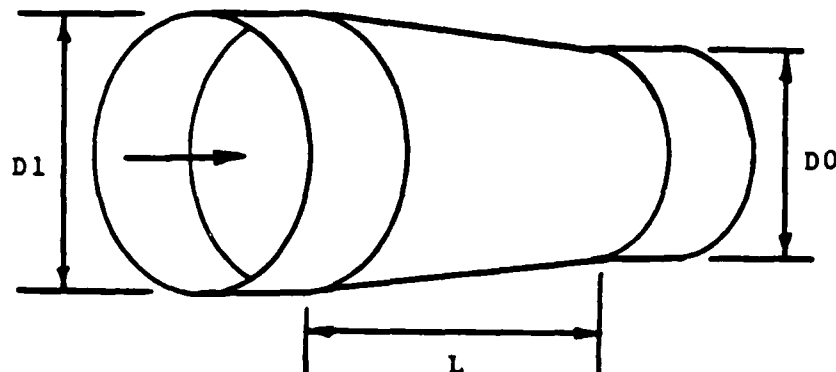
REFERENCE:

Handbook of Hydraulic Resistance, Idel'chik

FITTING NAME:
Circular contraction

NUMBER:
21

SKETCH:



INPUT REQUIREMENTS:

1. Length of the contraction
2. Upstream diameter
3. Downstream diameter

DUCT DATA FILE ENTRIES:

WORKR(I,1)
outlet
area

WORKR(I,2)
0.0

WORKR(I,3)
resistance
coefficient

WORKR(I,4)
inlet
area

REMARKS:

If you need a transitional contraction you could use this fitting or fitting 22. The area of the inlet or outlet would have to be converted to a circle or rectangle as required by the geometry for input to the program.

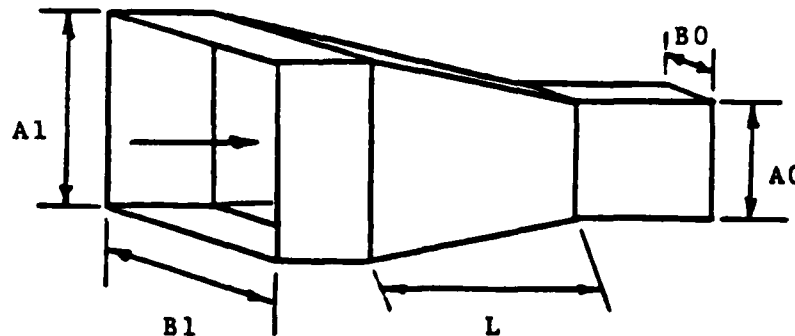
REFERENCE:

ASHRAE FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Rectangular contraction

NUMBER:
22

SKETCH:



INPUT REQUIREMENTS:

1. Length of the contraction
2. Upstream dimensions
3. Downstream dimensions

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
outlet	0.0	resistance	inlet
area		coefficient	area

REMARKS:

This fitting can be substituted for a transitional contraction. The inlet or outlet area should remain the same and the area for transition converted as required.

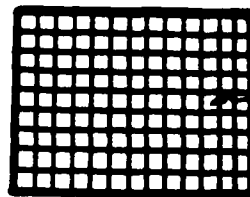
REFERENCE:

ASHRAE FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Screen

NUMBER:
23

SKETCH:



SCREEN AREA
(FREE FLOW MEANS HOLE SPACES)

DUCT AREA (OVERALL AREA)

INPUT REQUIREMENTS:

1. Overall duct cross section area
2. Screen free flow area

DUCT DATA FILE ENTRIES:

WORKR(I,1)
duct
area

WORKR(I,2)
0.0

WORKR(I,3)
resistance
coefficient

WORKR(I,4)
duct
area

REMARKS:

This fitting is useful for the screen in front of the engine inlet. The free flow area is the sum of all the holes in the screen.

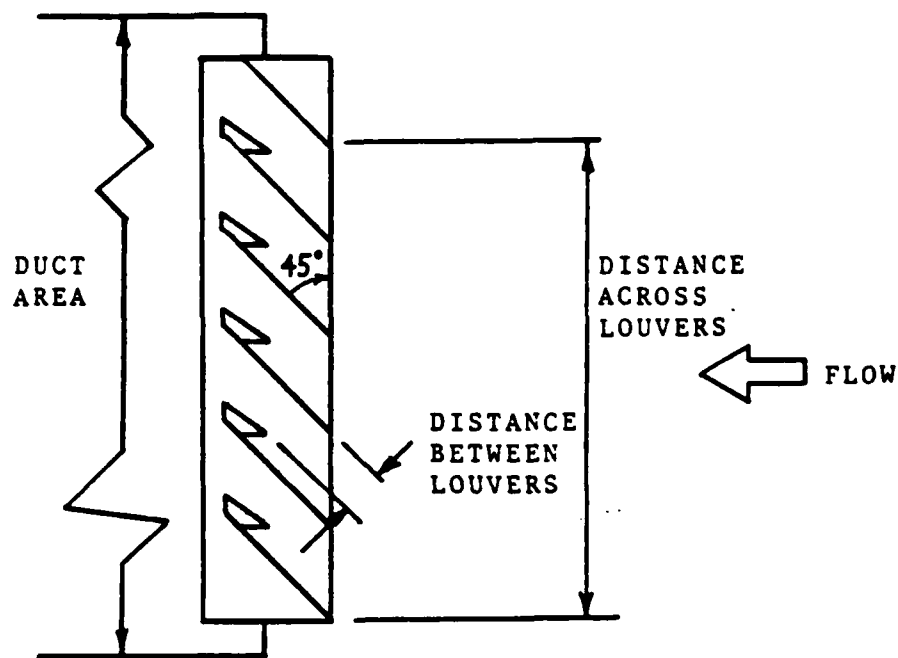
REFERENCE:

ASHRAE FUNDAMENTALS 1981, chapter 33

FITTING NAME:
Louver inlet

NUMBER:
24

SKETCH:



INPUT REQUIREMENTS:

1. Distance across the louver openings
2. Distance between the louvers
3. The number of openings between the louvers
4. Duct area just inside the louvers

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
duct	0.0	resistance	duct
area		coefficient	area

REMARKS:

The correlation is for flat louvers with the front edges flat with the face of the louvers. No friction is included in this correlation. Better models need to be developed for louvers with moisture separator edges. The louver angle is 45 degrees to the face.

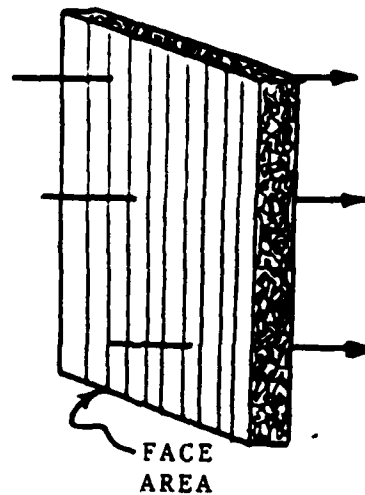
REFERENCE:

Handbook of Hydraulic Resistance, Idel'chik

FITTING NAME:
Filter

NUMBER:
25

SKETCH:



INPUT REQUIREMENTS:

None if the default value is used.
If another filter type is to be used then the user should provide pressure loss data as a function of face velocity. Only a few points are required for the power curve fit to work. The number of points is an input (two min.)

DJCI DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
filter face	multiplier	exponent	filter face
area	(A)	(B)	area

REMARKS:

The power curve fit is of the form:

$$\text{delta pressure (in WG)} = A * (\text{velocity}) ** B$$

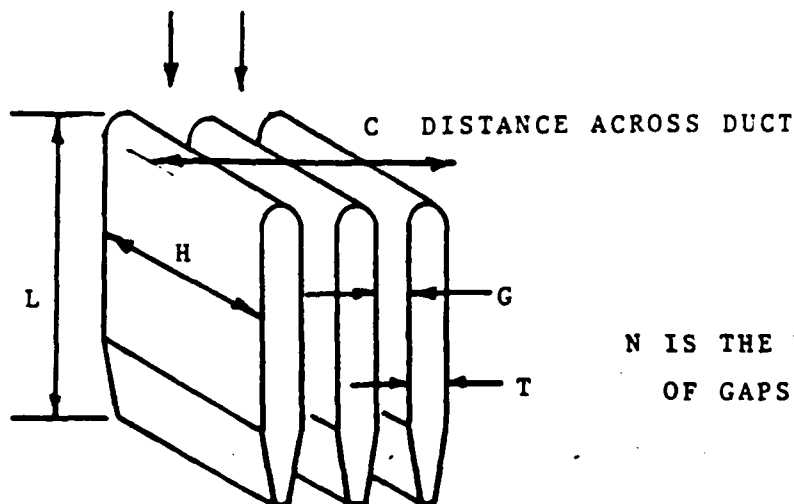
REFERENCE:

Filter manufacturer's data

FITTING NAME:
Multi-baffle type silencer

NUMBER:
26

SKETCH:



INPUT REQUIREMENTS:

1. Gap between baffles
2. Baffle thickness
3. Baffle length (with flow)
4. Duct dimension parallel to gap
5. Duct dimension across gaps
6. The number of gaps

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
duct	0.0	resistance	duct
area		coefficient	area

REMARKS:

This is a composite model. The resistance coefficient is modeled as a sudden contraction, friction along the length of the baffle, and a sudden expansion. It is not a very good model and a model based on experimental data would be better.

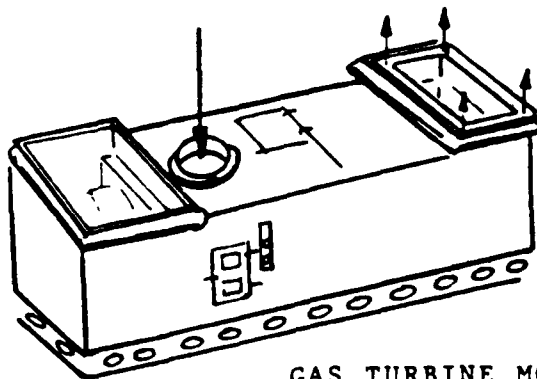
REFERENCE:

NAVSEA Inlet Design Handbook for Marine Gas turbines

FITTING NAME:
Gas turbine module

NUMBER:
27

SKETCH:



GAS TURBINE MODULE

** COOLING AIR PASSAGES ONLY **

INPUT REQUIREMENTS:
None

DUCT DATA FILE ENTRIES:
WORKR(I,1) WORKR(I,2) WORKR(I,3) WORKR(I,4)
1.0 1.0 1.0 1.0

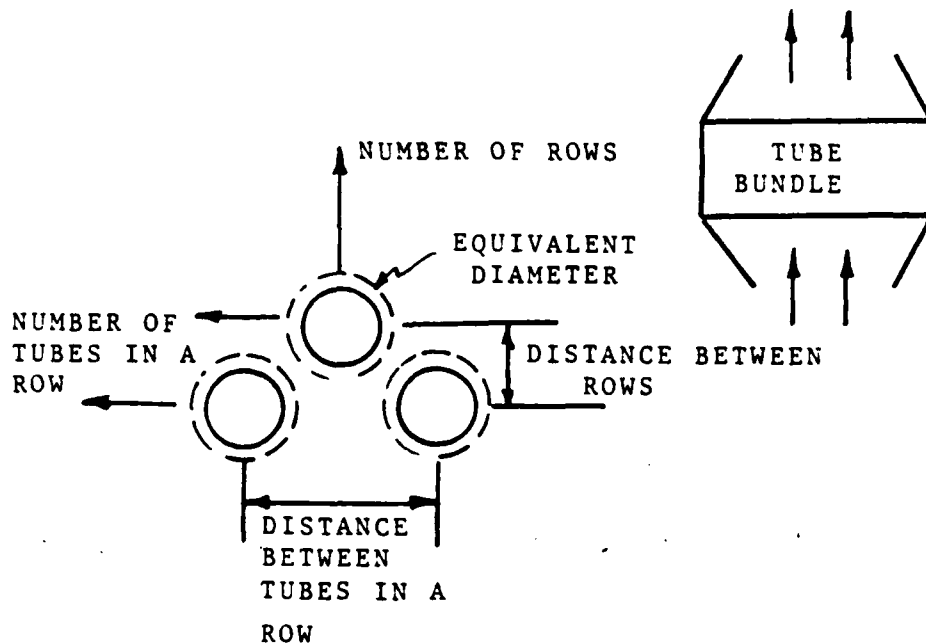
REMARKS:
This model is based on the mass flow rate of cooling air through the module. It is a power fit to data from General Electric Co. It should be good as long as entry and exit areas remain about the same. The 1.0's in the duct data file are there to prevent division by zero in the program and are not actually used.

REFERENCE:
Manufacturer's data

FITTING NAME:
Waste heat recovery boiler

NUMBER:
28

SKETCH:



INPUT REQUIREMENTS:

A default is available. It is based on current design considerations in the racer program. Should you choose not to use it several inputs are required. Read the reference and be prepared to enter the values shown on the sketch and a few you will have to compute yourself (i.e. tube equiv. dia. and hydraulic dia.).

DUCT DATA FILE ENTRIES:

WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
duct	0.0	resistance	duct
area		coefficient	area

REMARKS:

If the manufacturer will provide the data, write your own model, but this should be close for preliminary studies.

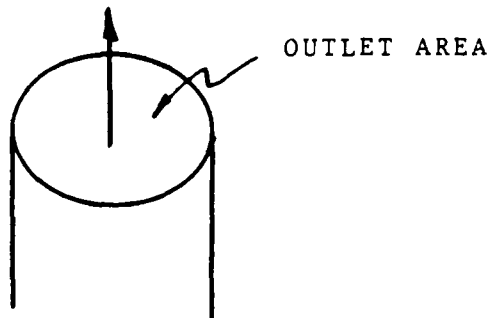
REFERENCE:

Extended Surface Heat Transfer, Kerns and Krauss
pages 582-589

FITTING NAME:
Abrupt Exit

NUMBER:
29

SKETCH:



INPUT REQUIREMENTS:
1. The exit area

DUCT DATA FILE ENTRIES:			
WORKR(I,1)	WORKR(I,2)	WORKR(I,3)	WORKR(I,4)
exit	0.0	1.0	exit
area			area

REMARKS:
All velocity energy is assumed lost after exiting the duct, hence a coefficient of 1.0.

REFERENCE:
ASHRAE FUNDAMENTALS 1981, chapter 33

C. EXECUTING THE PROGRAM

1. IBM 3033 at NPS

Issue the following commands to compile and execute the program.

FCRTHX filename

GLOEAL TXTLIB FORTMOD2 MOD2EEH NONIMSL

ICAD filename (START

"filename" is the name of the program in the user's files. NONIMSL is required because the program calls the NONIMSL library with FRTCMS when defining files and clearing the CRT screen. If the file has been compiled on the user's disk the lengthy compiling may be omitted and issue just the last two lines.

2. VAX-11 at NPS

The program version to be used is 1.1. This version is a modified version of the program listed in Appendix A. The modifications include elimination of all calls to FRTCMS. FRTCMS is used for two purposes in version 1.0. First to set up file definitions and second to clear the screen at appropriate times to prevent the format of the display from being chopped up. The file definitions in version 1.1 are set up using the standard OPEN statement of FORTRAN 77 used on the VAX-11/780 at NPS. All calls to FRTCMS to clear the screen were deleted and are not needed on the VAX because it scrolls the display from the bottom and does not cut off any continuous screen displays. One other change was made in the file definition area, all writes to the terminal were made to unit 5 and all reads from the terminal were made from unit 6. This agrees with the convention of FORTRAN 77 as implemented on the VAX. The program runs like any other program on the VAX, first the program must be compiled using the fortran compiler, then

linked and run. The program is still interactive on the VAX and about the only word of caution required is to remember to use CAPS ON or upper case input for logical replies. Using lower case leaves the user in a loop where the program keeps asking for for a correct reply. The duct geometry file information is on a file called duct.dat and the performance information is on a file called output.dat.

D. BUILDING A DUCT DATA FILE

The following pages are a recorded session at the terminal where the author entered a system in to the program. The system modeled is made up from drawings for the proposed Arleigh Burke class guided missile destroyer. The session has been annotated to point out features of the program.

GLOBAL TITLE CHSLIP FORTMOD2 MOD2EEN IMSLSP NONIMS2
 LOAD #HESSIS (START
 EXECUTION BEGINS...
 A ONE-DIMENSIONAL MODEL FOR THE SYSTEM PERFORMANCE
 OF A MARINE GAS TURBINE INSTALLATION

BY LCDR. STEPHEN M. EZZELL

OPTIONS: VERSION 1.0 MARCH 30, 1984
 BUILD A DATA FILE REPRESENTING THE DUCT SYSTEM
 EDIT OR CHANGE THE DUCT DATA FILE
 COMPUTE SYSTEM PERFORMANCE
 METHOD: INTERACTIVE INPUT OF DATA, BRANCHING TO DESIRED
 OPTION BY ANSWERING QUESTIONS

*** WARNING, TWO NULL ENTRIES ON NUMERICAL INPUT WILL ***
 *** KILL THE PROGRAM. ***

FIRST QUESTION:
 DO YOU HAVE A DATA FILE OF DUCT FITTINGS (Y/N)?

ⁿ DO YOU WANT LONG OR SHORT INSTRUCTIONS (L/S)?

^l YOU HAVE SELECTED THE LONG INSTRUCTIONS.
 ARE YOU WORKING ON A CRT OR TYPEWRITER TERMINAL (C/T)?

^c YOU ARE WORKING ON A CRT TERMINAL.
 DOES THE MODULE COOLING AIR BRANCH OFF THE MAIN INLET?
 (Y,N)

^y DOES THE MODULE COOLING AIR JOIN THE MAIN ENGINE EXHAUST?
 (Y,N)

^y IS THERE A COOLING FAN INSTALLED?

NOTE INCORRECT RESPONSE,
 ANSWER SHOULD HAVE BEEN
 Y OR N

^t YOU MUST ENTER A LETTER IN THE BRACKETS.
 IS THERE A COOLING FAN INSTALLED?

^y SYSTEM IS CLASS THREE, COMBINED INLETS AND EXHAUST
 FLOWS FOR THE ENGINE AND MODULE COOLING. A COOLING FAN IS
 INSTALLED. YOU WILL BE ENTERING FITTINGS FOR SIX BRANCHES.

1. COMBINED INLET TO THE COMBINED SECTION
 OF A DIVERGENT WYE.
2. MAIN SECTION OF THE DIVERGENT WYE TO THE ENGINE.
3. BRANCH SECTION OF THE DIVERGENT WYE TO THE COOLING FAN.
4. ENGINE EXHAUST TO MAIN SECTION OF A CONVERGENT WYE.
 AN INDUCTOR INSTALLED AT THE EXHAUST PLANE OF THE ENGINE
 IS CONSIDERED TO BE A CONTRACTION FOLLOWED BY THE MAIN
 SECTION OF A CONVERGING WYE FOR THE PURPOSES OF THIS
 PROGRAM.
5. COOLING FAN EXHAUST TO THE BRANCH SECTION
 OF A CONVERGENT WYE.
6. COMBINED SECTION OF A CONVERGENT WYE TO THE ATMOSPHERE.

ENTER ZERO TO CONTINUE

?

0

MENU LOOKS LIKE THIS
 IT APPEARS WITH EACH FITTING
 BUT IS OMITTED IN THIS LISTING TO
 CONSERVE SPACE

```

00 NO ICE FITTINGS THIS BRANCH
01 INTAKE SHAFT, RECT SECTION, SIDE
02 ORIFACES, WITH (OUT) LOUVERS
02 STRAIGHT DUCT
03 ELBOW, SMOOTH RADIUS ROUND
04 ELBOW, 90 DEG, 3 & 5 PCS, ROUND
05 ELBOW, MITERED, ROUND, W&W/O VANES
06 ELBOW, MITERED, RECTANGULAR
07 ELBOW, SMOOTH RADIUS, RECTANGULAR
08 ELBOW, SMOOTH RADIUS, WITH
   SPLITTERS, RECTANGULAR
09 ELBOW, MITERED WITH VANES, RECT
10 ELBOW, CONVERGING OR DIVERGING
   FLOW, RECTANGULAR
11 ELBOWS, 90 DEG, Z-SHAPED, RECT
12 ELBOWS, 90 DEG, IN DIFFERENT
   PLANES, RECTANGULAR
13 DIVERGING WYE, BRANCH SECTION
    *****USE TWO DIGIT NUMBER, PRESS ENTER*****
>> YOU ARE WORKING ON FITTING NUMBER >> 312201

25
YOU HAVE SELECTED THE INLET FILTER.
**FIRST QUESTION, WHAT IS THE TOTAL FACE AREA OF THE FILTER?
?
3
DO YOU WANT TO USE THE DD963 TYPE FILTER
  IN THE DRY CONDITION (Y/N)?
Y
NO MORE QUESTIONS.
DO YOU WANT TO ENTER THIS FITTING (Y/N)?
N
>> YOU ARE WORKING ON FITTING NUMBER >> 312201
24
<-- FITTING SELECTED
YOU HAVE SELECTED A LOUVERED ENTRANCE.
**FIRST QUESTION, WHAT IS THE DISTANCE ACROSS THE
  LOUVER OPENINGS?
?
25.5
WHAT IS THE DISTANCE BETWEEN THE LOUVERS, USE THE
  CLOSEST DISTANCE.
?
0.4021
HOW MANY OPENINGS ARE THERE BETWEEN THE LOUVERS?
?
17
LAST QUESTION, WHAT IS THE AREA OF THE DUCT
  JUST INSIDE THE LOUVER ENTRANCE?
?
197.75
DO YOU WANT TO ENTER THIS FITTING (Y/N)?
Y
>> YOU ARE WORKING ON FITTING NUMBER >> 312202

25
YOU HAVE SELECTED THE INLET FILTER.
**FIRST QUESTION, WHAT IS THE TOTAL FACE AREA OF THE FILTER?
?
197.75
DO YOU WANT TO USE THE DD963 TYPE FILTER IN
  THE DRY CONDITION (Y/N)?
Y
NO MORE QUESTIONS.
DO YOU WANT TO ENTER THIS FITTING (Y/N)?
Y
>> YOU ARE WORKING ON FITTING NUMBER >> 312203
  
```

MENU OMITTED

?
02 YOU HAVE SELECTED STRAIGHT DUCT. IT MAY BE ROUND
OR RECTANGULAR.
***FIRST QUESTION, IS THE DUCT CIRCULAR OR RECTANGULAR
(C/R)?
1 THE DUCT IS RECTANGULAR, ENTER FIRST CROSS-SECTIONAL
DIMENSION. (FEET)
18.83
2 SECOND DIMENSION (FEET)
10.5
3 ENTER THE LENGTH OF THIS DUCT SECTION. (FEET)
17.75
4 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
Y
5 >> YOU ARE WORKING ON FITTING NUMBER >> 312204
?
00
6 >> YOU ARE WORKING ON FITTING NUMBER >> 323101
?
14 YOU HAVE SELECTED THE MAIN SECTION OF A DIVERGING WYE.
THE AIR TO THE ENGINE SHOULD BE FLOWING THROUGH THIS SECTION.
JUST ONE QUESTION, WHAT IS THE CROSS-SECTIONAL AREA OF THE
MAIN SECTION? THIS SHOULD BE THE AREA JUST DOWNSTREAM OF THE
JUNCTION AND DIRECTS FLOW TO THE ENGINE. IT ALSO SHOULD BE
THE FIRST FITTING OF THE BRANCH.
81.375
7 >> YOU ARE WORKING ON FITTING NUMBER >> 323102
?
26 YOU HAVE SELECTED A MULTI-BAFFLE TYPE SILENCER.
EACH BAFFLE HAS A STREAMLINED SHAPE. IT IS THE TYPE
USED IN THE INLETS OF THE DD963.
**FIRST QUESTION, WHAT IS THE GAP BETWEEN THE BAFFLES?
0.333
8 WHAT IS THE THICKNESS OF THE BAFFLES?
0.666
9 WHAT IS THE LENGTH OF THE BAFFLES?
9.33
10 WHAT IS THE DIMENSION OF THE BAFFLES PARALLEL TO THE GAP?
7.75
11 WHAT IS THE DIMENSION OF THE MAIN DUCT ACROSS THE GAPS?
10.5
12 LAST QUESTION, HOW MANY GAPS ARE THERE?
1
13 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
Y
14 >> YOU ARE WORKING ON FITTING NUMBER >> 323103
?
22 YOU HAVE SELECTED A RECTANGULAR CONTRACTION.
**FIRST QUESTION, WHAT IS THE LENGTH OF THE CONTRACTION?
8.5
15 WHAT IS THE LEAST UPSTREAM CROSS-SECTION DIMENSION?

?
 7.75
 ? WHAT IS THE GREATER UPSTREAM CROSS-SECTION DIMENSION?
 ?
 10.5
 ? WHAT IS THE LEAST DOWNSTREAM CROSS-SECTION DIMENSION?
 ?
 6.807
 ? LAST QUESTION, WHAT IS THE GREATER DOWNSTREAM
 ? CROSS-SECTION DIMENSION?
 ?
 7.75
 ? DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 ? >> YOU ARE WORKING ON FITTING NUMBER >> 323104
 ?
 06
 ? YOU HAVE SELECTED A MITERED, RECTANGULAR CROSS-SECTION, ELBOW.
 ? **FIRST QUESTION, WHAT IS THE HEIGHT OF THE ELBOW?
 ? (THE DIMENSION PARALLEL TO THE TURN AXIS)
 ?
 6.67
 ? WHAT IS THE WIDTH OF THE ELBOW CROSS-SECTION?
 ? (THE DIMENSION IN THE PLANE OF THE TURN)
 ?
 7.75
 ? LAST QUESTION, WHAT IS THE ANGLE OF THE ELBOW TURN
 ? (0 - 90 DEGREES)?
 ?
 90
 ? DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 ? >> YOU ARE WORKING ON FITTING NUMBER >> 323105
 ?
 23
 ? YOU HAVE SELECTED A SCREEN OBSTRUCTION IN THE DUCT.
 ? **FIRST QUESTION, WHAT IS THE DUCT CROSS-SECTIONAL AREA?
 ?
 50
 ? LAST QUESTION, WHAT IS THE FREE FLOW AREA OF THE SCREEN?
 ?
 27.15
 ? DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 ? >> YOU ARE WORKING ON FITTING NUMBER >> 323106
 ?
 00
 ? >> YOU ARE WORKING ON FITTING NUMBER >> 324001
 ?
 13
 ? YOU HAVE SELECTED THE BRANCH SECTION OF A DIVERGENT WYE.
 ? THE MODULE COOLING AIR SHOULD BE BRANCHING OFF THE MAIN
 ? INLET AND FLOWING THROUGH THIS SECTION. THIS SHOULD BE THE
 ? FIRST FITTING OF THIS BRANCH.
 ? **FIRST QUESTION, WHAT IS THE ANGLE BETWEEN THE MAIN FLOW
 ? AXIS AND THE BRANCH FLOW AXIS (DEGREES)?
 ?
 90
 ? WHAT IS THE CROSS-SECTIONAL AREA OF THE COMBINED FLOW
 ? SECTION? THIS IS WHERE BOTH ENGINE AIR AND COOLING AIR FLOW
 ? JUST UPSTREAM OF THE BRANCH.
 ?
 197.75
 ? LAST QUESTION, WHAT IS THE CROSS-SECTIONAL AREA OF THE BRANCH?
 ?
 5.761
 ? DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y

2 >> YOU ARE WORKING ON FITTING NUMBER >> 324002
 02 YOU HAVE SELECTED STRAIGHT DUCT. IT MAY BE ROUND
 OR RECTANGULAR.
 ***FIRST QUESTION, IS THE DUCT CIRCULAR OR RECTANGULAR (C/R) ?
 C THE DUCT IS CIRCULAR, ENTER THE DIAMETER (FEET)
 2.708
 ENTER THE LENGTH OF THIS DUCT SECTION. (FEET)
 7.5
 DO YOU WANT TO ENTER THIS FITTING (Y/N) ?
 Y >> YOU ARE WORKING ON FITTING NUMBER >> 324003
 00
 >> YOU ARE WORKING ON FITTING NUMBER >> 335101
 02
 YOU HAVE SELECTED STRAIGHT DUCT. IT MAY BE ROUND OR RECTANGULAR.
 ***FIRST QUESTION, IS THE DUCT CIRCULAR OR RECTANGULAR (C/R) ?
 R THE DUCT IS RECTANGULAR, ENTER FIRST CROSS-SECTIONAL DIMENSION. (FEET)
 6.64
 SECCND DIMENSION (FEET)
 4.58
 ENTER THE LENGTH OF THIS DUCT SECTION. (FEET)
 1
 DO YOU WANT TO ENTER THIS FITTING (Y/N) ?
 Y >> YOU ARE WORKING ON FITTING NUMBER >> 335102
 16
 YOU HAVE SELECTED THE MAIN SECTION OF A CONVERGING
 WYE. THE ENGINE EXHAUST ALONE SHOULD BE FLOWING THROUGH
 THIS SECTION. IT SHOULD BE THE LAST FITTING OF THE BRANCH.
 **JUST ONE QUESTION, WHAT IS THE CROSS-SECTIONAL AREA OF THE
 MAIN BRANCH?
 20.19
 DO YOU WANT TO ENTER THIS FITTING (Y/N) ?
 Y >> YOU ARE WORKING ON FITTING NUMBER >> 335103
 00
 >> YOU ARE WORKING ON FITTING NUMBER >> 345001
 27
 YOU HAVE SELECTED THE GAS TURBINE MODULE AS A PART OF
 THE COOLING FLOW PASSAGE. NO QUESTIONS, JUST NEEDED
 TO KNOW WHERE YOU WANTED THE MODULE.
 DO YOU WANT TO ENTER THIS FITTING (Y/N) ?
 Y >> YOU ARE WORKING ON FITTING NUMBER >> 345002
 15
 YOU HAVE SELECTED THE BRANCH SECTION OF A CONVERGENT
 WYE. THE HOT MODULE COOLING AIR SHOULD BE JOINING THE MAIN
 ENGINE EXHAUST IN THIS WYE. THIS FITTING SHOULD BE THE LAST
 FITTING IN THE BRANCH.
 **FIRST QUESTION, WHAT IS THE ANGLE BETWEEN THE MAIN FLOW
 AXIS AND THE BRANCH AXIS (DEGREES) ?
 2

0
 WHAT IS THE CROSS-SECTIONAL AREA OF THE COMBINED FLOW
 SECTION? THIS IS WHERE ENGINE EXHAUST AND MODULE COOLING AIR
 FLOW JUST DOWNSTREAM OF THE BRANCH.
 30.46
 LAST QUESTION, WHAT IS THE CROSS-SECTIONAL AREA OF THE
 BRANCH?
 10.27
 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 >> YOU ARE WORKING ON FITTING NUMBER >> 345003
 00
 >> YOU ARE WORKING ON FITTING NUMBER >> 356201
 21
 YOU HAVE SELECTED A CIRCULAR CONTRACTION.
 **FIRST QUESTION, WHAT IS THE LENGTH OF THE CONTRACTION?
 9
 WHAT IS THE UPSTREAM DIAMETER?
 6.2374
 WHAT IS THE DOWNSTREAM DIAMETER?
 5.4667
 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 >> YOU ARE WORKING ON FITTING NUMBER >> 356202
 02
 YOU HAVE SELECTED STRAIGHT DUCT. IT MAY BE ROUND OR RECTANGULAR.
 ***FIRST QUESTION, IS THE DUCT CIRCULAR OR RECTANGULAR (C/R) ?
 C
 THE DUCT IS CIRCULAR, ENTER THE DIAMETER (FEET)
 5.4667
 ENTER THE LENGTH OF THIS DUCT SECTION. (FEET)
 7.11
 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 >> YOU ARE WORKING ON FITTING NUMBER >> 356203
 05
 YOU HAVE SELECTED A MITERED ROUND ELBOW.
 **FIRST QUESTION, WHAT IS THE CROSS-SECTIONAL DIAMETER?
 5.4667
 WHAT IS THE ANGLE OF THE ELBOW TURN?
 90
 LAST QUESTION, ARE OPTIMUM NUMBER OF CONCENTRIC VANES
 INSTALLED TO REDUCE RESISTANCE AND TURBULANCE (Y/N)?
 Y
 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 >> YOU ARE WORKING ON FITTING NUMBER >> 356204
 02
 YOU HAVE SELECTED STRAIGHT DUCT. IT MAY BE ROUND OR RECTANGULAR.
 ***FIRST QUESTION, IS THE DUCT CIRCULAR OR RECTANGULAR (C/R) ?
 C
 THE DUCT IS CIRCULAR, ENTER THE DIAMETER (FEET)
 ?

5.5667
 ENTER THE LENGTH OF THIS DUCT SECTION. (FEET)
 6.23
 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 >> YOU ARE WORKING ON FITTING NUMBER >> 356205
 05
 YOU HAVE SELECTED A MITERED ROUND ELBOW.
 **FIRST QUESTION, WHAT IS THE CROSS-SECTIONAL DIAMETER?
 5.4667
 WHAT IS THE ANGLE OF THE ELBOW TURN?
 90
 LAST QUESTION, ARE OPTIMUM NUMBER OF CONCENTRIC VANES
 INSTALLED TO REDUCE RESISTANCE AND TURBULANCE (Y/N)?
 Y
 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 >> YOU ARE WORKING ON FITTING NUMBER >> 356206
 02
 YOU HAVE SELECTED STRAIGHT DUCT. IT MAY BE ROUND OR RECTANGULAR.
 ***FIRST QUESTION, IS THE DUCT CIRCULAR OR RECTANGULAR (C/R)?
 C
 THE DUCT IS CIRCULAR, ENTER THE DIAMETER (FEET)
 5.4667
 ENTER THE LENGTH OF THIS DUCT SECTION. (FEET)
 3.033
 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 >> YOU ARE WORKING ON FITTING NUMBER >> 356207
 17
 YOU HAVE SELECTED A CONICAL DIFFUSER WITH CIRCULAR
 INLET AND OUTLET SECTIONS.
 **FIRST QUESTION, WHAT IS THE LENGTH OF THE DIFFUSER?
 2.967
 WHAT IS THE INLET DIAMETER?
 5.4667
 WHAT IS THE OUTLET DIAMETER?
 7.1667
 IS THERE A NON-UNIFORM VELOCITY DISTRIBUTION AT THE INLET (Y/N)?
 N
 SINCE THERE IS A WIDE DIVERGING ANGLE, THE PROPER
 INSTALLATION OF DIVIDING WALLS OR BAFFLES CAN REDUCE
 THE RESISTANCE OF THIS FITTING. DO YOU WANT TO INSTALL
 DIVIDING WALLS OR BAFFLES (Y/N)?
 N
 NO MORE QUESTIONS THIS FITTING.
 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 >> YOU ARE WORKING ON FITTING NUMBER >> 356208
 02
 YOU HAVE SELECTED STRAIGHT DUCT. IT MAY BE ROUND OR RECTANGULAR.
 ***FIRST QUESTION, IS THE DUCT CIRCULAR OR RECTANGULAR (C/R)?
 C
 THE DUCT IS CIRCULAR, ENTER THE DIAMETER (FEET)
 ?

7.1667
 ENTER THE LENGTH OF THIS DUCT SECTION. (FEET)
 ?
 1.7
 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 >> YOU ARE WORKING ON FITTING NUMBER >> 356209
 ?
 2.1
 YOU HAVE SELECTED A CIRCULAR CONTRACTION.
 **FIRST QUESTION, WHAT IS THE LENGTH OF THE CONTRACTION?
 ?
 0.1
 WHAT IS THE UPSTREAM DIAMETER?
 ?
 7.1667
 WHAT IS THE DOWNSTREAM DIAMETER?
 ?
 4.533
 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 >> YOU ARE WORKING ON FITTING NUMBER >> 356210
 ?
 2.9
 YOU HAVE SELECTED AN ABRUPT EXIT TO THE ATMOSPHERE.
 **JUST ONE QUESTION, WHAT IS THE AREA OF THE EXIT PLANE?
 ?
 16.1384
 DO YOU WANT TO ENTER THIS FITTING (Y/N)?
 Y
 >> YOU ARE WORKING ON FITTING NUMBER >> 356211
 ?
 00
 WHAT SERIAL NUMBER WOULD YOU LIKE TO GIVE THIS DUCT DATA FILE?
 YOU MAY USE UP TO A SIX DIGIT INTEGER NUMBER.
 ?
 510001
 DO YOU WANT TO COMPUTE WITH THE FILE OR QUIT (C/Q)?
 ?
 9

E. EDITING THE DUCT DATA FILE

This section demonstrates the editing capability of the program. The editor will be demonstrated by changing a fitting. The fitting chosen is an elbow in the exhaust duct. It has cascaded turning vanes installed. By using the editor the turning vanes will be removed and an ordinary mitered round elbow will be substituted. Any fitting that also serves the purpose could be substituted as well.

The program can also add or delete a fitting. It is somewhat limited in the addition ability. The program can not add a fitting to the first of a branch in one step. To add a fitting to the duct data file select the index of the fitting in the file that the fitting is to be placed after. The program will ask what fitting is to be added and then the user can enter the fitting directly or from the menu. To add a fitting at the first of a branch, first add the same first fitting presently in the branch after itself, then change the same index fitting as the first step to the desired new first fitting.

It should be emphasized that the editor does not change a system class. If the user wants a different duct arrangement a new file will have to be entered.

GLOBAL ENTRY IS CMSLIB FORTMOD2 MOD2EEN INSLSP NONINSL
 LOCAL THESS:USL START
 EXECUTION BEGINS
 A ONE-DIMENSIONAL MODEL FOR THE SYSTEM PERFORMANCE
 OF A MARINE GAS TURBINE INSTALLATION

BY LCDR. STEPHEN M. EZZELL

VERSION 1.0 MARCH 30, 1984
 OPTIONS: BUILD A DATA FILE REPRESENTING THE DUCT SYSTEM
 EDIT OR CHANGE THE DUCT DATA FILE
 COMPUTE SYSTEM PERFORMANCE
 METHOD: INTERACTIVE INPUT OF DATA BRANCHING TO DESIRED
 OPTION BY ANSWERING QUESTIONS

*** WARNING, TWO NULL ENTRIES ON NUMERICAL INPUT WILL ***
 *** KILL THE PROGRAM. ***

FIRST QUESTION:

DO YOU HAVE A DATA FILE OF DUCT FITTINGS (Y/N)?

Y

DO YOU WANT TO EDIT THE FILE OR USE IT FOR COMPUTATION (E/C)?

E

DO YOU WANT TO CHANGE, DELETE, OR ADD (C/D/A)?
 YOUR OLD FILE WILL BE PERMANENTLY CHANGED, DO YOU
 COPY THE OLD FILE UNDER A NEW NAME IF YOU WANTED TO
 SAVE IT? IF NOT, ENTER TWO NULL STRINGS TO KILL THE
 PROGRAM.

C

WHAT LINE DO YOU WANT TO EDIT?

19

DO YOU NEED A MENU (Y/N)?

Y

00 NO MORE FITTINGS THIS BRANCH	* 14 DIVERGING WYE, MAIN SECTION
01 INTAKE SHAFT, RECT SECTION, SIDE	* 15 CONVERGENT WYE, BRANCH SECTION
02 ORIFACES WITH (OUT) LOUVERS	* 16 CONVERGENT WYE, MAIN SECTION
03 STRAIGHT DUCT	* 17 DIFFUSER, CONICAL ROUND SECTION
04 ELBOW, SMOOTH RADIUS ROUND	* 18 DIFFUSER, PLANE, IN-LINE
05 ELBOW, 90 DEG, 3, 4, 5 PCS, ROUND	* 19 DIFFUSER, PYRAMIDAL, IN-LINE
06 ELBOW, MITERED, ROUND, NEW/O VANES	* 20 DIFFUSER, TRANSITIONAL (ROUND TO
07 ELBOW, MITERED, RECTANGULAR	* 21 RECT OR RECT TO ROUND)
08 ELBOW, SMOOTH RADIUS, RECTANGULAR	* 21 CONTRACTION ROUND
09 ELBOW, MITERED WITH VANES, RECT	* 22 CONTRACTION RECTANGULAR
10 ELBOW, CONVERGING OR DIVERGING	* 23 OBSTRUCTION SCREEN IN DUCT
11 ELBOW, RECTANGULAR	* 24 LOUVER ENTRANCE
12 ELBOWS, 90 DEG, Z-SHAPED, RECT	* 25 FILTER
13 ELBOWS, 90 DEG, IN DIFFERENT	* 26 MULTI-BAFFLE SILENCER
14 ELBOWS, 90 DEG, IN DIFFERENT	* 27 GR MODULE
15 ELBOWS, 90 DEG, IN DIFFERENT	* 28 WASTE HEAT EXCHANGER
16 ELBOWS, 90 DEG, IN DIFFERENT	* 29 EXIT ABRUPT
17 ELBOWS, 90 DEG, IN DIFFERENT	* 30 FITTING NOT LISTED
18 ELBOWS, 90 DEG, IN DIFFERENT	
19 ELBOWS, 90 DEG, IN DIFFERENT	
20 ELBOWS, 90 DEG, IN DIFFERENT	
21 ELBOWS, 90 DEG, IN DIFFERENT	
22 ELBOWS, 90 DEG, IN DIFFERENT	
23 ELBOWS, 90 DEG, IN DIFFERENT	
24 ELBOWS, 90 DEG, IN DIFFERENT	
25 ELBOWS, 90 DEG, IN DIFFERENT	
26 ELBOWS, 90 DEG, IN DIFFERENT	
27 ELBOWS, 90 DEG, IN DIFFERENT	
28 ELBOWS, 90 DEG, IN DIFFERENT	
29 ELBOWS, 90 DEG, IN DIFFERENT	
30 ELBOWS, 90 DEG, IN DIFFERENT	

<> YOU ARE WORKING ON FITTING NUMBER >> 356205

05

YOU HAVE SELECTED A MITERED ROUND ELBOW.

**FIRST QUESTION, WHAT IS THE CROSS-SECTIONAL DIAMETER?

5.4667

WHAT IS THE ANGLE OF THE ELBOW TURN?

90

LAST QUESTION, ARE CETIMUM NUMBER OF CONCENTRIC VANES
 INSTALLED TO REDUCE RESISTANCE AND TURBULANCE (Y/N)?

N

DO YOU WANT TO ENTER THIS FITTING (Y/N)?

y WANT TO CHANGE ANOTHER FITTING (Y/N)?
n WANT TO MAKE ANY OTHER CHANGES (Y/N)?
n WHAT SERIAL NUMBER WOULD YOU LIKE TO GIVE THIS DUCT DATA FILE?
YOU MAY USE UP TO A SIX DIGIT INTEGER NUMBER.
2
510002
DO YOU WANT TO COMPUTE WITH THE FILE OR QUIT (C/Q)?
q

F. COMPUTING SYSTEM PERFORMANCE

This section also contains a recorded terminal session. The computing section of the program was exercised here. The session has been annotated to point out program features.

GLOBAL TITLE: CMSLIB FORTMOD2 MOD2EEH IMSLSP JONIMSL
 LCAD THESIS (START
 EXECUTION BEGINS...
 A ONE-DIMENSIONAL MODEL FOR THE SYSTEM PERFORMANCE
 OF A MARINE GAS TURBINE INSTALLATION

BY LCDR. STEPHEN M. EZZELL

OPTIONS: VERSION 1.0 MARCH 30, 1984
 BUILD A DATA FILE REPRESENTING THE DUCT SYSTEM
 EDIT OR CHANGE THE DUCT DATA FILE
 COMPUTE SYSTEM PERFORMANCE
 METHOD: INTERACTIVE INPUT OF DATA, BRANCHING TO DESIRED
 OPTION BY ANSWERING QUESTIONS

*** WARNING, TWO NULL ENTRIES ON NUMERICAL INPUT WILL ***
 *** KILL THE PROGRAM. ***

FIRST QUESTION:

DO YOU HAVE A DATA FILE OF DUCT FITTINGS (Y/N)?

Y

DO YOU WANT TO EDIT THE FILE OR USE IT FOR COMPUTATION (E/C)?

C

THIS PORTION OF THE PROGRAM INPUTS THE ENVIRONMENTAL CONDITIONS.
 WHAT IS THE AMBIENT TEMPERATURE (DEGREES F)?

75

WHAT IS THE AMBIENT PRESSURE (PSIA)?

14.6

WHAT IS THE RELATIVE HUMIDITY (GRAINS PER POUND AIR)?

70

YOU HAVE SELECTED A SYSTEM WITH A COOLING FAN. THE
 DEFAULT SPECIFICATIONS ARE FOR THE FAN INSTALLED ON
 THE DD963 CLASS SHIP.

DO YOU WANT TO USE THE DEFAULT SPECIFICATIONS (Y/N)?

Y

INPUT THE POWER SETTING YOU DESIRE.

***WHAT IS THE HORSEPOWER?

20000

***WHAT IS THE POWER TURBINE SPEED (RPM)?

3600

THE RESULTS OF THIS RUN HAVE BEEN ENTERED
 INTO A FILE CALLED "OUTPUT DATA".

DO YOU WANT TO COMPUTE WITH DIFFERENT OPERATING CONDITIONS (Y/N)?

Y

INPUT THE POWER SETTING YOU DESIRE.

***WHAT IS THE HORSEPOWER?

10000

***WHAT IS THE POWER TURBINE SPEED (RPM)?

2200

THE RESULTS OF THIS RUN HAVE BEEN ENTERED
 INTO A FILE CALLED "OUTPUT DATA".

DO YOU WANT TO COMPUTE WITH DIFFERENT OPERATING CONDITIONS (Y/N)?

N

DO YOU WANT TO EDIT THE DUCT DATA FILE OR QUIT (E/Q)?

Q

G. EXAMINING THE OUTPUT

Included in this section are copies of two files. The first is a copy of the file the author built using the Arleigh Burke class example. The other one is a copy of the results from the runs made in the compute section using the sample file at two operating points.

THIS PERFORMANCE RUN WAS DEVELOPED FROM DUCT DATA FILE, 510001

INLET CONDITIONS: AMBIENT TEMP (DEG F) 75.00
 AMBIENT PRESS (PSIA) 14.60
 HUMIDITY (GRAINS) 70.00
 HORSEPOWER: 20000.0
 NPS (RPM): 3600.0

ENGINE DUCT LOSSES (IN.W.G.): INLET 1.98 EXHAUST 13.95

ENGINE PERFORMANCE PARAMETERS:

IMEP= 24.32 IHP/SEC
 MBHP= 122.71 IHP/SEC
 MBHP= 123.78 IHP/SEC
 PBP= 15.18 PSIA
 TBP= 1405.49 DEG R
 STFT= 0.406 LB(FUEL)/HP*HR
 TUS= 1827.1 DEG R
 NGM= 9827.0 RPM
 MODULE COOLING TEMP OUT= 250.3 DEG F

FITTING ID	FITTING TYPE	PRESSURE LOSS INCH W.G.	VELOCITY PRESSURE INCH W.G.	
3122201	24	0.42	0.02	LOUVER ENTRANCE
3122202	25	0.72	0.02	FILTER
3122203	26	0.00	0.02	STRAIGHT DUCT
3122204	14	0.08	0.09	MAIN SECT DIV WYE
3122205	26	0.09	0.09	SILENCER SECTION
3122206	22	0.01	0.23	CONTRACTION, RECT
3122207	22	0.01	0.23	ELBOW, MITERED, RECT
3122208	23	0.01	0.23	SCREEN IN DUCT
3122209	23	0.01	0.23	BRANCH DIV WYE
3122210	23	0.01	0.72	STRAIGHT DUCT
3122211	23	0.01	0.72	STRAIGHT DUCT
3122212	16	0.00	1.71	MAIN SECT CONV WYE
3122213	23	0.00	2.24	GAS TURBINE MODULE
3122214	23	0.00	0.00	BRANCH CONV WYE
3122215	23	0.00	0.00	CONTRACTION, ROUND
3122216	23	0.00	0.79	STRAIGHT DUCT
3122217	23	0.00	0.79	ELBOW, MITERED, ROUND
3122218	23	0.00	0.54	STRAIGHT DUCT
3122219	23	0.00	0.90	ELBOW, MITERED, ROUND
3122220	23	0.00	0.90	STRAIGHT DUCT
3122221	17	0.00	0.90	ELBOW, MITERED, ROUND
3122222	23	0.00	0.90	STRAIGHT DUCT
3122223	23	0.00	0.90	ELBOW, MITERED, ROUND
3122224	23	0.00	0.90	STRAIGHT DUCT
3122225	23	0.00	0.90	ELBOW, MITERED, ROUND
3122226	23	0.00	0.90	STRAIGHT DUCT
3122227	23	0.00	0.90	ELBOW, MITERED, ROUND
3122228	23	0.00	0.90	STRAIGHT DUCT
3122229	23	0.00	0.90	ELBOW, MITERED, ROUND
3122230	23	0.00	0.90	STRAIGHT DUCT
3122231	23	0.00	0.90	ELBOW, MITERED, ROUND
3122232	23	0.00	0.90	STRAIGHT DUCT
3122233	23	0.00	0.90	ELBOW, MITERED, ROUND
3122234	23	0.00	0.90	STRAIGHT DUCT
3122235	23	0.00	0.90	ELBOW, MITERED, ROUND
3122236	23	0.00	0.90	STRAIGHT DUCT
3122237	23	0.00	0.90	ELBOW, MITERED, ROUND
3122238	23	0.00	0.90	STRAIGHT DUCT
3122239	23	0.00	0.90	ELBOW, MITERED, ROUND
3122240	23	0.00	0.90	STRAIGHT DUCT
3122241	23	0.00	0.90	ELBOW, MITERED, ROUND
3122242	23	0.00	0.90	STRAIGHT DUCT
3122243	23	0.00	0.90	ELBOW, MITERED, ROUND
3122244	23	0.00	0.90	STRAIGHT DUCT
3122245	23	0.00	0.90	ELBOW, MITERED, ROUND
3122246	23	0.00	0.90	STRAIGHT DUCT
3122247	23	0.00	0.90	ELBOW, MITERED, ROUND
3122248	23	0.00	0.90	STRAIGHT DUCT
3122249	23	0.00	0.90	ELBOW, MITERED, ROUND
3122250	23	0.00	0.90	STRAIGHT DUCT
3122251	23	0.00	0.90	ELBOW, MITERED, ROUND
3122252	23	0.00	0.90	STRAIGHT DUCT
3122253	23	0.00	0.90	ELBOW, MITERED, ROUND
3122254	23	0.00	0.90	STRAIGHT DUCT
3122255	23	0.00	0.90	ELBOW, MITERED, ROUND
3122256	23	0.00	0.90	STRAIGHT DUCT
3122257	23	0.00	0.90	ELBOW, MITERED, ROUND
3122258	23	0.00	0.90	STRAIGHT DUCT
3122259	23	0.00	0.90	ELBOW, MITERED, ROUND
3122260	23	0.00	0.90	STRAIGHT DUCT
3122261	23	0.00	0.90	ELBOW, MITERED, ROUND
3122262	23	0.00	0.90	STRAIGHT DUCT
3122263	23	0.00	0.90	ELBOW, MITERED, ROUND
3122264	23	0.00	0.90	STRAIGHT DUCT
3122265	23	0.00	0.90	ELBOW, MITERED, ROUND
3122266	23	0.00	0.90	STRAIGHT DUCT
3122267	23	0.00	0.90	ELBOW, MITERED, ROUND
3122268	23	0.00	0.90	STRAIGHT DUCT
3122269	23	0.00	0.90	ELBOW, MITERED, ROUND
3122270	23	0.00	0.90	STRAIGHT DUCT
3122271	23	0.00	0.90	ELBOW, MITERED, ROUND
3122272	23	0.00	0.90	STRAIGHT DUCT
3122273	23	0.00	0.90	ELBOW, MITERED, ROUND
3122274	23	0.00	0.90	STRAIGHT DUCT
3122275	23	0.00	0.90	ELBOW, MITERED, ROUND
3122276	23	0.00	0.90	STRAIGHT DUCT
3122277	23	0.00	0.90	ELBOW, MITERED, ROUND
3122278	23	0.00	0.90	STRAIGHT DUCT
3122279	23	0.00	0.90	ELBOW, MITERED, ROUND
3122280	23	0.00	0.90	STRAIGHT DUCT
3122281	23	0.00	0.90	ELBOW, MITERED, ROUND
3122282	23	0.00	0.90	STRAIGHT DUCT
3122283	23	0.00	0.90	ELBOW, MITERED, ROUND
3122284	23	0.00	0.90	STRAIGHT DUCT
3122285	23	0.00	0.90	ELBOW, MITERED, ROUND
3122286	23	0.00	0.90	STRAIGHT DUCT
3122287	23	0.00	0.90	ELBOW, MITERED, ROUND
3122288	23	0.00	0.90	STRAIGHT DUCT
3122289	23	0.00	0.90	ELBOW, MITERED, ROUND
3122290	23	0.00	0.90	STRAIGHT DUCT
3122291	23	0.00	0.90	ELBOW, MITERED, ROUND
3122292	23	0.00	0.90	STRAIGHT DUCT
3122293	23	0.00	0.90	ELBOW, MITERED, ROUND
3122294	23	0.00	0.90	STRAIGHT DUCT
3122295	23	0.00	0.90	ELBOW, MITERED, ROUND
3122296	23	0.00	0.90	STRAIGHT DUCT
3122297	23	0.00	0.90	ELBOW, MITERED, ROUND
3122298	23	0.00	0.90	STRAIGHT DUCT
3122299	23	0.00	0.90	ELBOW, MITERED, ROUND
3122300	23	0.00	0.90	STRAIGHT DUCT
3122301	23	0.00	0.90	ELBOW, MITERED, ROUND
3122302	23	0.00	0.90	STRAIGHT DUCT
3122303	23	0.00	0.90	ELBOW, MITERED, ROUND
3122304	23	0.00	0.90	STRAIGHT DUCT
3122305	23	0.00	0.90	ELBOW, MITERED, ROUND
3122306	23	0.00	0.90	STRAIGHT DUCT
3122307	23	0.00	0.90	ELBOW, MITERED, ROUND
3122308	23	0.00	0.90	STRAIGHT DUCT
3122309	23	0.00	0.90	ELBOW, MITERED, ROUND
3122310	23	0.00	0.90	STRAIGHT DUCT
3122311	23	0.00	0.90	ELBOW, MITERED, ROUND
3122312	23	0.00	0.90	STRAIGHT DUCT
3122313	23	0.00	0.90	ELBOW, MITERED, ROUND
3122314	23	0.00	0.90	STRAIGHT DUCT
3122315	23	0.00	0.90	ELBOW, MITERED, ROUND
3122316	23	0.00	0.90	STRAIGHT DUCT
3122317	23	0.00	0.90	ELBOW, MITERED, ROUND
3122318	23	0.00	0.90	STRAIGHT DUCT
3122319	23	0.00	0.90	ELBOW, MITERED, ROUND
3122320	23	0.00	0.90	STRAIGHT DUCT
3122321	23	0.00	0.90	ELBOW, MITERED, ROUND
3122322	23	0.00	0.90	STRAIGHT DUCT
3122323	23	0.00	0.90	ELBOW, MITERED, ROUND
3122324	23	0.00	0.90	STRAIGHT DUCT
3122325	23	0.00	0.90	ELBOW, MITERED, ROUND
3122326	23	0.00	0.90	STRAIGHT DUCT
3122327	23	0.00	0.90	ELBOW, MITERED, ROUND
3122328	23	0.00	0.90	STRAIGHT DUCT
3122329	23	0.00	0.90	ELBOW, MITERED, ROUND
3122330	23	0.00	0.90	STRAIGHT DUCT
3122331	23	0.00	0.90	ELBOW, MITERED, ROUND
3122332	23	0.00	0.90	STRAIGHT DUCT
3122333	23	0.00	0.90	ELBOW, MITERED, ROUND
3122334	23	0.00	0.90	STRAIGHT DUCT
3122335	23	0.00	0.90	ELBOW, MITERED, ROUND
3122336	23	0.00	0.90	STRAIGHT DUCT
3122337	23	0.00	0.90	ELBOW, MITERED, ROUND
3122338	23	0.00	0.90	STRAIGHT DUCT
3122339	23	0.00	0.90	ELBOW, MITERED, ROUND
3122340	23	0.00	0.90	STRAIGHT DUCT
3122341	23	0.00	0.90	ELBOW, MITERED, ROUND
3122342	23	0.00	0.90	STRAIGHT DUCT
3122343	23	0.00	0.90	ELBOW, MITERED, ROUND
3122344	23	0.00	0.90	STRAIGHT DUCT
3122345	23	0.00	0.90	ELBOW, MITERED, ROUND
3122346	23	0.00	0.90	STRAIGHT DUCT
3122347	23	0.00	0.90	ELBOW, MITERED, ROUND
3122348	23	0.00	0.90	STRAIGHT DUCT
3122349	23	0.00	0.90	ELBOW, MITERED, ROUND
3122350	23	0.00	0.90	STRAIGHT DUCT
3122351	23	0.00	0.90	ELBOW, MITERED, ROUND
3122352	23	0.00	0.90	STRAIGHT DUCT
3122353	23	0.00	0.90	ELBOW, MITERED, ROUND
3122354	23	0.00	0.90	STRAIGHT DUCT
3122355	23	0.00	0.90	ELBOW, MITERED, ROUND
3122356	23	0.00	0.90	STRAIGHT DUCT
3122357	23	0.00	0.90	ELBOW, MITERED, ROUND
3122358	23	0.00	0.90	STRAIGHT DUCT
3122359	23	0.00	0.90	ELBOW, MITERED, ROUND
3122360	23	0.00	0.90	STRAIGHT DUCT
3122361	23	0.00	0.90	ELBOW, MITERED, ROUND
3122362	23	0.00	0.90	STRAIGHT DUCT
3122363	23	0.00	0.90	ELBOW, MITERED, ROUND
3122364	23	0.00	0.90	STRAIGHT DUCT
3122365	23	0.00	0.90	ELBOW, MITERED, ROUND
3122366	23	0.00	0.90	STRAIGHT DUCT
3122367	23	0.00	0.90	ELBOW, MITERED, ROUND
3122368	23	0.00	0.90	STRAIGHT DUCT
3122369	23	0.00	0.90	ELBOW, MITERED, ROUND
3122370	23	0.00	0.90	STRAIGHT DUCT
3122371	23	0.00	0.90	ELBOW, MITERED, ROUND
3122372	23	0.00	0.90	STRAIGHT DUCT
3122373	23	0.00	0.90	ELBOW, MITERED, ROUND
3122374	23	0.00	0.90	STRAIGHT DUCT
3122375	23	0.00	0.90	ELBOW, MITERED, ROUND
3122376	23	0.00	0.90	STRAIGHT DUCT
3122377	23	0.00	0.90	ELBOW, MITERED, ROUND
3122378	23	0.00	0.90	STRAIGHT DUCT
3122379	23	0.00	0.90	ELBOW, MITERED, ROUND
3122380	23	0.00	0.90	STRAIGHT DUCT
3122381	23	0.00	0.90	ELBOW, MITERED, ROUND
3122382	23	0.00	0.90	STRAIGHT DUCT
3122383	23	0.00	0.90	ELBOW, MITERED, ROUND
3122384	23	0.00	0.90	STRAIGHT DUCT
3122385	23	0.00	0.90	ELBOW, MITERED, ROUND
3122386	23	0.00	0.90	STRAIGHT DUCT
3122387	23	0.00	0.90	ELBOW, MITERED, ROUND
3122388	23	0.00	0.90	STRAIGHT DUCT
3122389	23	0.00	0.90	ELBOW, MITERED, ROUND
3122390	23	0.00	0.90	STRAIGHT DUCT
3122391	23	0.00	0.90	ELBOW, MITERED, ROUND
3122392	23	0.00	0.90	STRAIGHT DUCT
3122393	23	0.00	0.90	ELBOW, MITERED, ROUND
3122394	23	0.00	0.90	STRAIGHT DUCT

THIS PERFORMANCE RUN WAS DEVELOPED FROM DUCT DATA FILE, 510001

INLET CONDITIONS: AMBIENT TEMP (DEG F) 75.00
 AMBIENT PRESS (PSIA) 14.60
 HUMIDITY (GRAINS) 70.00
 HORSEPOWER: 10000.3
 WFT (REF): 2200.3

ENGINE DUCT LOSSES (IN.W.G.): INLET 1.40 EXHAUST 9.10

ENGINE PERFORMANCE PARAMETERS:

WC= 200.44 LBM/SEC
 W2= 99.45 LBM/SEC
 W8= 99.88 LBM/SEC
 F8= 14.97 PSIA
 T8= 1281.00 DEG R
 SFC= 0.508 LBM(FUEL)/HP*HR
 T54= 1549.0 DEG R
 NG= 8332.3 RPM
 MODULE COOLING TEMP OUT= 250.3 DEG F

FITTING ID	FITTING TYPE	PRESSURE LOSS INCH W.G.	VELOCITY INCH W.G.	PRESSURE INCH W.G.	
312201	24	0.39	0.32		LOUVER ENTRANCE
312202	200	0.55	0.42		FILTER
312203	200	0.00	0.42		STRAIGHT DUCT
312204	142	0.00	0.46		MAIN SECT DIV WYE
312205	226	0.06	0.06		SILENCER SECTION
312206	222	0.00	0.15		CONTRACTION, RECT
312207	226	0.12	0.15		EXPANSION, RECT
312208	200	0.00	0.16		SCREEN, IN DUCT
312209	133	0.74	0.79		BRANCH, DIV WYE
312210	200	0.00	0.79		STRAIGHT DUCT
312211	200	0.00	0.79		STRAIGHT DUCT
312212	102	0.00	0.33		STRAIGHT DUCT
312213	200	0.00	0.46		MAIN SECT CONV WYE
312214	200	0.00	0.50		GAS DIRECT LINE MODULE
312215	200	0.00	0.50		BRANCH, CONV WYE
312216	200	0.00	0.46		CONTRACTION, ROUND
312217	200	0.00	0.47		STRAIGHT DUCT
312218	200	0.00	0.47		ELBOW, FITTED, ROUND
312219	200	0.00	0.30		STRAIGHT DUCT
312220	200	0.00	0.48		ELBOW, FITTED, ROUND
312221	200	0.00	0.48		STRAIGHT DUCT
312222	200	0.00	0.48		DIFF, CONICAL
312223	200	0.00	0.84		STRAIGHT DUCT
312224	200	0.00	0.84		CONTRACTION, ROUND
312225	200	0.00	0.26		STRAIGHT DUCT
312226	200	0.00	0.27		EXIT, ABRUPT
312227	200	0.00	0.27		
312228	200	0.00	0.27		
312229	200	0.00	0.27		
312230	200	0.00	0.27		
312231	200	0.00	0.27		
312232	200	0.00	0.27		
312233	200	0.00	0.27		
312234	200	0.00	0.27		
312235	200	0.00	0.27		
312236	200	0.00	0.27		
312237	200	0.00	0.27		
312238	200	0.00	0.27		
312239	200	0.00	0.27		
312240	200	0.00	0.27		
312241	200	0.00	0.27		
312242	200	0.00	0.27		
312243	200	0.00	0.27		
312244	200	0.00	0.27		
312245	200	0.00	0.27		
312246	200	0.00	0.27		
312247	200	0.00	0.27		
312248	200	0.00	0.27		
312249	200	0.00	0.27		
312250	200	0.00	0.27		
312251	200	0.00	0.27		
312252	200	0.00	0.27		
312253	200	0.00	0.27		
312254	200	0.00	0.27		
312255	200	0.00	0.27		
312256	200	0.00	0.27		
312257	200	0.00	0.27		
312258	200	0.00	0.27		
312259	200	0.00	0.27		
312260	200	0.00	0.27		
312261	200	0.00	0.27		
312262	200	0.00	0.27		
312263	200	0.00	0.27		
312264	200	0.00	0.27		
312265	200	0.00	0.27		
312266	200	0.00	0.27		
312267	200	0.00	0.27		
312268	200	0.00	0.27		
312269	200	0.00	0.27		
312270	200	0.00	0.27		
312271	200	0.00	0.27		
312272	200	0.00	0.27		
312273	200	0.00	0.27		
312274	200	0.00	0.27		
312275	200	0.00	0.27		
312276	200	0.00	0.27		
312277	200	0.00	0.27		
312278	200	0.00	0.27		
312279	200	0.00	0.27		
312280	200	0.00	0.27		
312281	200	0.00	0.27		
312282	200	0.00	0.27		
312283	200	0.00	0.27		
312284	200	0.00	0.27		
312285	200	0.00	0.27		
312286	200	0.00	0.27		
312287	200	0.00	0.27		
312288	200	0.00	0.27		
312289	200	0.00	0.27		
312290	200	0.00	0.27		
312291	200	0.00	0.27		
312292	200	0.00	0.27		
312293	200	0.00	0.27		
312294	200	0.00	0.27		
312295	200	0.00	0.27		
312296	200	0.00	0.27		
312297	200	0.00	0.27		
312298	200	0.00	0.27		
312299	200	0.00	0.27		
312300	200	0.00	0.27		
312301	200	0.00	0.27		
312302	200	0.00	0.27		
312303	200	0.00	0.27		
312304	200	0.00	0.27		
312305	200	0.00	0.27		
312306	200	0.00	0.27		
312307	200	0.00	0.27		
312308	200	0.00	0.27		
312309	200	0.00	0.27		
312310	200	0.00	0.27		
312311	200	0.00	0.27		
312312	200	0.00	0.27		
312313	200	0.00	0.27		
312314	200	0.00	0.27		
312315	200	0.00	0.27		
312316	200	0.00	0.27		
312317	200	0.00	0.27		
312318	200	0.00	0.27		
312319	200	0.00	0.27		
312320	200	0.00	0.27		
312321	200	0.00	0.27		
312322	200	0.00	0.27		
312323	200	0.00	0.27		
312324	200	0.00	0.27		
312325	200	0.00	0.27		
312326	200	0.00	0.27		
312327	200	0.00	0.27		
312328	200	0.00	0.27		
312329	200	0.00	0.27		
312330	200	0.00	0.27		
312331	200	0.00	0.27		
312332	200	0.00	0.27		
312333	200	0.00	0.27		
312334	200	0.00	0.27		
312335	200	0.00	0.27		
312336	200	0.00	0.27		
312337	200	0.00	0.27		
312338	200	0.00	0.27		
312339	200	0.00	0.27		
312340	200	0.00	0.27		
312341	200	0.00	0.27		
312342	200	0.00	0.27		
312343	200	0.00	0.27		
312344	200	0.00	0.27		
312345	200	0.00	0.27		
312346	200	0.00	0.27		
312347	200	0.00	0.27		
312348	200	0.00	0.27		
312349	200	0.00	0.27		
312350	200	0.00	0.27		
312351	200	0.00	0.27		
312352	200	0.00	0.27		
312353	200	0.00	0.27		
312354	200	0.00	0.27		
312355	200	0.00	0.27		
312356	200	0.00	0.27		
312357	200	0.00	0.27		
312358	200	0.00	0.27		
312359	200	0.00	0.27		
312360	200	0.00	0.27		
312361	200	0.00	0.27		
312362	200	0.00	0.27		
312363	200	0.00	0.27		
312364	200	0.00	0.27		
312365	200	0.00	0.27		
312366	200	0.00	0.27		
312367	200	0.00	0.27		
312368	200	0.00	0.27		
312369	200	0.00	0.27		
312370	200	0.00	0.27		
312371	200	0.00	0.27		
312372	200	0.00	0.27		
312373	200	0.00	0.27		
312374	200	0.00	0.27		
312375	200	0.00	0.27		
312376	200	0.00	0.27		
312377	200	0.00	0.27		
312378	200	0.00	0.27		
312379	200	0.00	0.27		
312380	200	0.00	0.27		
312381	200	0.00	0.27		
312382	200	0.00	0.27		
312383	200	0.00	0.27		
312384	200	0.00	0.27		
312385	200	0.00	0.27		
312386	200	0.00	0.27		
312387	200	0.00	0.27		
312388	200	0.00	0.27		
312389	200	0.00	0.27		
312390	200	0.00	0.27		
312391	200	0.00	0.27		
312392	200	0.00	0.27		
312393	200	0.00	0.27		
312394	200	0.00	0.27		
312395	200	0.00	0.27		
312396	200	0.00	0.27		
312397	200	0.00	0.27		
312398	200	0.00	0.27		
312399	200	0.00	0.27		
312400	200	0.00	0.27		
312401	200	0.00	0.27		

LIST OF REFERENCES

1. T. Schweiger, "The Aerodynamic Aspects of Gas Turbine Ducting Design - Their Model Testing, Correlation of Model Testing and Full Scale Results", (paper presented at the Gas Turbine Conference, by the Institute of Marine Engineers, London, November 1975)
2. Irving H. Shames, Mechanics of Fluids, (New York 1982)
3. American Society of Heating, Refrigeration, and Air-conditioning Engineers, ASHRAE Handbook 1981 Fundamentals
4. 7LM2500 Marine Gas Turbine Performance Data, Report Number MID-TD-2500-8, Marine and Industrial Projects Department, General Electric Company, November 1978
5. I.E. Idel'chik, Handbook of Hydraulic Resistance, trans. from the Russian by A. Barouch (Israel, 1966)
6. Naval Ship Engineering Station Philadelphia, Gas Turbine Inlet Design Handbook, 25 April, 1983

INITIAL DISTRIBUTION LIST

	No. Copies
1. Defense Technical Information Center Cameron Station Alexandria, Virginia 22314	2
2. Library, Code 0142 Naval Postgraduate School Monterey, California 93943	2
3. Department Chairman, Code 69 Department of Mechanical Engineering Naval Postgraduate School Monterey, California 93943	2
4. Professor Paul F. Pucci (Code 69Pc) Department of Mechanical Engineering Naval Postgraduate School Monterey, California 93943	5
5. LCDR Stephen M. Ezzell 1388 Old Salem Road Kernersville, North Carolina 27284	1
6. Joseph Londino NAVSEA Code 56X11 Naval Sea Systems Command Washington, D.C. 20362	5
7. Dan Groghan NAVSEA Code 05R Naval Sea Systems Command Washington, D.C. 20362	1
8. Donald Tempesco NAVSEA Code 56X3 Naval Sea Systems Command Washington, D.C. 20362	1
9. General Electric Company Attn: Mr. Werner Hoelmer Aircraft Engine Group Cincinnati, Ohio 45215	1

END

FILMED

1-85

DTIC